

CONTROL

REEM MUIN



POWERUNIT



Control Systems Basics

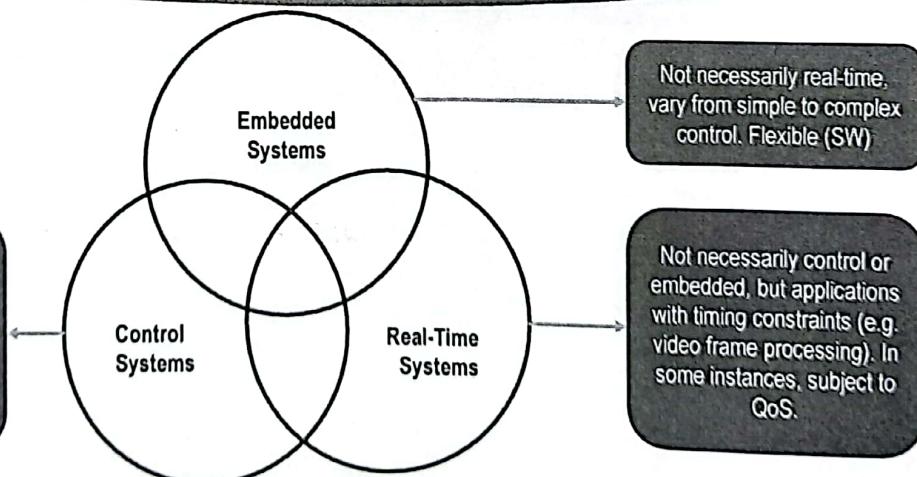
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Introduction

Classical Control Theory
(In the past, not necessarily digital, mostly analogue) → Modern Digital Control
Not necessarily embedded



Five Apple

Classical Control Development

Control Systems is not a human engineered discipline → Best controllers are found in nature!

Biochemical Controllers → Human body

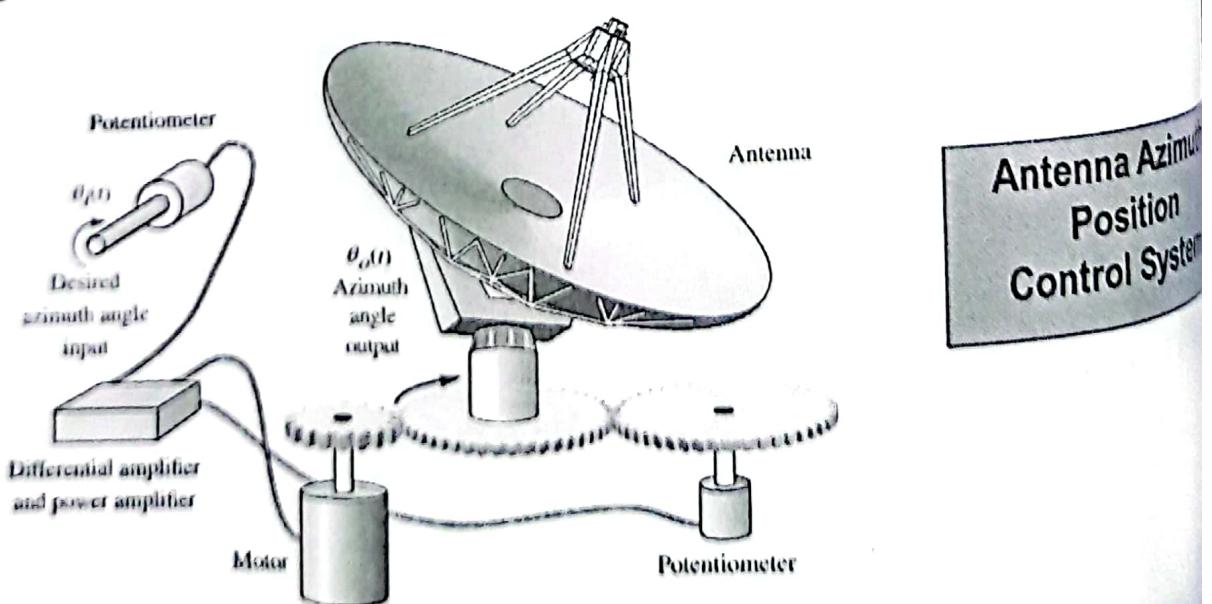
(numerous control systems (central and distributed), e.g., pancreas, immune system, eyes following an object, eye-hand coordination, fight or flight response, etc.)

Species declining or increasing population can be modeled as a self-regulating control system (wolves/rabbits population)

- Species declining or increasing population can be modeled as a self-regulating control system (wolves/rabbits population)
- First feedback control systems were developed in Greece in 300 B.C. (water clocks)
- Steam pressure and temperature control with safety valves as far back as 1681, windmill blade adjustment to wind speed 1745
- Foundation of Control Systems as we know today in 1868 (Maxwell, Routh-Hurwitz, Chebyshev) → system description, differential equations, criteria for stability → stability of steering ships and applying power through hydraulic systems, gun platforms for military ships (achieved 1922 by Minorsky) (Adaptive Control field)
- In 1930's and 40's: focus on developing of math to analyze modern systems (Bode, Nyquist) → Bode Plots and Root Locus

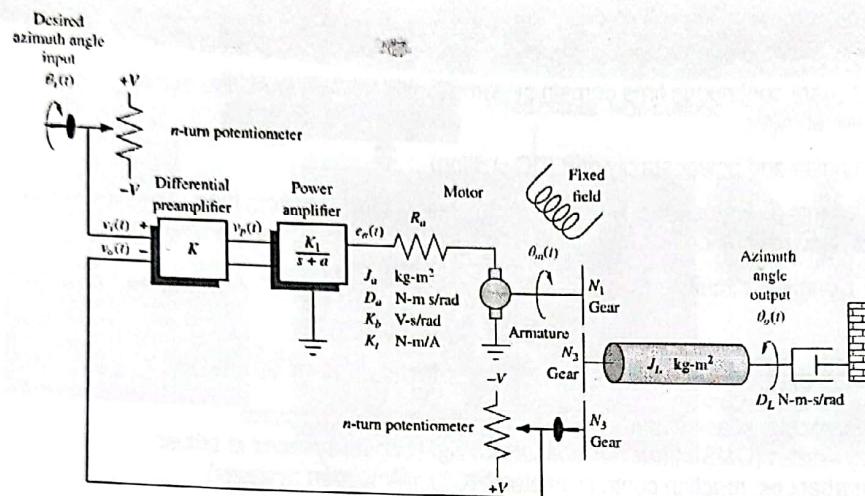
Simple Control System Example (All Analogue Components)

Layout



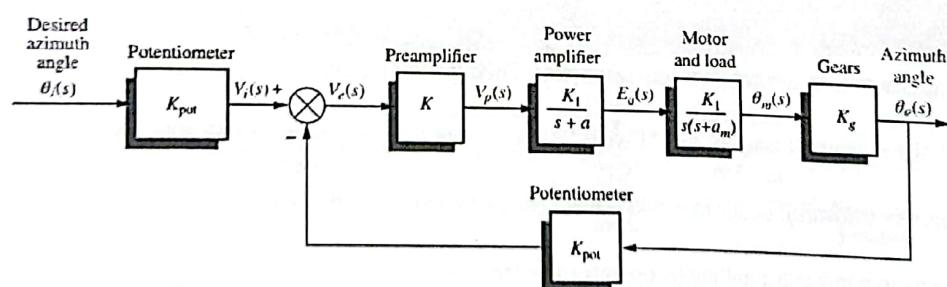
Simple Control System Example (All Analogue Components)

Schematic



Simple Control System Example (All Analogue Components)

Block Diagram



You can notice that there are no computers or digital controllers in the layout/schematic/block diagrams → analogue control

Digital (Computer) Control

Classical (Analogue) Control systems are continuous time domain systems (no sampling), the entire signal (voltage, current, processed). They have several disadvantages:

- ▶ highly susceptible (sensitive) to noise and power supply drift (DC shifting)
- ▶ Since they use analogue components (e.g., resistors and capacitors), their values vary from their nominal values due to tolerances (e.g., $1000 \pm 100\Omega$), and resistance changes for example by circuit/environment temperature.
- ▶ Difficult to modify or update → Complete circuit redesign



Last colour band: Tolerances

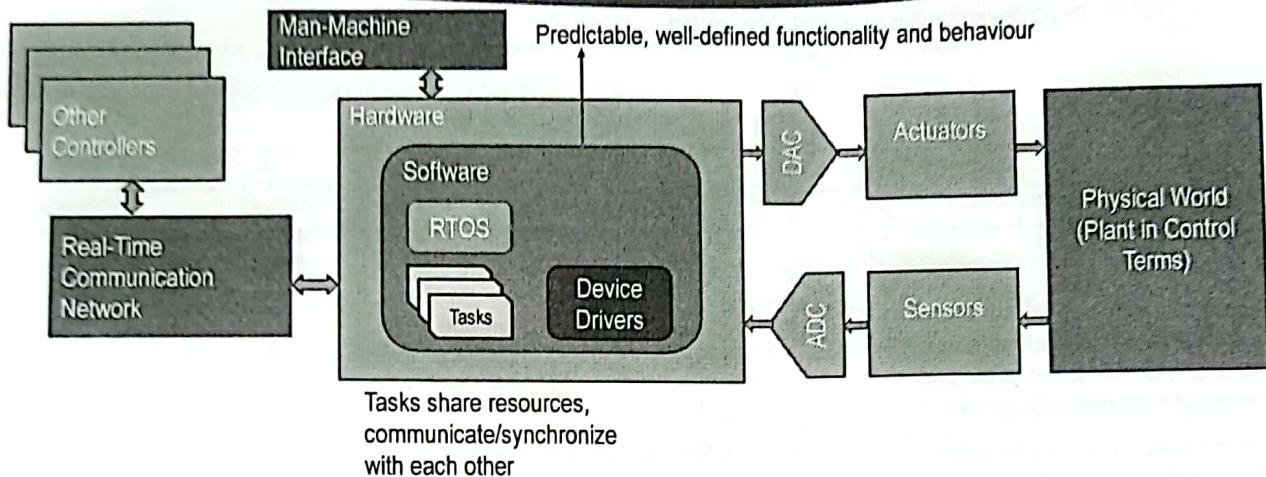
Most modern control systems are digital → They use main computer frames or embedded controllers

- ❖ Could be complex: Industrial robots, space craft/rover, chemical/nuclear process control (orbital maneuvering system (OMS), thrust for orientation, flight control systems to adjust for atmosphere disturbances, reaction control systems (RCS), life support systems)
- ❖ Could be simple: home automation, DVD player, HDD controller

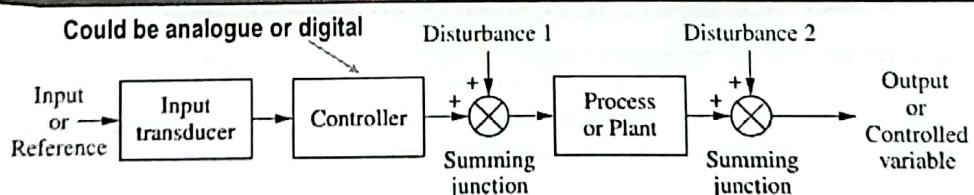
Digital (Computer) Control Advantages

- Discrete time signals and approximation always involved (**Sampling Theory : Shannon's Theory of Sampling / Nyquist Rate**)
- Accurate representation of digital signals using "0" and "1" with 12 bits or more for a single number → Negligible errors (depends on ADC resolution)
- Digital controllers (in firmware or software): easily modified without complete replacement of the original controller → no circuit redesign or re-wiring
- Complex digital controller require a few extra arithmetic operations or libraries
- Faster hardware allows short sampling periods (high sampling rate).
- With short sampling periods, digital controllers monitor controlled variables almost continuously
- Advances in VLSI technology provide better, faster and more reliable integrated circuits at lower prices.
- Many hardware circuits can be replaced by software code (e.g. Filters can be coded Instead of being built as analogue circuits)
- Can be real-time control
- Computer can schedule multiple control systems at the same time

Overview of a Real-Time Control System

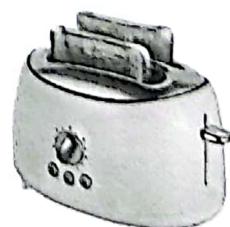


Open-Loop Control Systems

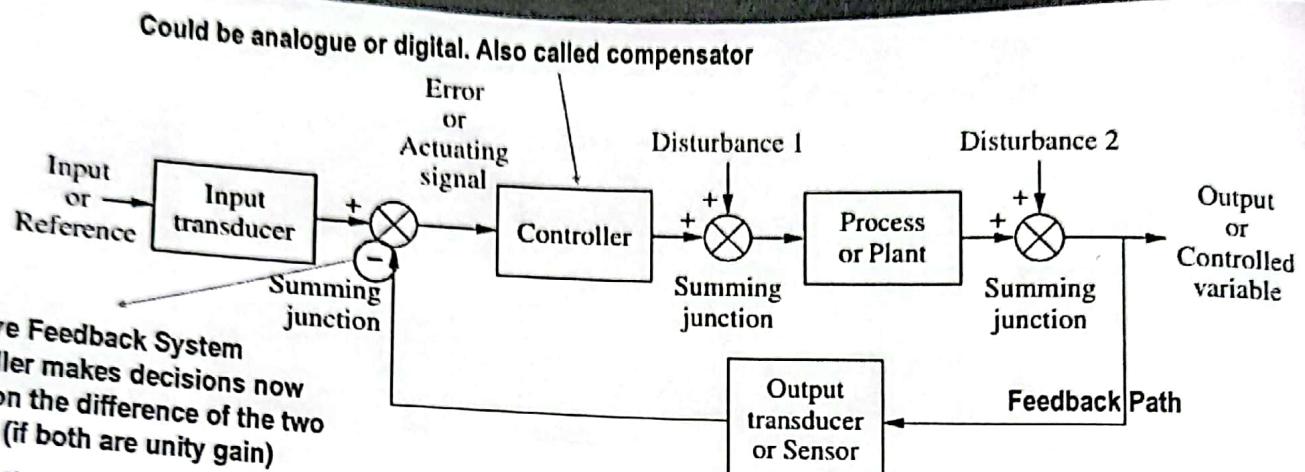


Main characteristic of open-loop systems is that they

- ❖ Are only commanded by the input and therefore sensitive to disturbance
- ❖ Cannot compensate/correct for any disturbances (noise) that add to the controller's driving signal (Disturbance 1), or disturbances at the output (Disturbance 2, for example a physical object in the way)
- ❖ A kitchen microwave or toaster is a basic example
- ❖ Very simple to design, but often times unstable for complex systems



Closed-Loop Control Systems I



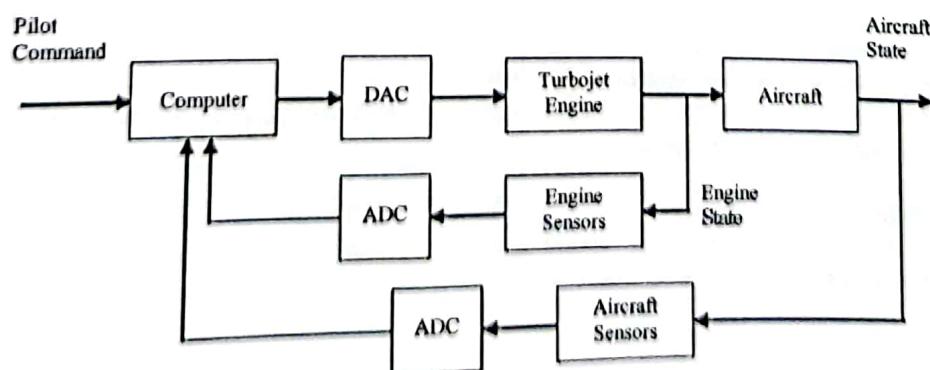
Note that there are positive feedback systems as well. A simple NAND latch that forces its output to 0s and 1s is a positive feedback system with desirable output. However, a microphone and speaker loop in a conference hall is a positive feedback system with undesirable output.

Closed-Loop Control Systems II

Could be multiple closed loops to one computer controller: Turbojet Engine Control System

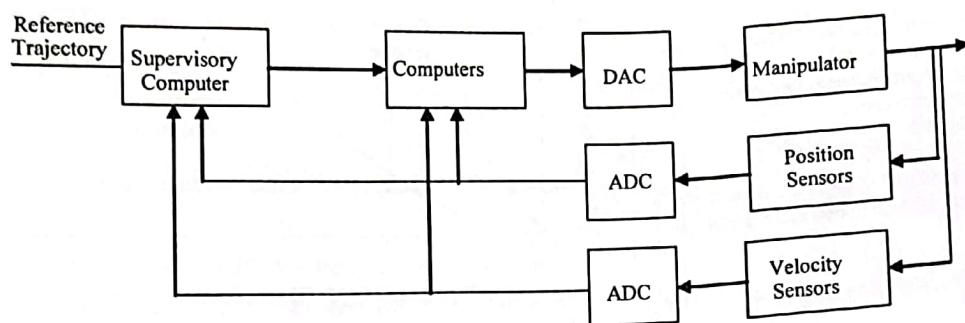
Feedback loops could be:

Multiple sensors connected to the same physical object in the plant (engine temperature, engine vibration, engine fuel flow)
Multiple sensors connected to the different physical objects in the plant



Closed-Loop Control Systems III

Could be multiple closed loops, to multiple computer controllers and a supervisory computer
 Controllers could be digital or analogue in the same system!



Closed-Loop Control Systems IV

- ▶ Have the obvious advantage of greater accuracy than open-loop systems
- ▶ Less sensitive to noise, disturbances, and changes in the environment
- ▶ Transient response and steady-state error can be controlled more conveniently and with greater flexibility in closed-loop systems (more in following slides)
- ▶ Closed-loop systems are more complex and expensive than open-loop systems (extra sensors, wiring, ADC if digital, and processing time)
- ▶ Question: What do you need to do to redesign the simple toaster as a closed-loop feedback system? What will you be sensing to get a perfect toasted piece of bread?
- ▶ Home Exercise: Think of another existing open-loop system at your home, and think of what you need to convert it to a closed-loop system? Is it worth it?

Analysis and Design of Control Systems (I)

Temporal Characteristics of the Plant

Input: Press the 4th floor button representing our desired output (reach 4th floor)

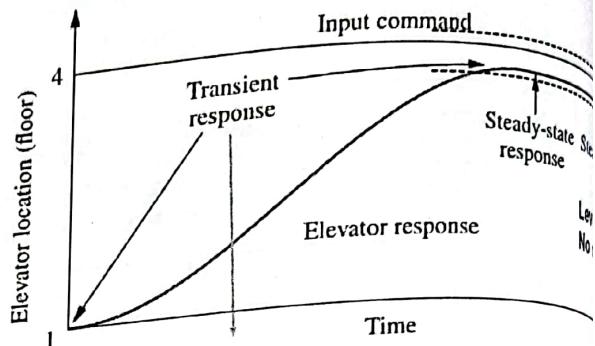
A button ideally represents a unit step function $u(t)$

Two major measures of performance are apparent:
(1) the transient response and
(2) the steady-state error.

Analysis Stage: find values for transient response and steady state error that are within specifications and acceptable

Design Stage: apply control system parameters that satisfy the values analyzed

Elevator Example



Too slow transient response: user patience is sacrificed

Too fast transient response : user comfort is sacrificed, could cause damage/death

Analysis and Design of Control Systems (II)

Temporal Characteristics of the Plant

The transient response should be as quick as possible (ideal case), but in many systems, this is not feasible either due to its catastrophic effects or simply it is physically impossible

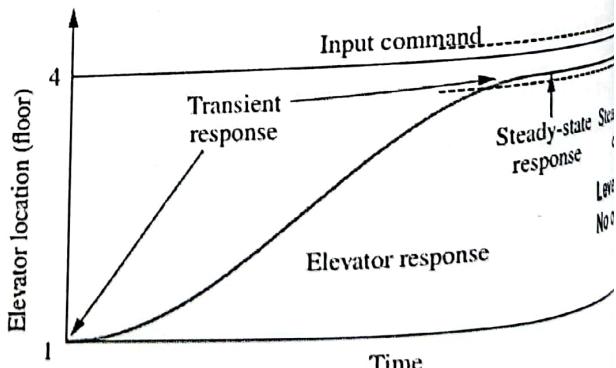
e.g., an elevator going from G-level to 4th or 100th floor incautiously will kill you; also, mechanical systems are slow and physically infeasible to have instantons response.

The steady-state means we should reach our desired output without error. However, error is always there.

Error band specifies the maximum acceptable margins for errors (0.01%, 1%, 5%)

Elevator could stop at the 4th floor with its level 1cm above the floor ground (acceptable), 5cm → You could trip and fall, 50 cm (something is visibly wrong, and it is dangerous)

Elevator Example



Too slow transient response: user patience is sacrificed

Too fast transient response : user comfort is sacrificed, could cause damage/death

Analysis and Design of Control Systems (III)

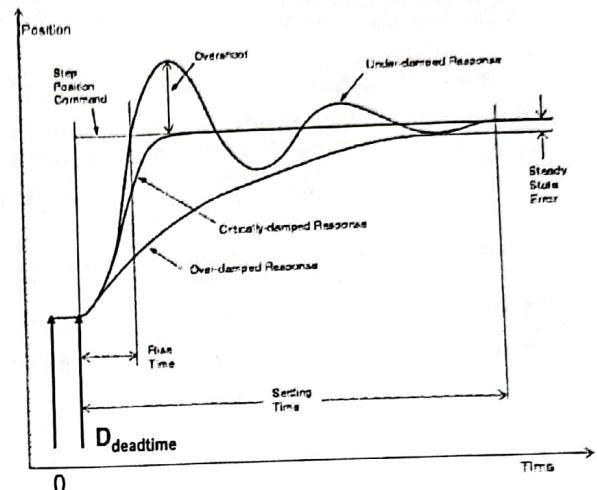
When responding to controllers' commands, changing the old state and reaching the new steady-state is not necessarily a smooth ride. It depends on the plant components, physical characteristics and equations

Given the system characteristics, we could have different type of responses:

1. Overdamped response
2. Critically damped response
3. Underdamped response → Overshoots

Overshooting means that the output oscillates until it settles on the steady-state value.

Imagine using an elevator to go to the 4th floor, but it first goes to the 5th floor, then 3rd floor, then goes to between 4th and 5th, then to between 3rd and 4th before eventually settling on the 4th floor.



Analysis and Design of Control Systems (IV)

Oscillations could be not only inconvenient but also dangerous, if they exceed the plant physical limitations.

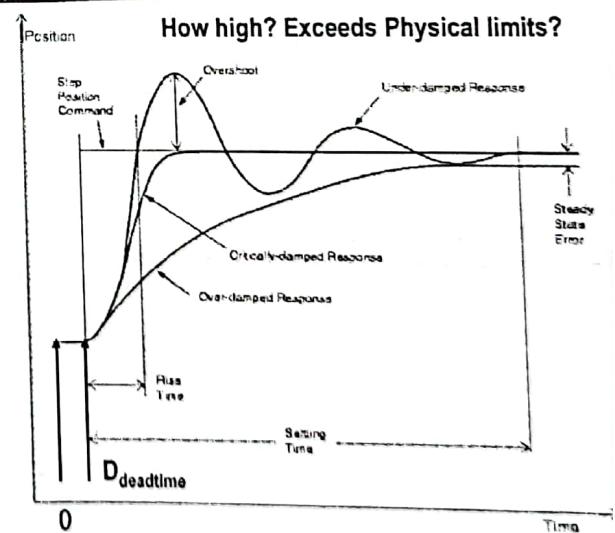
Imagine if the 4th floor is the last floor, if they elevator overshoots, then means it will hit the roof causing damage, injury, or even death.

Temporal characteristics of the physical plant:

Settling time: the time elapsed from the application of an ideal instantaneous step input to the time at which the output has entered and remained within a specified error band.

Rise time: the time required for the response to rise from 0% to 100% of its final value (Ideal Definition), In practice to 90%, or 95%

Dead time: Physical systems do not always respond immediately (e.g., motor). When you send a current to a motor, it takes time to generate a field to overcome the inertia and initial torque to start moving



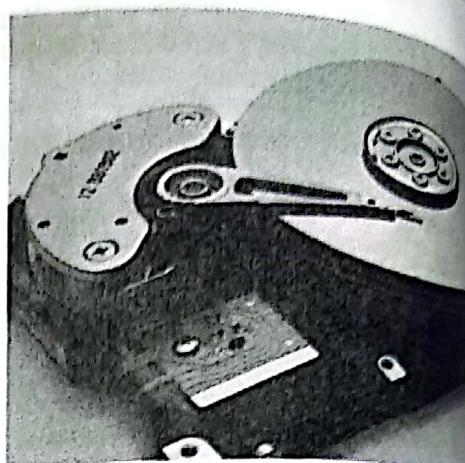
Analysis and Design of Control Systems (V)

Another Example is the HDD

- In a computer HDD, transient response contributes to the time required to read from or write to the computer's disk storage
- Since reading and writing cannot take place until the head stops, the speed of the read/write head's movement from one track on the disk to another influences the overall speed/performance of the computer. The faster the cylinder rotates; the quicker we move the information to the head to read.
- Steady state response: head of a disk drive finally stopped at the correct track → otherwise errors

Think: Why do we rarely see HDDs with speeds more than 7200 RPMs?

Think: What do you think are the error margins for the steady state?



Control System Stability

- Transient Response and Steady-State error is irrelevant if the system is not stable.
- To understand stability, one must understand the system total response.

$$\text{Total response} = \text{Natural response} + \text{Forced response}$$

Natural response:

Any physical system either mechanical or electrical has certain properties governed by physics equations, when it turns on, it responds to initial conditions and no other inputs (no forced inputs)

Examples: brick initially wired onto a spring, charged RC circuit

Forced response is the system response when there is a certain input.

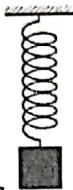
Examples: Adding additional weight to the brick/spring system or opening a switch in the charged RC circuit.

In any stable system, the natural response must decay, if not, and it must not grow indefinitely otherwise the system will become unstable.

A Simple Stability Example

Initially, suppose a certain spring can hold a maximum weight of 7KGs, beyond which, the spring loses its elasticity and cannot return back to its original shape, and is therefore damaged.

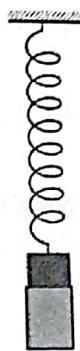
Consider that our system design initially requires attaching a weight of 3KGs. The spring stretches and reaches an equilibrium state. Now the spring system (spring + weight) is said to have a natural response.



During system operation, say we exert an external weight of 2KGs and the spring stretches. Since $2 + 3 < 7$, when we remove the 2KG weight, the system will bounce and return to its former state. The 2KG weight is a forced input which gave a forced response



During system operation, say we exert an external weight of 5KGs and the spring stretches beyond its physical limitations. Since $5 + 3 > 7$, when we remove the 5KG weight, the system will NOT bounce back and return to its former state. The 5KG weight is a forced input which gave a forced response that caused Instability and irrecoverable system damage



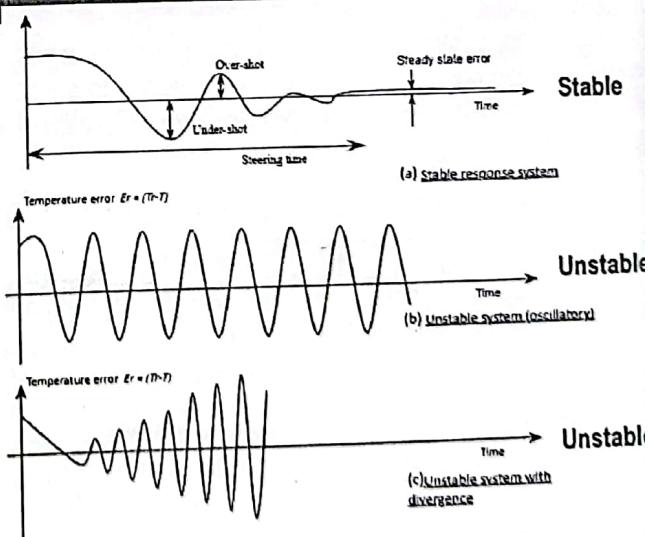
Instability

Instability, could lead to self-destruction of the physical device if limit stops (protection) are not part of the design.

Examples:

- an elevator would crash through the floor or exit through the ceiling;
- an aircraft would go into an uncontrollable roll;
- an antenna commanded to point to a target would rotate, line up with the target, but then begin to oscillate about the target with growing oscillations and increasing velocity until the motor or amplifiers reached their output limits or until the antenna was damaged structurally.

A time plot of an unstable system would show a transient response that grows without bound and without any evidence of a steady-state response.



١١ / ١٥ / ٢٠٢٢

* Control System basics :

١- Embedded System : * Complex * lots of processing * flexible (إنها بترجمة)

٢- Real time System : * hard real system : * every task has deadline (deadline) قبل المدة المحددة ستتم الكارثة → (لَا تم تأخير المهمة قبل deadline) → Examples: Cars or heart .

* Soft real system : * أو تتجاوز الـ deadline مابعد مدة كثيرة ولكن بأقل علاج performance مثل ما وصلت كل الفيديو للشاشة وتاخر جزء منها ، مابعد كارثة بس بتحبس المدة مفيدة تكون لـ ٢٩ او ٢٤

الـ embedded System هو شرط يكون طبقاً لها العلاقة بالـ "Real-time embedded System" : ولو كان إلـ "Real-time" اسمه .

* فيكون المخطوطة : ١- الـ gsm والشبكة (المكونات) ٢- الـ chips للبطارية (embedded)

٣- Control System : * embedded sys * أقدم من الـ sys * Classical Control (Signals and Mathematics) and Digital control → (digital signals) (بيانو شرط دامت تحكم الـ Real-time sys مثل عدد الخطوات .

* Classical Control (Slide 3) : هذو جزء الحيوانات في الغابات →

* أكبر مثال على الـ Classical هو الإنسان الطبيعة أيها مثال على الـ Classical (slide 4) : classical control System because it is Mechanical not digital .

* Laplace transform → بخط أبي محاذاة (أي) معادلة حورية بسيطة .

* Z transform → digital System ولكن الـ Laplace كانت بتتأثر وبتقرب على الـ transistors و capacitors للتحكم من المكافئ .

* فقط يعنى بالـ digital صل في هنا أكثر من الـ Filters وغيرها .

* التكامل Slide 7 ← المركبات الفحصائية في معاياres digital control systems خارج الغلاف الجوي .

* Impulse function : transient (حدث مفاجئ وحدث خاص اخشوقي)

ـ تغير مفاجئ → ((+) ٨)

* Step function : Switch (System (u(t)) . (احنة تشغيل وإطفاء الـ

* Control Systems & L- Open-Loop

2- Closed-Loop

(Slide 10):

- يكون بالعمليات البسيطة في المعايرة - بسيط و سهل التعامل
- open-Loop & input interface يدخل input و يطلع output بدون مانع (feedback) شو يعني
- جوا بالزبط مثل الميكروبيك والغسالة و حمامات الجبن وهذا (ما فيها feedback)
- * الـ input transducer يتربط المدخلات المصيغة التي يقدر يتقبلها System
- Controller تدخل لا

* control Systems Junction بكون + + أو - -

* plant : الاشيء اللي بيعي اتحكم فيه (حياتناها بالـ open-loop)

* له اجزاء عائنة داخلية او خارجية ملبيقت بأدفه او أفعاله بالـ open-loop

(Slide 11):

* Closed-Loops

- قوم أقيسوا إذا لا output طوله يعني صبح أو ظهر

- فيما ، مثل المكيفات اللي فيها Sensor بتقىس حرارة الغرفة و بتضبط المكيف على المناسب

* الـ output transducer يدخل الـ input ليتوافق مع الـ input لتحقق أى فر شو المستهدف

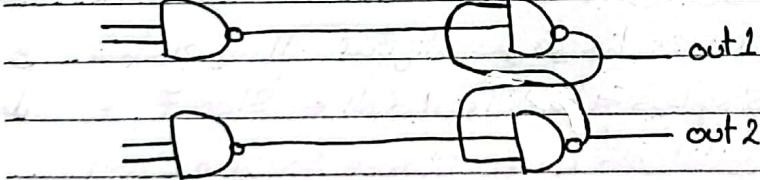
* أو كان فيه علامات - بين الـ output transducer والـ input transducer معندهما في عنا

(+) Positive feedback . (-) negative feedback

* مثل سباق الدراجات في اجتماع على الـ zoom والـ ktc يبحث

فالصوت يغوت ويدفع فونا هكذا يسبينا احنا ازعل لازم بطلع صوت تشويش او تدخين

* مثل جيد على الـ SR-latch % or positive feedback



* مش بالضرورة الـ feedback يكون جاكي من جهة ورقة يمكن يكون من أكثر من

جهة . * الـ closed more complex + hardware

* سؤال افتتاحي لو بيعي أحول التوستر لـ closed-loop شو بعمل بـ Sensor

light ابقيس درجة التحميص مثل Sensor نوع الجبن بـ هنا جيزيد التكلفة لـ انه لازم

مانعفـ Sensor لا نعم بالفاية جوا حمامات هم .

(Slide 15) :

* مثال المحدد: أداة الطابق الأول وبيغي أطلع الطابق الطابق.

- ممكن يطلع المحدد شوي شوي أو بسرعة أو بسرعة متوسطة.

- ال Switch بالعامية Signals Switch.

- لو كان سريعاً كثيراً هماشي هنبح بس جيطلعك بسرعة وهذا مش آمن ولو كان بطيف

كثير انت ربع تتحقق فلام ناخذ كل الأمر يعني الاختبار لنوازن كل شيء بالsystem.

- دقيقه هى بالمية بس لأنهم تكون Steady-state response.

- دقيقه هى بالمية بس لأنهم تكون أقرب ما يمكن للدقة.

(Slide 16) : setting time : قييمبه وقت من دماباش يطلع وينزل بعد ماشت \rightarrow

* في الحالات السيئة للاستجابة أسوأ هذه الحالات هي underdamped \rightarrow بحد المحدد

يطلع وينزل عن النتيجة الرغبانية تافتاك (هذا الطابق الرابع لما تطلع بالمحدد بدخل ينتقل بين الثالث والخامس وهيا).

لهم نحاول تقليل هذا الخطأ قدر الإمكان حتى لو صار مايلش كثيـر الحالة الثانية \leftarrow يكون بطيف كثيـر والحالة الثالثة \leftarrow

Critically \leftarrow وهي أضيقاً و الأقرب للدقة (95%) والعنصر \leftarrow إلى هنا منبع.

* لا dead time \leftarrow الوقت الميت الغير مستعمل قبل ما the system ييشغل منبع.

* ال Rise time \leftarrow الوقت منها توصلت أمر لا Control (عمر الوحدة) للـ Steady-State.

* مثال آخر هو ال Computer HDD underdamped \leftarrow سريـع والمـفـضـل بطـيـعـ

response \leftarrow يعني هو ال Rise time يكون أفضل ما يمكن بس لوعليـتـهـ كـثـيرـ الـ circular motion

time \leftarrow مـمـكـنـ يـقـيـمـتـ القرـصـ ويـخـرـجـ حـسـبـ قـوـاـدـ الـغـيـرـيـاءـ (Slide 20) :

الـ Steady state \leftarrow هـنـ اـذـيـ اـوـمـلـ الـ عـلـيـهـ الـ فـيـلـ الـ يـعـيـاهـ مـعـ القرـصـ (ما فيهـ منـعـ).

ـ Stability \leftarrow معناها إذا الـ System حـيـضـلـ تمامـ طـولـ الـ وـقـتـ ولا مـكـنـ يـحـسـ لـ شـيـ لـ قـدـامـ.

A- Natural Response :

ـ الـ يـتـجـوـيـ فـعـلـ الـ initial states الـ يـمـطـوـطـهـ عـلـ الـ inputs

B- Forced Response : My own input \rightarrow System حالـيـاـ باـيـيـ حالـيـاـ

ـ مـمـكـنـ يـخـرـجـ شـويـ ويـجـعـ ومـمـكـنـ يـخـرـجـ الـ System بالـكـاملـ حـسـبـ القـوـةـ الـ يـخـلتـ

ـ عـلـيـهـ هـنـاـ لوـكـانـ نـهـيـلـعـ حـامـلـ كـيلـ بـسـ وـأـنـاـ شـيـتـهـ وـصـارـ يـهـمـ أـكـثـرـ فـضـلـ

ـ يـتـحـمـلـ لـدـرـجـهـ مـعـيـنهـ بـعـيـنـ يـنـقـطـوـ أوـ يـخـرـجـ بالـكـاملـ.

Temporal Requirements in Computer (Digital) Control (I)

In Classical (Analogue)
Control

Plant Temporal
Characteristics
(settling time, rise time, dead time ...)

In Digital (Computer)
Control

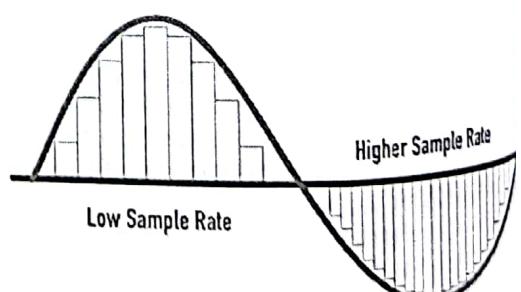
Plant Temporal
Characteristics
(settling time, rise time, dead time,
etc.)



Digital Controller Temporal
Characteristics
(sampling time, processing time,
deadlines, jittering and delays, etc.)

Temporal Requirements in Computer (Digital) Control (II)

- ▶ In a classical control system without digital controllers, the temporal requirements are restricted to analyzing the timing of the physical system (settling time, deadtime, risetime, transient time, etc.).
- ▶ In a digital system, more timing requirements are introduced → e.g., ADC sampling rate ($T_{sampling}$)
- ▶ Some sensors are digital (they have a built-in ADC and give us digital, (mostly serial) output. The programmer selects the output data rate (ODR, i.e. sampled rate) in software.
- ▶ Many other sensors are analogue which must be interfaced to an ADC, correct sampling rate must be chosen.



Temporal Requirements in Computer (Digital) Control (III)

- ▶ Choosing the sampling rate is not as easy as one thinks!
- ▶ Must comply with Shannon theory of sampling and the lower bound is Nyquist rate ($\min 2\pi f$). Sampling at higher frequencies is allowed (though not often necessary) and called oversampling.
- ▶ Sampling below this Nyquist rate is undersampling, and results in aliasing; that is, the correct original signal cannot be reconstructed
- ▶ Between each sample and the next, the controller might need to do online calibration, signal filtering, processing and control. We call this time ($T_{processing}$).
- ▶ $T_{processing} \leq T_{sampling}$
- ▶ In this case, the deadline to finish processing is before the next sample arrives

digital sensor جوال الـ ADC بـSampling Rate *
 كل فترة زمنية معينة بـيأخذ منه لاADC قطعة *

Temporal Requirements in Computer (Digital) Control (IV)

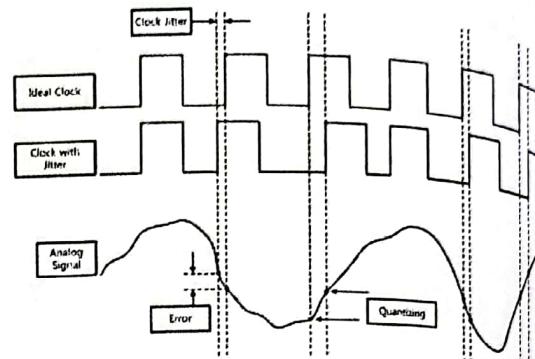
- حالات اخلاص بعد الـ "deadline" التي يراد منك ان تقدر أنتل بشرط.
- ▶ Can the controller finish the required tasks ($T_{processing}$) related to one sample before the new sample comes in? If not, what to do?
 - 1. Reduce the sampling rate to the lower bound if the specification allows you to * $\rightarrow (T_{sampling})$ increases giving us more time.
 - 2. Use compiler setting to optimize your code for speed (be careful as the compiler can remove variables necessary for code operation: Ports) * نكتب code Assembly
 - 3. Optimize your control code by using efficient algorithms or libraries, or in extreme cases, revert to assembly coding.
 - Remember some sorting algorithms are $O(n^2)$, others are $O(\log(n))$. \rightarrow subset of real Num. Algorithm ... نغير الـ
 - 4. If your control code uses real numbers, use fixed-point arithmetic instead of floating-point arithmetic. Fixed-point arithmetic uses integer hardware and is therefore faster. Many DSP and Control libraries are fixed-point.
 - ▶ Suppose sampling rate is 10KHz, processor is running at 50Mhz, the task to process one sample consists of 25000 assembly instructions, and on average, each instruction takes 1.5 clock cycles to execute, could it work?

$$T_{sampling} = \text{Deadline} = 1 / 10,000 = 0.0001 \text{ sec}$$

$$T_{processing} = 1 / 50,000,000 (\text{cycle time}) \times 25,000 (\text{Instructions}) \times 1.5 (\text{CPI}) = 0.0005 \rightarrow \text{More time than the deadline}$$

Temporal Requirements (Jittering Issues)

- Jittering as a word means to suffer from slight irregular movement, variation, or unsteadiness. We have two types:
 1. Jittering Δt due to uneven clock cycles that affect sampling (see adjacent figure)
 2. Jittering $\Delta t_{\text{computer}}$ due to uneven response time to a certain event
- If $\Delta t \ll T_{\text{sampling}}$ considered almost constant, system can handle it!
- If Δt is proportionally significant, that is the clock shifts between cycles is large compared to the cycle itself \rightarrow big problem
- Another source of jittering is $\Delta t_{\text{computer}}$ which is jittering in response time to an incoming event. If event A happens, it can interrupt the processing and call the ISR say in 10μs, but if the current task being processed is of a higher priority than event A, then the ISR of A cannot start until the current task finishes, this could be tens or hundreds of microseconds later.
- Some digital controllers can guarantee a maximum response time, useful to design for the worst case!



Temporal Requirements in Real-Time Control Systems

Some control systems might have real-time requirements. Real-time means:

1. Tasks must finish within a certain deadline
2. Tasks must not start before a certain time
3. System must respond to external events quickly

Timing Requirements (e.g. deadlines) are categorized into:

1. Absolute: response must occur at defined deadlines (an action must happen every 10 ms).

2. Relative: response must occur within a specified period of time following an event (when event A happens, we have 10 at most 10¹⁰s to finish associated task).

Real-Time Systems Classification by Deadline Type:

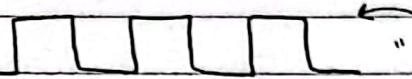
- | | |
|---------------------------|--------------------------------|
| 1. Hard Real-Time Systems | Strictest Deadline Enforcement |
| 2. Firm Real-Time Systems | ↓ |
| 3. Soft Real-Time Systems | Relaxed Deadline Enforcement |

Slide 28 Temporal Requirements in Computer Control:

لهم حبلى من تضييف خواص Control قبل كنائحي بـ Signal المدخلية

* بالـ ADC يذكر على الـ Signal frequency

* أي اتجاه لها : phase , frequency و amplitude

* Max frequency component is *  "Squarewave"

لهم يقسموا Sin waves لـ Sin wave ويشوف بين أكبر وحدة فيه وهي تسمى Max frequency

والباقي ينطلق على frequencies Fourier Series هو الـ (T)

Transform \leftarrow Max Frequency \rightarrow Sensor أبوجع الـ Cos waves أو Sin waves هي عبارة عن مركبة من signals

* كل signals يتطلبون Processing خلال الوقت المحدد فلو كانت signals مكتوبة بالـ codes ممكنها ما تقدر تعمل الـ Processing بشكل صحيح ويهمنا باختصار data استكمال سريع وهذا

* D sampling \geq D processing . (high sample Rate) سرعه اهتمان

لـ (D sampling) : آخر استوي (Slide 26) : يفترض عليه من 2πF (ما أزيد الـ 2πF)

$$\text{time} \leftarrow D_{\text{Sampling}} = \frac{1}{\text{Number of instruction}} \cdot \frac{1}{10 \text{ kHz}} \cdot \frac{1}{10,000}$$

$$D_{\text{Processing}} = \frac{1}{50,000,000} \times 25,000 \times 1.5 \rightarrow \text{each inst. takes } 1.5 \text{ cycle}$$

* إذا كان الـ D sampling أقل يزيد الـ D processing بـ (أقل اشتراك)

- D processing (clock cycle, optimization) instruction عدد الـ clock (وهذا يعني تقليل

Sampling frequency) لو حندي Signal الأعلى حندي Nyquist's

Slid 28 . 400 Hz

* الـ clock عادي مثل idle هنا يقدر ناخذها بشكل دقيق فهذا يعني إزاحت يمين

و شمال (clock with jitter) (AT)

* الـ first source الأول الـ jitter كروبي مادخلنا فيه ، أما الـ second source الثاني حاسبي

من الكمبيوتر نفسه من 50% off idle \leftarrow idle

* اذا عمل Sampling بشكل كبير بكل مرة والـ jitter مخذوقت كثير هؤلا يكون في مشكلة

* الـ deadline هو انه متوجه Sample ثانية * ممكن الـ System يبتلا بـ jitter

* اذا يجيء interrupt وبيستجيب له كل مرة يستجيب بوقت معين بـ (worst case response time)

* الجيد انه يعني controllers يتحصلوا على ما يجيء

Real-Time Systems Types (I)

1. Hard Real-Time (HRT) Systems

- Under all circumstances, ALL 'hard (critical)' tasks MUST meet ALL their deadlines.
- If not, system failure causes catastrophe or death.
- Imperative that responses occur and associated tasks finish within the required deadline.
- Response after deadline has no value!
- ▶ Guaranteed services required → functional correctness and timing correctness (response is worthless if you finish before deadline with the wrong result)
- ▶ Hard Real-Time Systems must be PREDICTABLE and DETERMINISTIC. Systems must always behave in the same way.
- ▶ Analysis of estimated worst-case time → Scheduling algorithm and system must pass schedulability test
- ▶ In practice, the time bounds for HRT ranges from microseconds to milliseconds.
- ▶ Deadline does not necessarily imply "imminency" → Hard real-time task does not need to be completed within the shortest time possible → Only within the bound and before the deadline

Real-Time Systems Types (II)

2. Firm Real-Time Systems

- Tasks missing their deadline **will not** result in a system failure, and no catastrophe or death
- But, Infrequent misses lead to **performance degradation (loss of QoS, quality of service)**
- Response following a deadline has no value (e.g., Video frame processing)

3. Soft Real-Time Systems

- Deadlines desired to be enforced, but they are not strict. (**Best-effort service** → deals with average response times)
- Frequent deadline misses do not cause errors, but the result of the task **might** no longer be as useful.
- Response following a deadline is not wasted, but degrades as more time passes.
- Usually specified by some probability? What is the probability that task A misses its deadlines 10% of the time?
- Probabilistic analysis → complexity at design time!
- Time bounds between fraction of a second to few seconds (example is Home IoT, if room temperature reading misses its deadline, no big deal)
- ▶ Complex real-time control systems could consist of subsystems of any of the three types (airplane control systems)

* Real time Means:

١) Deadline task قبل الـ deadline يكون كل مillisecond بحيس اشي (زي كل millisecond يعني) :
ثابت، كل وقت معين بحيس اشي (زي كل millisecond يعني) : ① اشي.
كل ما بحيس اشي لازم يحصل خلال وقت (عومنا ما يلش) ② من يوم مليجي event معاك
millisecond إنك تعلم اشي.
بـ input مثل اعش millisecond من وقتوها

٢) ما لازم الـ Task الثانية تلش قبل ما تخلص اللي فعلوا .

٣) استجابة interrupt تكون سريعة .

* Example: int x = 1; x, y, z Compiler هو روح يكتب جزء مكان الـ
int y, z = 1, m; فقط لما الـ m ما مستخدمو بس بالليمبيس هذا
y = z + x يمكن يسبب مشكلة لأن ملت المقادير بـ الـ hardware
cout << y فالـ Compiler مو فاهم إنه مثلاً كان لأنـ m يجيز ١٠ مكان.

* Imminency (السرعة)

• لازم ما يخلط بين السرعة وبين الـ deadline task قبل الـ deadline . (يعني عادي)

* Soft Real time → best effort Service .

* Firm Real time → مثل على الـ digital ، لازم يخلص قبل ما يجلس المشاهد بفرقة .

* Functional Requirements : « شوبينا الـ System يعمل »

A- Data collection .

c- Plant / process control .

B- Signal conditioning .

D- (optional) Alarm monitoring / Data logging
/ Man Machine interaction .

* الـ noise مثل كبركش وما يقرب لـ the current أو العرض حتى وجود العرض عادي

عندي في الـ digital أما بالـ Analog ما يقدر نشيل الـ noise فـ ما يقدر أعرف القيمة الأصلية للـ Analog .

الـ noise ممكن تكون استعلامات أو Temperature أو غيرها .

الـ noise ممكن تختلفوا بين ما بتتشيلوا لو اثنان .

انتهينا الـ digital فإنه حتى هو وجود الـ noise بنقدر نعرف القيمة الأصلية لـ الـ noise عادي إلا

وـ ١ ، يمكن بنقدر ن Detect the noise بالـ error detection and correction .

ماهنة المعايير الـ Control Systems *
 نحن انه بنتبه للـ Dependability dia timing وار Functional req.
 وار Robustness وار Performance وار plant or ADC

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Control Systems Requirements I

A. Data Collection / Acquisition

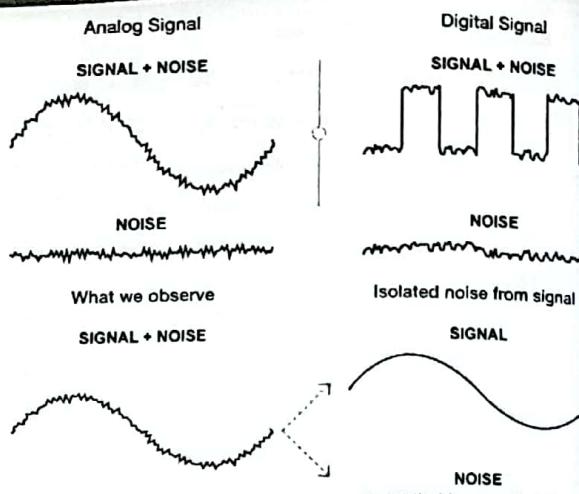
B. Signal Conditioning

Signal conditioning is used to refer to all the processing steps that are necessary to obtain meaningful measured data of plant from the raw sensor data.

- Sensors produce raw data! (e.g., voltages, currents).
- Scaling to required values (e.g., sensor voltage to input voltage of port pin)
- Inherent Measurement errors (e.g. A/D Quantization).
- Sensors values also need calibration.
- Noise effects → Must be filtered out (Anti-Aliasing filters, DC Filters, Digital Filters, etc.)

C. Plant / Process Control

D. (Optional) Alarm Monitoring / Data Logging / Man Machine Interaction



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Control Systems Requirements II

1. Dependability Requirements: بعد ما يمر وقت معين هل حيكلن دا

A- ► Reliability (survivability): probability that a control system will provide the plant control task until time t , given that the system was operational at $t = t_0$

$$e.g. R(t) = e^{-\lambda(t-t_0)}, \text{ where } \lambda \text{ is constant failure rate in failures/hour}$$

كلما كانت أوقاف كلها على بحال طيارات Ultra-high reliability when $\lambda < 10^{-9}$ (automotive-avionics)

السيارات إلخ Mean Time To Failure (MTTF) = $1/\lambda$ more reliable

If a car company produces 1 million cars in one year, and on average each car drives two hours a day each day of the year, and only one car fails during this whole year:

$$\text{total hours} \leftarrow 1,000,000 \times 2 \times 365 = 730,000,000 \text{ hours}$$

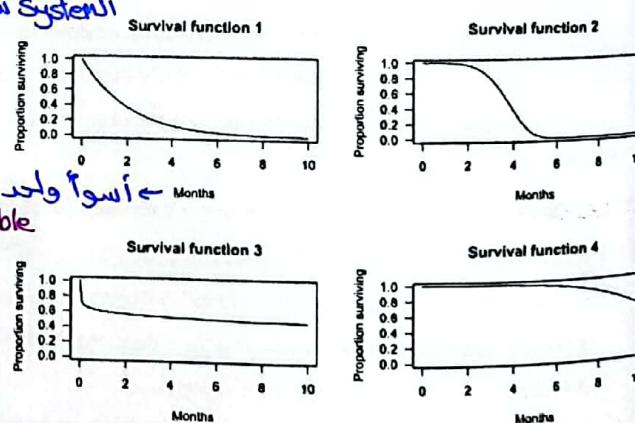
$$\frac{1}{730,000,000} = 1.36 \times 10^{-9}, \text{ which means that the reliability } \lambda \text{ is close to the order of } 10^{-9}$$

قريب جداً من الـ Standard

للسيارات الرئيسية 1 وهذا

معناه انه Reliable

* لوميل فالك خربت معاً سارة فتنقسم 1000 على 730,000,000. بنسوى يعنيها لو كانت Reliable ولا لا.



لـ أعلى الـ survival دا

ذلك شغال منبع قربة الـ 6 أشهر يعني بلش ينزل شوي.

* لويني أخطاء معينة يكتشفها

(detection + correction) (parity bits, ecc codes)
System (لويني يتصفح بعدها critical failure)

Control Systems Requirements III

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1. Dependability Requirements (continued):

تحدد حسب معيينه
موافقنا بخدمتها.

- B- ▶ **Safety:** defined as 1. responses to protect the system from harm (e.g., error detection)
2. reliability against critical failure modes (e.g., plane crash, self-driving car accident)

Safety requires certification

For example, safERTOS is a certified safe version of freeRTOS

components must successfully pass certain tests like the FCC, CE, EMC

Must comply with certified industry safety standards (e.g., aviation or automotive safety standards)

C- ▶ **Fault-tolerance:** protection from design and operational faults? How? → odd Number of sensors

Hardware redundancy → e.g., Two lock-step processors in tandem (e.g., ARM Cortex-R processors) / multiple sensors
In software, roll back/recovery and checkpoints (similar to computer games) however; in hard real-control systems:

1. Difficult to guarantee a deadline when error occurs → roll-back and recovery can take unpredictable time.
2. The error could have caused irrevocable action (remember we are connected to other hardware which affects the plant controlled)
3. Temporal accuracy of the checkpoint data is invalidated by passage of time

D- ▶ **Security:** protect system from intentional harm or access

ما يحير
هكـ.

يعملوا نفس العمليات

كل القطع ولازم يتطابقوا ليكونوا
سلبيين ١٠٠٪

مثل الطائرة التي وقفت لأنها في
25 sensors و خربانس

أحدى الحلول أنه يتحقق ذلك قبل ما يخرب
بعض الألعاب بين حل هعب وأساساً هيكل الغلط هار وبيث وأثير حتى لورجت لا data قيمة

Control Systems Requirements IV

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يستجيب ويعدكم الشفف
بوقت سريع

2. Performance: timing of responses or throughput necessary? →

3. Robustness: protection from external interference and perturbations

(مثل الروبوت اصطدم بالهواء لازم نعمله يكون قادر
على حله وما يخرب).

Must remain at 30m



4. Scalability: Perform reasonably in an environment with added load. We have a swarm of three drones doing a task, and we add more drones to help to complete a task that the three cannot do on their own.

مثلًا لو روبوت اجا يدفع قطعة لقلها ثقيلة
بنادي روبوت ثاني (تفعل الروبوتات مع بعض)

Fail-safe and Fail-Operational Control Systems

- يجرب أن يعود إلى معياره **fail-safe system** عندما ينجز المهمة
- يابوتفقاً إذا أخترعنى fail كيف بيبي
- Some control systems can have safe states (fail-safe) → When system fails, go to safe state
- Examples:**
- Electronic → Simple Electric Fuse
 - In control and embedded software → Watchdog timers
 - In Nuclear reactors → Magnetically held lead rods
- لـ الهايسي يعتمد كل النيونات التفاعل معه و
- * Requires high error-detection coverage → the probability that an error is detected, provided it has occurred, must be close to one ما يطفئ يعلم restart
- لـ الهايسي يعتمد كل النيونات فـ كل مدة معينة هناك ما يطير
- لـ الهايسي يعتمد كل النيونات فـ Overflow في Fuse (أصعب من الأ
- In certain applications, you cannot identify a safe state!
 - Example: Flight control system of airplane or space craft!
 - Must provide minimum level of service to avoid catastrophe even if failures occur, or sound alarms.
 - If main power shuts down, switch to auxiliary power in hospitals.
 - These systems are called fail-operational
- لـ الهايسي يكون fail يعني الطباعة مثلًا لا يقدر بـ كل المشكلة أو هيل.
- الفرق الأساسي بينهم أنه هنا لـ the system يخليش النوعية من المشاكل التي
- يعني Mechanism معيته يانحططني Alarm تبقي ساعي
- Restart فـ
- يتحم الشغل فـ الفرق الأساسي بينهم أنه هنا لـ the system يخليش
- يتحم الشغل فـ النوعية من المشاكل التي

Control System Block Diagram Exercise

The control of the recording head of a dual actuator hard disk drive (HDD) requires two types of actuators to achieve the required high real density:

- The first is a coarse voice coil motor with a large stroke but slow dynamics
- The second is a fine piezoelectric transducer (PZT) with a small stroke and fast dynamics.

A sensor measures the head position, and the position error is fed to a separate controller for each actuator.

Draw a block diagram for a dual actuator digital control system for the HDD.

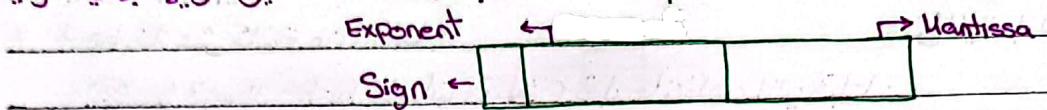
مواصفات للـ block diagram لـ hard disk او Sensors ، تصميم المخرج

* جزء أساسي من الـ Control System ADC

* Sensors (few Cents) أصغرها تتراوح بين المئات المدروجات feedback

* Slide 26.8 Analog/digital وبنطاقها عاليتين

* Computer floating point الـ بحسب المطريقه وقت كثير لنقريها برمجي الطريقة.



* بالـ Python والماثلاب يمكن معرفة direct ازواجا "double"

* لا يدرك تعرف بالـ double / float بداخل وقت تحويل مساحة power Control system

* زويمما أخذنا بالـ logic Fixed-point

* Ex: → 1 0 0 0 1 1 float لـ ملحوظ يعتقدون بتعريف الـ double

$$= 8.75 \text{ لـ double يشتمل على وقت طول.}$$

* FPU (Floating point unit) مشتملة على بقية كل المعايير Control system float/doubles hardware

* الـ Fixed-point بخلافه تعريف الـ float/double لكنه أصبح يستخدم لذلك إلا أنه

* أسع ، ويكتنأ تحويل الـ float/double ، الـ صيغة الـ Fixed-point باستثناء

* المكتبات الخارجية في الـ C وـ C++ . (هو أسع لكنه يستخدم الـ integer hardware)

* CMSIS DSP ← مكتبة خارجية ← f32 ← الـ floating point الـ (طولية)

* كل نفس الوظيفة ليس هي كممثل والفرق بينهم " → (32 bits) Fixed-point - 931 - (16 bits). Fixed-point - 915 - Parallel " .

* لم استخدموها بقدر أعلى Storage Processing بـ 931 ، الـ 9.31 أدق بـ 931 بـ 32 bits .

* الـ 64 bits بـ 32 bits . الـ floating point بـ 32 bits .

* (Self learning Material 2) Signal to noise Ratio ← SNR *

* ← لوبيي أحظل شيئاً منيح بـ مناسب .

* الـ Amplitude 2 frequencies نفس بعض تكون الـ Amplitude على أنها لوكان

* الـ Amplitude يكون الـ 2 frequencies أدق .

* Proportional-Integral - ← PID controller *

* derivative controller

* convolution ←



Computer Controlled Systems (Digital Control)

University of Jordan

School of Engineering

Department of Computer Engineering

Material prepared by Dr. Ashraf Suyyagh

Review of Signals, Fourier Transform, and ADCs

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In embedded, cyber-physical, and control systems, the controller interfaces with the real physical world to either sense or actuate the plant. In feedback control systems, sensors are attached to the plant to provide the controller with measurements about the state of control to enable the controller to tune its actuating/control signal to achieve the desired output. The physical plant could be interfaced to either analogue or digital sensors. Analogue sensors must be interfaced to an intermediate ADC before their value is used by the digital controller/computer. Digital sensors have their ADCs built-in, and through software they are configured to provide data at a given output data rate (ODR); that is provide data at a certain sample rate.

* إذا زلنا عن Nyquist بحiss underSampling وهذا مشكلة .

The Sampling Rate

* لا يعطيني الـ Nyquist لـ Sampling frequency المناسبة للـ

Per Shannon's Theorem of sampling, we must sample the analogue signal at a minimum signal sampling frequency. This frequency should be at least twice the maximum frequency component that exists in the finite bandwidth signal. This sufficient sampling rate is called the Nyquist rate. Sampling below this rate is called undersampling and this means we did not capture the entire information (e.g., shape) of the signal, and thus when we recreate the original signal, it will be distorted. We call this problem aliasing. Sampling at above the Nyquist rate is called oversampling. A signal is said to be

لـ أخذ قدرات أكثر resolution قليلاً مثل bit ما تكافئ أخذ قدرات قليلة resolution على

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تكلفة

Page 1 of 15

* لـ الـ 200Hz = Nyquist ← الـ 100Hz = Signal ← هون الشج بوكھلak
كيف لو الاستغلت rates أعلى وليه مرات بشغل علوي .

oversampled by a factor of N if it is sampled at N times the Nyquist rate. Oversampling can improve resolution, signal-to-noise ratio (SNR), and simplify building anti-aliasing filters. We shall discuss the first two benefits of oversampling:

Reducing SNR →

الـ Signal من الـ noise نجح أكثر

When we oversample by a factor of N , the SNR improves by a factor of \sqrt{N} . Suppose we have a signal with a maximum frequency of 100 Hz; this means the minimum sampling frequency should be 200Hz. If we sample the signal at 800 Hz, this means that we oversampled by a factor of 4 and improved the SNR by a factor of $\sqrt{4} = 2$.

* استخدامي لـ Samples أكثر بعده بـ SNR خير دار

Improving Resolution

ADCs output is a digital set of binary bits. ADCs can have either a fixed-width output (You buy them with only one fixed output width), or configurable width output (they can provide different output widths that you configure in software). ADCs nominally have 10, 12, 14, 16, 20, 24, 32 bits among other configurations. High-resolution ADCs are more expensive, slower in converting an analogue sample to digital representation, yet more accurate and precise. ADCs also vary at their maximum sampling speeds that range from kSPS (kilo samples per second) to tens or hundreds of MSPS (mega samples per second). ADCs with GSPS (giga samples per second) also exist but are often used in high-speed communication systems. The higher the digital resolution and the higher the sampling rate, the more expensive is the ADC. ADCs could cost from few cents to hundreds of dollars.

Oversampling is a technique that allows the use of lower-resolution ADCs to give a comparable performance of higher-resolution ADCs. That is, we can sample our data at many times the Nyquist rate using a 10-bit ADC to give similar performance of a 12-bit ADC.

Formally, the extra number of samples we need to take to increase the resolution by one bit n is:

$$\text{No. Samples} = 2^{2n} \quad (1)$$

So, if we have a 16-bit ADC, the number of samples we need to have a resolution similar to that of a 18-bit ADC is $2^{2x2} = 2^4 = 16$ times the samples. So, if our signal has a maximum frequency of 1000 Hz, the Nyquist Rate is 2000 Hz. We can either use an 18-bit ADC at 2000 Hz, or a 16-bit ADC at 32,000Hz.

As we learnt in class, choosing a sampling rate is not an easy task. It should at least meet the Nyquist rate for that sampled signal, and the time between the sample and the next should be sufficient to finish sample processing (e.g., calibrating, filtering, control, etc). Yet, we have just learnt that to reduce the cost of using a more expensive high-resolution ADC and increase the SNR, we can use a cheaper lower-resolution ADC but with oversampling. But oversampling means taking more samples with less time in between making our deadline constraints stricter.

Exercise 1

In a certain control application, suppose that the plant is connected to an analogue sensor whose signal is fed back to a digital controller through an ADC. The sensed signal has a maximum frequency component of 15 kHz.

Given the following ADC specifications, answer the questions that follow:

	Cost	Resolution	Max. Samples
ADC 1	5 \$	12-bit	24 kSPS (kHz)
ADC 2	6 \$	12-bit	50 kSPS (kHz)
ADC 3	8 \$	12-bit	1 MSPS (MHz)
ADC 4	10 \$	14-bit	50 kSPS (kHz)
ADC 5	14 \$	16-bit	100 kSPS (kHz)

- Suppose that it is sufficient for the control application purposes to use a resolution of 12-bits, which one of the ADCs would you use?
- Suppose that our control application requires processing data samples that have a resolution no less than 14-bits or lower resolution with *equivalent* performance. Which one of ADC3, ADC4, or ADC5 would you choose assuming that the control task time will always finish by the deadline?
- Suppose that the control task takes 0.00001 seconds on the digital controller used in our system, and that our control application requires processing data samples that has a resolution no less than 14-bits or lower resolution with *equivalent* performance. Which one of ADC3, ADC4, or ADC5 would you choose?

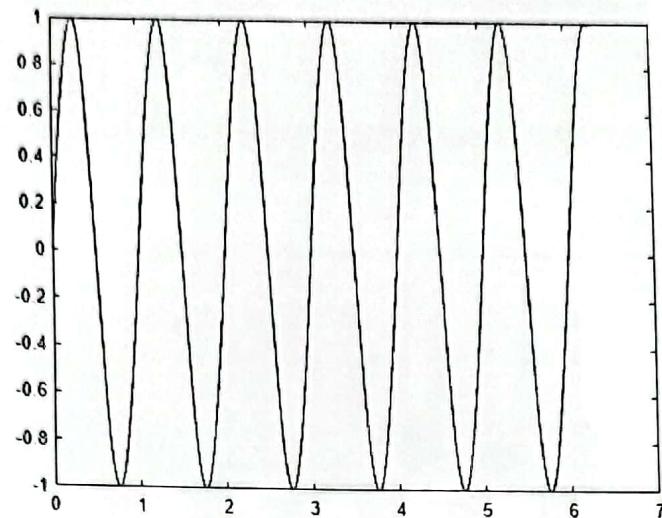
Review of Periodic Signals and Basic Analysis

As we have seen in the previous section, all our decisions fundamentally require us to have some knowledge of the properties of the signal that we want to sense, more critically we need to know its bandwidth (BW); that is the **maximum frequency component** that exists in the signal.

لـ *هـيـ الـسـاسـيـ لـأـفـرـقـةـ الـنـاقـصـيـ وـكـلـ اـشـيـ*

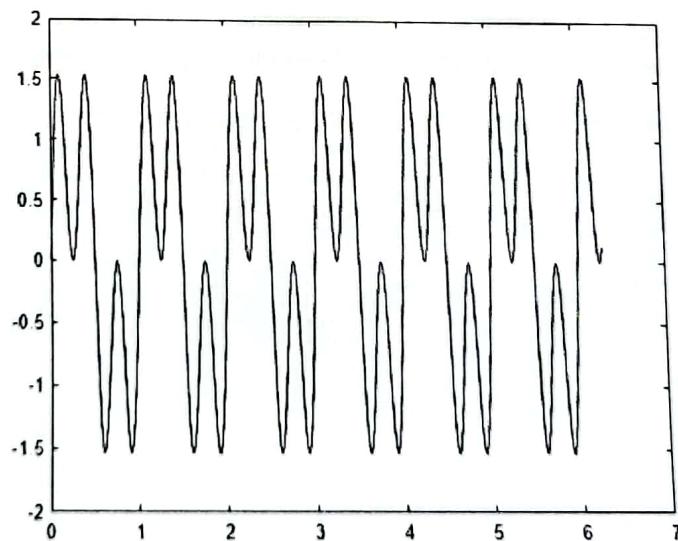
As you have studied in the *Signals and Systems* course, periodic signals can be constructed from the summation of sinusoidal waves. For example, a **periodic square wave** can be constructed by adding the odd harmonics of a sine wave, the more harmonics we add, the closer we get to a square wave. Let us draw a sine wave over the range of 0 to 2π with steps of 0.01 and use MATLAB's plot command to view the signal:

```
Ts = 0.01;
x = 0:Ts:2*pi;
y =sin (2*pi*x);
plot (x,y)
```



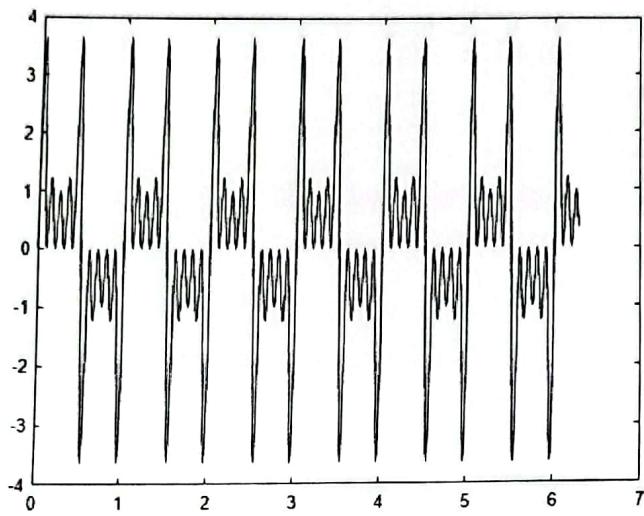
Now, let us add the third harmonic, that is a sine wave with three times the original frequency and plot the signal again:

```
Ts = 0.01;  
x = 0:Ts:2*pi;  
y =sin (2*pi*x) + sin(2*pi*3*x);  
plot (x,y)
```



Now let us add the fifth, seventh, and ninth harmonics, and plot the signal again:

```
Ts = 0.01;
x = 0:Ts:2*pi;
y =sin (2*pi*x) + sin(3*2*pi*x)+ sin(5*2*pi*x)+ sin(7*2*pi*x)+ sin(9*2*pi*x);
plot (x,y)
```

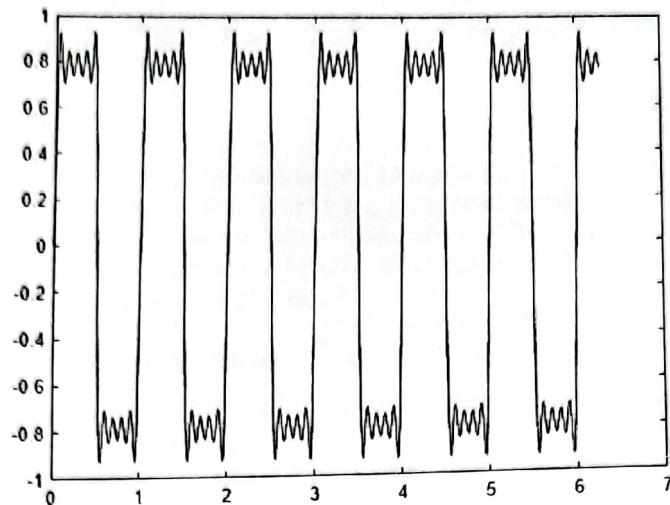


"لقد همن فصلنا لا نه كذا نغير بس عال"

As you can see, the square wave that we got does not look like as the square wave we expect it to be; this is because we changed the frequencies (harmonics) but not the amplitude of the additional sine waves. Normally, we can construct different waveforms from sinusoidal waveforms through changing **frequency**, **amplitude** and the **phase** of the underlying sinusoidal waveforms.

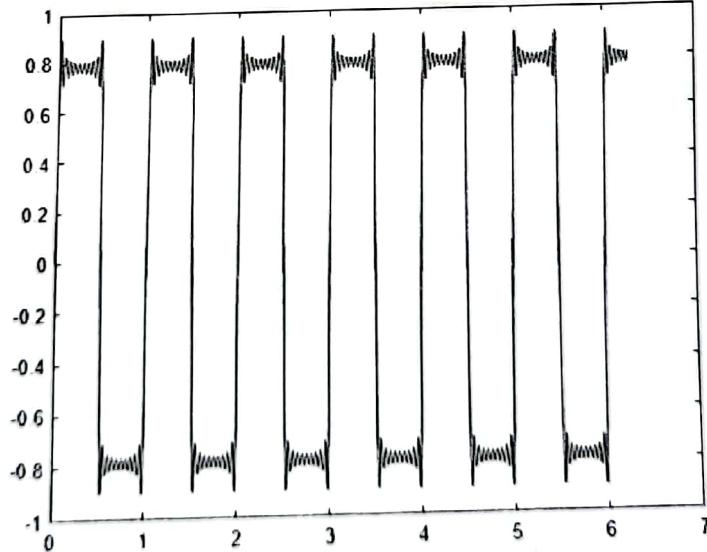
Now let us add the fifth, seventh, and ninth harmonics with varying amplitudes, and plot the signal again:

```
Ts = 0.01;
x = 0:Ts:2*pi;
y =sin (2*pi*x) + (1/3)* sin(3*2*pi*x) + (1/5)*sin(5*2*pi*x) +
(1/7)*sin(7*2*pi*x)+ (1/9)*sin(9*2*pi*x);
plot (x,y)
```



The more harmonics we add, the closer we get to a square wave:

```
Ts = 0.01;
x = 0:Ts:2*pi;
y = sin (2*pi*x) + (1/3)* sin(3*2*pi*x)+ (1/5)*sin(5*2*pi*x) +
(1/7)*sin(7*2*pi*x)+ (1/9)*sin(9*2*pi*x) + (1/11)*sin (11*2*pi*x)...
+ (1/13)* sin(13*2*pi*x)+ (1/15)*sin(15*2*pi*x)+ (1/17)*sin(17*2*pi*x) +
(1/19)*sin(19*2*pi*x);
plot (x,y)
```



Do not concern yourself at this stage how we got the correct amplitudes and frequencies of the square wave. We simply got them from an equation from a book. You will always be given any equations if necessary.

The Fourier Transform

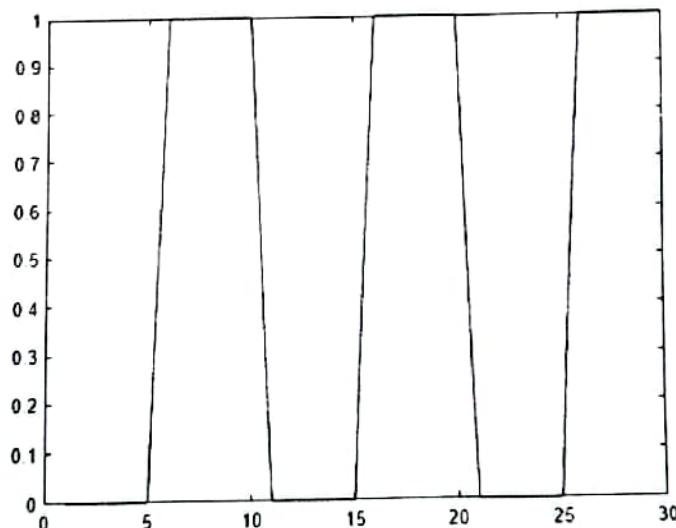
In the previous section, we have seen the underlying characteristic that periodic waveforms can be constructed from the summation of multiple underlying sinusoidal waveforms. This is fundamental to understand the basics of Fourier transform. But initially, let us review the basic of convolution. You learnt that in convolution, we flipped one of the signals, then started sliding it onto another signal while multiplying and summing (or integrating) them together.

Quick Visual Review of Convolution

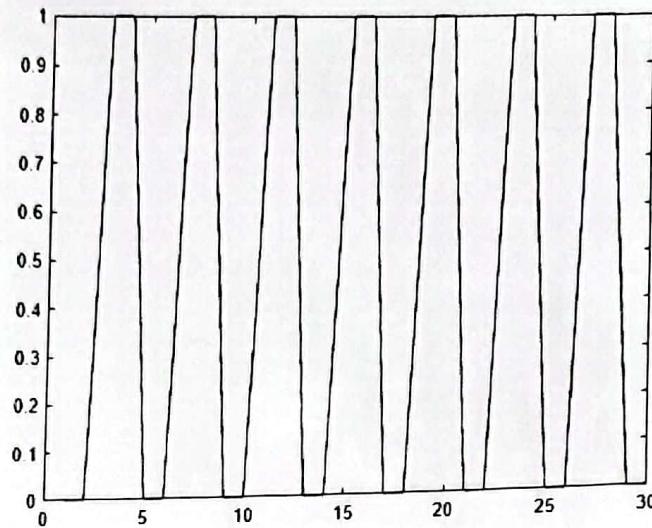
أهم العمليات بالهندسة = Signal و وحدة ثانية
يعمل وحدة معنوم وبهرين أمشيغ ذوق بعض واخرهم
أو جمعهم أو هكذا!

Suppose we have these two waveforms which differ in their frequency. To plot each of the plots in a separate window, use the figure command

```
x = 1:30;
y1 = [0, 0, 0, 0, 1, 1, 1, 1, 0, 0, 0, 0, 1, 1, 1, 1, 1, 0, 0, 0, 0, 0,
1, 1, 1, 1, 1];
y2 = [0, 0, 1, 1, 0, 0, 1, 1, 0, 0, 1, 1, 0, 0, 1, 1, 0, 0, 1, 1, 0, 0, 1, 1, 0,
0, 1, 1, 0, 0];
figure
plot (x, y1)
```

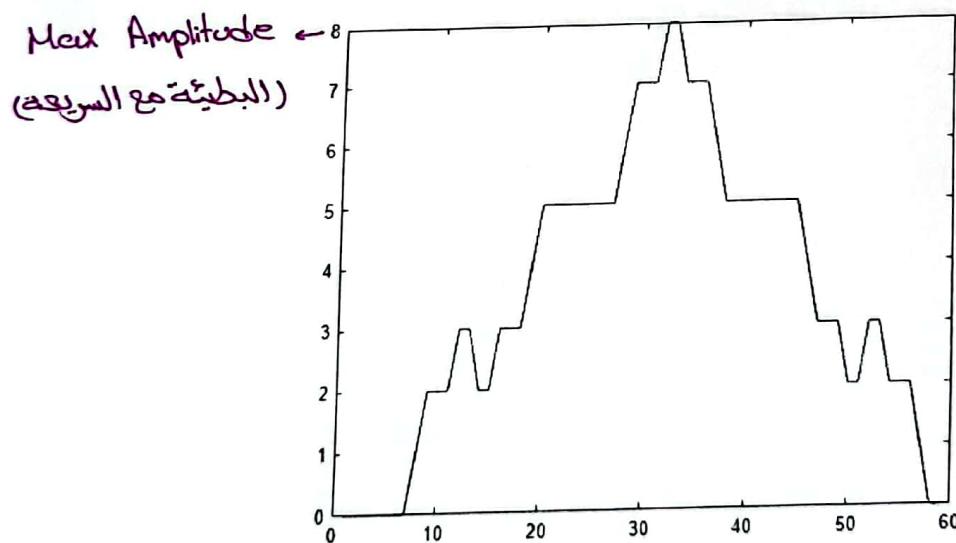


```
figure  
plot (x, y2)
```



Now, let us do a convolution between the two different signals and plot the result:

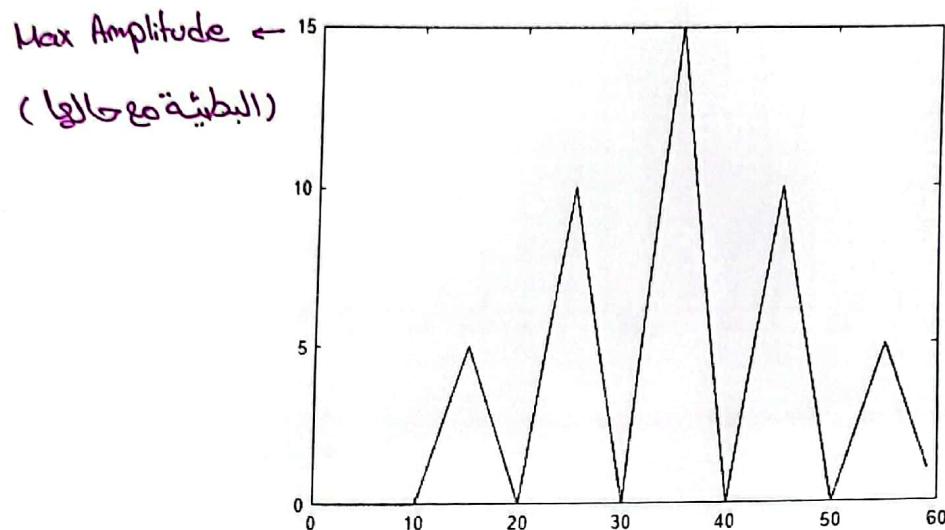
```
z = conv (y1, y2);  
plot (1:length(z), z);
```



Notice that the frequency of y_1 is different from y_2 , and the maximum amplitude in the convolved signal is 8.

Let us do a convolution between the first signal $y1$ and itself and plot it on a new figure:

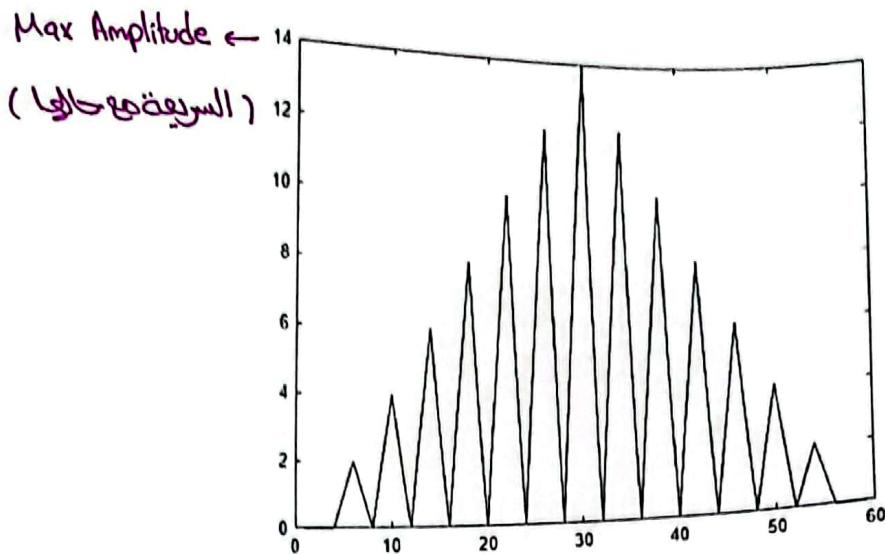
```
figure  
z = conv (y1, y1);  
plot (1:length(z), z);
```



Notice that the frequency of $y1$ and the convolved signal $y1$ (obviously), and the maximum amplitude in the convolved signal is 15.

Let us do a convolution between the second signal $y2$ and itself and plot it on a new figure:

```
figure  
z = conv (y2, y2);  
plot (1:length(z), z);
```



Notice that the frequency of y_2 and the convolved signal y_2 (obviously), and the maximum amplitude in the convolved signal is 14.

What we want to emphasize from the previous discussion is that we obtain the maximum amplitude when we convolve the signal with itself, that is we are convolving a signal at some frequency with itself at the same frequency, and the resulting signal has a higher amplitude than if we convolve a signal with another signal at a different frequency. This forms the basis of the Fourier transform.

The Fourier Transform

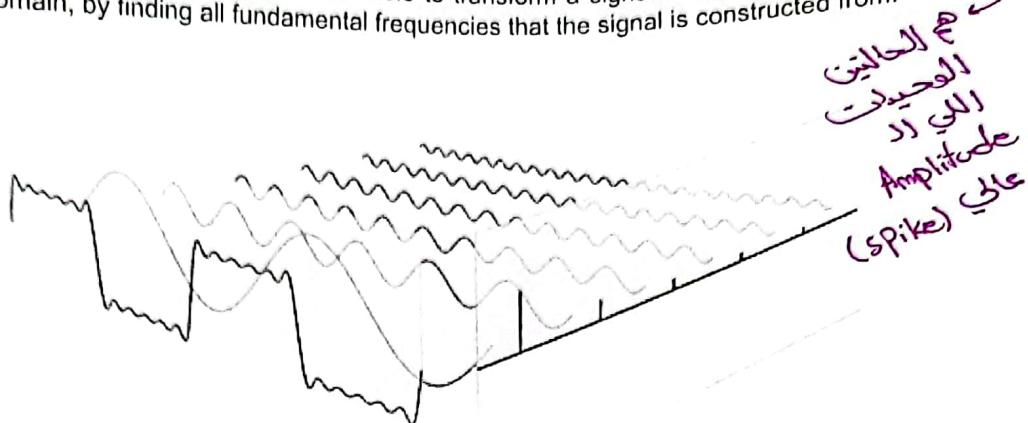
The Fourier transform is a mathematical formula that transforms a signal sampled in time or space to the same signal sampled in temporal or spatial frequency. In signal processing, the Fourier transform can reveal important characteristics of a signal, namely, its frequency components. Take a look at the definition of the Fourier transform in the continuous-time domain:

$$\text{Signal } \xrightarrow{\text{Convolution}} \text{Convolution of } x(t) \text{ with } \sin/\cos \xrightarrow{\text{Fourier Transform}} X(\omega) = \int_{-\infty}^{+\infty} x(t) e^{-j\omega t} dt \xrightarrow{\text{Fourier Transform}} \sin + j \cos$$

What it basically means, is that we are doing something similar to a convolution. We are taking a signal $x(t)$ and convolving it with sinusoids $e^{-j\omega t}$ at virtually all frequencies from $-\infty$ to ∞ . Remember that $x(t)$ is a periodic signal, and all periodic signals can be constructed from the summation of sinusoids at different frequencies, phases, and amplitudes. So, what happens with the Fourier transform, is that since we are integrating from $-\infty$ to ∞ , it means we are scanning for the underlying sinusoids at all different frequencies.

Suppose our signal $x(t)$ is constructed from the summation of sinusoids at 30, 40, and 80 Hz. The equation of the Fourier transform means that we will convolve this signal with sine waves at all frequencies. So, if the frequency is 1Hz, and we convolve it with the signal, it will give a certain amplitude, same thing for a signal of 2Hz, and 3Hz, and 10 Hz. But when we integrate $x(t)$ with a sine

wave of frequency of 30Hz, since the signal $x(t)$ is constructed from a sine wave of 30Hz, here the convolution will give a high amplitude, or a spike. The same thing happens with the 40Hz and 80Hz cases. This is how the Fourier transform is able to transform a signal from the time domain to the frequency domain, by finding all fundamental frequencies that the signal is constructed from.



You can watch this video for a more visual explanation.



Fourier Transform in MATLAB

The **fft** function in MATLAB® uses a fast Fourier transform algorithm to compute the Fourier transform of data. Consider the square wave that we constructed earlier from the summation of multiple sinusoidal waveforms (first three terms)

```
Ts = 0.01;
x = 0:Ts:2*pi;
y = sin(2*pi*x) + (1/3)*sin(3*2*pi*x) + (1/5)*sin(5*2*pi*x);
```

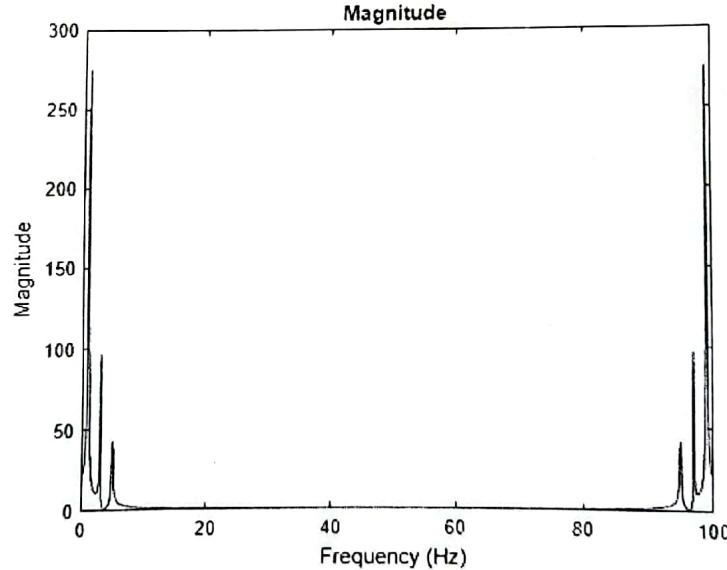
We can compute the Fourier transform of the signal and create the vector **f** that corresponds to the signal's sampling in frequency space.

```
z = fft(y);
fs = 1/Ts;
f = (0:length(z)-1)*fs/length(z);
```

When you plot the magnitude of the signal as a function of frequency, the spikes in magnitude correspond to the signal's frequency components of 1 Hz, 3 Hz and 5 Hz. Yet, since the output of the **fft** function is not zero-centered, you will see these frequencies shifted.

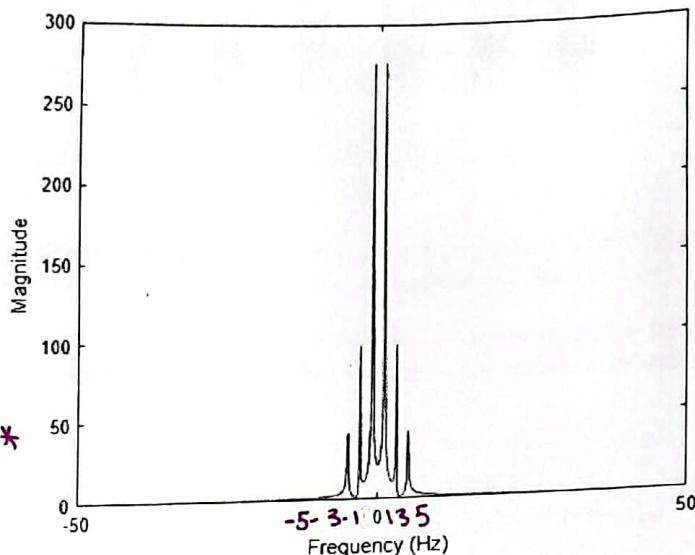
```
figure
plot(f, abs(z))
xlabel('Frequency (Hz)')
ylabel('Magnitude')
title('Magnitude')
```

Frequencies



The transform also produces a mirror copy of the spikes, which correspond to the signal's negative frequencies. To better visualize this periodicity, you can use the `fftshift` function, which performs a zero-centered, circular shift on the transform.

```
n = length(y);
fshift = (-n/2:n/2-1)*(fs/n);
yshift = fftshift(z);
plot(fshift,abs(yshift))
xlabel('Frequency (Hz)')
ylabel('Magnitude')
```

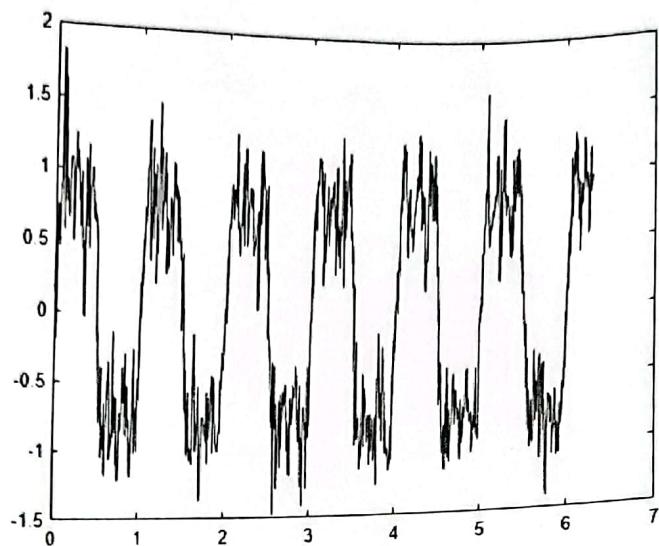


Fourier Transform and Noisy Signals

In scientific applications, signals are often corrupted with random noise, disguising their frequency components. The Fourier transform can process out random noise and reveal the frequencies.

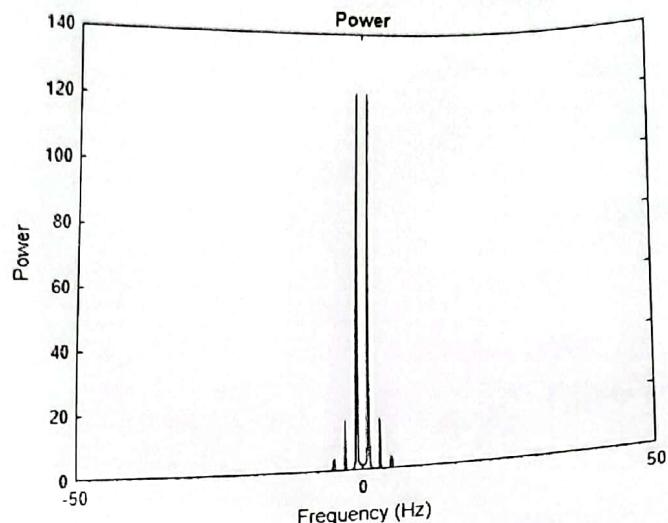
For example, create a new signal, `xnoise`, by injecting Gaussian noise into the original signal, `x`.

```
rng('default')
Ts = 0.01;
x = 0:Ts:2*pi;
y = sin(2*pi*x) + (1/3)*sin(3*2*pi*x) + (1/5)*sin(5*2*pi*x);
ynoise = y + 0.25*randn(size(y)); % We are adding random Gaussian noise to the
original signal
plot(x, ynoise)
```



Signal power as a function of frequency is a common metric used in signal processing. Power is the squared magnitude of a signal's Fourier transform, normalized by the number of frequency samples. Compute and plot the power spectrum of the noisy signal centered at the zero frequency. Despite noise, you can still make out the signal's frequencies due to the spikes in power.

```
n = length(x);
fshift = (-n/2:n/2-1)*(fs/n);
ynoise = fft(ynoise);
ynoiseshift = fftshift(ynoise);
power = abs(ynoiseshift).^2/n;
plot(fshift,power)
title('Power')
xlabel('Frequency (Hz)')
ylabel('Power')
```



Determining the Nyquist Rate

Now that we have used the Fourier transform analyse the signal in the frequency (spatial) domain, we can look for the maximum frequency. This in turns determines the Nyquist rate and all subsequent analysis.

Exercise 2

The file *signal.mat* contains a signal synthesized in MATLAB. The signal also has white Gaussian Noise. You can load the file *signal.mat* by double clicking it, or drag and dropping it into MATLAB, or through the Home Import Wizard. The variable *ynoisy_signal* will appear in the workspace which contains the signal.

- Draw the signal in the time domain from [0, 1] and assume the spacing is 0.0001.
- You can zoom in the figure to see the signal closely.
- You are required to find the maximum frequency in this signal and determine the Nyquist rate.
- You can zoom in the figure to see the signal closely

Answer: The signal has two frequency components, one at 400 Hz, another at 950Hz. Therefore, Nyquist's rate should not be less than 1.9 kHz.

Chapter 2 : Introduction to System Modelling :

* خاصية قيود concept والتطبيق الرياضي يكون على المايكرو.

* من الممكن تفريغ مقاييس System تكون على Stable وكيف يمكن time rising time وما المعايير.

* لبناء وتحليل System لازم أعرف المعادلات المكونة بحاجة وبعد ذلك يمكن طرفيتين :

1- Transfer function in frequency domain. (خاصية)

(زوج Fourier transform وهكذا .)

2- State space equations in the Time Domain. (مخصوص)

بنظر عالم العدالة المعقولة وبنطاق معهنا إنما هو صعب .

* Kirchhoff laws & KCL + KVL \rightarrow مجموع كل المقاومات يساوي صفر

التيار الداخلي I_{in} حين يساوي التيار الذي يخرج منها .

* Ohm's law & V=IR فرق الجهد الكهربائي بين المinals ذاقت يتاسب طردياً مع وصلة التيار الكهربائي المار فيه (V=IR)

* وجود التكامل والاستدراك بالمعادلات هي المشكلة التي خلقتنا ناحية transfer لتبسيط المعادلات .

* \leftarrow output \rightarrow input linear System

\leftarrow linear time Invariant System \leftarrow LTI

فقط هنا بنجاح التقارب لـ linear باقل درجة خطأ ممكنة لتبسيط المشكلات .

* Slide 3 & 4 فوق محولات السيارات تتألف من System يتميز معين يسمى يسمى السيارة

أو طلاق عصبي على أو هيك . (معاملاته مقدرة وكلها مستقرات) .

* " RIC " & Slide 4 \leftarrow التيار $I(t)$ من C له قيمة V يشترى على Circuit (يوجد)

" شاليم "
 " الدكتور "
 " حللاهم "
 " دعوه "
 " Slide 11+12 "

فيها تفاصيل ونكت المعاشرة .

\leftarrow الرسمة التي تحت Amplifier \leftarrow يبارك عن التيار في جهاز Circuit وظيفتها تحمل

تكاليف تفاصيل وجهاز وطروح و Amplification بـ computers قبل اخراج ال Analog .

تطبيق لها في الصوتيات بالهواتف وغيرها (يوجد فيها تفاصيل وتكامل) .

2 modes of operation transistor & linear Systems & Slide 5 *

\leftarrow Digital أو بل (0) أو بلا Saturation وما كانت يعني \leftarrow يحيى

فرق طيفية الموجة الى بالذى بينهم .

\leftarrow Analog (Amplification) \leftarrow plant \leftarrow cutoff \leftarrow saturation \leftarrow ما بينهم بالذى

مثلًا ما يعني أنزيد الموجات بزيد ال Volt ليزيد التيار .

* \leftarrow non-linear curve \leftarrow كافر أننا يأخذ الجزء ال linear كتفير ويشغل كلية للتوصيل .

* الـ Non-linear System (الـ non-linear) هو الذي يغير مفهومه ما يساويها هو أن
ـ مستحيل تقارب لـ linear

* linear System (الـ linear) كيف أعرف إنما هو؟ Slide 6+7 *

ـ مجموع الـ 2 inputs مجموع الـ 2 الأوليّات لأنّه يعطيه نفس

ـ الخطاب يكلّ واحد على جدي مجموعين

Example: $y = 5x$

Example: $y = 5c^x$ (non linear)

$$* x_1 = 3 \rightarrow y_1 = 15$$

$$* x_1 = 3 \rightarrow y_1 = 100.4277$$

$$* x_2 = 4 \rightarrow y_2 = 20$$

$$* x_2 = 4 \rightarrow y_2 = 272.9908$$

$$* x_3 = 7 \rightarrow y_3 = 35$$

$$* x_3 = 7 \rightarrow y_3 = 5483.165792$$

If $15 + 20 = 35$ and is therefore linear

$$100.4277 + 272.9908 \neq 5483.165792$$

and is therefore non-linear

* Scaling & Homogeneity (2)

ـ Superposition (الـ Superposition)

Example: $y = 5x$

Example: $y = 5x + 4$ (Scaling with shift)

$$* x_1 = 3 \rightarrow y_1 = 15$$

$$* x_1 = 3 \rightarrow y_1 = 19$$

$$* x_2 = 4 \rightarrow y_2 = 20$$

$$* x_2 = 4 \rightarrow y_2 = 24$$

$$* x_3 = 7 \rightarrow y_3 = 35$$

$$* x_3 = 7 \rightarrow y_3 = 39$$

$15 + 20 = 35$ and is therefore linear

$$19 + 24 \neq 39 \text{ and is therefore non-}$$

homogeneous \rightarrow non-linear.

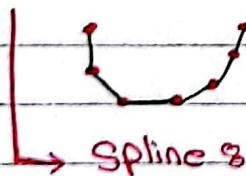
- Scaling (الـ scaling) هي حالة خطية لكن من حيث المفهوم هي غير خطية

* 2nd order & 1st order (الـ 2nd order & 1st order)

ـ درجة المشتق هي التي تحدد الـ order

ـ "if $f''(x) = 0 \rightarrow$ linear" ؟ linear

* Matlab \rightarrow linear Interpolation & non-linear Interpolation



ـ non-linear (الـ non-linear)



Slide 9 : Time Variant and Invariant System

- ① Time Invariant System : (مثل قانون أوم، مثل لاتيار 5 و المقاومة 5 فالقيمة دائماً 25 أشواكاً في الوقت). (متغير)
 يكون $A + B$ بالمعادلة برق (خارجية و مدخل بحد ذاتها من function input يتغير مع تغير الوقت). (محدد)
- ② Time Variant System : (أنا هيك بنسب المعدلات قدر الامكان يشان نغير نحلاً (إذن مثل بقولون أوم لو حطيت المقاومة و حطيت فيها تيار و بتتغير مقاومتها) بس أنا اكتبنا حاجة التغيرات طيفية تكون لا تؤثر).

← معادلات ← Slide 10+11+12 *

كيف بنا نحل المعدلات المختلطة هي؟ هل ألا computers حلولت تحلها. ← Slide 13 *

· laplace هي حالة خاصة من fourier Series ← Slide 14 *

· بقدر يعزمي signals لما تكون fourier ← fourier *

لأن يكون $f(t)$ function e^{-jwt} ، الاختلافاته $\frac{df}{dt} = -wf$ بل سبز

ـ S مورقم بل $(\alpha + j\omega)$: Complex

* لو كانت $f(t)$ Signal منتظم \Rightarrow laplace أفضل وأعم . حسنت \Rightarrow laplace

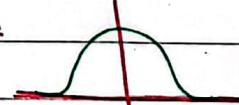
لتحل المعدلات المختلطة والتكاملية لـ المعدلات جبرية بسيطة (تربيع، تكعيب، خطي، ...)

ـ samples ← continuous time domain ← laplace *

table ← المهم $5+6+7$ (حسنت جانوالكتور بتحليل المعدلة بتطلعها table)

ويتحول المعدلة المختلطة العامة $f(t)$ إلى $f(s)$ ← وما كلما الدخول يتغير

* if my signal is periodic → like :



(\circ)

→ بقسم R and C Sub Systems System كل واحدة

يجدوا $\frac{1}{R}$ شان نحصل من التكامل والاشتقاق.

* دائمًا $\frac{1}{R}$ System يجيء بي أعني $input$ \rightarrow $output$ يعطي $output$ ارتكابه، حيسير يعني أي شيء

اسمه $\frac{1}{R}$ transfer function وهو حمل $input$ \rightarrow $output$ على $input$.

* $\frac{1}{C}$ System \rightarrow $input$ \rightarrow $output$ \rightarrow $output$ \rightarrow $output$... وهو ناتج بوجود التكامل.

فبناءً $\frac{1}{R}$ \rightarrow $input$

· أصعب معدلة جبرية فشول أزيدوا وهيك بقدر أقوى كل الشروط.

→ $\frac{1}{s^2 + \frac{1}{R^2}}$ ← يوجد مقلمات ونحوه ، $\frac{1}{s^2 + \frac{1}{R^2}}$ هي الصيغة الوطيدة

لهذا الموضوع ، * النهاية بحسب s و التكامل بقيمة s .

Five Apple



Modelling of Control Systems

Dr. ASHRAF E. SUYYAGH

THE UNIVERSITY OF JORDAN
DEPARTMENT OF COMPUTER ENGINEERING
FALL 2022

Systems *
التي تتبع معايير
linear
superposition
homogeneity

اكتب المعادلات ميكانيكياً وربما هنا
بعض

Frequency domain - 2

Introduction to System Modelling

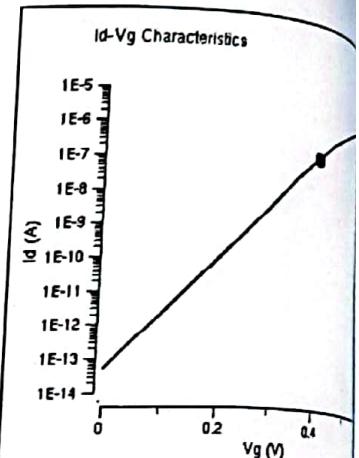
2

- ▶ In Chapter 1 we discussed the analysis of temporal and functional requirements among others than are required in control systems design. We have also seen basic schematics and block diagram of a sample Azimuth angle system.
- ▶ Next step is to develop the mathematical model in order to analyze the systems:
 1. Transfer Function in Frequency Domain (This course)
 2. State Space Equations in the Time Domain (Another Advance Course)
- ▶ Modelling requires applying the fundamental physical laws of science and engineering: Ohm's law, Kirchhoff's laws, Newton's laws, etc.
- ▶ Many of these governing equations have derivatives, integrals, and many are described in differential equations
- ▶ We concern ourselves in this course with linear systems, many systems can be treated as linear if their inputs remain within a certain range.
- ▶ Yet, in general most of nature is not always linear. We deal with **approximations** to simplify the design. Non-linear systems are handled in advanced control systems.

Quick Review of Linear Systems (I)

- A great majority of physical systems are linear within some range of the variables. For example, the operation of a FET transistor:
 - Linear operation (amplification) if V_g between 0 – 0.4 Volts
 - Non-Linear beyond $V_g = 0.4$ Volts → Goes towards saturation
- In general, many systems ultimately become nonlinear as the variables are increased without limit (e.g. spring response)
- The linearity of many mechanical and electrical elements can be assumed over a reasonably large range of the variables. This is not usually the case for thermal and fluid elements, which are more frequently nonlinear in character

A linear system satisfies the properties of superposition and homogeneity.



Quick Review of Linear Systems (II)

This Principle of Superposition

In general, a necessary condition for a linear system can be determined in terms of an excitation $x(t)$ and a response $y(t)$.

- When the system at rest is subjected to an excitation $x_1(t)$, it provides a response $y_1(t)$.
- Furthermore, when the system is subjected to an excitation $x_2(t)$, it provides a corresponding response $y_2(t)$.
- For a linear system, it is necessary that the excitation $x_1(t) + x_2(t)$ result in a response $y_1(t) + y_2(t)$.

Example: $y = 5x$

$$\begin{aligned} \diamond x_1 = 3 &\rightarrow y_1 = 15 \\ \diamond x_2 = 4 &\rightarrow y_2 = 20 \\ \diamond x_3 = 7 &\rightarrow y_3 = 35 \end{aligned}$$

$15 + 20 = 35$ and is therefore linear

Example: $y = 5e^x$

$$\begin{aligned} \diamond x_1 = 3 &\rightarrow y_1 = 100.4277 \\ \diamond x_2 = 4 &\rightarrow y_2 = 272.9908 \\ \diamond x_3 = 7 &\rightarrow y_3 = 5483.165792 \end{aligned}$$

$100.4277 + 272.9908 \neq 5483.165792$ and is therefore non-linear

Quick Review of Linear Systems (III)

This Property of Homogeneity

The magnitude scale factor must be preserved in a linear system.

Example: $y = 5x$

$$\begin{aligned} \diamond X_1 = 3 &\rightarrow Y_1 = 15 \\ \diamond X_2 = 4 &\rightarrow Y_2 = 20 \\ \diamond X_3 = 7 &\rightarrow Y_3 = 35 \end{aligned}$$

$15 + 20 = 35$ and is therefore homogenous

Example: $y = 5x + 4$

$$\begin{aligned} \diamond X_1 = 3 &\rightarrow Y_1 = 19 \\ \diamond X_2 = 4 &\rightarrow Y_2 = 24 \\ \diamond X_3 = 7 &\rightarrow Y_3 = 39 \end{aligned}$$

$19 + 24 \neq 39$ and is therefore non-homogenous \rightarrow non-linear System

Quick Review of Linear Systems (IV)

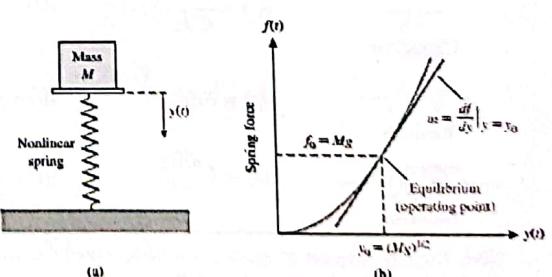
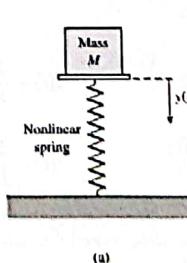
$y(t) = x^2(t)$ (non-linear, does not satisfy the superposition property)
 $y(t) = mx(t) + b$ is not linear, because it does not satisfy the homogeneity property

$y' + p(x)y = r(x)$ is a first order LINEAR differential equation
 if $r(x) = 0$, for all values of x , then homogenous, else non-homogeneous

$y'' + p(x)y' + q(x)y = r(x)$ is a second order LINEAR differential equation
 if $r(x) = 0$, for all values of x , then homogenous, else non-homogeneous

What if the system is non-linear? Then we must apply techniques for linear approximation before analyzing the system is linear.

Approximations are as accurate as the underlying assumptions \rightarrow Model errors



* ال RLC سيركت شفناه تعالوا وطبقنا كانون كيرشوف عليه

Quick Review of Time Variant and Invariant Systems

7

Any system has inputs and outputs.

A time invariant system is a system in which the output does not depend on the time at which the input arrives. That is; regardless of when the input comes in, now or delayed; the output to this input remains always the same result.

We can spot a time invariant system if the equation has no dependence on time as A SEPARATE variable.

$$V = IR \quad (\text{Time-invariant})$$

$$V(t) = I(t)R \quad (\text{Also time invariant})$$

A time variant system is a system in which the output changes if the same input arrives at different times. That is; if the input $x = 5$ arrives at $t = 0$, it will produce different result if $x = 5$ arrives delayed by 10 seconds.

We can spot an time variant system if the equation has dependence on time as A SEPARATE variable.

$$Y = tX \quad (\text{Time-variant})$$

$$y(t) = tx(t) \quad (\text{Time-variant})$$

$$V(t) = I(t)R(t) \text{ where } R(t) \text{ changes due to component temperature over time} \quad (\text{Time-variant})$$

تغير حسب input مع اختلاف الوقت

دخلت عليه او input دخلت عليه او System

بأي وقت حبيطني نفس الـ output

* In the Signals and Systems Course, and in this Control course, we assume we are working with only Linear Time-Invariant Systems (LTI)

* مشهود اليك بتحليل السيركت الدكتور يعطيك
ياها وقبلك هي بالـ Time Domain equations تبعها

8

Simple Dynamics of Electrical Components

Component	Voltage-current	Current-voltage	Voltage-charge	Impedance $Z(s) = V(s)/I(s)$	Admittance $Y(s) = I(s)/V(s)$
 Capacitor	$v(t) = \frac{1}{C} \int_0^t i(\tau) d\tau$	$i(t) = C \frac{dv(t)}{dt}$	$v(t) = \frac{1}{C} q(t)$	$\frac{1}{Cs}$	Cs
 Resistor	$v(t) = Ri(t)$	$i(t) = \frac{1}{R} v(t)$	$v(t) = R \frac{dq(t)}{dt}$	R	$\frac{1}{R} = G$
 Inductor	$v(t) = L \frac{di(t)}{dt}$	$i(t) = \frac{1}{L} \int_0^t v(\tau) d\tau$	$v(t) = L \frac{d^2q(t)}{dt^2}$	Ls	$\frac{1}{Ls}$

Note: The following set of symbols and units is used throughout this book: $v(t)$ – V (volts), $i(t)$ – A (amps), $q(t)$ – Q (coulombs), C – F (farads), R – Ω (ohms), G – Ω (mhos), L – H (henries).

Simple Dynamics of Mechanical Components (I)

9

Component	Force-velocity	Force-displacement	Impedance $Z_M(s) = F(s)/X(s)$
Spring	$f(t) = K \int_0^t v(\tau) d\tau$	$f(t) = Kx(t)$	K
Viscous damper	$f(t) = f_v v(t)$	$f(t) = f_v \frac{dx(t)}{dt}$	$f_v s$
Mass	$f(t) = M \frac{dv(t)}{dt}$	$f(t) = M \frac{d^2x(t)}{dt^2}$	$M s^2$

Simple Dynamics of Mechanical Components (II)

10

Component	Torque-angular velocity	Torque-angular displacement	Impedance $Z_M(s) = T(s)/\theta(s)$
Spring	$T(t) = K \int_0^t \omega(\tau) d\tau$	$T(t) = K\theta(t)$	K
Viscous damper	$T(t) = D\omega(t)$	$T(t) = D \frac{d\theta(t)}{dt}$	$D s$
Inertia	$T(t) = J \frac{d\omega(t)}{dt}$	$T(t) = J \frac{d^2\theta(t)}{dt^2}$	$J s^2$

Note: The following set of symbols and units is used throughout this book: $T(t)$ – N-m (newton-meters), $\theta(t)$ – rad (radians), $\omega(t)$ – rad/s (radians/second), K – N-m/rad (newton-meters/radian), D – N-m-s/rad (newton-meters-seconds/radian), J – kg-m² (kilograms-meters² – newton-meters-seconds²/radian).

Mechanical Example of System Modelling

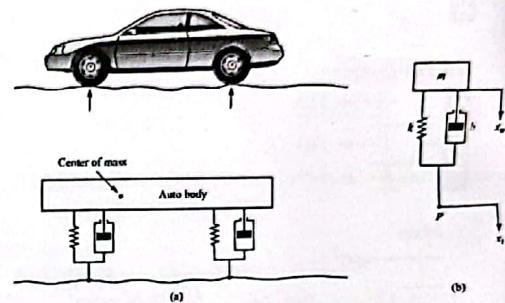
11

Automobile Suspension System

A very simplified version of the suspension system is shown. Assuming that the motion (displacement) x_i at point P is the input to the system and the vertical motion x_o of the body is the output, then, the equation of motion for the system is:

$$m\ddot{x}_o + b(\dot{x}_o - \dot{x}_i) + k(x_o - x_i) = 0$$

$$m\ddot{x}_o + b\dot{x}_o + kx_o = b\dot{x}_i + kx_i$$



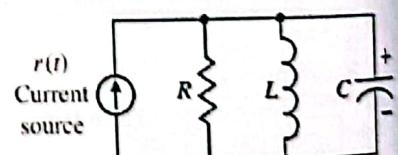
Electrical Example of System Modelling

12

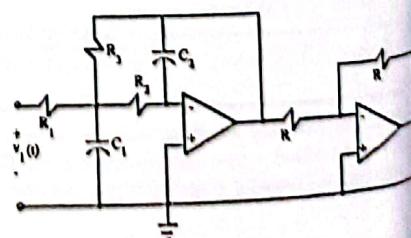
Simple RLC Circuit

One may describe the electrical RLC circuit shown in the figure by utilizing Kirchhoff's current law. We will obtain the following integrodifferential equation:

$$\frac{v(t)}{R} + C \frac{dv(t)}{dt} + \frac{1}{L} \int_0^t v(t) dt = r(t)$$



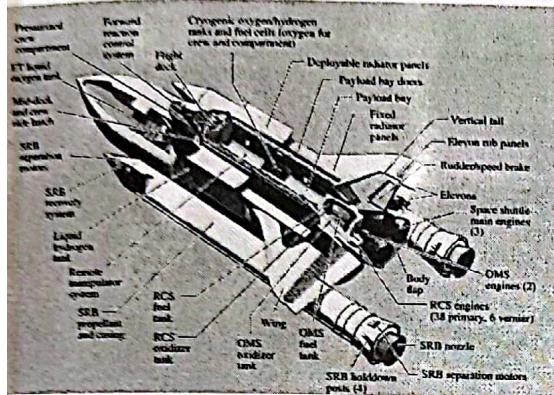
$$v''_2(t) + \left(\frac{1}{R_1 C_1} + \frac{1}{R_2 C_1} + \frac{1}{R_3 C_1} \right) v'_2(t) + \frac{1}{R_1 R_2 C_1 C_2} v_2(t) = \frac{1}{R_1 R_2 C_1 C_2} v_1(t)$$



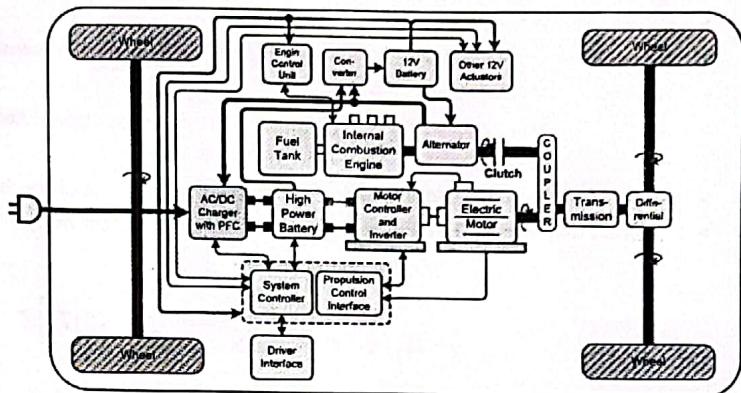
Modelling Complexity

13

Real systems have many electrical, mechanical, hydraulic and pneumatic components. Putting them all together and writing their equations in the time domain then trying to solve the system (e.g., finding the relationship between the output and the input) in the time domain can be difficult.



Ex.1: Shuttle Control



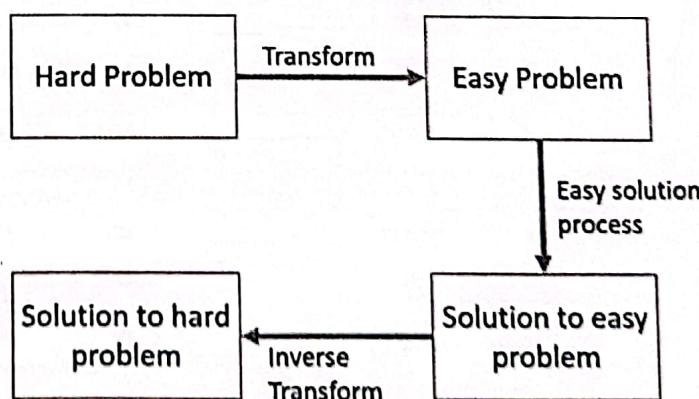
Ex.2: Hybrid Car Control

Complex Dynamics of Control Systems

14

- Most systems even when constructed from simple components will quickly turn to be more complex to analyze in the time domain, as many of the basic components are governed by integrodifferential equations.
- Need a much-simplified solution to transfer the hard problem into simpler one
- Laplace transform to the rescue

پیشنهاد می‌شود
Fourier یا
transform.



Laplace Transform

The Laplace transform mathematically resembles Fourier transform:

$$\mathcal{L}_t [f(t)](s) \equiv \int_0^{\infty} f(t) e^{-st} dt,$$

- But here, s is a complex number, so s is $a + j\omega$
- Fourier Transform is a special case of Laplace, when $a = 0$, that is there are no underlying exponentials in the signal.
- Laplace is a more generic transform
- Laplace's Transform beauty lies in its ability to transform functions, even integral, differential into simple ALGEBRAIC notation.
- Algebra is much easier to solve
- Notice that Laplace is a continuous time-domain transformation into the spatial (frequency domain) (Analogue Systems)
- In discrete time systems, where we have samples fed into digital controllers, we have the equivalent z-transform

$$X(z) = \sum_{n=0}^{\infty} x[n] z^{-n}$$

- MATLAB has functions for Fourier, Laplace, and z-transforms and their inverse transformations.

Laplace Transform Table

S.no	$f(t)$	$\mathcal{L}[f(t)]$	S.no	$f(t)$	$\mathcal{L}[f(t)]$
1	1	$\frac{1}{s}$	11	$e^{at} \sinh bt$	$\frac{b}{(s-a)^2 - b^2}$
2	e^{at}	$\frac{1}{s-a}$	12	$e^{at} \cosh bt$	$\frac{s-a}{(s-a)^2 - b^2}$
3	t^n	$\frac{n!}{s^{n+1}}$	13	$t \cos at$	$\frac{s^2 - a^2}{(s^2 + a^2)^2}$
4	$\sin at$	$\frac{a}{s^2 + a^2}$	14	$t \sin at$	$\frac{2as}{(s^2 + a^2)^2}$
5	$\cos at$	$\frac{s}{s^2 + a^2}$	15	$f'(t)$	$sF(s) - f(0)$
6	$\sinh at$	$\frac{a}{s^2 - a^2}$	16	$f''(t)$	$s^2 F(s) - sf(0) - f'(0)$
7	$\cosh at$	$\frac{s}{s^2 - a^2}$	17	$\int_0^t f(u) du$	$\frac{1}{s} F(s)$
8	$e^{at} t^n$	$\frac{n!}{(s-a)^{n+1}}$	18	$t^n f(t)$ Where $n = 1, 2, 3, ..$	$(-1)^n \frac{d^n}{ds^n} \{F(s)\}$
9	$e^{at} \cos bt$	$\frac{(s-a)^2 + b^2}{(s-a)^2 - b^2}$	19	$\frac{1}{t} \{f(t)\}$	$\int_s^{\infty} F(s) ds$
10	$e^{at} \sin bt$	$\frac{b}{(s-a)^2 + b^2}$	20	$e^{at} f(t)$	$F(s-a)$

Laplace Transform Table Example

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*PROBLEM: Find the Laplace transform of $f(t) = te^{-5t}$.

From the Laplace Table, we can find this relationship:

$$e^{at} t^n \quad | \quad \frac{n!}{(s-a)^{n+1}} = 1 \rightarrow \text{Power } + \text{تعدد}$$

So, in our case, $a = -5$ and $n = 1$, substituting in the Laplace transform, we get:

$$\text{ANSWER: } F(s) = 1/(s+5)^2$$

*PROBLEM: For the following differential equation, find the LaPlace trans

$$\frac{d^2x}{dt^2} + 4 \frac{dx}{dt} + 5x = 1$$

If the initial conditions are $x(0) = 1$, and $x'(0) = -1$

$\hookrightarrow x=0$ القيمة المعرفة لـ x
لـ x' أدخلنا لها خالص بعثها

ANSWER: From the Laplace Table, we can find this relationship:

$f'(t)$	$sF(s) - f(0)$
$f''(t)$	$s^2F(s) - sf(0) - f'(0)$
$x' = t^n$	$\frac{n!}{s^{n+1}} = 1$

So, for the 2nd derivative we get:

$$s^2F(s) - s(1) - (-1) = s^2F(s) - s + 1$$

for the 1st derivative we get:

$$4(sF(s) - 1)$$

For the 3rd term, $L(5x)$, here $n = 1$, so

$$\frac{5}{s^2}$$

And the right-hand term is 1, so $L(1) = \frac{1}{s}$

$$s^2F(s) - s + 1 + 4sF(s) - 4 - \frac{5}{s^2} = \frac{1}{s}$$

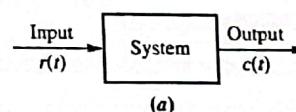
Then simplify

The Transfer Function

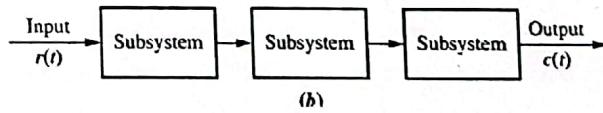
العلاقة التي تربط input بـ output هي ما كانت العلاقة تكون.

18

A control system has an input signal $r(t)$ which is going to provide an output (a control signal) $c(t)$ as you can see in the simple open-loop control system (Figure a).



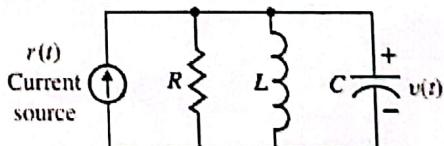
When a system is complex, and it has many electrical or mechanical components, we divide it into subsystems. But still, we have an input signal $r(t)$ which is going to provide an output (a control signal) $c(t)$ as you can see in the simple open-loop control system (Figure b).



For example, in the adjacent RLC circuit, the current i (i.e., $r(t)$) controls the output voltage $v(t)$ (i.e., $c(t)$)

The relationship between the output and the input is governed by the equations of the system.

We are interested to know the relationship of the output to the input; that is: $G(t) = \frac{c(t)}{r(t)}$ which we call the transfer function.



But in the time domain (complex) $\frac{v(t)}{R} + C \frac{dv(t)}{dt} + \frac{1}{L} \int_0^t v(t) dt = r(t)$

The Transfer Function

This is why we use the Laplace transform and express the Transfer Function as:

$$G(s) = \frac{c(s)}{r(s)}$$

1. Use Algebra instead of integrodifferential equations
2. Spatial domain provides means to analyze the system in ways unavailable to us in the time-domain. Exposes and allows using tools to determine transient response, steady-state, stability.

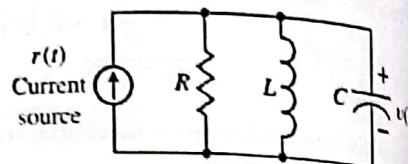
Assuming no initial conditions, and using Laplace Tables, we can rewrite the system equations as:

$$\frac{V(s)}{R} + sCV(s) + \frac{V(s)}{sL} = R(s)$$

$$V(s) \left(\frac{1}{R} + sC + \frac{1}{sL} \right) = R(s)$$

$$G(s) = \frac{V(s)}{R(s)} = \frac{1}{\left(\frac{1}{R} + sC + \frac{1}{sL} \right)}$$

$$G(s) = \frac{V(s)}{R(s)} = \frac{sRL}{s^2 RLC + sL + R}$$



$$\frac{v(t)}{R} + C \frac{dv(t)}{dt} + \frac{1}{L} \int_0^t v(t) dt = r$$

Transfer Functions are valid when the system is LINEAR; that is, the governing equations are linear.

If not, we need to apply linearization to nonlinear components first; then retrieve the transfer function.

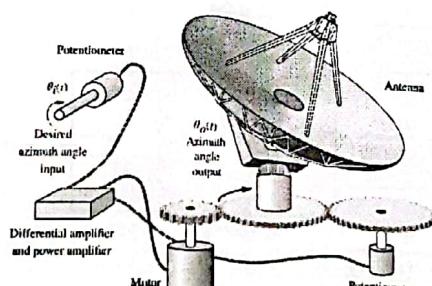
(closed feedback) Closed-loop System * إفلاط الـ Systems

The Transfer Function of Complex Systems

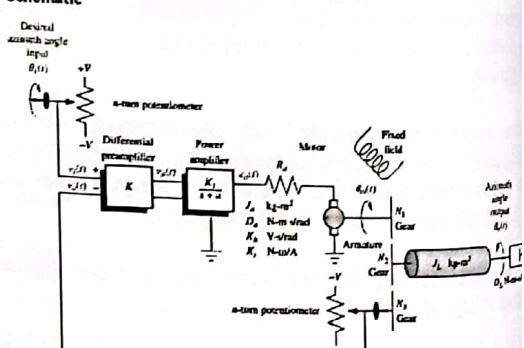
Most control systems are far more complex.

1. Divide them into subsystems
2. Retrieve the transfer function for each subsystems individually
3. Simplify using block diagram reduction techniques or by maths

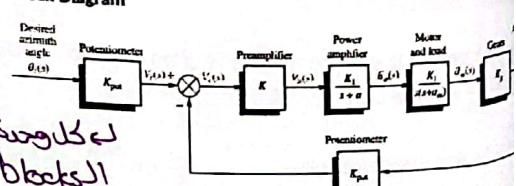
Layout



Schematic



Block Diagram



لـ كل وحدة من هذه
الـ blocks

(الـ معادلة كلـ يـ علىـها Laplace transfer function)

بالـ S domain مـ كـ تـ بـ لـ الـ Time Domain (مـ كـ تـ بـ لـ وـ كـ وـ بـ اـ تـ ةـ) ، كـ ثـ بـ تـ بـ لـ

بالـ S domain للـ تنـ هـ يـ لـ .

* مُوْمِقٌ أَوْ جَدِيدُ الْعَلَاقَةِ الْفَرَاعِيَّةِ بَيْنَ الْأَنْتَرِنَسِ وَالْأَنْتَرِنَسِ (مُعَادَلَةُ السِّرْكِيْكِتِ)

Transfer Function of Systems with Negative Feedback

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- The input to this system is the signal $x(s)$
 - Since this is a negative feedback system, the actual controlling signal is the difference (error) between the input signal and the feedback signal (say previous reading) which is $C(s)H(s)$
 - Therefore, the difference (error) signal is $R(s) - C(s)H(s)$
 - The new output signal $C(s)$ is the result of applying the control (system) $G(s)$ to the difference (error) signal $G(s)(R(s) - C(s)H(s))$
- So:

$$C(s) = G(s)(R(s) - C(s)H(s))$$

$$C(s) = G(s)R(s) - G(s)C(s)H(s)$$

$$C(s) + G(s)C(s)H(s) = G(s)R(s)$$

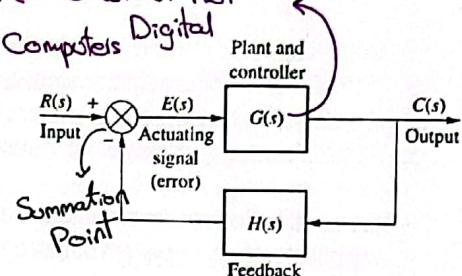
$$C(s)(1 + G(s)H(s)) = G(s)R(s)$$

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$

من هنا نقدر نقييم كل الأجزاء
ونجد كل block وحده هيواه
المطلوبة.

actual desired سُقْلَانِ الْفَرَقِ بَيْنَ الْأَنْتَرِنَسِ *
كَمَا شافَ فِي فُوقِ بَخْلِيَّتِكُمْ هَذِهِ لِيُوَجِّلَنَا إِلَى بَيْنِ يَمَاهَا output

لأنه ليس ما يعنيه
Digital Computer
بتكون $G(z)$



A positive feedback system will have the same transfer function, except that the sign will be negative in the denominator

Examples

22

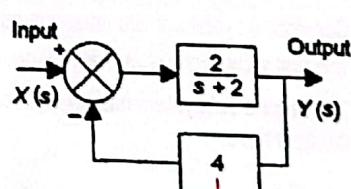
Example 1) Determine the overall transfer function for the adjacent control system which has a negative feedback loop with a transfer function of a gain equal 4 and a forward path transfer function of $2/(s+2)$.

The overall transfer function of the system is:

$$G_{\text{overall}}(s) = \frac{\frac{2}{s+2}}{1 + 4 \times \frac{2}{s+2}} = \frac{2}{s+10}$$

تعويض بالمعادلة
اليتى

لبيان رقم العدد مع
Signal gain
ألا يزيد من قيمة الـ gain
Amplifier يكتب
بسقطه.

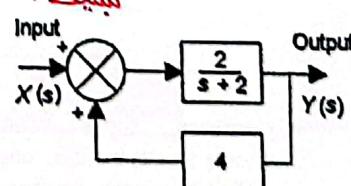


Example 2) Determine the overall transfer function for the adjacent control system which has a positive feedback loop with a transfer function of a gain equal 4 and a forward path transfer function of $2/(s+2)$.

The overall transfer function of the system is:

$$G_{\text{overall}}(s) = \frac{\frac{2}{s+2}}{1 - 4 \times \frac{2}{s+2}} = \frac{2}{s-6}$$

تعويض
 $(\frac{G}{1-HG})$
Positive w/ feedback



* مدخلات input هي إلكترونية Time Domain || output هي إلكترونية فرقة فرط لها

Frequency Domain

- سوء كهرباء أو هيكانيكية * Multiple components مكون من System ||

Classical \leftarrow Laplace / for Digital \leftarrow Z-transform *

حالياً هن في sys كله أوكله Digital دايماً ما يرجعه sys الواحد *

(Mechanic + Computer Control) و بعد ذلك Classical و Digital

Classical \leftarrow Digital

$$Y(s) \text{ (output)} = E(s) G(s)$$

Slide 21

error or differences \rightarrow System

$\rightarrow X(s) - Y(s) H(s)$ جرف الـ S لغير انه المدخلات مكتوبة بصيغة Laplace

$\rightarrow Y(s) = (X(s) - Y(s) H(s)) G(s)$ الى أصلها كانت Time domain

System | Time domain \leftarrow feedback مكتوبة بصيغة Somain

* $Y = (X - yH) G \rightarrow Y = XG - yHG \rightarrow Y + yHG = XG$

$$\rightarrow Y(1 + HG) = XG \rightarrow Y = \frac{G}{1 + HG} X$$

أنا بالـ S هو نسبة المدخل إلى المخرج transfer function || negative feedback sys

. feedback على 1 + الـ G معرفة بالـ controller(s)

لـ sys اشارة الـ feedback

G

$\frac{1}{1 + HG}$: Positive feedback تكون المعادلة

* في فرقابس الـ Matlab و Simulink

. Plots و يدخل فيه شوية Command lines

control system و مدخلين بمكتبة اسمها commands for control

بالـ Python

لـ ما نكتب الـ Python تفتح الـ Matlab و المايلز فعلياً منكتب كلـ sys لحال

transfer funct. هو الحال بعطيك الـ Simulink

و بحل.

Time domain لـ negative feedback systems ٨ slide 22

يستخدم المعلمات لـ Laplace transform table

بشكل هباء : Positive feedback

→ grows exponential

ما ينقط ولا يوصل لقيمة

(unstable)

Stability is ↓

Steady state J

العمليات كلها ترتب مش تهويدي ، يا Time domain يا S Domain *

بالنسبة

لبسيط System يا اما بـ حل المعادلة وبنطلي X بـ جمعه و لا يوجد يا اما يستخدم او

. (block Manipulation) block Simplification

Transfer Function Simplification using Block Manipulation

23

- Control systems may have many elements and sometimes more than one input. \rightarrow systems العالي
- A single input-single output system is often termed a SISO system.
- A multiple input-single output system is a MISO system.
- A multiple input-multiple output system is a MIMO system.

- We need to find relationships (transfer functions) between each input and each output. That is; how each input will affect any given system output based on the circuit/system design and controller(computer) decisions.
- We need to simplify the control system block diagram to find these transfer functions
- The following are some of the ways we can reorganise the blocks in a block diagram of a control system in order to produce simplification and still give the same overall transfer function for the system:

1. Merging Block in Series and Parallel Forms (Cascade Form)
2. Moving take-off points (distribution points)
3. Moving a summing point
4. Eliminating Feedback loops

* بالـ output هو علـة الـ transfer funct \leftarrow SISO
 * بالـ output هو علـة كلـ input لـ all \leftarrow MISO
 * بالـ output كلـ input لـ all هو علـة كلـ transfer funct \leftarrow MIMO
 . 9 transfer functions بـ 3 outputs وـ 3 inputs لـ SISO

* ١ * Block Manipulation: Transfer Function of Systems in Series

24

كلما شوف block او أكـثر بـشـلـوم وـبـخـلـوم \leftarrow 2 blocks in series وـ بـخـرـجـم بـعـدـه.

Consider a system of two subsystems in series.

The first subsystem $G_1(s)$ has an input of $X(s)$ and an output of $Y_1(s)$; thus, $G_1(s) = \frac{Y_1(s)}{X(s)}$.

The second subsystem has an input of $Y_1(s)$ and an output of $Y_2(s)$ thus, $G_2(s) = Y_2(s)/Y_1(s)$

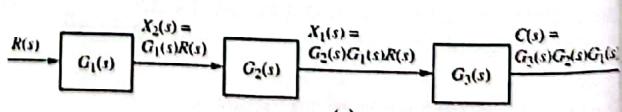
We thus have:

$$Y_2(s) = G_2(s)Y_1(s) = G_2(s)G_1(s)X(s)$$

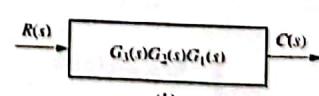
The overall transfer function $G(s)$ of the system is $Y(s)/X(s)$ and so:

$$G_{\text{overall}}(s) = G_1(s)G_2(s)$$

The overall transfer function of a system composed of elements in series is the product of the transfer functions of the individual series elements



(a)



(b)

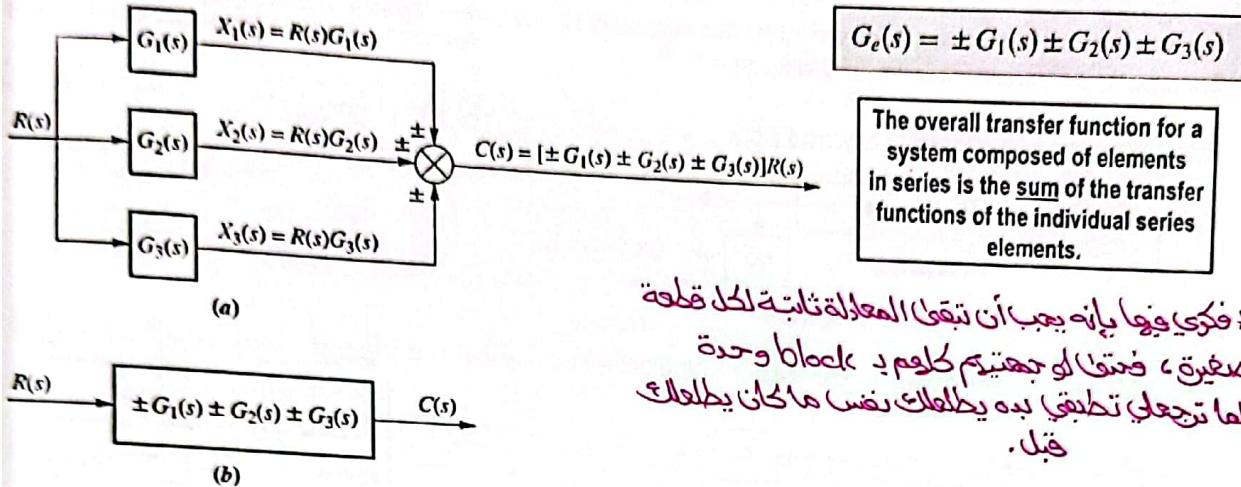
Determine the overall transfer function for a system which consists of two elements in series, one having a transfer function of $1/(s + 1)$ and the other $1/(s + 2)$.

$$G_{\text{overall}}(s) = \frac{1}{s+1} \times \frac{1}{s+2} = \frac{1}{(s+1)(s+2)}$$

2 نقلات النفع ، كثيرة و كثير مستخدمة .

Block Manipulation: Transfer Function of Systems in Parallel

25

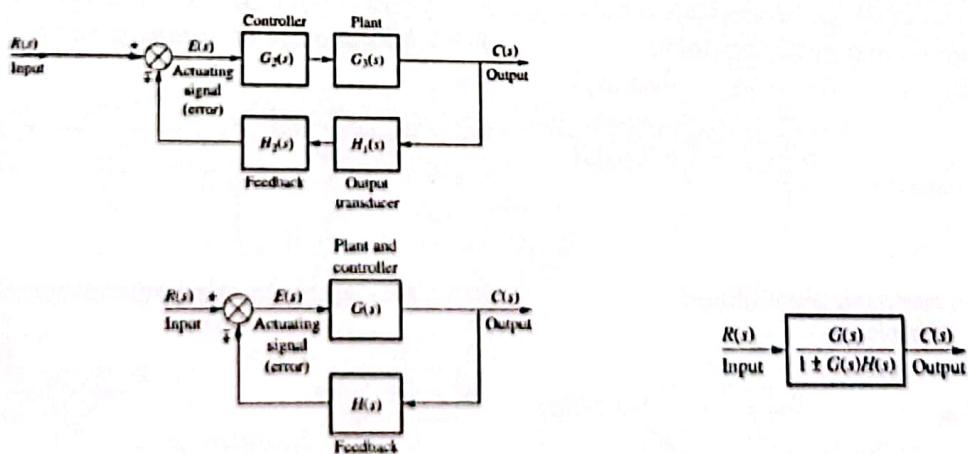


* احنا ما بنغير بال System كثوبانية او هيكلانيكي احنا زي اهنا بنعبر عنه بطريقة أسهل .

3 Block Manipulation: Changing Feedback Loops

26

We follow the rule for the feedback loop transfer function, but we need to make sure that we do not have any distribution points in the middle of our forward or feedback paths.



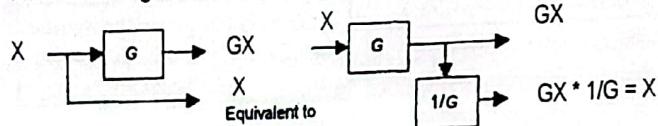
بخلد نكزى ما هو احنا التبسيط هنا نظرنا فقط.

4 Block Manipulation: Moving Take-off Points

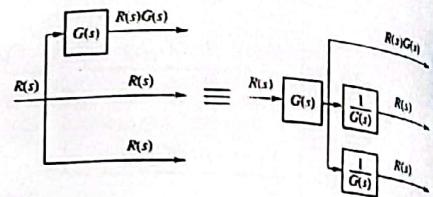
لتحتها نقطتين تقع أيهما، هم متساوية

As a means of simplifying block diagrams, it is often necessary to move take-off points. The following figures demonstrate the basic rules for such movements:

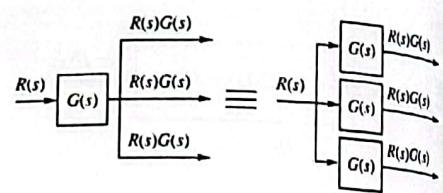
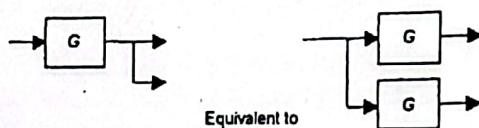
Rule 1: Moving a take-off point to beyond a block



لتحتها نقطتين يطلع على X زعيم ما كان يطلع قبل



Rule 2: Moving a take-off point to ahead of a block (distributive)



* بالعادة يابخرب يا بقسم G(s)

* هايل Slides تكون معططة لانا
بالامتحان.

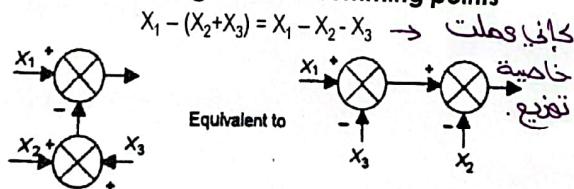
5 Block Manipulation: Moving a Summing Point

As a means of simplifying block diagrams, it is often necessary to move summation points. The following figures demonstrate the basic rules for such movements:

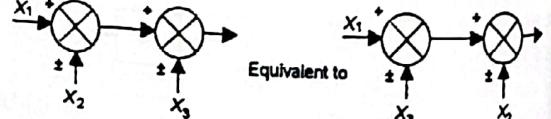
خاصية التبادل بين الجمع والطرح (بشرط ما في خطوط متفرعة وطالعه من اى

blocks)

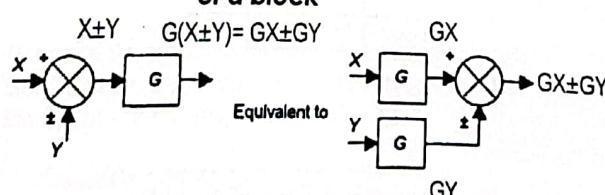
Rule 1: Rearrangement of summing points



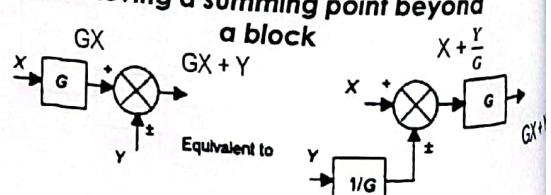
Rule 2: Interchange of summing points



Rule 3: Moving a summing point ahead of a block



Rule 4: Moving a summing point beyond a block

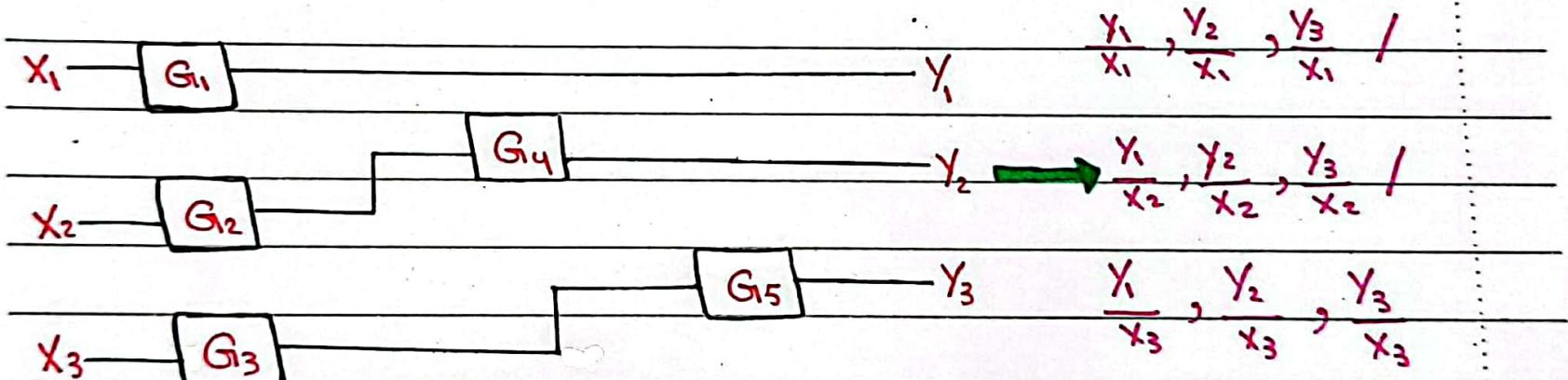


3/11/2022

- * الحالة المعقّدة هي التي تكون فيها أحزمة ميكانيكية وهيدروليكية وكهربائية لها خواص فيزيائية معينة وبالتالي معادلات معينة.
- * إذا تم التوزيع ليطلع به يوم بكل Subsystems ولازم نتأكد أنه هادي الشيطة اللي ببننا لها.

- * لازم أعمل دائمًا لأننا input يكون كلها بجعة ولا output كلها بجعة أخرى لتحقق أقصى Transfer function.

Slide 23 : MIMO → if we have 3 inputs + 3 outputs = 9 transfer functions :



- * ممكن تكون input وoutput وكذا Sensors، كبسات Switches أو أحمر من ثلاثة.
- * motor يتحرك أو يوصل طاقة معينة أو ينزل درجة حرارة معينة أو يحرك أشياء ثقيلة وهكذا.

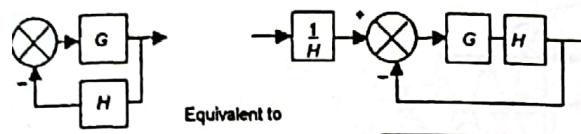
Five Apple

6 Block Manipulation: Changing feedback and forward paths

29

لـ 2 forward paths وـ 1 feedback path لـ 1 feedback path
وـ 1 forward path

Rule 1: Removing a block from a feedback path

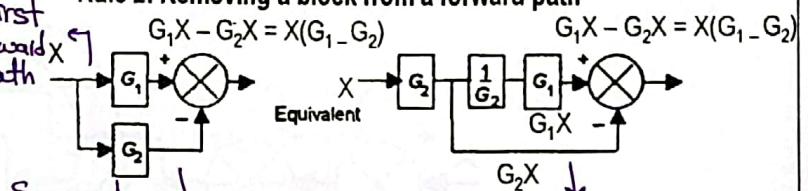


حولناه إلى مجموعتين في سلسلة

Rule 3: Eliminating feedback loop →

الـ block العلوي مقصوم على 1 محفوظ الإشارة
بالـ block العلوي بالـ block الأسفلية.
like Slide 21+22+26

Rule 2: Removing a block from a forward path



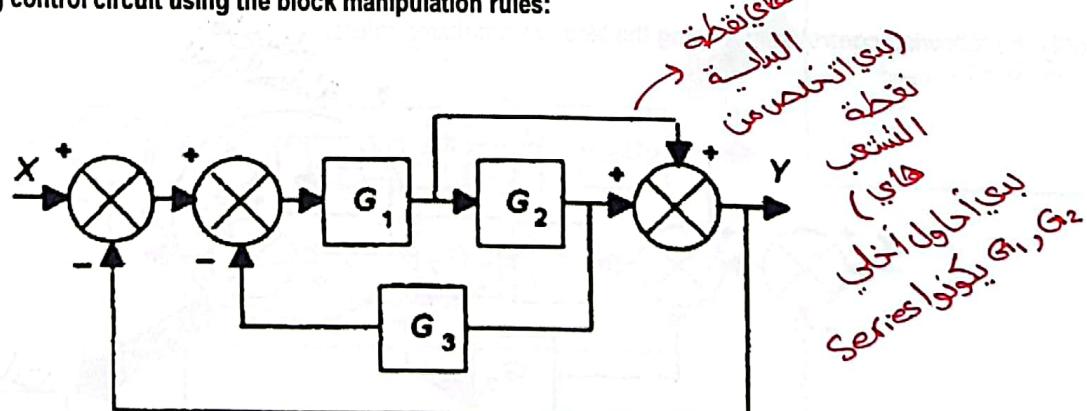
الـ Path الذي يمليها أي block سواء سلسلة unity gain Path أو feedback forward path كانت

فعلياً ما ينادي Signal system الـ مارقة زمي ماهي، وفي كأنه المـ System يدار عن block في حارقـ ١.

Example 1 on Control System Block Simplification

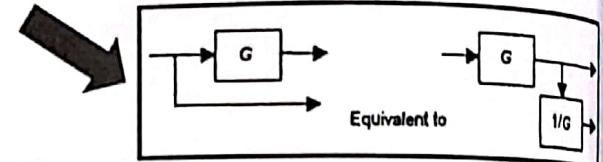
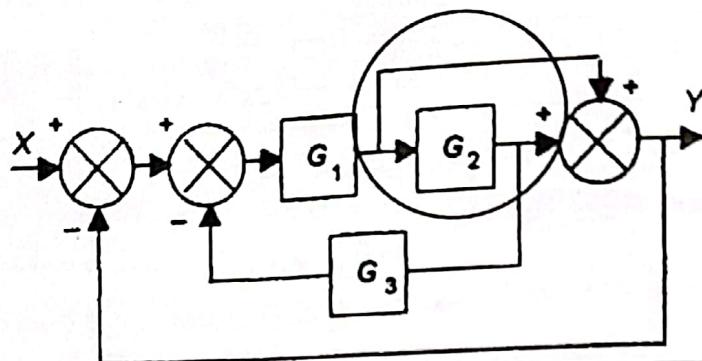
30

Simplify the following control circuit using the block manipulation rules:



Example 1 on Control System Block Simplification

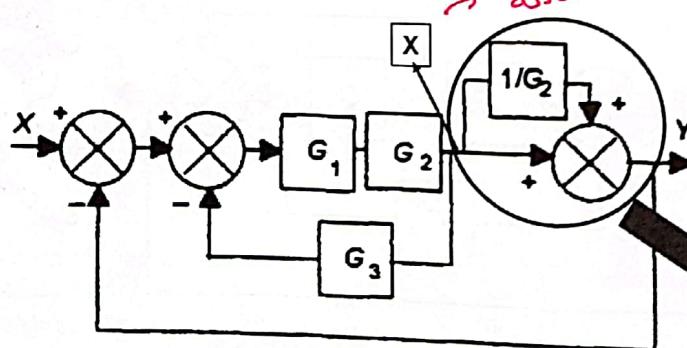
Simplify the following control circuit using the block manipulation rules:



Example 1 on Control System Block Simplification

Simplify the following control circuit using the block manipulation rules:

افتراض
نخل الحالة



حلينا بالرياضيات

$$Y = X + \frac{X}{G_2}$$

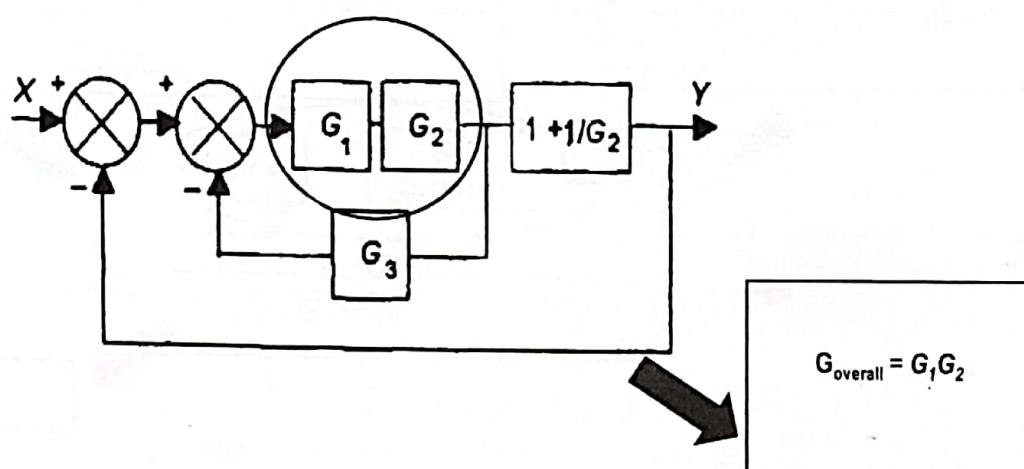
$$Y = X \left(1 + \frac{1}{G_2}\right)$$

$$\frac{Y}{X} = 1 + \frac{1}{G_2}$$

Example 1 on Control System Block Simplification

33

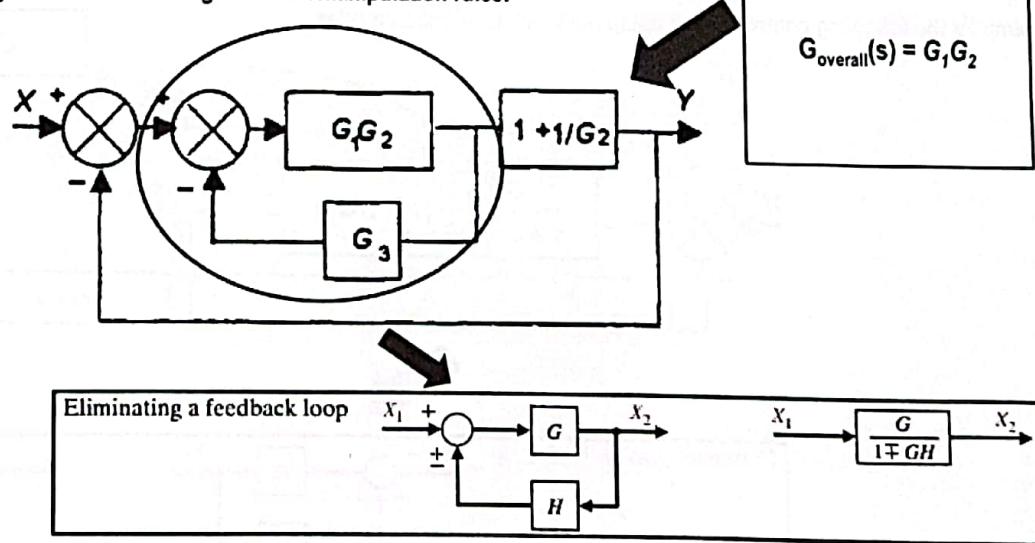
Simplify the following control circuit using the block manipulation rules:



Example 1 on Control System Block Simplification

34

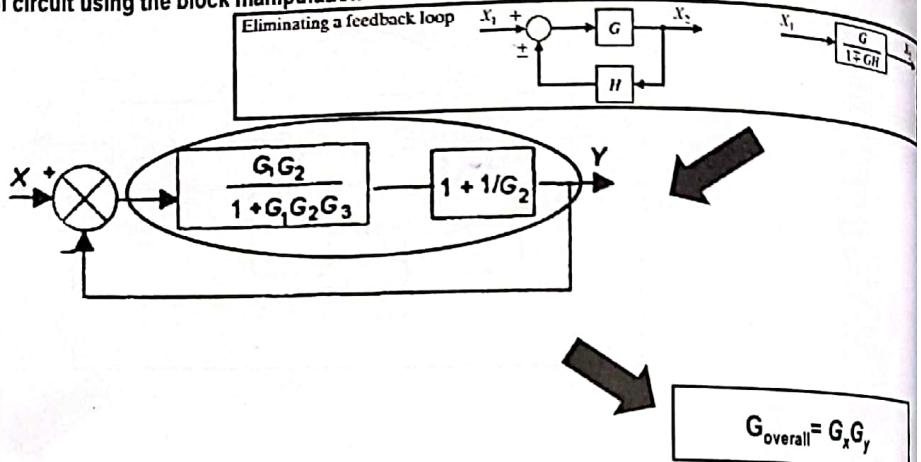
Simplify the following control circuit using the block manipulation rules:



Example 1 on Control System Block Simplification

35

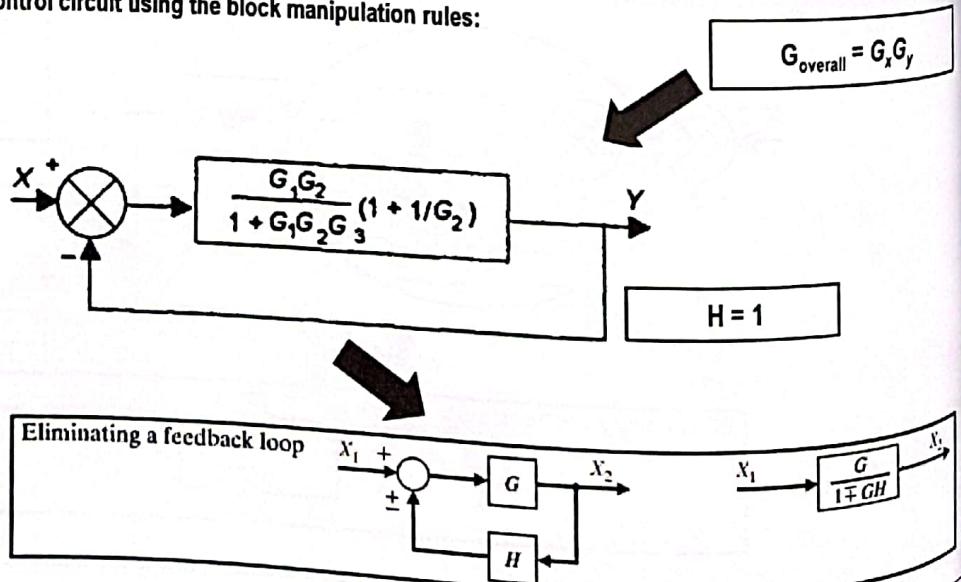
Simplify the following control circuit using the block manipulation rules:



Example 1 on Control System Block Simplification

36

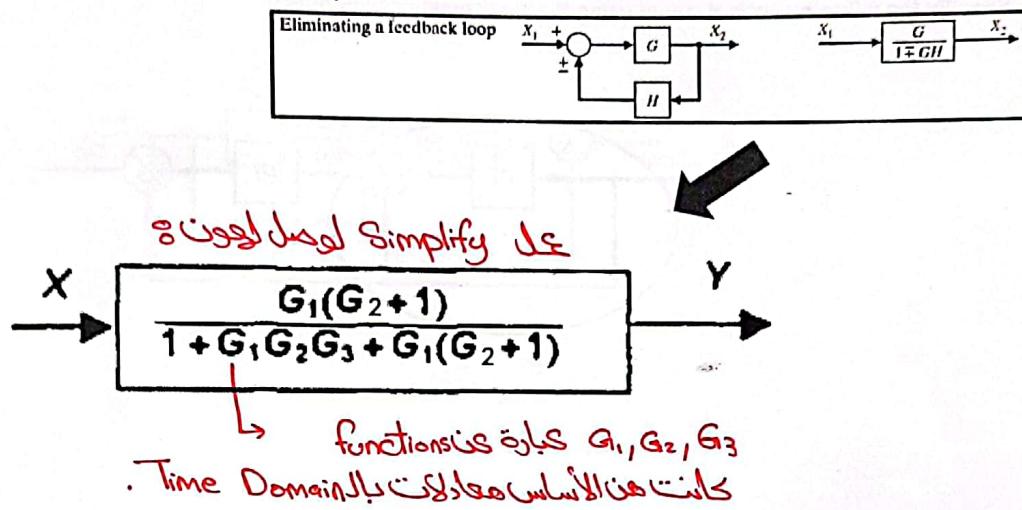
Simplify the following control circuit using the block manipulation rules:



Example 1 on Control Block Simplification

37

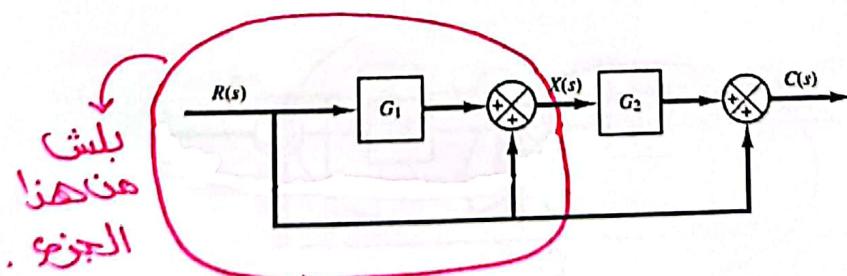
Simplify the following control circuit using the block manipulation rules:



Example 2 on Control System Block Simplification

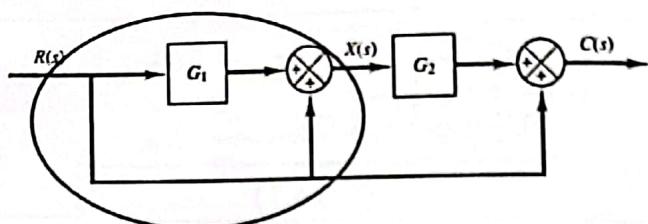
38

Simplify the following control circuit using the block manipulation rules:



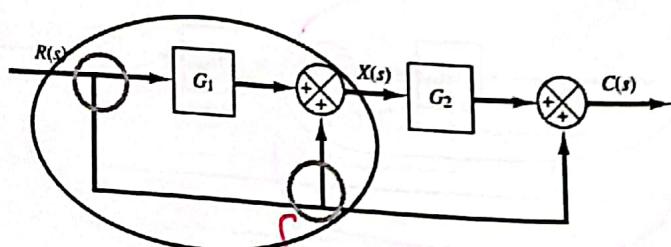
Example 2 on Control Block Simplification

Simplify the following control circuit using the block manipulation rules:



Example 2 on Control Block Simplification

Simplify the following control circuit using the block manipulation rules:

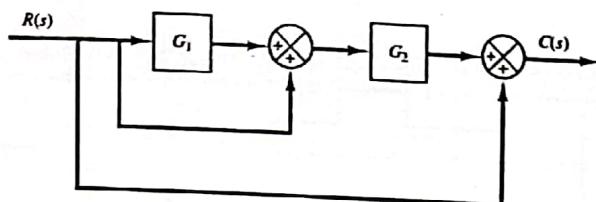


ما إنتم الآتین زعي يعنى
ربّهم بطريقته أفشل.

Example 2 on Control Block Simplification

41

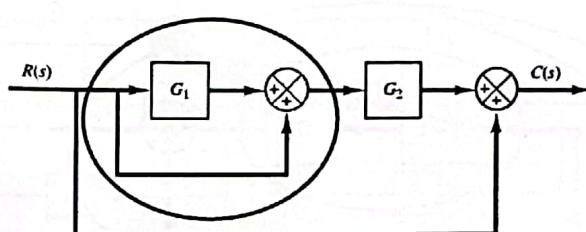
Simplify the following control circuit using the block manipulation rules:



Example 2 on Control Block Simplification

42

Simplify the following control circuit using the block manipulation rules:



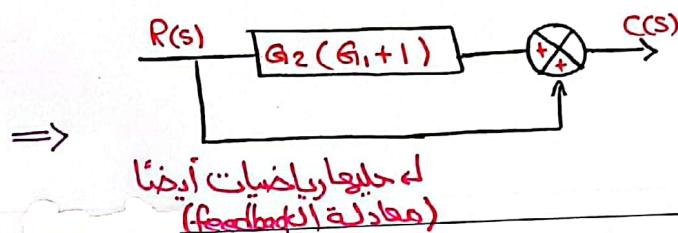
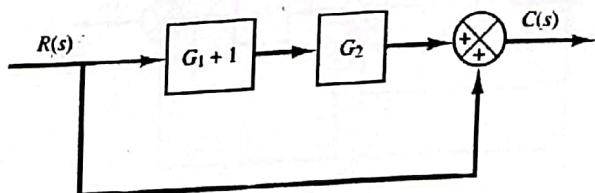
$$* Y = XG_1 + X$$

$$\rightarrow Y = X(G_1 + 1)$$

$$\rightarrow \frac{Y}{X} = (G_1 + 1)$$

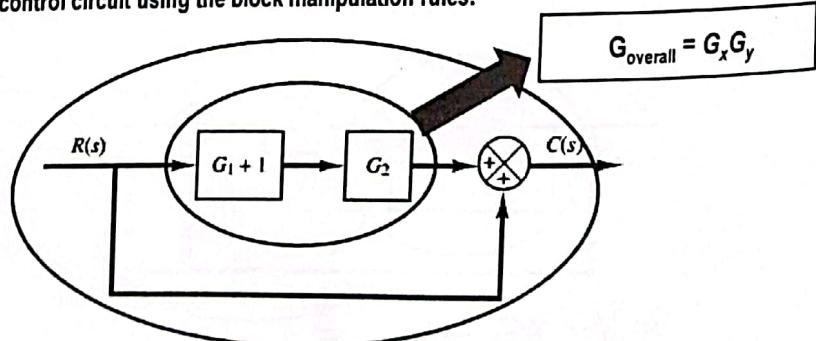
Example 2 on Control Block Simplification

Simplify the following control circuit using the block manipulation rules:



Example 2 on Control Block Simplification

Simplify the following control circuit using the block manipulation rules:

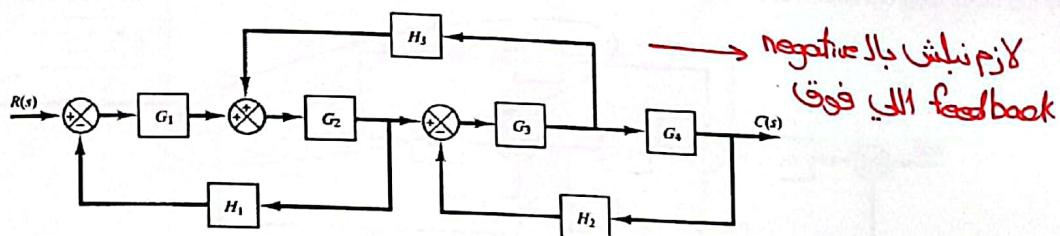


$$R(s) \rightarrow [G_1 G_2 + G_2 + 1] \rightarrow C(s)$$

Example 3 on Control Block Simplification

45

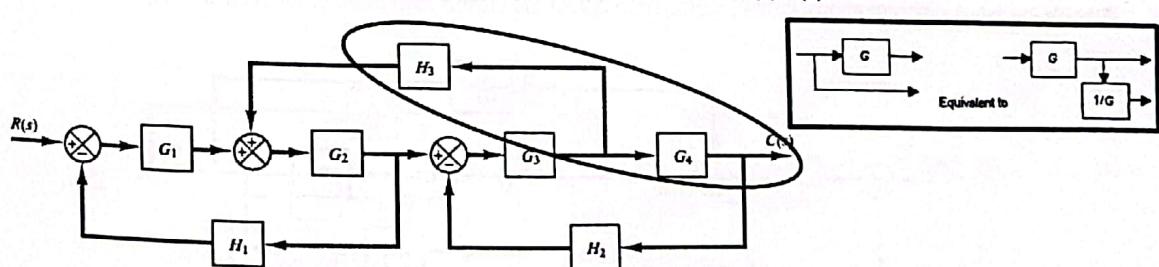
Simplify the block diagram shown in the Figure. Then obtain the closed-loop transfer function $C(s)/R(s)$.



Example 3 on Control Block Simplification

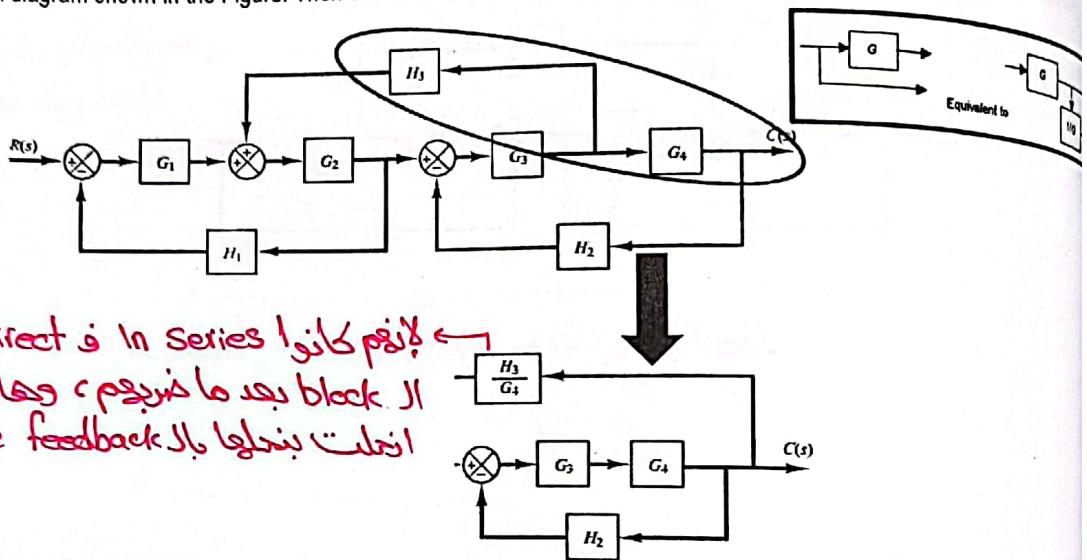
46

Simplify the block diagram shown in the Figure. Then obtain the closed-loop transfer function $C(s)/R(s)$.



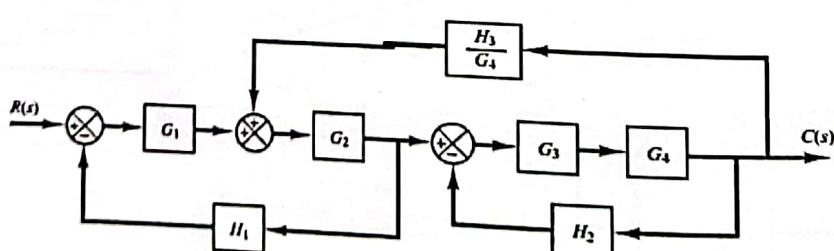
Example 3 on Control Block Simplification

Simplify the block diagram shown in the Figure. Then obtain the closed-loop transfer function $C(s)/R(s)$.



Example 3 on Control Block Simplification

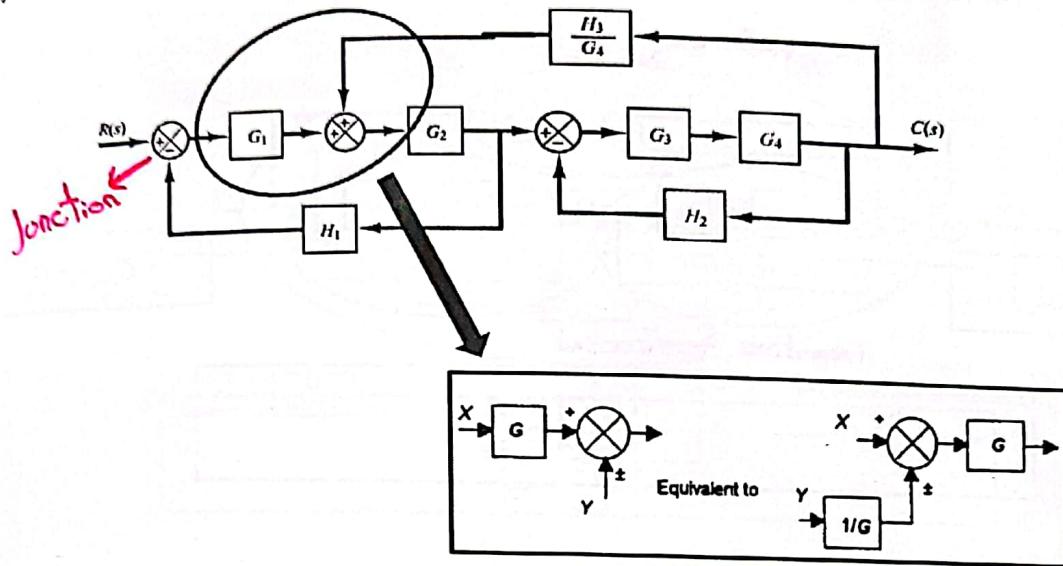
Simplify the block diagram shown in the Figure. Then obtain the closed-loop transfer function $C(s)/R(s)$.



Example 3 on Control Block Simplification

49

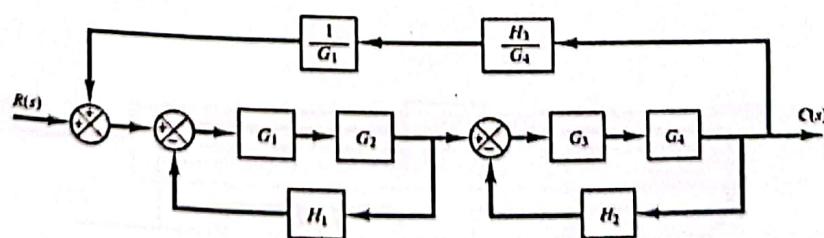
Simplify the block diagram shown in the Figure. Then obtain the closed-loop transfer function $C(s)/R(s)$.



Example 3 on Control Block Simplification

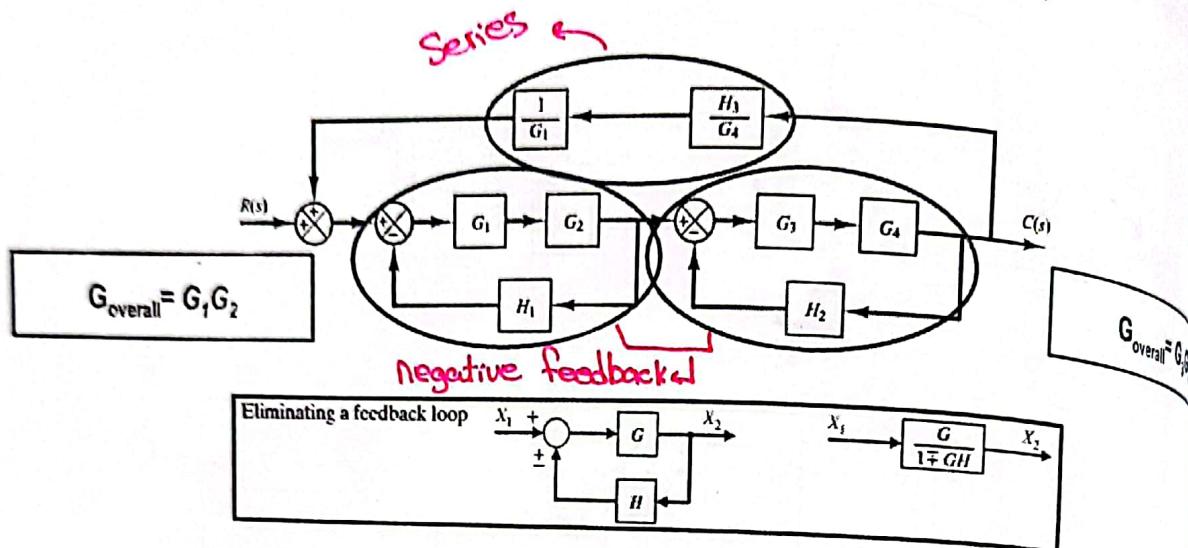
50

Simplify the block diagram shown in the Figure. Then obtain the closed-loop transfer function $C(s)/R(s)$.



Example 3 on Control Block Simplification

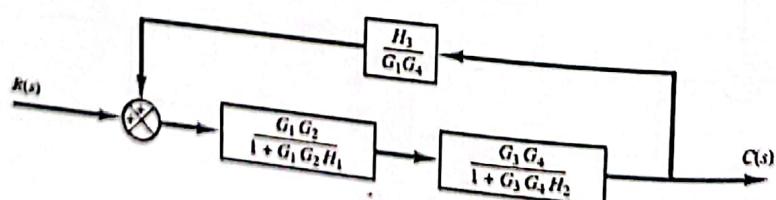
Simplify the block diagram shown in the Figure. Then obtain the closed-loop transfer function $C(s)/R(s)$.



Example 3 on Control Block Simplification

52

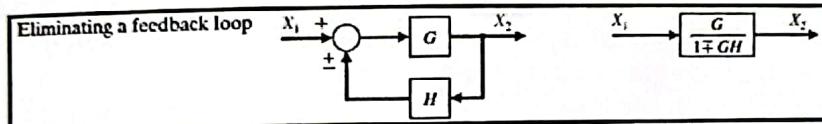
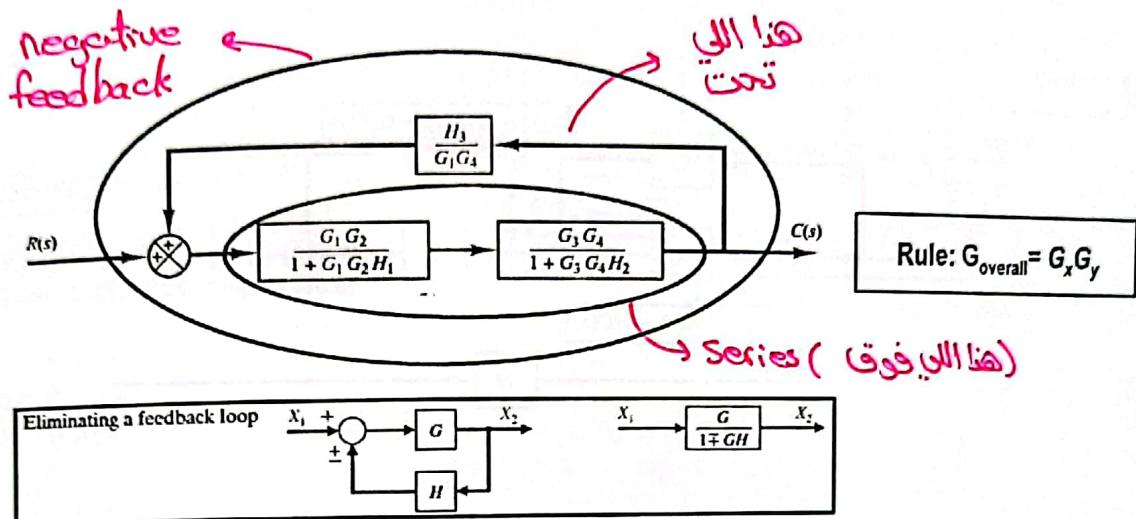
Simplify the block diagram shown in the Figure. Then obtain the closed-loop transfer function $C(s)/R(s)$.



Example 3 on Control Block Simplification

53

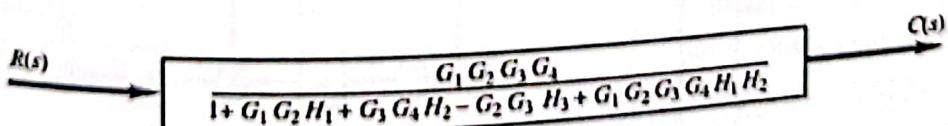
Simplify the block diagram shown in the Figure. Then obtain the closed-loop transfer function $C(s)/R(s)$.



Example 3 on Control Block Simplification

54

Simplify the block diagram shown in the Figure. Then obtain the closed-loop transfer function $C(s)/R(s)$.



* ملحوظة بنحتاج نجمع بين المدخل الرياحي والعمل blocks Simplified أو كان $\frac{1}{G}$ أو الرسمة كثيرة Complex

* معاوقة الـ loop = negative feedback loop

* أي block مثلاً معروفة بالـ unity gain block جواها رقم 1 (unity gain block)

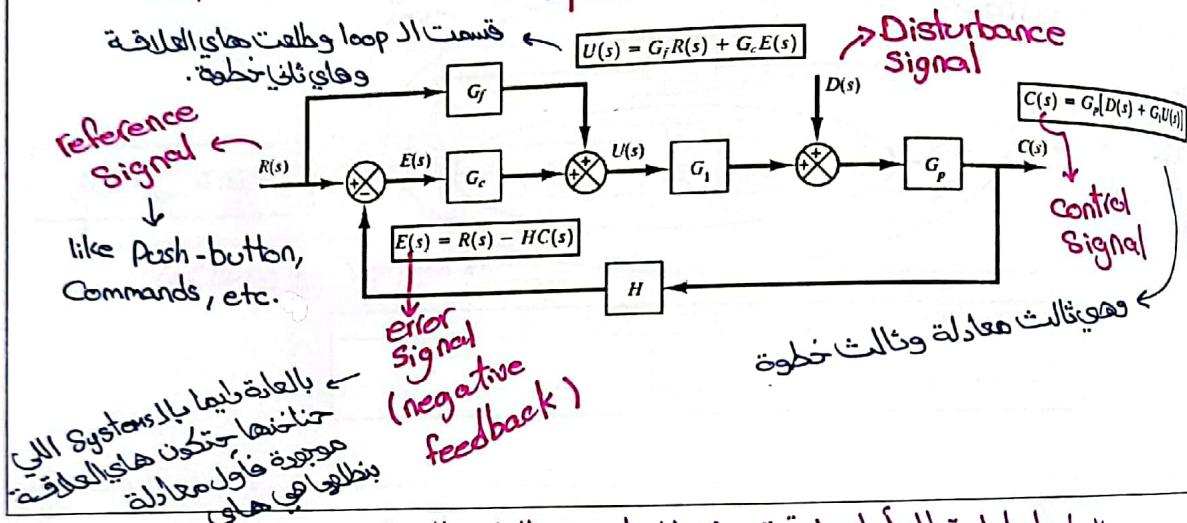
Example 4 on Control Block Simplification MISO

Using Equations

Mathematically obtain transfer functions $C(s)/R(s)$ and $C(s)/D(s)$ of the system shown in the Figure:

"حله باستخاد المعادلات"

Idle مدخلات قديمون فيها
noise.



* دائماً حاول تطبيق أول علاقة بين الـ input ولا feedback input أي loop بعين أي loop بالذريعة ورماعته بمعادلة وبعين أفرطهم بالعلاقات.

Example 4 on Control Block Simplification

Using Equations

Mathematically obtain transfer functions $C(s)/R(s)$ and $C(s)/D(s)$ of the system shown in the Figure:

Substitute U_s into C_s :

$$C(s) = G_p D(s) + G_i G_p [G_f R(s) + G_c E(s)]$$

Substitute E_s into the above equation:

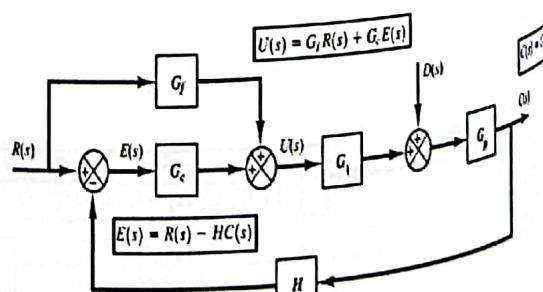
$$C(s) = G_p D(s) + G_i G_p \{G_f R(s) + G_c [R(s) - H C(s)]\}$$

Solve for C_s :

$$C(s) + G_i G_p G_i H C(s) = G_p D(s) + G_i G_p (G_f + G_c) R(s)$$

Rearrange:

$$C(s) = \frac{G_p D(s) + G_i G_p (G_f + G_c) R(s)}{1 + G_i G_p H}$$



* دائماً ابتعد عن الـ output على جملة ولا input على جملة ثانية.

Building this block or others in Matlab → commands (language) (بنية) (لغة)
Simulink → GUI (متقدمة بالمناعة) (متقدمة بالمناعة)

Example 4 on Control Block Simplification Using Equations

Mathematically obtain the transfer functions $C(s)/R(s)$ and $C(s)/D(s)$ of the system shown in the Figure:

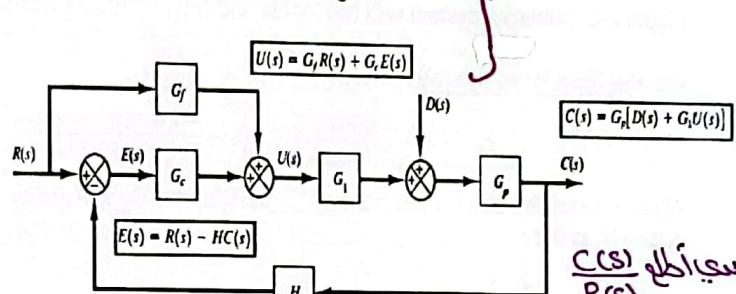
$$C(s) = \frac{G_p D(s) + G_1 G_p (G_f + G_c) R(s)}{1 + G_1 G_p G_c H}$$

Notice that this is a multiple input system, one for the actual Input $R(s)$, and another for some disturbance $D(s)$

To obtain the transfer functions $C(s)/R(s)$ and $C(s)/D(s)$, we must only let one input present, and all other inputs are 0, therefore:

$$C(s)/R(s) = \frac{C(s)}{R(s)} = \frac{G_1 G_p (G_f + G_c)}{1 + G_1 G_p G_c H}$$

$$C(s)/D(s) = \frac{C(s)}{D(s)} = \frac{G_p}{1 + G_1 G_p G_c H}$$



لوب يأخذوا باستخدام
ألا blocks مزة بصفه
R(s) و مزة بصفه D(s)

57

يعطي أطلع $\frac{C(s)}{R(s)}$
لباقي $D(s)$ متش
موجود (بعضه)
يعطي أطلع $\frac{C(s)}{D(s)}$
لباقي $R(s)$ متش
موجود (بعضه)

* بالعملية الأولى تحل ألا blocks ماستخدام الـ Multiple inputs
* بالعملية الثانية تحل ألا blocks ماستخدام الـ Single input

Example 5 on Control Block Simplification Using Equations MIMO

الطريقة الأولى :

Figure 2-24 shows a system with two inputs and two outputs. Derive $C_1(s)/R_1(s)$, $C_1(s)/R_2(s)$, $C_2(s)/R_1(s)$, and $C_2(s)/R_2(s)$

Solution:

$$\begin{aligned} C_1 &= G_1(R_1 - G_2 C_2) \\ C_2 &= G_4(R_2 - G_3 C_1) \end{aligned}$$

أخذ نقطة معينة
وحلوة يفتر أقوفه
C2 فينعا.

أخذ نقطة معينة
وحلوة يفتر أقوفه
C1 فينعا.

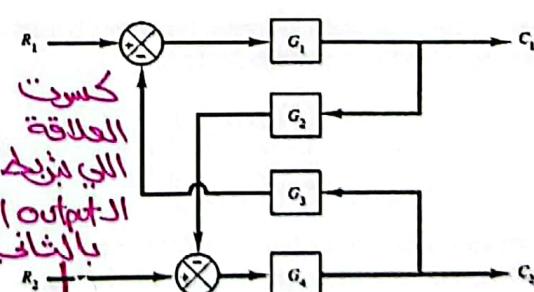
Substitute C_2 into C_1

Substitute C_1 into C_2

Solve the equation for C_1

$$\begin{aligned} C_1 &= G_1[R_1 - G_2 G_4(R_2 - G_3 C_1)] \\ C_1 &= G_1[R_1 - G_2 G_4(R_1 - G_1 C_1)] \end{aligned}$$

$$C_1 = \frac{G_1 R_1 - G_1 G_2 G_4 R_1}{1 - G_1 G_2 G_3 G_4}$$



Solve the equation for C_2

$$C_2 = \frac{-G_1 G_2 G_4 R_1 + G_4 R_2}{1 - G_1 G_2 G_3 G_4}$$

حارات كل معاشرة
عبارة عن واحد
output واحد
و عدد هن الـ blocks

58

2 inputs

2 outputs

Example 5 on Control Block Simplification Using Equations

Figure 2-24 shows a system with two inputs and two outputs. Derive $C_1(s)/R_1(s)$, $C_1(s)/R_2(s)$, $C_2(s)/R_1(s)$, and $C_2(s)/R_2(s)$. We now have these two relationships:

$$C_1 = \frac{G_1 R_1 - G_1 G_2 G_3 G_4 R_2}{1 - G_1 G_2 G_3 G_4} \quad C_2 = \frac{-G_1 G_2 G_4 R_1 + G_4 R_2}{1 - G_1 G_2 G_3 G_4}$$

We can derive the transfer functions of $C_1(s)/R_1(s)$ and $C_2(s)/R_1(s)$ by setting R_2 to 0, so

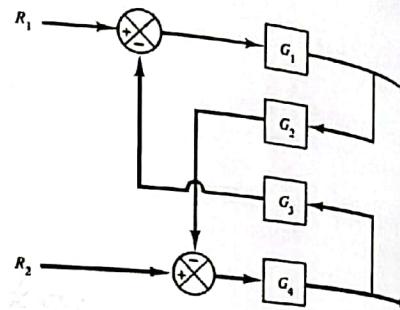
$$\frac{C_1(s)}{R_1(s)} = \frac{G_1}{1 - G_1 G_2 G_3 G_4},$$

$$\frac{C_2(s)}{R_1(s)} = -\frac{G_1 G_2 G_4}{1 - G_1 G_2 G_3 G_4}.$$

Similarly, we can derive the transfer functions of $C_1(s)/R_2(s)$ and $C_2(s)/R_2(s)$ by setting R_1 to 0, so

$$\frac{C_1(s)}{R_2(s)} = -\frac{G_1 G_3 G_4}{1 - G_1 G_2 G_3 G_4}$$

$$\frac{C_2(s)}{R_2(s)} = \frac{G_4}{1 - G_1 G_2 G_3 G_4}$$



Example 5 on Control Block Simplification Using Equations

Figure 2-24 shows a system with two inputs and two outputs. Derive $C_1(s)/R_1(s)$, $C_1(s)/R_2(s)$, $C_2(s)/R_1(s)$, and $C_2(s)/R_2(s)$.

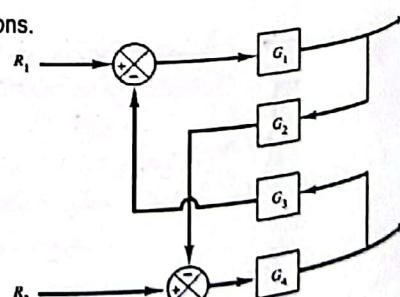
In control systems, we prefer to present the system in matrix/vector notations. The equations can be combined in this form:

$$\begin{bmatrix} C_1 \\ C_2 \end{bmatrix} = \begin{bmatrix} \frac{G_1}{1 - G_1 G_2 G_3 G_4} & -\frac{G_1 G_2 G_4}{1 - G_1 G_2 G_3 G_4} \\ \frac{G_1 G_3 G_4}{1 - G_1 G_2 G_3 G_4} & \frac{G_4}{1 - G_1 G_2 G_3 G_4} \end{bmatrix} \begin{bmatrix} R_1 \\ R_2 \end{bmatrix}$$

Then the transfer functions $C_1(s)/R_1(s)$, $C_1(s)/R_2(s)$, $C_2(s)/R_1(s)$ and $C_2(s)/R_2(s)$ can be obtained as follows:

$$\frac{C_1(s)}{R_1(s)} = \frac{G_1}{1 - G_1 G_2 G_3 G_4}, \quad \frac{C_1(s)}{R_2(s)} = -\frac{G_1 G_2 G_4}{1 - G_1 G_2 G_3 G_4}$$

$$\frac{C_2(s)}{R_1(s)} = -\frac{G_1 G_3 G_4}{1 - G_1 G_2 G_3 G_4}, \quad \frac{C_2(s)}{R_2(s)} = \frac{G_4}{1 - G_1 G_2 G_3 G_4}$$



Example 5 on Control Block Simplification

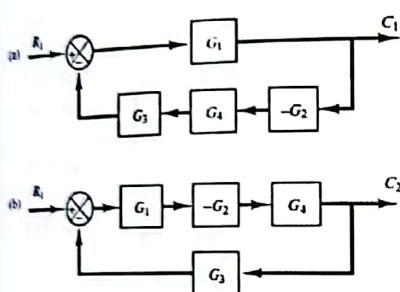
Using Equations – Alternative Solution

When setting $R_2 = 0$, the block diagram can actually be simplified to:

طريقة ثانية
للحل

$$R_1 \rightarrow \text{مدة صفرنا } R_2 \text{ وظلت معادلة} \rightarrow \frac{C_1}{R_1}, \frac{C_2}{R_1}$$

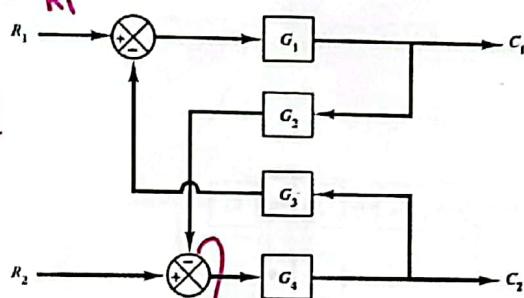
$$R_2 \rightarrow \text{مدة صفرنا } R_1 \text{ وظلت معادلة} \rightarrow \frac{C_1}{R_2}, \frac{C_2}{R_2}$$



$$\frac{C_1}{R_1} = \frac{G_1}{1 - G_1 G_2 G_3 G_4}$$

$$\frac{C_2}{R_1} = \frac{-G_1 G_2 G_4}{1 - G_1 G_2 G_3 G_4}$$

$$\frac{C_2}{R_2} = \frac{G_4}{1 - G_1 G_2 G_3 G_4}$$

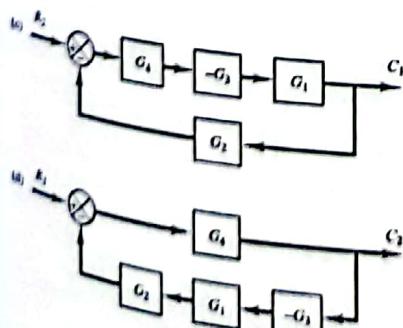


شائكة 6.2 بقيت
 موجودة بعد
 موت الـ junction.

Example 5 on Control Block Simplification

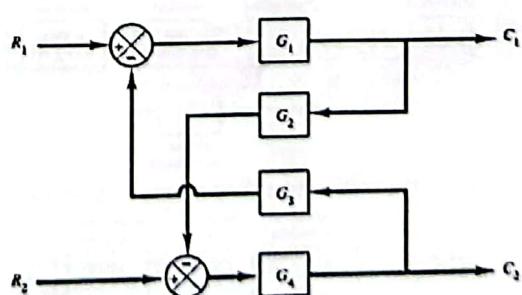
Using Equations – Alternative Solution

When setting $R_1 = 0$, the block diagram actually can be simplified to:



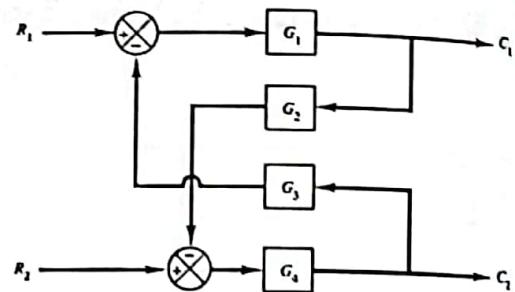
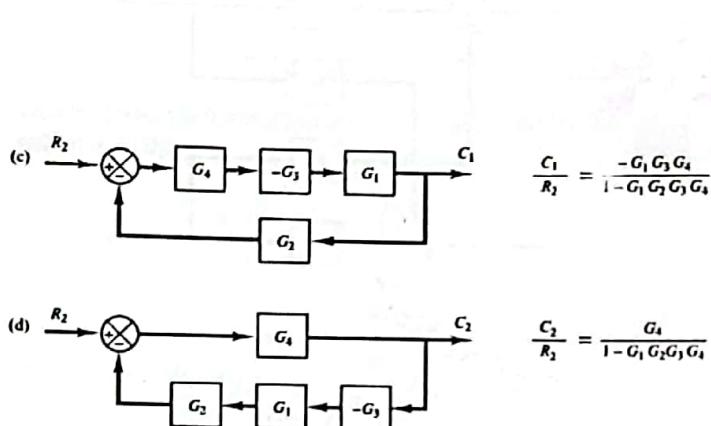
$$\frac{C_1}{R_2} = \frac{-G_1 G_3 G_4}{1 - G_1 G_2 G_3 G_4}$$

$$\frac{C_2}{R_1} = \frac{G_2}{1 - G_1 G_2 G_3 G_4}$$



Example 5 on Control Block Simplification Using Equations – Alternative Solution

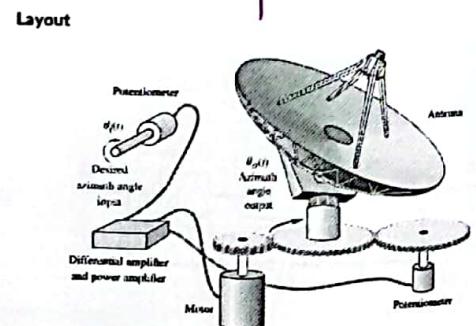
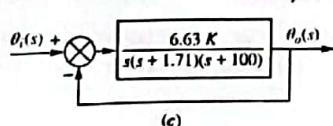
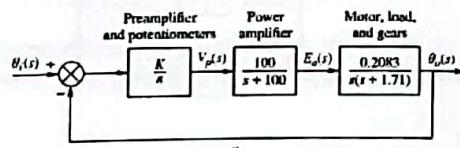
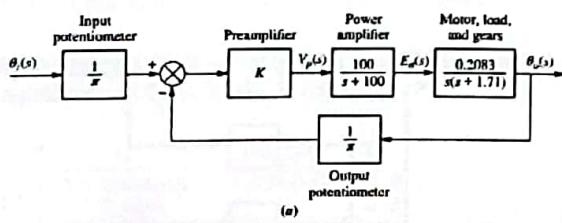
When setting $R_1 = 0$, the block diagram actually can be simplified to:



يسوفى أننا وصلت معه ولا معلومات تجده.
تبعد Antenna ومن آل Antenna بخطى مشان feedback بخطى Antenna ←
Amplifier بخطى Motor الي بعله gears ←

كل معاشرات كهربائية وmekanikite ←
وهكذا، وعلنا كل وعده خطى S-Domain ←

Example 6 on Control Block Simplification Antenna Azimuth Control System



وعلنا العلاقة بين
الزاوية الي بعلها في الـ
Potentiometers والزاوية الي
بتذرلها المستلبيت

$$\frac{\theta_o(s)}{\theta_i(s)} = \frac{6.63 K}{s^3 + 101.7 s^2 + 171 s + 6.63 K}$$

لـ العرق سانخلص إني أكتب كل واحد بالـ S-Domain هشنان الحسيمة الزوايا
الي درملها حتى بعين الـ System كاهم بالـ S-Domain .

Partial Fractions Form

بعطوك النتيجة المكافئة وانت
فكلي لها ورجوها لأصولها.

65

Now that we have found transfer functions that relate the control system outputs to its inputs, this transfer function can be analyzed to find the control system parameters that we have discussed in Chapter 1: settling time, overshoot, stability, etc.

But first, we need to mathematically transform the function $\frac{1}{s(s+4)(s+8)}$ into the partial fractions form, from which it is even easier to do an inverse Laplace transform to get back to the time-domain. It is mathematically easy to do so, we shall do it in MATLAB in self-learning material.

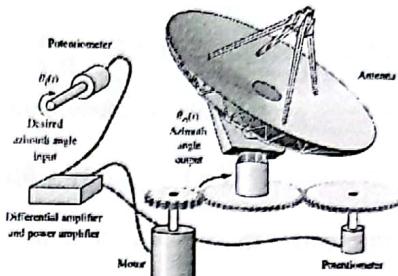
For example: the function $Y(s)$ below can be rewritten as:

$$Y(s) = \frac{32}{s(s+4)(s+8)} = \frac{1}{s} - \frac{2}{(s+4)} + \frac{1}{(s+8)}$$

(علاقة تكاملية)

Partial Fractions Form is عكس توحيد المقام والعودة إلى الكسور الأصلية
المعادلات مثل المعادلة
الكافائية (الذكجية)

Layout



$$\theta_r(s) \xrightarrow{\text{Differential amplifier and power amplifier}} \frac{6.63 K}{s^2 + 101.71 s^2 + 171 s + 6.63 K} \xrightarrow{\text{Motor}} \theta_d(s)$$

(d)

تحتكر شغنا بالـ dh
الباقي كشان نعمل
الـ System

Math Review: Partial Fractions Example 1 (Optional)

66

Determine the partial fractions of:

$$\frac{s+4}{(s+1)(s+2)}$$

The partial fractions are of the form:

$$\frac{A}{s+1} + \frac{B}{s+2}$$

Then, for the partial fraction expression to equal the original fraction, we must have: $\frac{s+4}{(s+1)(s+2)} = \frac{A(s+2) + B(s+1)}{(s+1)(s+2)}$

And consequently: $s+4 = A(s+2) + B(s+1)$

Pick values of s that will enable some of the terms involving constants to become zero and so enable other constants to be determined:

Let $s = -2$ then $(-2)+4 = A(-2+2) + B(-2+1)$
so $B = 2$

Let $s = -1$ then $(-1)+4 = A(-1+2) + B(-1+1)$
so $A = 3$

Therefore $\frac{s+4}{(s+1)(s+2)} = \frac{3}{s+1} - \frac{2}{s+2}$

Math Review: Partial Fractions Example 2 (Optional)

67

Determine the partial fractions of: $\frac{3s+1}{(s+2)^3}$

The partial fractions are of the form: $\frac{A}{(s+2)} + \frac{B}{(s+2)^2} + \frac{C}{(s+2)^3}$

Then, for the partial fraction expression to equal the original fraction, we must have: $\frac{3s+1}{(s+2)^3} = \frac{A}{(s+2)} + \frac{B}{(s+2)^2} + \frac{C}{(s+2)^3}$

and consequently: $3s+1 = A(s+2)^2 + B(s+2) + C = A(s^2 + 4s + 4) + B(s+2) + C$

We start equating the same power terms on each side to determine A, B, and C

Equating s^2 terms gives $0 = A$. Equating s terms gives $3 = 2A + B$ and so $B = 3$. Equating the numeric terms gives $1 = A + 2B + C$ and so $C = -5$. Thus:

$$\frac{3s+1}{(s+2)^3} = \frac{3}{(s+2)^2} - \frac{5}{(s+2)^3}$$

Math Review: Partial Fractions Example 3 (Optional)

68

Determine the partial fractions of: $\frac{2s+1}{(s^2+s+1)(s+2)}$

The partial fractions are of the form: $\frac{As+B}{s^2+s+1} + \frac{C}{s+2}$

Then, for the partial fraction expression to equal the original fraction, we must have: $\frac{2s+1}{(s^2+s+1)(s+2)} = \frac{As+B}{s^2+s+1} + \frac{C}{s+2}$

and consequently: $2s+1 = (As+B)(s+2) + C(s^2+s+1)$

We start equating the same power terms on each side to determine A, B, and C

With $S = -2$ then $-3 = 3C$ and so $C = -1$. Equating s^2 terms gives $0 = A + C$ and so $A = 1$. Equating s terms gives $2 = 2A + B + C$ and so $B = 1$. Thus

$$\frac{2s+1}{(s^2+s+1)(s+2)} = \frac{s+1}{s^2+s+1} - \frac{1}{s+2}$$

Math Review: Partial Fractions Example 4 (Optional)

69

Determine the partial fractions of: $\frac{2s^2 + 2}{(s+4)(s-2)}$

Notice the power of numerator and denominator is the same

We must first use division:

$$s^2 + 2s - 8 \overline{)2s^2 + 2}$$
$$\underline{2s^2 + 4s - 16}$$
$$-4s + 18$$

And the expression becomes:

$$2 + \frac{-4s + 18}{(s+4)(s-2)}$$

Then, for the partial fraction expression to equal the original fraction, we must have: $\frac{-4s + 18}{(s+4)(s-2)} = \frac{A}{s+4} + \frac{B}{s-2}$

and consequently: $-4s + 18 = A(s-2) + B(s+4)$

We start equating the same power terms on each side to determine A, B, and C

With s = 2, then B = 5/3. With s = -4, then A = -17/3. Thus, the expression can be written as:

$$2 - \frac{17}{3(s+4)} + \frac{5}{3(s-2)}$$

References and Textbook Material

70

The material in these slides are based on:

Control Systems Engineering, Norman S. Nise, 7th Edition (2014), John Wiley And Sons

- **Chapter 2 – System Modelling in the Frequency Domain**

Sections 2.1, 2.2, 2.3, 2.4, 2.5, 2.6 (Students kindly note that these sections involve lots of math, and we only described the ideas as we will use MATLAB instead)

- **Chapter 5 – Reduction of Multiple Subsystems**

Sections 5.1, 5.2

Instrumentation and Control Systems, 1st Edition , 2004, Elsevier (Newer versions available 3rd edition 2021, but I used the first one)

- **Chapter 9 – Transfer Function**

Sections 9.1, 9.2, 9.3, 9.4, 9.5

6/11/2022

(Self-learning Material 3+4)

Partfrac ($F(s)$) ← Partial Fractions ← Page 5

ويعطيك لها بالشكل البسيط .

* بأول بحثك كيف تعرف العناصر باستخدام الـ Functions وـ Syms وكيف يدخل
Multiple vars وـ Symbolic var وكيف يمكن تعيين
تقديراتك مثل وظائف المتغيرات المختلفة ، Sub Index وـ Variables

Part 2 *

Time Domain وـ laplace ← ينكره معرف المعادلة بالـ Time Domain ← laplace ←
S-Domain الـ

مهمة جداً إذا أردت laplace Function وـ بحثك العبار بنفس
الـ input أو laplace الـ input أو عامل خارج انت مهتم
الـ S-Domain الـ ibplace ← تعطيك المعادلة الـ Time Domain

* آخر مراجعة للـ Part Polynomials

* أمان بحثك إذا أردت تشغيل الـ symbolic (التحول
مع البسط لـ الـ symbolic) في Command بتحول بين الـ symbolic والـ
numerical command ، numerical command ، symbolic كل واحد به متغير
Symbolic Numerical Partial fraction does ← residue *



Time Response

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THE UNIVERSITY OF JORDAN

DEPARTMENT OF COMPUTER ENGINEERING
FALL 2022

2

Introduction

- After the engineer obtains a mathematical representation of a subsystem, the subsystem is analyzed for its transient and steady-state responses to see if these characteristics yield the desired behavior.
- We already learnt how transfer functions can represent linear, time-invariant systems in the s-domain.
- Engineers need to evaluate the response of a subsystem prior to inserting it into the overall closed-loop system.
- Engineers need to evaluate the response of the overall system as well.

Poles, Zeros, and System Response

- The output response of a system is the sum of two responses:
 - Forced response (also called steady-state response, or particular solution)
 - Natural response (also called homogenous solution)
- Solving in the time-domain or inverse Laplace transform is time-consuming and laborious.
- We use the transfer function, and the new concepts of poles and zero to reach qualitative and quantitative solutions to learn quickly all about our system

Poles and Zeros of a Transfer Function

- The poles of a transfer function are
 1. The values of the Laplace transform variable, s , that cause the transfer function to become infinite or
 2. Any roots of the denominator of the transfer function that are common to roots of the numerator.
- The zeros of a transfer function are
 1. The values of the Laplace transform variable, s , that cause the transfer function to become zero, or
 2. Any roots of the numerator of the transfer function that are common to roots of the denominator

$s = -2$ (zero)

$$G(s) = \frac{s+2}{s+5}$$

$s = -5$ (pole)

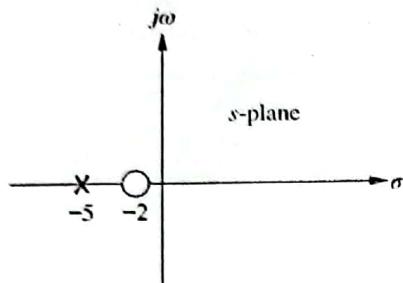
$$H(s) = \frac{(s+2)(s+4)(s-1)}{(s-1)(s-2)(s+5)(s+1)}$$

Zeros of $H(s) \rightarrow -4, -2, 1$

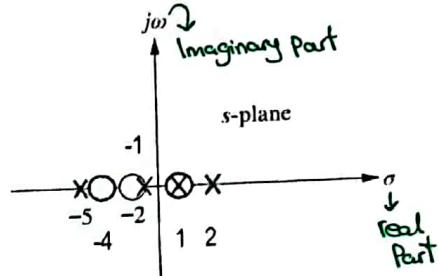
Poles of $H(s) \rightarrow -5, -1, 1, 2$

Poles and Zeros - S-Plane

$$G(s) = \frac{s+2}{s+5}$$



$$H(s) = \frac{(s+2)(s+4)(s-1)}{(s-1)(s-2)(s+5)(s+1)}$$

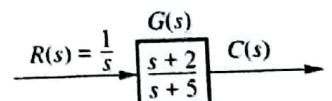


Poles and Zeros of a First-Order System I

- The system $G(s)$ is a first-order system, given that the denominator has a maximum power for s as 1
- Suppose that the input of this system is a simple input switch, which in the time domain is represented as a unit step function, which in the s -domain is represented as $\frac{1}{s}$
- The total system response is therefore:

$$C(s) = \frac{(s+2)}{s(s+5)} = \frac{2/5}{s} + \frac{3/5}{s+5} \quad c(t) = \frac{2}{5} + \frac{3}{5}e^{-5t}$$

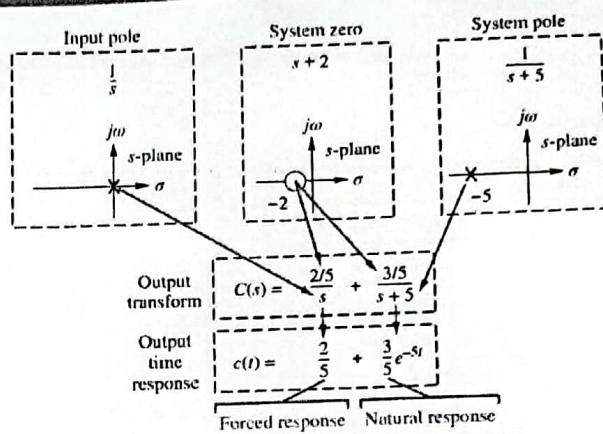
$$G(s) = \frac{s+2}{s+5}$$



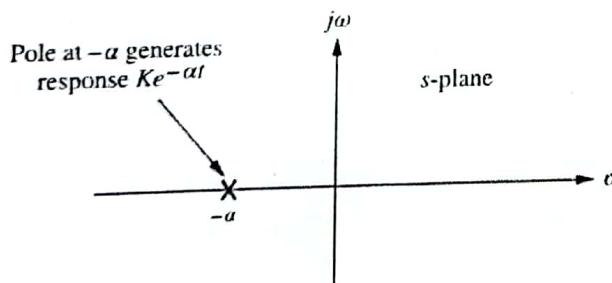
Poles and Zeros of a First-Order System II

► Analysis:

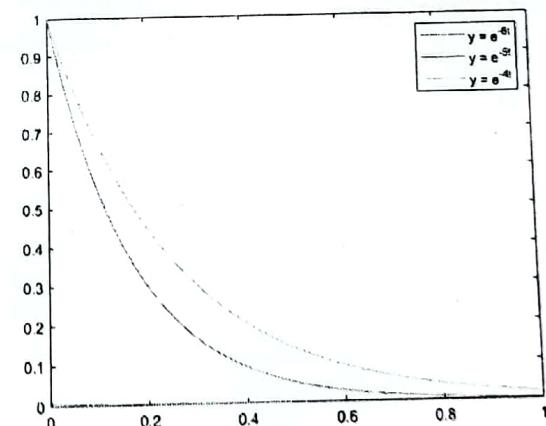
1. A pole of the input function generates the form of the forced response (that is, the pole at the origin generated a step function at the output).
2. A pole of the transfer function generates the form of the natural response (that is, the pole at -5 generated e^{-5t}).
3. A pole on the real axis generates an exponential response of the form $e^{-\alpha t}$, where $-\alpha$ is the pole location on the real axis.
4. The zeros and poles generate the amplitudes for both the forced and natural responses



Effect of a real-axis pole upon transient response



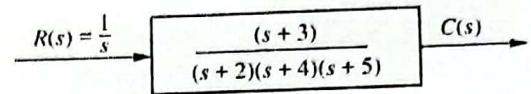
Thus, the farther to the left a pole is on the negative real axis, the faster the exponential transient response will decay to zero. Remember the natural response of the system must decay at some point to only leave the system with the forced response, otherwise, it will be an unstable system



Evaluating Response Using Poles – Example I

- Given the system in the adjacent figure, write the output, $c(t)$, in general terms by inspection. Specify the forced and natural parts of the solution.

$$C(s) \equiv \underbrace{\frac{K_1}{s}}_{\text{Forced response}} + \underbrace{\frac{K_2}{s+2} + \frac{K_3}{s+4} + \frac{K_4}{s+5}}_{\text{Natural response}}$$



$$c(t) \equiv \underbrace{K_1}_{\text{Forced response}} + \underbrace{K_2 e^{-2t} + K_3 e^{-4t} + K_4 e^{-5t}}_{\text{Natural response}}$$

- Notice that the transfer function quickly made us understand how the output would be like.
- To know the amplitudes of each term (The K s), write $C(s)$ in MATLAB and use the **partfrac (symbolic)** command or **residue (numeric)** command

Evaluating Response Using Poles – Example II

- Given the system in the adjacent figure, write the output, $c(t)$, in general terms by inspection if the input is a unit step. Specify the forced and natural parts of the solution.

- Given that the input is $\frac{1}{s}$, $C(s) = \frac{1}{s} G(s)$

$$G(s) = \frac{10(s+4)(s+6)}{(s+1)(s+7)(s+8)(s+10)}$$

$$c(t) \equiv A + Be^{-t} + Ce^{-7t} + De^{-8t} + Ee^{-10t}$$

Analysis of First Order Systems without Zeros

- If the input is a unit step, where $R(s) = 1/s$, the Laplace transform of the step response $C(s) = R(s)G(s)$

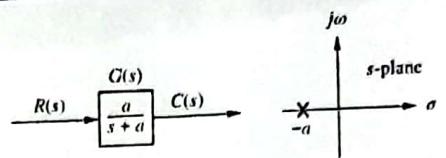
$$C(s) = R(s)G(s) = \frac{a}{s(s+a)}$$

- The inverse Laplace Transform is

$$c(t) = c_f(t) + c_n(t) = 1 - e^{-at}$$

Where the input pole at the origin generated the forced response $c_f(t) = 1$, and the system pole at $-a$ generated the natural response $c_n(t) = -e^{-at}$

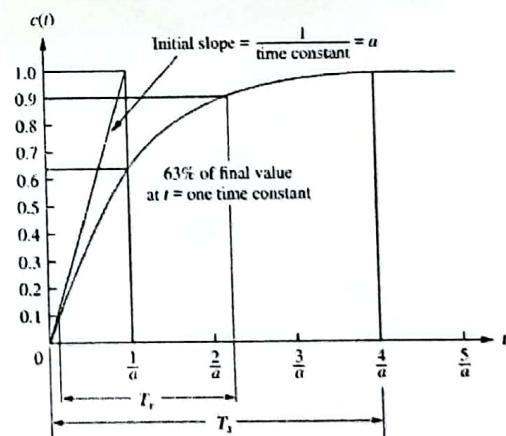
when $t = \frac{1}{a}$ the exponential $e^{-at} = e^{-a/a} = e^{-1} = 0.37$. And $c(t) = 1 - 0.37 = 0.63$



```
>> syms a s
>> syms G(s)
>> G(s) = a / (s*(s+a))
G(s) =
a/(s^2*(a+s))
>> ilaplace(G(s))
ans =
1 - exp(-a*t)
```

Time Constant of a First Order System

- We call $\frac{1}{a}$ the time constant of the response.
- The time constant can be described as the time for e^{-at} to decay to 37% of its initial value, and that $c(t)$ reaches 63% of its final value.
- The reciprocal of the time constant has the units (1/seconds), or frequency. Thus, we can call the parameter 'a' the exponential frequency.
- Thus, the time constant can be considered a transient response specification for a first-order system, since it is related to the speed at which the system responds to a step input.
- The time constant can also be evaluated from the pole plot. Since the pole of the transfer function is at a , we can say the pole is located at the reciprocal of the time constant, and the farther the pole from the imaginary axis, the faster the transient response.



First-order system response to a unit step

Rise Time and Settling Time of First Order Systems

- **Rise time (T_r)** is defined as the time for the waveform to go from 10% to 90% of its final value.
- To derive the equation, we know that the time-domain equation of a first-order system is $C(t) = 1 - e^{-at}$, and that at $t = 0$, $c(t)$ equals 0, and as time goes by, $c(t)$ will near to be 1, so the range is 0 to 1
 - ❖ 90% of the final value is $0.9 = 1 - e^{-at_{90}} \rightarrow e^{-at_{90}} = 0.1$ then solve for t by taking the natural logarithm for both sides
 - ❖ 10% of the final value is $0.1 = 1 - e^{-at_{10}} \rightarrow e^{-at_{10}} = 0.9$ then solve for t by taking the natural logarithm for both sides
 - ❖ $t_{90} = \frac{2.31}{a}$
 - ❖ $t_{10} = \frac{0.11}{a}$
 - ❖ $\text{Rise time } (T_r) = t_{90} - t_{10} = \frac{2.2}{a}$
- **Settling time (T_s)** is defined as the time for the response to reach, and stay within, 2% of its final value.
 - ❖ $T_s = \frac{4}{a}$

Rise Time and Settling Time Example

PROBLEM: A system has a transfer function, $G(s) = \frac{50}{s + 50}$. Find the time constant, T_c , settling time, T_s , and rise time, T_r .

ANSWER: $T_c = 0.02$ s, $T_s = 0.08$ s, and $T_r = 0.044$ s.

(Time Division)

(Time Response)

* هو خريط Ch1 بالبي وصلاته بـ Ch2 .

* ممكناً تطبقه خصائص هذا chapter على كل block على حدة أو لا block المترابطة.

* All the Systems المترابطة كلها تكون Time-Invariant linear و Zeros تكون Poles .

* من خلالهم ينقر نحل System Poles and Zeros (8 Slide 3) .

* جاء من Input بطيء System (خارجي) .

* $\text{Total response} = \text{Forced response} + \text{Natural response}$

(initial State) .

* يومنا إنما System ينبع المخرج المترابط أنه يعتمد على forced response .

* مثل شغلاته ومعالجاته المترابطة وأي response اله يموت مع الزمن لينقر

. System Stable .

* في أطلاع Time-Domain بال Natural response وال Forced response .

. laplace .

. System response النظر الأفضل للنتائج ← qualitative *

. أطلاع response (أوصي) ← quantitative *

: (Poles and Zeros examples) : Slide 4 *

$$* S+2 \rightarrow s = -2 \text{ (Zeros)}$$

$$S+5 \rightarrow s = -5 \text{ (Poles)}$$

$$* H(s) = \frac{(s+2)(s+4)(s-1)}{(s-1)(s-2)(s+5)(s+1)}$$

$$\text{Zeros of } H(s) \rightarrow -4, -2, 1$$

← (s-1) مشترك بين البسط والمقام فأجمعوا

انه يحسبوا بال Zeros وبال Poles أيهما

أهم فالبسط هي ال Zeros وأهم للمقام هي ال Poles .

Poles and Zeros on S-plane (complex plane) : Slide 5 *

fourier & laplace ال S بـ $s = \sigma + j\omega$ = laplace . σ تعرف ال Real part و $j\omega$ ال Imaginary part .

فبنكون ال X-axis (σ) Real part ($j\omega$) Imaginary part .

zeros = 0 ← . X-axis على ال Poles وال Zeros بالعادة يرسموا على S-plane *

Poles = x ← ← يحيط بالنظر لأعلى سرعة المترابط .

لـ كانه كثبان صارفة

فتشطب ال System

Steady-State بالوinkel للـ

Second order / First order System كمان ← 8 Slide 6 *

وهما ينبعونا من شكل الـ $H(s)$ ، بنطاع عالمقام أنا أكبر درجة /

بالمقام 2 فهذا First order ولو s^2 فهذا Second order ولو s^3 فهذا Third order . ولـ s^4 وهذا Fourth order وهكذا ، كل ماتريه المقام يكون System يكون أكثر . (لاحظ بالعلاقة خططي 1st and 2nd order) .

* الـ Signal الذي يستخدمها لنظامنا الـ System ← بالعادة هذول المستخدمين .

→ $U(t)$ (unit step function) → Switch (on/of)

حيث سريع ايجاد وتحشيد نواتجه → $S(t)$ (Pulse function)

لـ مثلاً حيث ردي بالجرو أدق ، الذي يقابض جهاز Computer فشو التأثير

الى سببه هذا الحد : المفاجئ السريع .

$G(s)$

$$* R(s) = \frac{1}{s} \rightarrow \boxed{\frac{s+2}{s+5}} C(s)$$

↳ transfer function of $U(t)$ كيف يتغير لما أكبـ → System بـ 5 بـ سـ تـ .

$$\rightarrow C(s) = \frac{(s+2)}{s(s+5)} = \frac{2/5}{s} + \frac{3/5}{s+5} \quad (\text{partial fraction}) \quad (\text{make it in Matlab})$$

يمكن نزع المقامات قبل توحيد المقام

لـ قدر أرجوه !! (laplace table)

$$\rightarrow \text{from table } \Rightarrow C(t) = \frac{2}{5} + \frac{3}{5} e^{-5t} \rightarrow 1: \frac{2}{5}, 2: \frac{3}{5} e^{-5t}$$

الطريق للأول عن $C(s)$ ، إلى Matlab

* العزم الذي حيـوت من معادلة $\ddot{C}(t)$ مع الزمن

$$G(s) = (s+2) / (s+5) \quad \text{لهـ } e^{-5t} \quad \text{لـ } t=0 \quad \text{لـ } C(0) \quad \text{لـ } 2$$

$$C(s) = \frac{1}{s} * G(s) \quad \text{لـ } 1 = \frac{2}{5} + \frac{3}{5} \quad \text{لـ } s \text{ كل ما يغير وقت يا } 1 \text{ !! Sys}$$

بتظل تقل بـ كل مـعوا العزم الثاني لـ اـخـدرـة زـمـنـية ثـوـلـة (ilaplace (partfrac (C(s))))

لـ حـصـرـ الجـزـيـ الثـالـيـ فيـ مـلـفـ فيـ سـ 2 ← وـ يـشـتـ علىـ 0.4 فيـ سـ يـسـ بالـ وـ كـلـ

شيـ transient (respons) between Steady-state (respons)

· Zeros and poles بالـ Slide 6 إلىـ Sys

0 = Pole zero = 1/5 / Poles = -5 / zeros = -2 *

* بالـ Sys اللي المقام فيـ اـنـدـ الروـجـةـ الأولىـ لوـ أـنـهـ يـلـيـهاـ (U(t)) تـأـثـيرـ الـ (U(t)) معـ الـ Sys

هوـ الليـ يـجيـلـيـ الـ Steady-stateـ . والـ poleـ transientـ .

* دـائـيـاـ بـهـاـ النـوعـ منـ الـ Sysـ يـحـلـوـ 2 termsـ واحدـ Steadyـ .

$$* C(t) = \frac{2}{5} + \frac{3}{5} e^{-5t}$$

Forced response \rightarrow Natural response \rightarrow Transient \rightarrow إنه يموت

\rightarrow Steady-State \rightarrow هو الذي يدخل.

* قيمك بديع وقت لأوكل المثلث Natural response ؟ بناء على (القيمة Natural response) \rightarrow Steady-State \rightarrow إلى بار e^{-5t} بالمثل هون " الذي يتبعي من العام "

8 Slide 8 *

سر

Pole at $-a$ generates response

$$k e^{-at}$$

s-plane

$$\times \quad \times$$

كلما تبعد عن الأصل تكون بـ

بسقة وكل ما نقرب له تكون تنزلاً بطيئاً. فلأنها من الرسمة بدون

مكتن sys كغيرياني

أي رياضيات تعرف إن(sys) تنزل بسرعة أو ببطء.

رسو بعضاً $\rightarrow -9, -10 \rightarrow$

طريق بعضاً $\rightarrow -1, -0.5 \rightarrow$

لولا Pole موجدة مكان الأصل المركبة \rightarrow معناها sys

مكتن sys ميكانيكي أو هيدروليكي.

Natural Response تبعه خاتمال كبير ما يكون

(Unstable) Stable

بطيء سريع (نم (ah 3rd order sys العادة) كان مطبق عليه وزايد).

$$R(s) = \frac{1}{s} \xrightarrow{(s+3)} \frac{C(s)}{(s+2)(s+4)(s+5)}$$

EX 1 & Slide 9 *

$$C(s) = \frac{k_1}{s} + \frac{k_2}{s+2} + \frac{k_3}{s+4} + \frac{k_4}{s+5}$$

Poles $\approx -2, -4, -5 \rightarrow$

$e^{-5t}, e^{-4t}, e^{-2t}$

للوحدة مفعم عبارته \rightarrow

فهي تدل بسرعة \rightarrow zeros ≈ -3

Forced Response \rightarrow Natural response \rightarrow Matlab 11 is " "

$$\rightarrow C(t) = k_1 + k_2 e^{-2t} + k_3 e^{-4t} + k_4 e^{-5t} \rightarrow$$

ينظعوا " "

Forced Response \rightarrow Natural Response \rightarrow قيم k يطلع بس هلا

ما يهمنا منهم إلا k_1 الذي يأتي بـ يموتوا.

$$G(s) = \frac{10(s+4)(s+6)}{(s+1)(s+7)(s+8)(s+10)}$$

← EX 2 & Slide 6 *

$$\rightarrow C(t) = A + B e^t + C e^{-7t} + D e^{-8t} + E e^{-10t} \rightarrow \text{from Matlab}$$

Steady-state \rightarrow

(First order System without zeros)

→ 8 Slide 1 *

$$\rightarrow \text{constant} \rightarrow C(s) = R(s) G(s) = \frac{9}{s(Sta)}$$

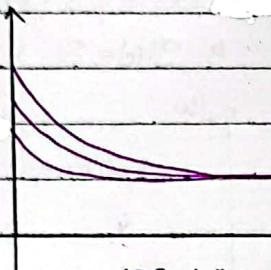
بالبداية كانت بذرة السط →

$$\rightarrow C(t) = C_F(t) + C_n(t) = 1 - e^{-\alpha t}$$

والقائم نفس (s) من Matlab constant

* ما يعنيها pole وحدة عدد (-9).

* أكثر اشغاله لجهة هون أنه يمكن الوصول إلى State من طريقتين ، التي يحدد سرعة الوصول هي



• (Time constant of the sys) $\frac{1}{\alpha}$ 8 Slide 1 *

• (Frequency) ω ←

الفrequeny المئوية العادي لها فيها sys

لـ 37% من اقيمة الخواص (يموت فيها بنسبة 37%) وبقيت 63%

* الرسمة خاصة بال First order unit step funct.

* لـ 5% من اقيمة Time const. إذا $e^{-5t} = 0.2$ يعني بعد $\frac{1}{5}$ ثانية(sys)

وصل لـ 63% واركانت State-Steady sys

(كان جايني من pole) أكبر قيمة لها يحدد سرعة الوصول

First order خاصية لو كانت First order كلما كبر الرقم كلما ماحصلت أسرع .

(Rise Time and Settling Time) 8 Slide 13 *

* Rise Time → الفرق الزعنية الذي يستغرقها sys ليصل من 10% لغاية 90%.

$$Tr = t_{90\%} - t_{10\%} = \frac{2.2}{\alpha} \rightarrow 2.2 * \text{Time constant}$$

* Settling Time → Final Value من sys (within 1-2%.)

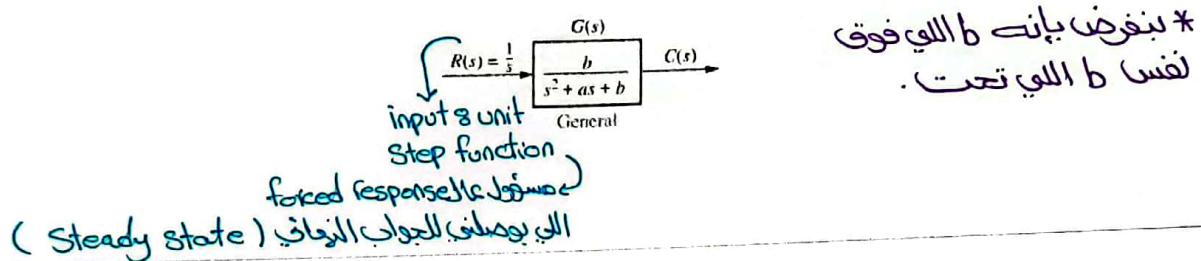
$$Ts = \frac{4}{\alpha} \rightarrow 4 * \text{Time constant}$$

Rise Time , Settling Time , Time const sys

• order system

Second Order Systems: Introduction

- Varying a first-order system's parameter a simply changes the speed of the response.
- Changes in the parameters of a second-order system can change the form of the response
 - Can display characteristics much like a first-order system
 - Display damped or pure oscillations for its transient response.



* لما نشوف معاملات order 2 (المقام فيه تربيعى) وتطبع الجذر بتقىعه حقيقية (real not complex)
فبتتحقق المدفعة المزدوجة يكون فيها 2 terms exponential بالاضافة الى القيمة الثالثة (Steady-state) شكلها بشبه كثير ال 1st order.

Second Order Systems: Overdamped

This function has a pole at the origin that comes from the unit step input and two real poles $-\sigma_1$, $-\sigma_2$ that come from the system.

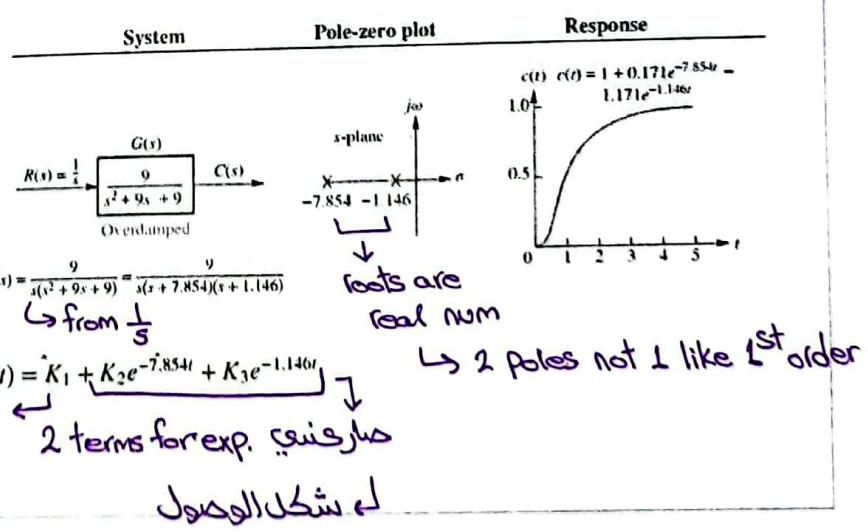
The input pole at the origin generates the constant forced response;

Each of the two system poles on the real axis generates an exponential natural response whose exponential frequency is equal to the pole location.

The output can be estimated as

$$c(t) = K_1 e^{-\sigma_1 t} + K_2 e^{-\sigma_2 t}$$

لـ لـ لـ لـ لـ



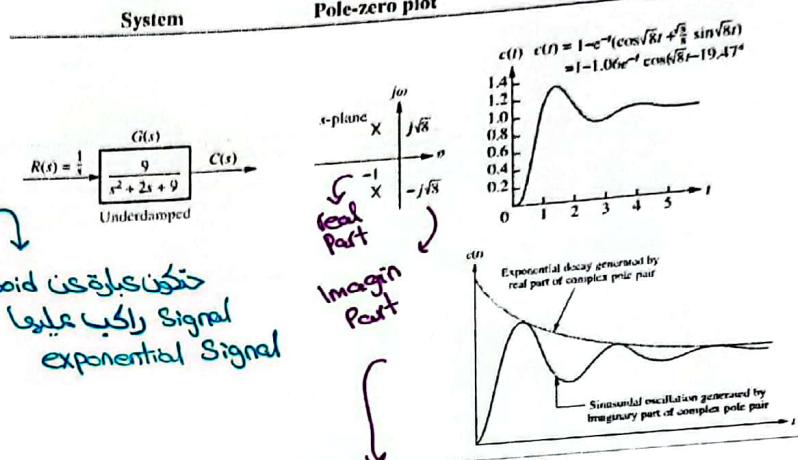
Second Order Systems: Underdamped I

This function has a pole at the origin that comes from the unit step input and two complex poles $-\sigma_d \pm j\omega_d$ that come from the system.

The poles that generate the natural response are at $s = -1 \pm j\sqrt{8}$.

real \leftrightarrow **Imagin**

The real part of the pole matches the exponential decay frequency of the sinusoid's amplitude. Sinusoid



The imaginary part of the pole matches the frequency of the sinusoidal oscillation.

*الرسن كنا مربوطة لأنها بالأساس Sinusoid $\leftarrow e^{j\omega t}$ / exponential $\leftarrow e^{\sigma t}$ *

أول ما نشوف الـ Poles هيك
فيها إزاحة عن الـ real Part

بنفع إن لهذا الـ System لما أعطيه
ملاع يوصل الـ output بسرعة

بعد يحصل عمليات بعدين يوصل الـ output.

Second Order Systems: Underdamped II

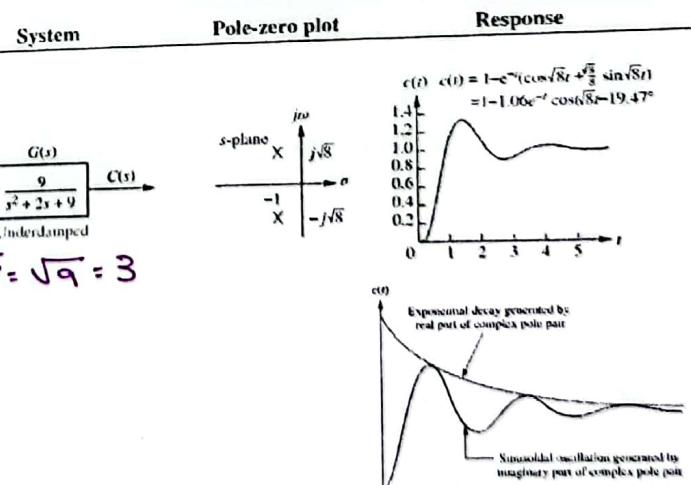
The time constant of the exponential decay is equal to the reciprocal of the real part of the system pole.

The value of the imaginary part is the actual frequency of the sinusoid. This sinusoidal frequency is given the name damped frequency of oscillation, ω_d (radian frequency)

$$\rightarrow \sqrt{(-1)^2 + (\sqrt{8})^2} = \sqrt{1+8} = \sqrt{9} = 3$$

The output can be estimated as

$$c(t) = Ae^{-\sigma_d t} \cos(\omega_d t - \phi)$$



"Revision"

* unit step function يعبر عن بدء (1/5) وهو التعريف بتنشيل الـ System أو كبسه التنشيل (هو اللي ينطلق من حالة لحالة) (يعبر عن الـ Forced response) وهو مسؤول عن الـ Steady state لكن شكل الانقال من حالة لحالة مسؤول عنه الـ System المعتبر عنه بالـ Transfer function (الـ poles تأثيرها على poles).

* Form 1 ينبع رقم بالـ Speed بينما II speed ينبع بالـ 2nd order sys 1st order exponential part.

* الـ poles المعرفة بـ 1st order sys يعبر عن مختلف الأشكال.

* شكل الانقال بالـ 1st order system هو سطح واحد بينما بالـ 2nd order مختلف للانقال.

* العدد اللي يهادين عليه حالياً من slides وظاهر انه رقم اللي بالبسه نفسه اللي يهادين.

* Slide 16 المعلقة بشكلها هباء صعب تحليلاً وتفهمها من شكلها وبنحتاج لتحليلها أو لاستخراج الجذور، انطلع الجذور بالمثلث (هذا حل من الحلول).

$eq = [1, 9, 9]$ بعد التأكيد انه المعادلة صريحة، vector \rightarrow roots(eq) \rightarrow يعطيني الجذور وديدهم 2 لازوا معادلة تربيعية \rightarrow ممكن يكونوا real لو كان تحت الجذر سالب.

* Slide 17

exponential II & Note *

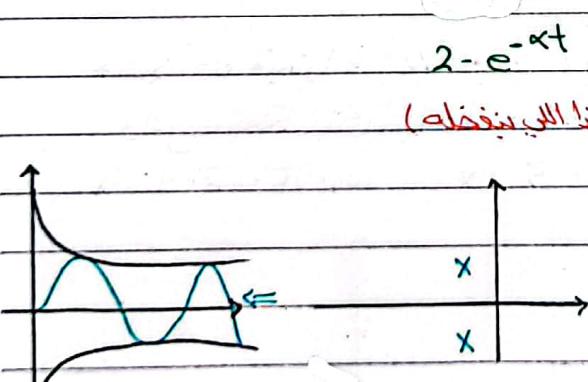
؟ Sinwave

Sin wave نتائج خبراء به \rightarrow الـ Sys يدخل يتخرج \rightarrow ما يهادين

$1 - e^{at}$  \rightarrow لعد ما يهادين حالة هادين \rightarrow لم يمكنني منحة لها الموجة أو يتحقق او كوبائي

$2 - e^{-at}$  \rightarrow تكون مادة unstable اعلي ما كاننا نحب نشتوف poles على الموجة المعنون

الـ Sys ينزل اخر ما يهادين (هذا اللي يهادين)

 * افهمنا الـ poles كانوا هباء \rightarrow الـ poles قرينة من الـ X-axis (X-axis Frequency)

* بينما لو كانت لا يوجد بعية على X-axis Poles

* يحدنا على real-axis

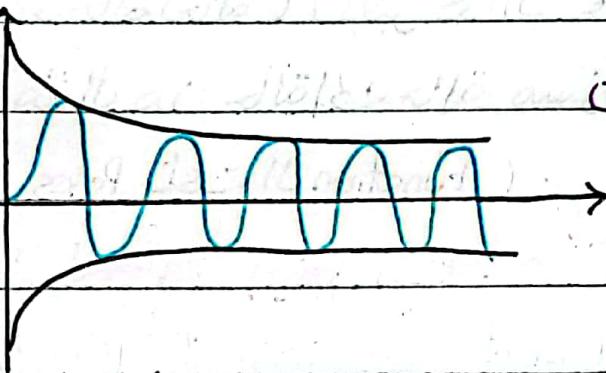
بوضعي ال Frequency

الأول oscillation

Steady-state II

Frequency ←
الربيعية

سرعات



* Frequency هو أسيست ال تدل على سرعة أو بطء الموج (لكن Frequency تقسموا ، هي تدل على سرعة أو بطء الموج (لكن (Slide 17) استرواها .)

* الفرق الأساسي بين ال 1st order وال 2nd order أنه ال 1st order له شكل الاستغاثة Undamped بينما ال 2nd order له شكل مختلفة الاستغاثة (over-damped / critically damped / under-damped /

Signal كل ال signals exponential فلكل part real part oscillations ←
Sinusoidal is

ما تكتب الا Roots فيها بس Part real part ما فيها Part

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Second Order Systems: Undamped

This function has a pole at the origin that comes from the unit step input and two imaginary poles $\pm j\omega_n$ that come from the system.

The input pole at the origin generates the constant forced response.

The two system poles on the imaginary axis at $j\beta$ generate a sinusoidal natural response whose frequency is equal to the location of the imaginary poles $\sqrt{b} = 3 \rightarrow$ (موجة حسب المطالبات) $\omega_n = \sqrt{b}$ (radian)

The output can be estimated as $c(t) = A \cos(\omega_n t - \phi)$

System	Pole-zero plot	Response
$R(s) = \frac{1}{s} \xrightarrow{G(s)} \frac{9}{s^2 + 9} \xrightarrow{\text{Undamped}} C(s)$	s-plane: $\text{j}\omega$ axis, poles at $\pm j\beta$	$c(t) = 1 - \cos 3t$
$R(s) = \frac{1}{s} \xrightarrow{G(s)} \frac{b}{s^2 + as + b} \xrightarrow{\text{General}}$		$c(t) = K_1 + K_4 \cos(3t - \phi)$

جذور ايجادي، ايجادي، Stability State
وهي موجة حسب المطالبات \rightarrow بالعكس يكون مطلوب الا out

هذا الا Stable System يعتمد على الا System تبعي شو بعمل ، قد يكون مطلوب الا out يكون الا Sinusoidal فهذا ينطبق النتيجة فتكون Stable وقد يكون مطلوب الا out يكون الا unstable .

الجذر (real) مافي complex بين الجذرين متساويين لبعض في القيمة .

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Second Order Systems: Critically damped

This function has a pole at the origin that comes from the unit step input and two real poles $-\sigma_1$ that come from the system.

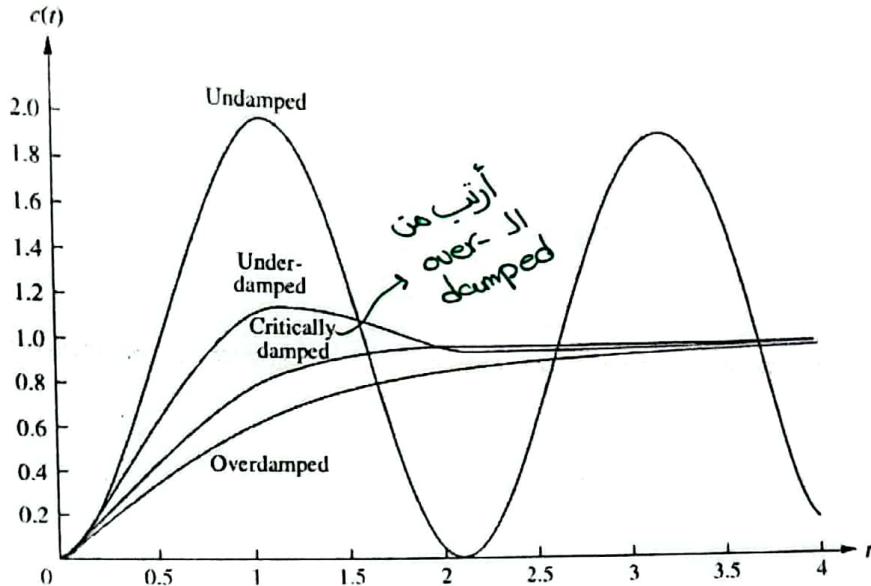
The input pole at the origin generates the constant forced response

The two poles on the real axis at 3 generate a natural response consisting of an exponential and an exponential multiplied by time, where the exponential frequency is equal to the location of the real poles

The output can be estimated as $c(t) = K_1 e^{-\sigma_1 t} + K_2 t e^{-\sigma_1 t}$

System	Pole-zero plot	Response
$R(s) = \frac{1}{s} \xrightarrow{G(s)} \frac{9}{s^2 + 6s + 9} \xrightarrow{\text{Critically damped}}$	s-plane: $\text{j}\omega$ axis, poles at $-3, -3$	$c(t) = 1 - 3te^{-3t} - e^{-3t}$ overdamped

exp. all terms \rightarrow
بكل مذروب بـ t
(time dependent)



(مقارنتهم بعض)

The General Second Order System

* من ζ لحالها يمكن تعرف شكل الـ System بدون الـ Transfer function

Natural Frequency, ω_n of a second-order system is the frequency of oscillation of the system without damping

Damping Ratio, ζ compares the exponential decay frequency of the envelope to the natural frequency

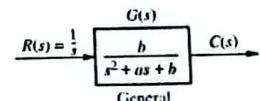
$$\zeta = \frac{\text{Exponential decay frequency}}{\text{Natural frequency (rad/second)}} = \frac{1}{2\pi} \frac{\text{Natural period (seconds)}}{\text{Exponential time constant}}$$

تائير
الـ radian
الموجود.

For the underdamped system, the complex poles have a real part σ , equal to $-a/2$. \rightarrow Frequency = Poles

إيجاد Freq ما إذا عرف بالسالب.

$$\zeta = \frac{\text{Exponential decay frequency}}{\text{Natural frequency (rad/second)}} = \frac{|\sigma|}{\omega_n} = \frac{a/2}{\omega_n} \rightarrow a = 2\zeta\omega_n$$



Radian \rightarrow تفاصيل بالـ Radian خلا حاجة لتحويلها إلى
بالماثلاب لأنها أساساً بالـ Radian

$$\text{Frequency (Natural)} \leftarrow \omega_n = \sqrt{b}$$

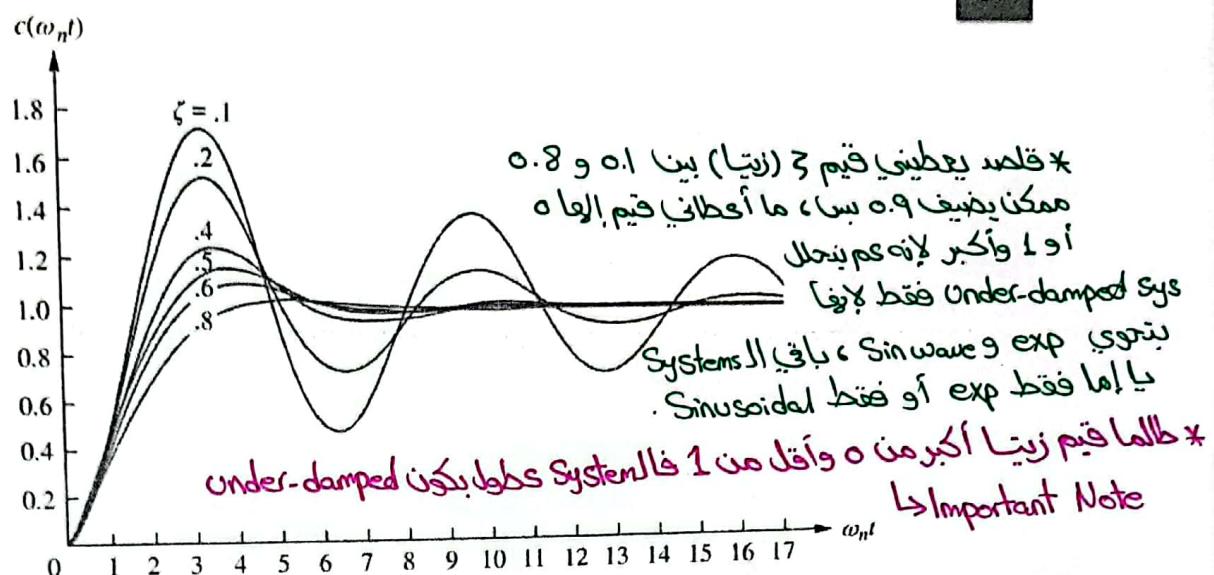
$$s_{1,2} = -\zeta\omega_n \pm \omega_n\sqrt{\zeta^2 - 1}$$

كتب المعادلة بدلاة ζ و ω_n و s

(عاد كتابة المعادلة من s إلى ω_n و ζ)

Effect of ζ (dampening ratio) Visualized

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- * لما يدرس أي System بالدنيا كشان شوف تأثير الـ Variable اللي قيد المراقب عن المـ System
لازم نشت باقـ الـ Variables المـ الباقيـ.
- * هون هو almost all Sin waves have similar frequency (الـ التـ زيتـ ζ)
كلـ ما كانت قليلـة بكون المـ Frequency \exp قليلـ فيـنـكـنـ الـ dampening
ـ طـيـيـيـ وـ كـلـ ماـ نـادـتـ بـ زـيـدـ الـ dampeningـ وـ بـ حـيـرـ سـرـعـ.

24

Finding ζ and ω_n For a Second-Order System - Example

Given the transfer function of $G(s)$, find ζ and ω_n .

$$G(s) = \frac{36}{s^2 + 4.2s + 36}$$

$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

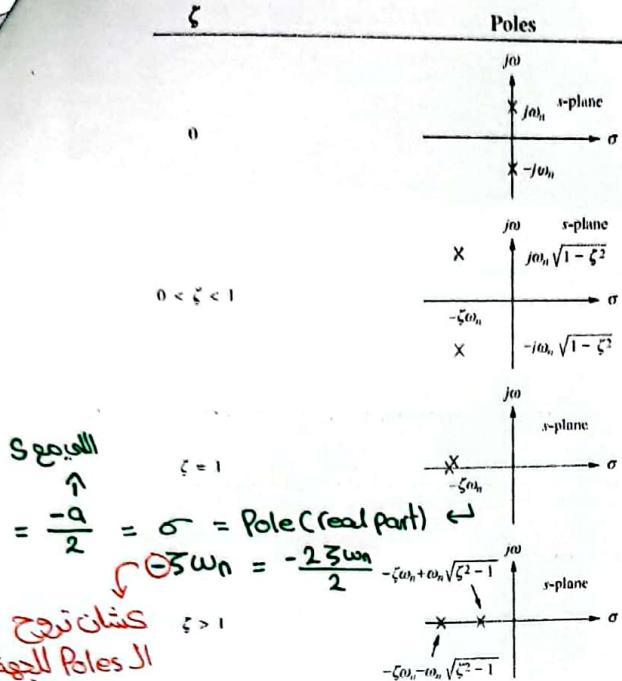
$$\omega_n^2 = 36, \text{ therefore } \omega_n = 6$$

Also, $2\zeta\omega_n = 4.2$, so after substituting the value of $\omega_n = 6$, we get $\zeta = 0.35$ (underdamped System)

* سـكـلـ الـ اـمـتـحـانـةـ بـ جـيـلـكـ 2 systems
ـ قـارـيـنـوـمـ وـ اـجـكـيلـهـ شـوـ نـوـمـ (undamped)
ـ وـ هـكـذـاـ (underdamped).

* سؤال امتحان: بعطيك رسمة Poles وينكل أي وحة من
الـ S-Domain plots بطبق مواصفات رسمة Poles.

25



البسط بمعادلة الـ $\ddot{x} + 2\zeta\omega_n x + \omega_n^2 x = 0$ يعني ما في oscillation Part س في undamped Part فلان هو Part we have complex Poles (we don't have real)

Poles and Zeros 8 Complex

$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$\zeta_{1,2} = -\zeta\omega_n \pm \omega_n\sqrt{\zeta^2 - 1}$$

الحالة الخامدة اللي يكون ζ فيها بساوي 1.

كترت ζ فنفريها إنلطف تأثير oscillation وصل exponential هو الـ Dominant

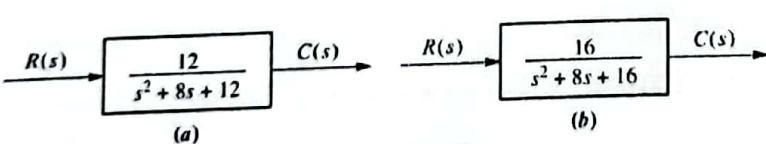
* لما تدخل زيتا أكبر كثير يعني كل تأثير oscillation في ترجمة الحالات اللي قبلت خبرها exponential.

26

Characterizing Response from the Value of ζ - Example

For each of the systems shown, find the value of ζ and report the kind of response expected.

$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$



$$\text{Since } a = 2\zeta\omega_n \text{ and } \omega_n = \sqrt{b}, \quad \zeta = \frac{a}{2\sqrt{b}}$$

We find:

- $\zeta = 1.155$ for system (a), which is thus over-damped, since $\zeta > 1$.
- $\zeta = 1$ for system (b), which is thus critically damped.
- $\zeta = 0.894$ for system (c), which is thus under-damped, since $\zeta < 1$.

$$\rightarrow \text{Poles in } -4$$

$$= -a/2$$

$$= -8/2$$

$$= -4$$

* بالماناب ممكن ترسمهم بس يعاني الحالة هيكل أصول.

* ممكن ينزلكي حدلي أي مكان الـ Poles.

١٧/١١/٢٠٢٢

* Slide 22 :

Suppose that we have :

$\frac{4}{6s^2 + 2s + 16}$

* هون كنا مش زوي ما فحستنا قبل، هون

البسد لا يساوي المقام ($4 \neq 16$)

وعلب بقدر نحال العازمي باقي الـ Systems

بنحاول نطلع عامل مشترك، لنغير نخلي القيمة اللي بالبسد تساوي القيمة اللي بالمقام

$$\frac{\frac{1}{4}(4 \times 4)}{6s^2 + 2s + 16} \rightarrow \frac{\frac{1}{4} * 16}{6s^2 + 2s + 16} \rightarrow \frac{\frac{1}{4}}{1} \rightarrow \frac{G(s)}{1} \rightarrow 2 \text{ subsystems}$$

* كل اتفهم حملوا

لـ $\frac{1}{4}$ أصبحت كأنها gain من

ناتحة الـ Control System

لـ هنا إجراء رياضي فقط مش واقعي وحقيقي

-1 is Pole $\Rightarrow -1 = -2/2 = -1/2$ * هون \Rightarrow قيمتها =

والـ Frequency بتكون $+1 = 1 - (-1) = 2$ معادلة

under-damped System $\leftarrow \frac{1}{4} = \zeta * \omega_n = \sqrt{b} = \sqrt{16} = 4 *$

* Slide 26 : under-damped under-damped undamped under-damped

\Rightarrow Note :	1-	2-	3-	4-
	x	x	x	x
	x	x	*	x

له بجيبلات ١١ System وبجيبلات ياهم وبجيبلات اختار Poles اللي يتمثل الـ System المعطى.

Characterizing Response from the Value of ζ - Exercise

"Self Exercise"
+ Find the poles of each system.

For each of the systems shown, find the value of ζ and ω_n and report the kind of response expected.

a. $G(s) = \frac{400}{s^2 + 12s + 400}$

b. $G(s) = \frac{900}{s^2 + 90s + 900}$

c. $G(s) = \frac{225}{s^2 + 30s + 225}$

d. $G(s) = \frac{625}{s^2 + 625}$

$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

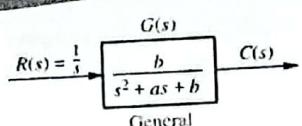
$$\zeta = \frac{a}{2\sqrt{b}}$$

Underdamped Second-Order Systems - Parameters

يؤمنني كثيراً إنها موجودة كثيرة بالطبيعة

We have seen that in the underdamped case, it is assumed that $\zeta < 1$

Other parameters associated with the underdamped response are rise time, peak time, oscillation (أو ارتجاع)، Peak (مаксимум)، First overshoot (أول ارتجاع)، Settling time (نقطة بالاستقرار)، System (نظام).



$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

1. Rise time, T_r . The time required for the waveform to go from 0.1 of the final value to 0.9 of the final value.
 2. Peak time, T_p . The time required to reach the first, or maximum, peak.
 3. Percent overshoot, %OS. The amount that the waveform overshoots the steady-state, or final, value at the peak time, expressed as a percentage of the steady-state value.
 4. Settling time, T_s . The time required for the transient's damped oscillations to reach and stay within 2% of the steady-state value. \rightarrow Steady-state (نقطة الاستقرار) (System) (النظام) (Arising) (ارتفاع) (Settling) (Settling time)
- Notice that the definitions for settling time and rise time are basically the same as the definitions for the first-order response. All definitions are also valid for systems of order higher than 2.*
- هذه التعاريفات مشتملة على مقتصرة على الأنظمة ذات الرتبة الثانية 2^{nd} order system، وإنما بنفس المفهوم تطبق على الأنظمة ذات الرتبة العالية.*

$$C(s) = \frac{\omega_n^2}{s(s^2 + 2\zeta\omega_n s + \omega_n^2)} = \frac{K_1}{s} + \frac{K_2 s + K_3}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

General

General

Underdamped Second-Order Systems - Parameters

- * Rise time, peak time, and settling time yield information about the speed of the transient response.

لهم بعطنا معلومات لسرعة
وصلنا الى state → Steady-state

This information can help a designer determine if the speed and the nature of the response do or do not degrade the performance of the system.
يساعدنا لا designer هنا حيث يتأثر response بالجهد المفاجئ

For example, the speed of an entire computer system depends on the time it takes for a hard drive head to reach steady state and read data; passenger comfort depends in part on the suspension system of a car and the number of oscillations it goes through after hitting a bump.

الـ motor يزبح الى disk ما يجبي عالي السرعة بالضبط به وقت ليستقر بالمكان المقصود

العلاقة تسرع المروان فيه يحيي قريب على الـ steady-state

الوقت حتى الاستقرار وأصل 98% للـ steady-state

وهكذا بينك انهم كيف يمكن أن يؤثروا بسرعة الـ system
للوصول الى الشيحة النهائية.

Underdamped Second-Order Systems – Parameters Evaluation I

* كل معطياتنا ببعضها البعض

Peak time

$$T_p = \frac{\pi}{\omega_n \sqrt{1 - \zeta^2}}$$

يعتمد على
frequency
و damping

Percentage
Overshoot

$$\% OS = e^{-(\zeta \pi / \sqrt{1 - \zeta^2})} \times 100$$

ينقص من الا
exp. \ln
رجل \ln

$$\zeta = \frac{-\ln(\% OS / 100)}{\sqrt{\pi^2 + \ln^2(\% OS / 100)}}$$

معادلة
ثانية لـ ζ

Settling Time

$$T_s = \frac{4}{\zeta \omega_n}$$

مالطبق

* الـ rise-time ما قرروا يطلعوا معادلة (علاقة) بسبب كثرة المتغيرات فعملوا التالي (Slide 31)

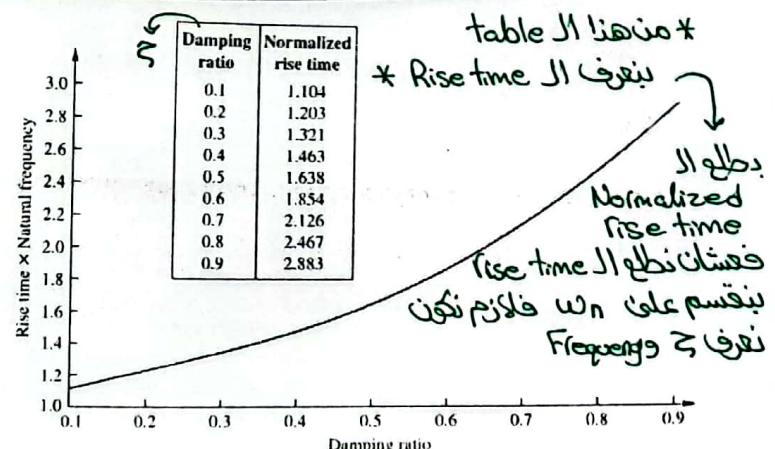
Underdamped Second-Order Systems – Parameters Evaluation II

An analytical relationship to derive T_r from the other system parameters is not easily available.

An experimental relationship exists.

- We set the value of $c(t)$ to 0.9 and solved for $\omega_n t$.
- Then, we set the value of $c(t)$ to 0.1, and solve again for $\omega_n t$.
- The rising time is the time required for the waveform to go from 0.1 of the final value to 0.9 of the final value, so T_r is the difference between the two values.

Note; however, that we solved for $\omega_n t$ not t . This $\omega_n t$ is called the normalized rise time. We plot the resulting equation with respect to different value of ζ



- * الـ 90% = Rise Time
- لـ 90% من المدة المدروسة تكون المدة المائية المطلوبة = Signal
- Normalized Rise time = ازاحت المدة المائية المطلوبة بالنسبة لـ $\omega_n t$ تحت معنوي
- * مهم كثير المقاييس الرياضية لأنها بالذاتية حجمية بالـ (Matlab).
- * سبيلاً الحال أو Visual إنه مش دقيق (إجابات متقاربة لكن ليست دقيقة).
- * فبشكلها رياضيتاً بالـ Matlab باستخدام الـ Interpolation

Underdamped Second-Order Systems – Parameters Evaluation III

To find the rising time for an underdamped system,

First, we determine ζ , then we see to which value in the plot

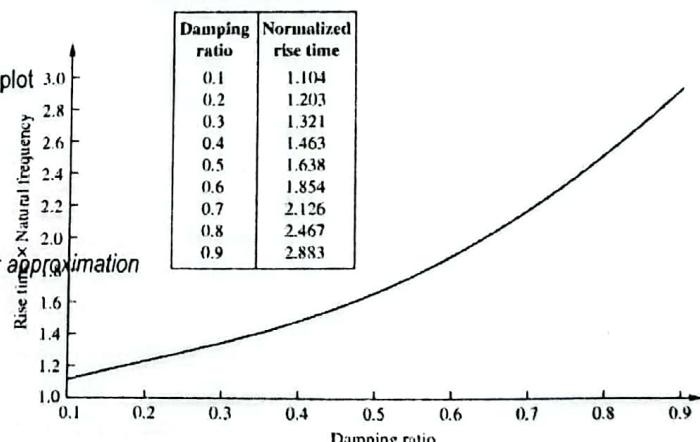
It corresponds to $\omega_n t$

Finally, divide by ω_n

Visual is not accurate

Use MATLAB commands interp1 or spline to get a better approximation

```
X = 0.1 : 0.1 : 0.9
Y = [1.104, 1.203, 1.321, 1.463, 1.638 ...
     1.854, 2.126, 2.467, 2.883]
Interp1 (X, Y, your zeta) // linear interpolation
Spline (X, Y, your zeta) // cubic interpolation
// Then divide by omega_n
```



أدق بكثير → non-linear interpolation

* مثلاً لو طلبنا الـ System | Rise time | damping ratio (zeta) = 0.85 وـ 5 = Frequency (Frequency) ← بخط قيم X وقيم Y

الثانية من الجدول يدعين يستخدم Spline أو Interp1 (أدق)، وبنفس قيمة zeta = 0.85 والناتج يقسم إلى 5 ليعطينا الـ Rise time

Underdamped Second-Order Systems – Parameters Evaluation Ex.

$$\begin{aligned} * \sigma &= \frac{\omega}{2} \\ * \xi &= \frac{\sigma}{\omega_n} \end{aligned}$$

Finding T_p , %OS, T_s , and T_r from a Transfer Function

PROBLEM: Given the transfer function

$$G(s) = \frac{100}{s^2 + 15s + 100}$$

find T_p , %OS, T_s , and T_r .

SOLUTION: ω_n and ξ are calculated as 10 and 0.75, respectively. Now substitute ξ and ω_n into Eqs. $\rightarrow \sqrt{100}$ $\rightarrow \sigma/\omega_n$ \rightarrow underdamped find, respectively, that $T_p = 0.475$ second, %OS = 2.838, and $T_s = 0.533$ second. Using the table \rightarrow , the normalized rise time is approximately 2.3 seconds. Dividing by ω_n yields $T_r = 0.23$ second. This problem demonstrates that we can find T_p , %OS, T_s , and T_r without the tedious task of taking an inverse Laplace transform, plotting the output response, and taking measurements from the plot.

$$* \sigma = \frac{15}{2} = 7.5$$

لـ 7.5
Pole بالقطة

* الي ينبع هو الـ rising time \rightarrow بينما بالـ معالة.

* أول خطوة لازم نطلع ω_n و ξ لأنها المعادلات بتحتاج علهم بعدين تطبيقات مباشرة.

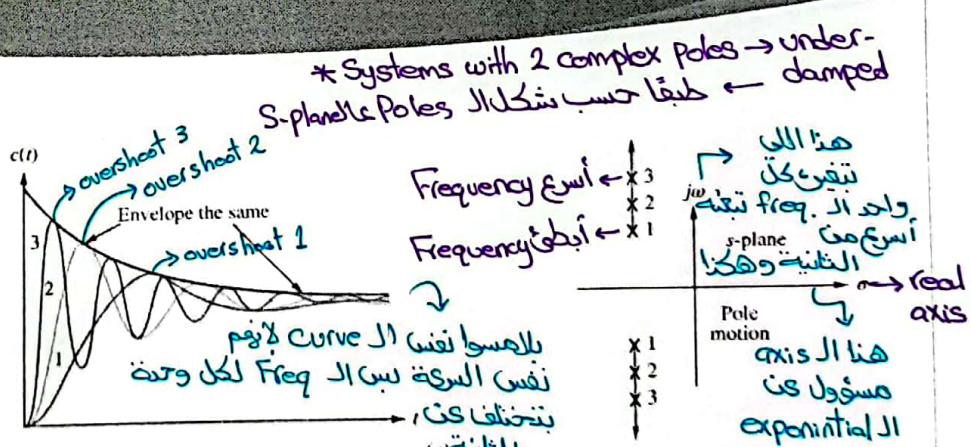
* Setting time \rightarrow Important \rightarrow لا overshot ولا oscillation \rightarrow Peak time \rightarrow لا oscillation \rightarrow يكون فيه System يكون فيه \rightarrow يمكن تقدّر تطبيق عليه هـ المعاـلات.

Pole Motion Effect on the Underdamped System Response I

As the poles move in a vertical direction, the frequency increases, but the envelope remains the same since the real part of the pole is not changing.

The figure shows a constant exponential envelope, even though the sinusoidal response is changing frequency.

Since all curves fit under the same exponential decay curve, the settling time is virtually the same for all waveforms.



Note that as overshoot increases, the rise time decreases. \rightarrow ينبع إـ الـ 90% من خط الـ steady-state (مورسوم هوـ بالرسمـة)

* تغيير مكان الـ poles ما يـنـعـافـ الـ settling time

لـ كلـمـونـ وـدـلـواـ نفسـ المـكـانـ \rightarrow rise time \leftarrow overshoot \leftarrow Frequency \leftarrow الـ اـخـاعـ الـ

(Envelope \rightarrow نفسـ الـ سـرـيـةـ)

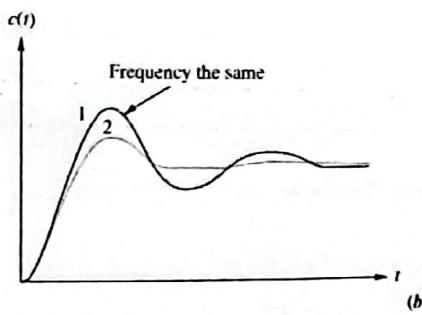
(الـ اـخـاعـ الـ)

Pole Motion Effect on the Underdamped System Response I

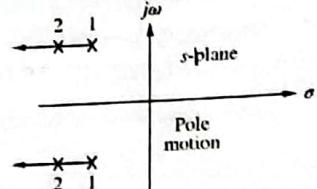
Let us move the poles to the right or left. Since the imaginary part is now constant, movement of the poles yields the responses shown in the figure.

Here the frequency is constant over the range of variation of the real part. As the poles move to the left, the response damps out more rapidly, while the frequency remains the same.

Notice that the peak time is the same for all waveforms because the imaginary part remains the same.



* المقارنة هي بناء على الـ left side لورقتليمين يمكن يغير σ حالـة الـ System Unstable.



* هـذا الـ Frequency ثابتة

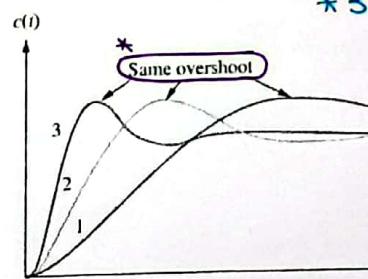
بسـ سـرـة الـ اـذـنـسـافـ بـشـفـرـ كلـما بـعـدـنـا بـلـكـونـ أـسـعـ .

* الـ Peak Time هو الـ الوقت من أول ما يـلـشـنـا لـغاـيـةـ منـدـفـ أـولـ Peak ، مـاـحـ يـخـلـفـ الـ Peak time يـغـيرـ سـرـةـ الـ اـذـنـسـافـ .

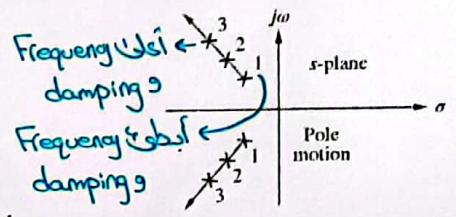
Pole Motion Effect on the Underdamped System Response I

مـاـبـحـرـكـ سـكـلـىـشـوـافـيـ * مـهـلـ شـعـاعـ
diagonal Moving the poles along a constant radial line وـبـمشـفـاـتـ كـلـيـةـ آـلـ سـيـسـمـ Here the percent overshoot remains the same.

Notice also that the responses look exactly alike, except for their speed. The farther the poles are from the origin, the more rapid the response



* 3 Systems with different Poles *



* هـونـ كـمـ يـغـيرـ فـرـقـيـنـ وـبـالـاـ

* بـقـارـنـ بـقـارـنـ (qualitative) وـلـيـسـ رـيـاضـيـاـ (quantitative) .

* كـيفـ يـجيـيـ عـلـيـاـ بـالـامـتـحـانـ هـوـ ماـبـعـدـيـ الرـسـمـةـ بـشـكـلـهـ باـشـ ، بـعـطـيـاـ الـحـيـفـةـ (ـالـعـادـلـاتـ) لـ 2 systems وـبـقـلـكـ هـنـاكـ هـذـلـ هـذـلـ الـعـمـلـنـفـسـ الـأـ وـهـيـكـ ، يـاـ بـتـحـلـ بـالـمـعـدـلـاتـ (ـquanti~tative) أوـ بـتـرـسـمـ الـ نـتـلـابـ نـتـلـابـ qualiti~tive .

System Response with Additional Poles and Zeros

37

- * We analyzed systems with one or two poles. The formulas describing percent overshoot, settling time, and peak time were derived only for a system with two complex poles and no zeros. الـ Pole المـيـانـهـمـاـ
 - * If a system has more than two poles or has zeros, we cannot use the formulas to calculate the performance specifications. ماـ الـ مـهـاـنـهـمـاـ يـغـرـبـهـاـ
 - * However, under certain conditions, a system with more than two poles or with zeros can be approximated as a second-order system that has just two complex **dominant poles**. \rightarrow **2 dominant poles** يـكـفـيـنـ فـيـ هـذـهـ الـ كـوـكـهـ 6 8 9
 - * Only then, the formulas for percent overshoot, settling time, and peak time can be applied to these **higher-order systems** by using the **location of the dominant poles**. * هـدـفـنـاـ هـوـنـ اـنـ نـظـرـ
 - * We will restrict our analysis to a system of three poles (3rd degree), or a 2nd degree system with a zero عـلـىـ الـ مـهـاـنـهـمـاـ

$$\text{new exponential has extra pole } \text{if } * \quad C(s) = \frac{A}{s} + \frac{B(s + \zeta\omega_n) + C\omega_d}{(s + \zeta\omega_n)^2 + \omega_d^2} + \frac{D}{s + \alpha_r}$$

Steady-
state

$$\text{new exponential has extra pole } \Re s \\ (\text{equation}) \text{ Systematic term} \quad C(s) = \frac{A}{s} + \frac{B(s + \zeta\omega_n) + C\omega_d}{(s + \zeta\omega_n)^2 + \omega_d^2} + \frac{D}{s + \alpha_r}$$

$$c(t) = \underbrace{Au(t)}_{\text{so}} + e^{-\zeta\omega_n t} (B \cos \omega_d t + C \sin \omega_d t) + De^{-\alpha t}$$

مع الزمن ينافي الـ exponential terms . Steady-State وبشكل الـ

• Steady-State وبحال ثابت

* هدفنا هو أن نطلع على هذا الـ System

الى خinde extra poles and zeros
ونقدر لو بنفس بناء و Criteria

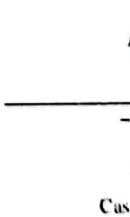
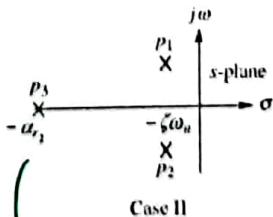
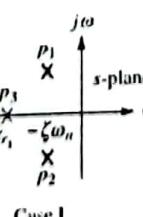
معينة أقول انه يقدر اقرب
هذا System واتصال مده ك

- * جنرس فقط حالات انه يكون عندي 3 poles او 1 zero .
- * بال Systems قبل ما كلان فيينا zeros لانه البسط كان Constant .

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System Response with an Additional Real Pole I

- ▶ Suppose that in the denominator, we have a third real pole α_r , we will consider three cases:
 - ▶ Case I: $\alpha_r = \alpha_{r1}$ where it is not much larger than $\zeta \omega_n$
 - ▶ Case II: $\alpha_r = \alpha_{r2}$ where it is much larger than $\zeta \omega_n$
 - ▶ Case III: $\alpha_r = \infty$



كhan قیب کیش ما تقدیر ائمه

لقد أتماء ignore؟ حسب Criteria مكينة وعلاقة
رياضية مكينة

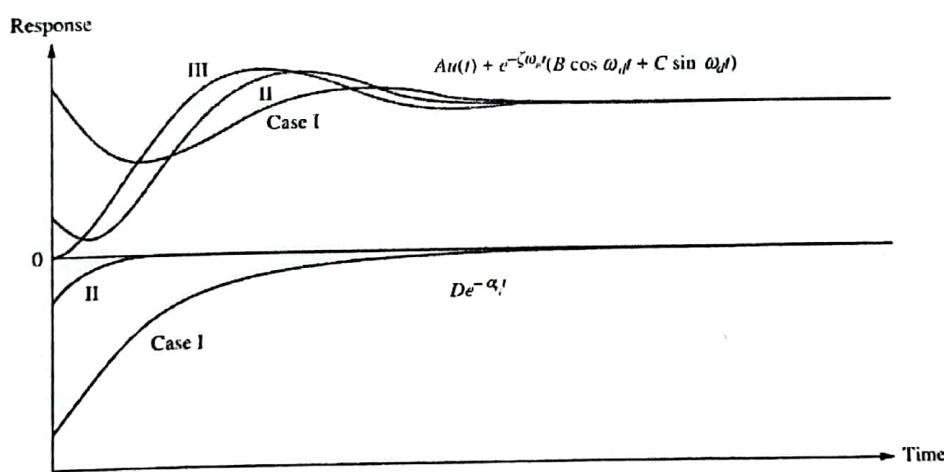
هذا الوضع الطبيعي من الممكن وجود إلأ third pole الذي كل ما بعدناه كل ما كان أفضل للـ approx.

→ 2 dominant نظائر ممكن

System Response with an Additional Real Pole II

- * ► Case III → Pure Second Order System
- * ► Case II → If $\alpha_r \gg \zeta \omega_n$, the pure exponential will die out much more rapidly than the second-order underdamped step response. If the pure exponential term decays to an insignificant value at the time of the first overshoot, such parameters as percent overshoot, settling time, and peak time will be generated by the second-order underdamped step response component. Thus, the total response will approach that of a pure second-order system.
- * ► If If $\alpha_r > \zeta \omega_n$ but is not much greater than $\zeta \omega_n$, the real pole's transient response will not decay to insignificance at the peak time or settling time generated by the second-order pair. In this case, the exponential decay is significant, and the system cannot be represented as a second-order system.
- * ► How much farther from the dominant poles does the third pole have to be for its effect on the second-order response to be negligible?
- * ► Depends on the accuracy desired. The textbook book assumes that the exponential decay is negligible after five time constants. \rightarrow **لما يكون 5 مرات بعيد عن الـ Dominant pole**
- * ► Thus, if the real pole is five times farther to the left than the dominant poles, we assume that the system is represented by its dominant second-order pair of poles
- * ► Magnitude od the exponential term has less effect on the system as the pole moves further away to the left

System Response with an Additional Real Pole III



Exercise

$$\omega_n = 4.954, \sigma = \frac{9}{2} = 2, \zeta = \frac{\sigma}{\omega_n} = \frac{2}{4.954} = 0.4037$$

- * Find the step response of the following transfer functions and compare them. Which one of T_2 or T_3 can we apply the performance equations we learnt to?

* It is clear that T_1 is a second-order system while T_2 and T_3 are third order systems

* There is a frequency component $\omega_n = \sqrt{24.542} = 4.954$, therefore $\zeta = 4/(2 \cdot 4.954) = 0.4037$, so T_1 is an underdamped system

* We notice that T_2 and T_3 have identical complex poles like T_1 , but with extra real poles, one at -10 for T_2 , and one at -3 for T_3 , so expect the shapes of T_2 and T_3 to have similar shapes to T_1 , but the effect of the dampening by the extra real pole will change them.

* We must know if the new shape will still resemble a second order system, so that we can approximate the third-order system as a second-order system and use the equations or not.

$$T_1(s) = \frac{24.542}{s^2 + 4s + 24.542} \rightarrow \text{second order system}$$

$$T_2(s) = \frac{24.542}{(s+10)(s^2 + 4s + 24.542)} \quad] \rightarrow 3^{\text{rd}} \text{ order}$$

$$T_3(s) = \frac{73.626}{(s+3)(s^2 + 4s + 24.542)} \quad] \rightarrow 3^{\text{rd}} \text{ order}$$

أي وحدة مفهوم يغيرني آنتمام مدة زدي T_1 . كشان أقدر آنتمام مدة T_2 زي T_1 .

لأنه لا Pole الإثناي يكون 5 أدنعاف بعيد عن الـ Poles الأصلين عالاً.

* سؤال انتقال = بطلبك

متلا overshoot المعدلة
قبل ما تبلي (تحل تأكيد إنها
أو بتعق نتعامل
Second order معها كـ ما بهم تطبق
غيرهيك ما بهم تطبق
العادلات.

$$T_2: \frac{-10}{-3-2} \rightarrow \frac{T_2(\text{pole})}{T_1(\text{pole})} = \frac{-10}{-2} = 5 \rightarrow \text{تقرب} T_2 \text{ لـ } T_1 \text{، وكأنه } -10 \text{ غير موجود.}$$

$$T_3: \frac{-3}{-2} = 1.5 \rightarrow \text{ما بعد 5 أدنعاف تقرب} \rightarrow \text{تقرب} T_3 \text{ لـ } T_1.$$

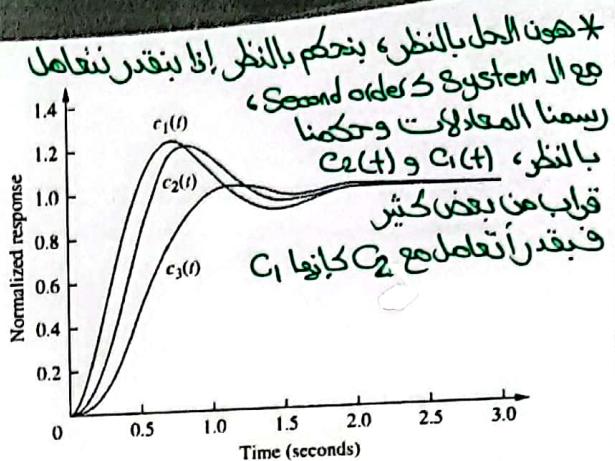
* نتائج المعادلات T_2 مش درقة % صارى T_1 ، ولكنها أقرب ما يمكن المقدمة.

Solutions- Numerical

- * We should find the location of the complex poles , and see if the real pole is five times further away to the left
- * We know from the equations that the complex pole location will be at $-\zeta \omega_n = -0.4037 \times 4.954 = -2$
- * For T_2 , $\alpha_r = -10$ which five times larger than -2, so this system can be approximated as a 2nd order system. We can use the equations we learnt for settling time, overshoot, rise time, and peak time to APPROXIMATE the solutions.
- * For T_3 , $\alpha_r = -3$ which 1.5 times larger than -2, this system cannot be approximated as a 2nd order system. We cannot use the equations we learnt for settling time, overshoot, rise time, and peak time.

Solutions- Visual

- Using plots, you can do an inverse Laplace transform on each of the transfer functions, and you get the time domain equation. Once you do that, plot the equations.
- We can see how $c_2(t)$ is close to and resembles the shape of the second order function $c_1(t)$, while $c_3(t)$ differs in shape, and yields more error if we use the performance equations on it.



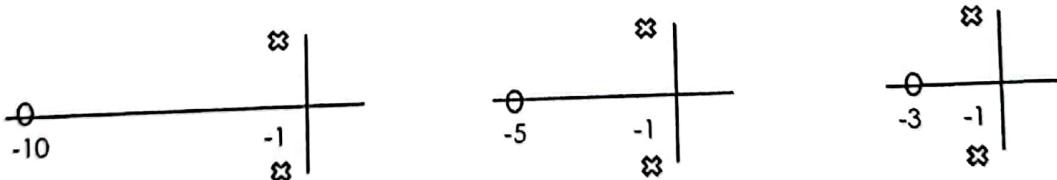
* الـ Poles هم المغ بيدوا شكل الـ response ولا response Speed تتبع المغ $c_1(t)$ ولا $c_2(t)$ ولا $c_3(t)$.
 * unstable ، كل الشغلات الأساسية بالـ System موجهة نحوها بـ Poles ، فكتاً لـ Zeros .
 * لا Zeros تأثيرها الـ zero على قيمة الـ final value (Forced response) (أيضاً الـ Zeros به يومن) عش شكل وسرعة الوصول والـ Stability .

* لعيك كنا بـ Poles أكثر من الـ zeros

System Response with an Additional Real Zero |

البسط بـ poles جيـس فيـ zeros .

- All the second order systems we have analyzed so far had no zeros, only poles. What if the transfer function had a zero as well?
- We saw (see slide 7) that the zeros of a response affect the residue, or amplitude, of a response component but do not affect the nature of the response—exponential, damped sinusoid, and so on.
- Suppose we have the system with the complex poles $-1 \pm 2.828j$ and we add each time a different zero at -3, -5, and -10



* نفس الـ analysis اللي استخدمناه بالـ poles حينستخدمنها بالـ zeros ، انه لو كان الـ zero بعيد عن الـ pole عاًقل 5 أمتار فيغير نوكه وننعامل مع الـ system زي ما ننطرنا في (ما في عرق بالبسط)

$$* \text{ Example 8 } 1 - \frac{7(s-10)}{s^2 + 6s + 7}$$

لـ هذا اللي بقدر تلفي فيه

الماج وننعمل

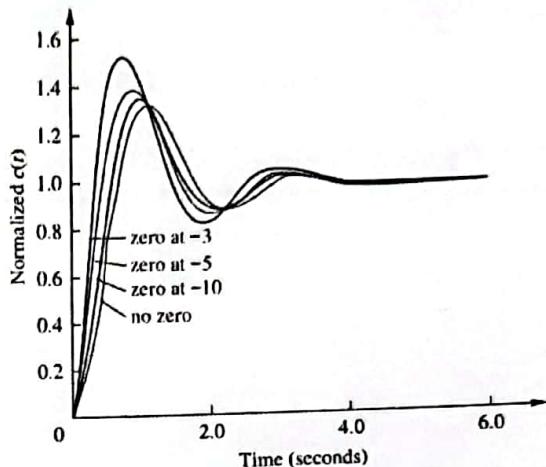
$$2 - \frac{7(s-0.5)}{s^2 + 6s + 7}$$

هون الا zero قريب \rightarrow
صعب نعمله.

System Response with an Additional Real Zero II

- We can see that the closer the zero is to the dominant poles, the greater its effect on the transient response. As the zero moves away from the dominant poles, the response approaches that of the two-pole system.

Transfer function (النسبة المئوية) معزز أي. زéro اضافي (أي. زéro إضافي) معنده كأنه يغير Transfer funct. مخترق مختلفة مخترق لا نفس مخترق بـ gain مخترق function



S → Transfer funct. مخرب أي.

$$T(s) = \frac{90}{s^2 + 2s + 90} (s-2)$$

* Laplace { T'(s) } = ST(s)

* مخرب أي. Transfer funct. (نسبة المئوية) معزز أي. gain (أي. زéro إضافي)

، (Amplifier) دخلها Signal

ويمكننا أن نكتب gain تكون

loss (ت تكون نسبة مثل 0.5 0.25 ونحو ذلك)

↓ reduction

Another way to think of systems with a zero

- Suppose we have a transfer function of a 2nd order function $T(s) = \frac{90}{s^2 + 2s + 90}$, if $T(s)$ were to have a zero, it would be for example in the shape of $T(s) = \frac{90(s+a)}{s^2 + 2s + 90}$ where a is any number. Basically, we multiplied $T(s)$ by $(s+a)$
- If we think of it, by using the laws of distributions, the version of a second order system with a zero is $sT(s) + aT(s)$
- $sT(s)$ means the first derivative of $T(s)$, while $aT(s)$ is a scaled version of $T(s)$, so in other words, adding a zero to a second order system basically has the effect of adding the derivative of the system to a scaled version of itself. For a unit step response, usually this derivative is positive at the beginning.
- When a is large, the scaled version overcomes the derivative part, and the derivative part become negligible, this is why we noticed that as the zero value increases, it approximates a second-order system
- When a is small, the derivative effects are more noticeable

* شكل هذه المعادلة أنه تكون المسنقة بالعلاقة بالبيانية
لبتها موجبة، إذا صرفت وكانت قيم \propto سالبة تكون البيانية سالبة (Input اموجب والبيانية سالبة)
هذا معنده بال Control، أنا أسوق السيارة كطريق Order يروح حالياً يمين راحت نارت السيارة عالي شمال.

* فلو كان الـ 2nd المي اندیف بالبسط سالب بدأ تدين بالـ 1st، انه هنا الـ System عارض الواقع ويرجعه وكيف اتجه الـ order المطلوب.

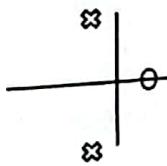
وهذا بالغالب مذموم.

* sys Control الملي يعطي order ويروح بعدها
وكذلك بالبداية حين ما يرجع sys يديه.

47

Nonminimum-phase system

- ▶ What if the zero a is on the right-hand plane?
- ▶ We see that the derivative term, which is typically positive initially, will be of opposite sign from the scaled response term.
- ▶ Notice that the response begins to turn toward the negative direction even though the final value is positive. A system that exhibits this phenomenon is known as a **nonminimum-phase system**.
- ▶ If a motorcycle or airplane was a nonminimum-phase system, it would initially veer left when commanded to steer right.



كتشاف تحفيز المعاشرة Constant بالبسط وعما لا
تريبيعة بالمقام وتهيئ نتعامل معها في قبل.
وتحقق لها كلها متطلبات ١٠٠% من قلب هليقرا أكسنل
كنا قبل نجيلى لوفي $(s+2)$ مثلًا بالبسط والمقام فنكون Pole
Zero ١٢ - السؤال هون هل يغير أكسنل $(s+2)$ من البسط والمقام؟

48

Zero – Pole Cancellation I

- ▶ Suppose we have systems with many poles and zeros (high-order systems). Can we cancel zeros and poles so that we reach a system that approximates a 2nd order system that we already know how to analyze?
- ▶ Can we cancel the zero $(s + 4)$ and the pole $(s + 3.5)$ which are the closest to each other, so that we end up with a second order system?
هليقرا أشتغل بال $(s+4)$
مع الـ $(s+3.5)$
- ▶ We must do a partial fractions expansions and evaluate the residue, we notice that the residue of $(s + 3.5)$ term is so close to the others, therefore, no cancellation.

$$C_1(s) = \frac{26.25(s+4)}{s(s+3.5)(s+5)(s+6)}$$

Constant $\xrightarrow{\text{zero}}$
unit $\xrightarrow{\text{Step funct.}}$ Poles

أولاً لا ندخلها بال Partial fraction
Syms S C(s)
 $C(s) = 26.25 * (s+4) / (s * (s+3.5) * (s+5) * (s+6))$
Partfrac (C(s))

الآن ندخلها بال Matlab

$$C_1(s) = \frac{1}{s} \left[\frac{3.5}{s+5} + \frac{3.5}{s+6} - \frac{1}{s+3.5} \right]$$

unit step

قيرب علا 3.5 و 3.5 معناه الوزن تبع الـ ١
عليه هنا يغير أشتغل بال Matlab مثلًا بتحليبي
الـ $(s+3.5)$ مع الـ $(s+4)$ اللي فوق فما يقدر أطبق المعاشرات اللي تختلفوا عليه حيصله من الدرجة الثالثة وفيه
Zero بالبسط.

فانت بس بسطها واقسم الـ $1/2$ + وزن المعاشرة.

Zero – Pole Cancellation II

ریاضیاً ما بحیث نشطبم احنا بنشطب تغییرها

- ▶ Can we cancel the zero ($s + 4$) and the pole ($s + 4.01$) which are the closest to each other, so that we end up with a second order system?

$$C_2(s) = \frac{26.25(s+4)}{s(s+4.01)(s+5)(s+6)} \rightarrow * R = \frac{1}{s} \xrightarrow{\text{TF}} C(s)$$

- ▶ We must do a partial fractions expansions and evaluate the residue, we notice that the residue of ($s + 4.01$) term is so far away from the others, therefore cancellation is possible

$e^{-6}, e^{-5} \xrightarrow{\text{as time goes to }} \pi(\infty) \rightarrow$ $C_2(s) = \frac{0.87}{s} - \frac{5.3}{s+5} + \frac{4.4}{s+6} + \frac{0.033}{s+4.01}$

لحدى عن الأداء دلماً البسط الباقي على $\frac{0.87}{s}$ هو الذي أقرب لـ 4.4، فنقسم لـ 4.4 $C_2(s) \approx \frac{0.87}{s} - \frac{5.3}{s+5} + \frac{4.4}{s+6}$.
علاوة على ذلك، إذا كانت حامل $\frac{0.033}{s+4.01}$ المتصل بالزمن، وبطبيعته، فهو ينبع من معادلة تربيعية بغير أطبق علىها المعادلات.

References

- ▶ The material in these slides are based on:
Control Systems Engineering, Norman S. Nise, 7th Edition (2014), John Wiley And Sons
- Chapter 4 – Time Response
 - ▶ Sections 4.1, 4.2, 4.3, 4.4, 4.5, 4.6, 4.7 and 4.8 (Students kindly note that some sections involve math derivations that we did not cover in class)

* يعنـا الـ Chapter ما ذكرـاـتـه زـيـ المـنـاـلـاـتـيـ تـهـ ، حـنـتـرـفـ مـحـرـقـ

ريـاضـيـةـ يـسـرقـيـ شـوـالـمـاهـمـاتـ الـيـ أـحـطـهـواـعـاـلـاـ Component الجـبـيـةـ لـجـنـقـ يـقـيـ المـلـكـ

* الوـظـيـقـةـ الـأـسـاسـيـةـ بـوـزـاـلـاـ Chapter لـغـاـيـاتـ الـدـيـزـيـنـ ، إـنـهـ هـذـاـ لـوـدـنـتـ

جـرـبـيـةـ الـسـيـسـتـمـ (like Amplifier, etc...) هـذـاـ سـيـقـلـاـلـاـ System Stable ولاـ لـاـ .

System Stability

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* Suppose we have this system $\frac{S}{S^2 + 5S^2 + 3S + 2}$ هل هو أو لا؟ ،

Nomical form بـنـوـلـهـ المـقـامـ . Roots بـتـجـدـ حـسـبـ المـقـامـ (Poles) وـالـيـهـيـ الـRـo~otsـ .

In Matlab & Vector: [1, 0, 0, 0, 0, 5, 3, 2] * Roots (vector) عدد الـ rootsـ لـ الـ stableـ بـنـوـلـهـ المـقـامـ (Poles) .

انتـهـ هـنـاـوـ بـنـوـلـهـ Matlab Complex بـنـوـلـهـ Matlab بـنـوـلـهـ . Roots بـنـوـلـهـ المـقـامـ .

Roots * 6+0.000j → Real 0+7.2j → Img-axis يـاـ Re~al-axis يـاـ Re~al-part يـاـ Re~al .

Roots * 2+j → complex . Img-part يـاـ Im~ag-axis يـا~ Im~ag-part .

S-Domain = (left half plane / Right half plane)

2

مـعـدـ مـاـيـ مـاـلـاـ Sys.Poles RHP Sys. فالـ الـRـe~alـ

Definitions

أـهمـ وـدـةـ الـمـقـالـ مـنـ دـلـلـةـ اـنـ تـرـوـتـ .

* In Chapter 1, we saw that three requirements play a major role in the design of a control system: transient response, stability, and steady-state errors.

* Stability is the most important system specification. If a system is unstable, transient response and steady-state errors are moot points. An unstable system cannot be designed for a specific transient response or steady-state error requirement.

* We can control the output of a system if the steady-state response consists of only the forced response. But the total response of a system is the sum of the forced and natural responses $c(t) = c_{forced}(t) + c_{natural}(t)$

* Using these concepts, we present the following definitions of stability, instability, and marginal stability.

✓ A linear, time-invariant system is stable if the natural response approaches zero as time approaches infinity.

✓ A linear, time-invariant system is unstable if the natural response grows without bound as time approaches infinity.

✓ A linear, time-invariant system is marginally stable if the natural response neither decays nor grows but remains constant or oscillates as time approaches infinity.

* Thus, the definition of stability implies that only the forced response remains as the natural response approaches zero

هـيـ النـارـيـفـ مـرـبـوـطـةـ بـنـوـلـهـ الـS~ystemـ (Natural response) بيـنـ

عـشـرـ اـبـطـلـهـ بـلـاـ inputـ ، بـلـكـيـلاـ Sysـ دـىـنـدـ اـلـiـنـيـجـيـاـ فـرـقـ

عـسـتـوـهـ وـنـاـعـشـ مـرـبـوـطـ بـنـوـلـهـ ، لـيـكـ اـلـيـدـ سـيـلـفـهـ هـيـ التـرـوـنـ.

المـطـلـوبـاتـ بـسـماـخـنـ بـسـماـخـنـ الـS~ystemـ

الـS~ystemـ

"Bounded Input / Bounded Output"

3

Definitions II : BIBO Stability

- * ► The previous definition only takes into account the natural response and not the total response. What if the input (forced response) is from the beginning unbounded? BIBO Stability definition means:
 - ✓ 1. A system is stable if every bounded input yields a bounded output.
 - ✓ 2. A system is unstable if any bounded input yields an unbounded output
- * ► Under this definition, marginal stability, means that the system is stable for some bounded inputs and unstable for others.
- * ► Physically, an unstable system whose natural response grows without bound can cause damage to the system, to adjacent property, or to human life. From the perspective of the time response plot of a physical system, instability is displayed by transients that grow without bound and, consequently, a total response that does not approach a steady-state value or other forced response.

* Poles in LHP \rightarrow Stable.

* Poles in Img-axis \rightarrow undamped \rightarrow Marginally Stable.

* Poles in RHP \rightarrow Unstable.

4

How do we determine if a system is stable?

- * ► Stable systems have closed-loop transfer functions with poles only in the left half-plane (LHP).
- * ► Unstable systems have closed-loop transfer functions with at least one pole in the right half-plane and/or poles of multiplicity greater than 1 on the imaginary axis.
 - ✓ Poles in the right half-plane yield either pure increasing exponentials or exponentially increasing sinusoidal natural responses. These natural responses approach infinity as time approaches infinity.
 - ✓ Poles of multiplicity greater than 1 on the imaginary axis lead to the sum of responses of the form $A t^n \cos(\omega t + \phi)$, where $n = 1, 2, \dots$, which also approaches infinity as time approaches infinity
- * ► Marginally stable systems have closed-loop transfer functions with only imaginary axis poles of multiplicity 1 and poles in the left half-plane

$$*\frac{1}{(s+2)(s+2)} \rightarrow \begin{array}{c} \times \\ -2 \end{array}$$

٢ بعدين مطبقين يعني
(نفس بعض)

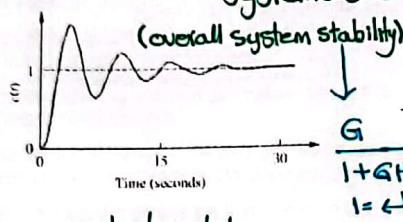
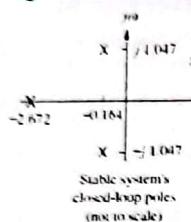
* Multiplicity n &
n Poles (نفس بعض)
(راكبين فوق بعض)

$\hookrightarrow * e^{-2t} + t e^{-2t} \rightarrow$ System هل لها اصل
BIBO \rightarrow Stable or not
بعض الكتب تذكرها
بعض الكتب في المقام في t فـ unstable وهي كتب
لا يتعلموا بعد ما يتعلمن.

Examples

لـ $G(s) = \frac{1}{(s+1)(s+2)}$ لو طلعت الأـ s -poles بـ $(0, -1, -2)$ بتـ $G(s) = \frac{1}{s+1}$ بتـ $G(s) = \frac{1}{(s+1)(s+2)}$

(Subsystem stability).

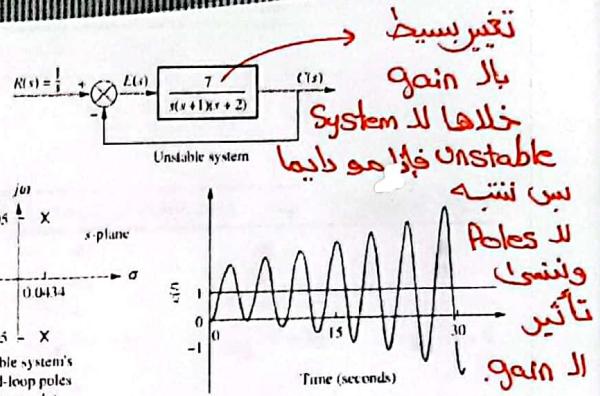


لـ $G(s) = \frac{1}{(s+1)(s+2)}$ لا يـ $G(s) = \frac{1}{s+1}$ أو لا يـ $G(s) = \frac{1}{(s+1)(s+2)}$ لـ $G(s) = \frac{1}{s+1}$ لـ $G(s) = \frac{1}{(s+1)(s+2)}$

Transfer funct. or

System

لـ $G(s) = \frac{1}{(s+1)(s+2)}$



$$\frac{G}{1+GH} = \frac{3}{s(s+1)(s+2)} = \frac{3}{s^3 + 3s^2 + 2s}$$

هـ $G(s) = \frac{1}{s+1}$ هـ $G(s) = \frac{1}{(s+1)(s+2)}$ وـ $G(s) = \frac{1}{s+2}$ هـ $G(s) = \frac{1}{s+1}$ هـ $G(s) = \frac{1}{(s+1)(s+2)}$

* المـ $G(s) = \frac{1}{s+1}$ بـ $G(s) = \frac{1}{(s+1)(s+2)}$ بـ $G(s) = \frac{1}{s+2}$ بـ $G(s) = \frac{1}{s+1}$ بـ $G(s) = \frac{1}{(s+1)(s+2)}$

$\boxed{3} \rightarrow \frac{1}{s(s+1)(s+2)}$: (Amplifier) يـ $G(s) = \frac{1}{s+1}$ بـ $G(s) = \frac{1}{(s+1)(s+2)}$

قـ $G(s) = \frac{1}{s+1}$ وـ $G(s) = \frac{1}{(s+1)(s+2)}$ بـ $G(s) = \frac{1}{s+2}$ بـ $G(s) = \frac{1}{s+1}$ بـ $G(s) = \frac{1}{(s+1)(s+2)}$

بتـ $G(s) = \frac{1}{s+1}$ بـ $G(s) = \frac{1}{(s+1)(s+2)}$ بـ $G(s) = \frac{1}{s+2}$ بـ $G(s) = \frac{1}{s+1}$ بـ $G(s) = \frac{1}{(s+1)(s+2)}$

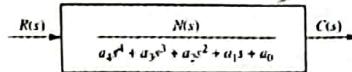
Routh-Hurwitz Criterion for Stability

الـ $G(s) = \frac{1}{s+1}$ تـ $G(s) = \frac{1}{(s+1)(s+2)}$ تـ $G(s) = \frac{1}{s+2}$ تـ $G(s) = \frac{1}{s+1}$ تـ $G(s) = \frac{1}{(s+1)(s+2)}$

- * Routh-Hurwitz Criterion is a method that yields stability information without the need to solve for the closed-loop system poles. Using this method, we can tell how many closed-loop system poles are in the left half-plane, in the right half-plane, and on the $j\omega$ -axis. (Notice that we say *how many*, not *where*.) We can find the number of poles in each section of the s -plane, but we cannot find their coordinates.
- * The method requires two steps: (1) Generate a data table called a Routh table and (2) Interpret the Routh table to tell how many closed-loop system poles are in the left half-plane, the right half-plane, and on the $j\omega$ -axis
- * If we can use computers to easily get the roots (poles), why would we need this method? The power of Routh-Hurwitz criterion does not lie in the analysis stage but in the design stage, as it can let us know how can we change the range of a parameter and the system remains stable (check last slide).

Generating a Basic Routh Table

- Suppose we have the following equivalent closed-loop transfer function:



- Label the rows with powers of s from the highest power of the denominator of the closed-loop transfer function to s^0 . **الخطوة الأولى**
- Next start with the coefficient of the highest power of s in the denominator and list, horizontally in the first row, every other coefficient. **الخطوة الثانية**
- In the second row, list horizontally, starting with the next highest power of s , every coefficient that was skipped in the first row.
- Each entry is a negative determinant of entries in the previous two rows divided by the entry in the first column directly above the calculated row.
- The left-hand column of the determinant is always the first column of the previous two rows, and the right-hand column is the elements of the column above and to the right.
- We can multiply/divide any row, if needed, by a positive constant without changing the analysis.

TABLE 6.1 Initial layout for Routh table

s^4	a_4	a_2	a_0
s^3	a_3	a_1	0
s^2			
s^1			
s^0			

TABLE 6.2 Completed Routh table

s^4	a_4	a_2	a_0
s^3	a_3	a_1	0
s^2	b_1	b_2	0
s^1	c_1	c_2	0
s^0	d_1	d_2	0

* المتطلبات الرئيسية هي الجذور اليدين من كل

الجذول ملحوظ

* مهم نعرف نحل الأ

Example 1

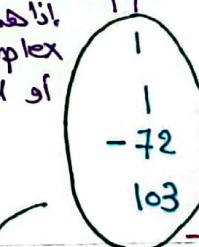
- Make the Routh table for the system shown in the right.
- First, we need to find the equivalent transfer function for the entire system, then fill in the table

قسم على رقم موجب "ما يغير سالب"

لتحق ببساط الأرقام ويسهل الحل.

s^3	1	31	0
s^2	40	103	0
s^1	$\begin{vmatrix} 1 & 31 \\ 1 & 103 \end{vmatrix} = -72$	$\begin{vmatrix} 1 & 0 \\ 1 & 0 \end{vmatrix} = 0$	$\begin{vmatrix} 1 & 0 \\ 1 & 0 \end{vmatrix} = 0$
s^0	$\begin{vmatrix} 1 & 103 \\ -72 & 0 \end{vmatrix} = 103$	$\begin{vmatrix} 1 & 0 \\ -72 & 0 \end{vmatrix} = 0$	$\begin{vmatrix} 1 & 0 \\ -72 & 0 \end{vmatrix} = 0$

ما يغير أعرف
إذا هم
Complex
أو real



من خلاله نعرف إذا المثلث
أو لا.

stable

ما يغير
بعض
الأماكن
يعرف
qualitative

يعني أول عود، إذا كانت جميع القيم
موجبة فعنده جميع ال Zeros بال LHP يعني العا

System stable أو في قيم سالبة فعدد مرات تغير الإشارة هو عدد
الجذور الموجبة بالجزء اليدين (RHP)، (ما يغير القيم بالآماكن) Sys unstable

هذا الطريقة اخري تewan زمان ما كانوا يقدروا
يحلوها بأجهزة والآن حاسة وهكذا

9

Routh-Hurwitz Basic Interpretation

- The number of roots of the polynomial that are in the right half-plane is equal to the number of sign changes in the first column.
- If an entire row of zeros appear in the Routh Table, then we have $j\omega$ poles
- In the previous example, we have a cubic equation, so we expect three roots.
 - We have two sign changes in the first column, this means two poles in the RHP \rightarrow System Unstable
 - We have $3 - 2 = 1$ root in the LHP
 - No row is entirely zero, no pure $j\omega$ poles

```
>> roots([1 10 31 1030])
ans =
-13.4136 + 0.0000i
1.7063 + 8.5950i
1.7063 - 8.5950i
```

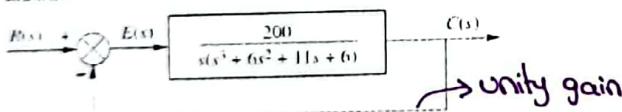
النهاية ←
النهاية ←
النهاية ←

لوبنا نربطوا بالواقع نقدر نتمنى إن هذا عبارة عن إشارة تتيحني أتحكم فيها بـ motor
اللي الـ معلنة Amplifier ويرجع بـ unity gain feedback على قيم error وبحسب الـ
يشكل مستقر.

10

Example 2

- Find the number of poles in the left half-plane, the right half-plane, and on the $j\omega$ -axis for the



- First, we find the equivalent transfer function then complete the Routh Table

$$F(s) = \frac{200}{s^4 + 6s^3 + 11s^2 + 6s + 200}$$

- * At the s^1 row there is a negative coefficient; thus, there are two sign changes. The system is unstable, since it has two right-halfplane poles and two left-half-plane poles. The system cannot have $j\omega$ poles since a row of zeros did not appear in the Routh table

System ← 2 poles in Right half plane ← 2 changes
unstable

قسم أرقام موجبة
للتبسيط.

s^4	1	11	200
s^3	-6	1	1
s^2	-10	1	200
s^1	-19		
s^0	20		

```
>> roots([1 6 11 6 200])
```

```
ans =
```

```
-4.2763 + 2.5409i
-4.2763 - 2.5409i
1.2763 + 2.5409i
1.2763 - 2.5409i
```

الأماكن
الفاصلية

0 =

المشكلة بالسطر الثالث طلع معناه فاللوي بعده يكون بقسم 5
وهذا ما يحصل.

11

Routh-Hurwitz Criterion: Special Cases I

- Determine the stability of the closed-loop transfer function

$$T(s) = \frac{10}{s^5 + 2s^4 + 3s^3 + 6s^2 + 5s + 3}$$

• تتشبه بـ $\lim_{\epsilon \rightarrow 0}$.

s^5	1	3	5
s^4	2	6	3
s^3	$\theta - \epsilon$	$\frac{7}{2}$	0
s^2	$\frac{6\epsilon - 7}{\epsilon}$	3	0
s^1	$\frac{42\epsilon - 49 - 6\epsilon^2}{12\epsilon - 14}$	0	0
s^0	3	0	0

Second assumption

Label	First column	$\epsilon = +$	$\epsilon = -$
s^5	1	+	+
s^4	2	+	-
s^3	$\theta - \epsilon$	-	-
s^2	$\frac{6\epsilon - 7}{\epsilon}$ بخط أول من 7	$(-) (-)$	\oplus
s^1	$\frac{42\epsilon - 49 - 6\epsilon^2}{12\epsilon - 14}$	$(+) (-)$	\oplus
s^0	3	+	+

المفروض بعد
الـ changes
 تكون نفسها
 بالذموديات لأنـ
 نفس الـ System.
 (ينتـ تغيرـون
 بالإشارة فإذاـ فيـ

الـ 3 poles بالـ RHP حالـ System يكونـ بالـ LHP

* الـ Computers غيرـ قادرـ علىـ تمثـ الأـ رقمـ بـ سـ طـ لـ حـ يـقـ.

* قدـ تكونـ قـيـمةـ مـوجـيـةـ أوـ سـالـبةـ فـلـ حـنـقـزـ الـ اـتـمـالـيـنـ وـ نـعـرـفـ الـ إـشـارـاتـ لـ الـ عـمـوـدـ الـ أـولـ.

* هـنـاكـ ماـ يـحـتـمـلـ مـعـاـ بـ أـرـقـامـ بـ سـتـخـمـواـيـشـانـ نـعـرـفـ الـ إـشـارـاتـ

فـقطـ.

- إذاً طبعـ معـناـيـ 5 بـ بـ دـعـ
- (Epsilon) تـبـرـيـدـ رقمـ مـغـيـبـ جـداـ قـيـبـاـ مـنـ الصـفـرـ بـ مـشـ
- صـفـرـ، حـنـقـانـ حـسـانـ ماـ يـخـبـ
- الـ S~ystemـ أوـ اـنـطـرـ أـقـسـمـ 0ـ Z~eroـ.
- أـمـيـرـ رقمـ مـعـكـ يـمـثـ الـ C~omputerـ الـ C~omputerـ يـشـعـمـ مـقـواـمـ علىـ
- أـنـجـاـ Constantـ.

12

Routh-Hurwitz Criterion: Special Cases II

الـ سـطـرـ الـ أـولـ

بـ سـلـوـيـ

الـ سـطـرـ الثـانـيـ

لـ فـوـيـكـ

كـلـ المـمـلـاتـ

مـيـكـونـ

الـ نـاتـجـ

فـيـواـ 0ـ (لـ الـ يـسـطـ)

الـ S~3~

الـ خطـ الأـحـمـرـ الـ

يـفـعـلـ فـيـ التـعـلـيلـ

هـوـ الـ خطـ الـ يـفـعـ

الـ مـشـقـةـ (S~4~)

- Determine the number of right-half-plane poles in the closed-loop transfer function

$$T(s) = \frac{10}{s^5 + 7s^4 + 6s^3 + 42s^2 + 8s + 56}$$

- We stop at the third row, since the entire row consists of zeros

- We return to the row immediately above the row of zeros and form an auxiliary polynomial, using the entries in that row as coefficients. The polynomial will start with the power of s in the label column and continue by skipping every other power of s. $P(s) = s^4 + 6s^2 + 8$ \rightarrow كـيـنـاـعـدـلـةـ بـنـاءـ الـ مـلـحـرـ

- Next, we differentiate the polynomial with respect to s and obtain

$$\frac{dP(s)}{ds} = 4s^3 + 12s + 0$$

- We use the coefficients of the differential to replace the row of zeros

- All all entries in the first column are positive. Hence, there are no right-half-plane poles. We need to understand the concept of symmetry before moving forward in our analysis.

- طـرـيـقـ التـعـلـيلـ بـلـتـ زـيـ أـولـ لـ إـنهـ تـغـيـرـتـ الـ M~at~h~ (بـطـلـاـ نـتـطـلـ بـ سـعـاـلـ يـمـودـ)
- * سـيـمـ تقـسـمـ التـعـلـيلـ إـلـيـ قـسـمـيـنـ قـسـمـ مـاقـبـلـ الـ اـشـفـاقـ وـ قـسـمـ مـاـ بـعـدـ، التـعـلـيلـ أـسـفـلـ الـ اـشـفـاقـ يـشـمـلـ قـوـادـيـاـلـ الـ Caseـ، أـمـاـ أـعـلـ الـ اـشـفـاقـ بـكـوـنـ التـعـلـيلـ زـيـ هـيـ.
- * العـلـ بـيـنـ الـ أـعـلـ وـ الـ أـسـفـلـ بـيـنـ الـ مـارـلـعـ الـ أـمـمـ (فـوقـ سـطـرـ الـ اـشـفـاقـ).

s^5	1	6	8
s^4	7	1	$\frac{42}{56}$
s^3	$\frac{4}{7}$	$\frac{1}{2}$	0

s^2	3	8	0
s^1	$\frac{1}{3}$	0	0
s^0	8	0	0

١: أسفل الاشتغال: ما في تغير
4 poles in Img. axis ←
بالإشارة ←

حل لا Case السابعة بناء على التحليل: ٢: أعلى الاشتغال: ما في تغير
1 pole in LHP ←
بالإشارة ←

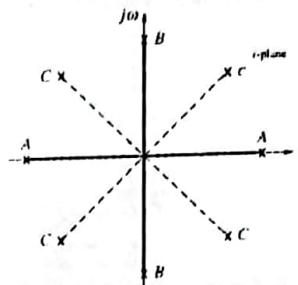
13

More about Rows of Zeros

- Let us look further into the case that yields an entire row of zeros. An entire row of zeros will appear in the Routh table when a purely even or purely odd polynomial is a factor of the original polynomial. For example, $s^4 + 5s^2 + 7$ is an even polynomial; it has only even powers of s.

- Even polynomials only have roots that are symmetrical about the origin. This symmetry can occur under three conditions of root position:

- The roots are symmetrical and real,
- The roots are symmetrical and imaginary, or
- The roots are quadrantal.



A: Real and symmetrical about the origin
B: Imaginary and symmetrical about the origin
C: Quadrantal and symmetrical about the origin

لـ تحمل فقط حال حدهم الي خربوا سطر كاحد بساوي و حلية بطرق المشقة.

- * لو كانوا كلهم موجب يكون حالاً *
1- اذا ما في ولا تغير بالإشارة (كل العدد موجب)
فاحـ دا Something like this: $0+2j$
- $0-3j$ خارـ Poles يكونـ اعـالـ اـيمـنـ او لـيسـارـ. j
- 2- لو في تغيرات بالإشارة فعدد التغيرات يكون عدد Poles اللي حال RHP ولـكن بسبب خاصـيـةـ الـSymmetryـ فـيـكـونـ فـيـ نـفـسـوـمـ عـالـ LHPـ يـخـطـاـ.

14

Example 1

20						
$T(s) = \frac{1}{s^8 + s^7 + 12s^6 + 22s^5 + 39s^4 + 59s^3 + 48s^2 + 38s + 20}$						
For the transfer function						
in the left half-plane, and on the $j\omega$ -axis						
No sign changes exist from the s^4 row down to the s^0 row. Thus, the even polynomial does not have right-half-plane poles. Since there are no right-half-plane poles, no left-half-plane poles are present because of the requirement for symmetry. Hence, the even polynomial, must have all four of its poles on the $j\omega$ -axis.						
Since the polynomial is of degree 8, four roots remain. From the s^8 row down to the s^4 row, there are two sign changes, this means, two poles in the RHP. The two remaining poles will therefore be in the LHP.						
Symmetry due to the requirement of symmetry.						

لـ المـقـدـلـ بـطـيـكـ الـta~bleـ جـزـءـاتـ
لـنـمـ تـحـلـلـهـ وـكـنـشـفـهـ هـوـ أـيـ Caseـ وـيـنـجـ
كـلـيـاـ تـكـمـلـ تـحـلـيلـكـ.

Example I

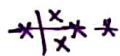
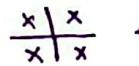
- For the transfer function $T(s) = \frac{20}{s^8 + s^7 + 12s^6 + 22s^5 + 39s^4 + 59s^3 + 48s^2 + 38s + 20}$ how many poles are in the right half-plane, in the left half-plane, and on the $j\omega$ -axis
- No sign changes exist from the s^4 row down to the s^0 row. Thus, the even polynomial does not have right-half-plane poles. Since there are no right-half-plane poles, no left-half-plane poles are present because of the requirement for symmetry. Hence, the even polynomial must have all four of its poles on the $j\omega$ -axis.
- Since the polynomial is of degree 8, four roots remain. From the s^8 row down to the s^4 row, there are two sign changes, this means, two poles in the RHP. The two remaining poles will therefore be in the LHP.

```
>> roots([1 12 22 39 59 48 38 20])
```

ans =

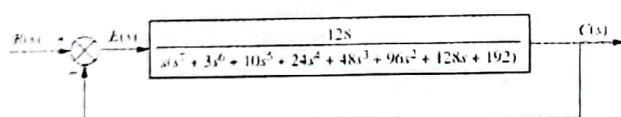
$$\begin{aligned} 0.5000 &= 3.12351 & \xrightarrow{\text{RHP}} \\ 0.5000 &= 3.12351 \\ 0.0000 &= 1.61421 * \\ 0.0000 &= 1.61421 * \\ -1.0000 &= 0.00001 & \xrightarrow{\text{LHP}} \\ -1.0000 &= 0.00001 & \xrightarrow{\text{Imaginary}} \\ -0.0000 &= 1.00001 * \\ -0.0000 &= 1.00001 * \end{aligned}$$

* سؤال امتحان ٢ يحمل الجدول ويطلب تطبيق أمكننا الـ Poles وبخطيك ٤ أشكال لـ المثلث



Example II

- Find the number of poles in the left half-plane, the right half-plane, and on the $j\omega$ -axis for the system



The closed loop transfer function for the system is

$$T(s) = \frac{128}{s^8 + 3s^7 + 10s^6 + 24s^5 + 48s^4 + 96s^3 + 128s^2 + 192s + 128}$$

Example II - Continued

- Two sign changes exist from the s^6 row down to the s^0 row. Thus, the even polynomial has two right-half-plane poles. Since there are two right-half-plane poles, there are two left-half-plane due to symmetry.
- Since the degree of the even polynomial is 6, and we have found four poles on the left and right, this means we have two poles remaining on the jw axis
- Since the degree of the overall polynomial is 8, and we have found six, we look at the sign changes from s^8 row down to the s^6 row, since there are no sign changes, this means we have all remaining poles in the LHP

s^8	1	10	48	128	128
s^7	-3	1	-24	8	48
s^6	-2	1	-16	8	48
s^5	-8	-3	-48	16	-48
s^4	8	1	64	8	64
s^3	3		3	8	24
s^2	-4	-1	-16	-5	
s^1	-2	1	-24	8	
s^0	3				

because we have 2 changes

لدينا سطر
المشتقه

RHP	$\left[\begin{array}{c} 1.0000 + 1.7321i \\ 1.0000 - 1.7321i \\ 0.0000 + 2.0000i \\ 0.0000 - 2.0000i \end{array} \right]$
LHP	$\left[\begin{array}{c} -1.0000 + 1.7321i \\ -1.0000 - 1.7321i \\ -2.0000 + 0.0000i \\ -2.0000 - 0.0000i \end{array} \right]$
Imaginary	$\left[\begin{array}{c} 0.0000 + 2.0000i \\ 0.0000 - 2.0000i \end{array} \right]$

لدينا 2 RHP
لدينا 2 LHP
لدينا 2 Imaginary
لدينا Symmetric

في مجموع Real وفي مجموع Imaginary

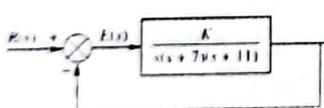
لأنه طلعت 4
وبكون حنائي 2 لأن الجزء الذي تحت لازم يطلع 6 عدد لا
، فاللي ضليل بكون أكيد عالي

Imaginary

Stability Design via Routh-Hurwitz Criterion

→ Amplifier → مخرج بالمعادله لزيادة القوة → Increase current or voltage.

- Find the range of gain, K, for the system shown in the figure that will cause the system to be stable, unstable, and marginally stable. Assume $K > 0$.



- The closed-loop transfer function is

$$T(s) = \frac{K}{s^3 + 18s^2 + 77s + K}$$

Transfer funct.
(نطاعه تتبع مع وظيفه ثالثه)

تعزى القيمه ل K
وهي تقدر
نستخدموها
وماتخرب
System

استخراج
Routh-Hurwitz
بخطوات

$$\begin{matrix} s^3 & 1 & 77 \\ s^2 & 18 & K \\ s^1 & \frac{1386 - K}{18} & \end{matrix}$$

طقوامن
القوانين

Since K is assumed positive, we see that all elements in the first column are always positive except the s^1 row. This entry can be positive, zero, or negative, depending upon the value of K. If $K < 1386$, all terms in the first column will be positive, and since there are no sign changes, the system will have three poles in the left half-plane and be stable.

بتخلق قيم أول عمود موجبة فالـ System Stable

If $K > 1386$, the s^1 term in the first column is negative. There are two sign changes, indicating that the system has two right-half-plane poles and one left-half-plane pole, which makes the system unstable.

* بالمايلاب يمكن يكون في دايموند بروجيكشن قيم الـ K الممكنة ليظل المـ System Stable دون الحاجة لحل قوانين وحدريمه (بالصور والرسم).

Stability Design via Routh-Hurwitz Criterion

- If $K = 1386$, we have an entire row of zeros, which could signify jw poles.
- Returning to the s^2 row and replacing K with 1386, we form the even polynomial

$$P(s) = 18s^2 + 1386$$

- Differentiating with respect to s, we have

$$\frac{dP(s)}{ds} = 36s + 0$$

- Replacing the row of zeros with the coefficients of the differential, we obtain the new Routh-Hurwitz table for the case of $K = 1386$

undamped
(Marginally
stable)

s^3	1	77
s^2	18	1386
s^1	-0	36
s^0		1386

الخط الأحمر (فوق
الخط الأزرق طبعنا فيه المسئلة)

Since there are no sign changes from the even polynomial (s^2 row) down to the bottom of the table, the even polynomial has its two roots on the jw-axis of unit multiplicity. Since there are no sign changes above the even polynomial, the remaining root is in the left half-plane.

Therefore, the system is marginally stable.

Step 1 & K ↴ Vector
for loop

how we solve these Problems in Matlab :

for $K = 1:2000$

$R = \text{roots}([1 18 77 K])$

* $\text{my_roots} = \text{real}(R)$

if $\max(\text{my_roots}) \geq 0$

disp("System unstable")

Ki ↩ break;

end

* لو رسمت المخطط لـ Step response لـ K مختلفة لا يتحقق ذلك في وقت.

يعني أن الخط
الـ Poles لميغنا ولا
شعلان.

لنفرض عند النهاية كانت الأجهزة
الـ myroots بلا real
يكون غالبيته
فيكون unstable

References

قيمة
الـ k
الـ sys
منها
unstable

فينوقف بـ K الـ sys

لـ $\text{K} = 1$ لا ينعد متصال

The material in these slides are based on:

unstable Control Systems Engineering, Norman S. Nise, 7th Edition (2014), John Wiley And Sons

• Chapter 6 – Stability

- Sections 6.1, 6.2, 6.3, 6.4, (Students kindly note that some sections involve math derivations that we did not cover in class)



Steady State Errors

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FALL 2022

- * يوضح في هذا الـ chapter أخطاء عمليات (MatLab)
- * تعلم 3 أنواع لـ Control systems
- * ما هي شروط الـ Steady-State و Transient و Stability

* Time \rightarrow Steady-state \leftarrow Transient response ، كل المأذوقات ، كل الحالات

* يمثل الـ Steady-state-error وبين الـ forced input يمكن أخذ ما يهمك

2

Topics

* باللي درسناه له إمكان (unit step) $R(s) = \frac{1}{s}$ و دعوه State Second (ثانية)

- ▶ Definitions and Signal Inputs
- ▶ Sources of Error
- ▶ The Final Value Theorem
- ▶ Static Error Constants k_p , k_v and k_a
- ▶ Steady State Errors in Closed-Loop Unity Feedback System
- ▶ Steady State Errors in Closed-Loop Unity Feedback System with Disturbance
- ▶ Steady State Errors in Closed-Loop Non-Unity Feedback System
- ▶ Steady State Errors in Closed-Loop Non-Unity Feedback System with Disturbance
- ▶ Extra Examples

* ملاحظة: **Satellite** به يتبع عادةً أجرامٍ صناعية أو هيلك،
استخدام الـ **Unit step function** يعني **Satellite** به يتبع
(محل ما أطلقناه وحطيناه بهله بمكانه) **geostationary orbit**
(مثل نايلسات)، فهنية التتبع هي **on/off**.

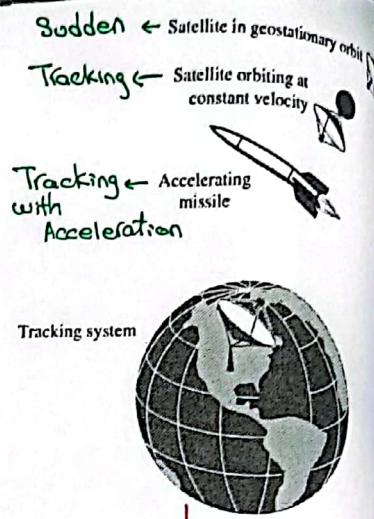
Definition and Test Inputs

Steady-state error is the difference between the input and the output for a prescribed test input as $t \rightarrow \infty$.

Step inputs represent constant position and thus are useful in determining the ability of the control system to position itself with respect to a stationary target, such as a satellite in geostationary orbit.

Ramp inputs represent constant-velocity inputs to a position control system by their linearly increasing amplitude. These waveforms can be used to test a system's ability to follow a linearly increasing input or, equivalently, to track a constant-velocity target. For example, a position control system that tracks a satellite that moves across the sky at a constant angular velocity.

Parabolas, whose second derivatives are constant, represent constant acceleration inputs to position control systems and can be used to represent accelerating targets, such as the missile in, to determine the steady-state error performance.



* ملاحظة: يكون **Satellite System** بطريقة **on/off** مثل **العنتر** (محطة الفضاء الدولية) ← يبلغ كل 90 دقيقة بخلاف المدار لذا **Satellite** يخلي **Space** ويتابع، هرول **System** ما يخلوم **Step funct** يخلي **Space** ثابتة
عشرن ما يزيد بعد ما يطلع بي المدار) (هذا **input** (constant velocity) ↓ $t u(t)$

Test Waveforms

TABLE 7.1 Test waveforms for evaluating steady-state errors of position control systems

Waveform	Name	Physical interpretation	Time function	Laplace transform
	Step	Constant position	1	$\frac{1}{s}$
	Ramp	Constant velocity	t	$\frac{1}{s^2}$
	Parabola	Constant acceleration	$\frac{1}{2}t^2$	$\frac{1}{s^3}$

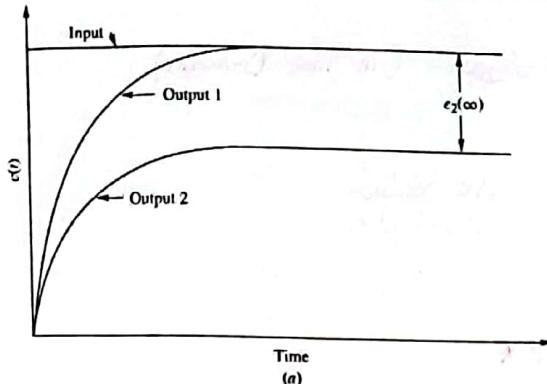
* النوع الثالث من **systems** مثل المدار (ما ي تكون سريعاً ثابت يتوجه تسللاً فالـ **Ramp**).

ما يكتنوا يستخدم الـ **Parabola** (Constant acceleration).

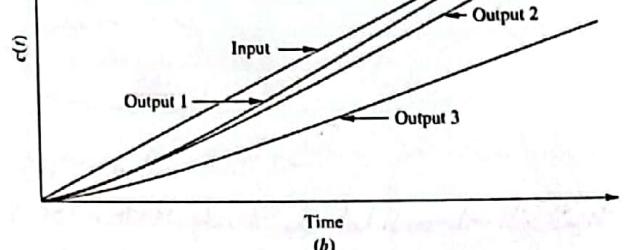
$$\leftarrow t^2 u(t)$$

لـ **بنديفون** شان نتأكد إنها في قيم Time سالبة.

Steady State Errors for Unit Step and Ramp Input



* أقصى حالة تكون ميل الخط المستقيم نفسه ميل خط الـ output عشان يكون فرق بينهم يساوي بالطبع يعني المفرق بينهم بسيط ويحصل الى output يقرب لحد ما يتطابق مع input.

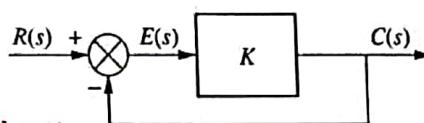


* لو الـ output مني بعدى الى input محتاج
الـ error كلها كم يزيد .

الـ Control Systems → Proportional (gain/Amplification) : (يعنى يفواخطة)
ولو بنسبة بسيطة .
→ Integral : وجودها بالـ System بتخليلي الأخطاء zero

Sources of Errors

- Many steady-state errors in control systems arise from nonlinear sources, such as backlash in gears or a motor that will not move unless the input voltage exceeds a threshold.
- The steady-state errors we study here are errors that arise from the configuration of the system itself and the type of applied input. Consider a system with only a gain, and a system with an Integrator (Laplace of Integration has division by s)



Sources of errors :-

أول سبب انه الطبيعة non-linear فكل شغنا افترضات عالـ linear system ، هيا لا errors هنا حطوا انه كثير مختلفة .

ثاني سبب هو التحريم نفسه ومرات بنكون مجبوبين بـ (inherent errors)

* عشان يكون الى error قليلة جداً
بالـ System لازم تكون k (كبيرة جدًا) ومستعمل تكون ∞
الـ amikun في error لـ (يدخل
هـ اي الشـ لـ block Integrator
يـ نـ عـ دـ هـ ايـ الاـ خطـ وـ هـ ايـ الا~ block هـ ايـ
ناسـ الا~ PID controller
 $\frac{1}{s}$ = Integrator block
(unit step)

Ex: Laplace $\int f(t) = \frac{F(s)}{s}$ * SF(s)
عمليات اشتغال بالـ System [معمليات تكميل بالـ]

The Final Value Theorem

- From the definition of the steady-state error, steady-state error is the difference between the input and the output for a prescribed test input as $t \rightarrow \infty$.
- We are interested in $e(\infty) = \lim_{t \rightarrow \infty} e(t)$ مفهوم تغير بالذاتية (in Time Domain)
- But since we are working in the s-domain, the final-value theorem is helpful

$$e(\infty) = \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} sE(s)$$

تحسّن على ها هي هنا.

* كيفية حساب الخطأ *

* يوفّرنا الا-Steady-state error بآليّة المحصلة النهايّة.
للّتّر تزوج لا ∞ يعني أوصل لأنّه الا error يكون 0.

Example for Forward Path System

* نظام forward path system يكون مثاليًّا Stable لو ما كان كل أفراده ذات ايجابي خاصّة.

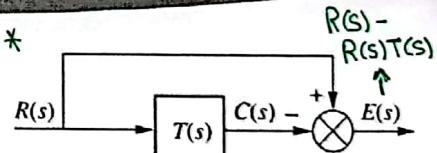
- Find the steady-state error for the system shown if $T(s) = 5/(s^2 + 7s + 10)$ and $R(s)$ is a unit step response $1/s$

$$E(s) = \frac{s^2 + 7s + 5}{s(s^2 + 7s + 10)}$$

- To find the error $e(\infty)$, we apply $\lim_{s \rightarrow 0} sE(s)$

- We solve the limit in MATLAB:

$$\begin{aligned} * R(s) * (1 - T(s)) &\rightarrow R(s) * \left(1 - \frac{5}{s^2 + 7s + 10}\right) \\ \rightarrow R(s) * \left(\frac{s^2 + 7s + 5}{s^2 + 7s + 10}\right) &\rightarrow \frac{1}{s} * \left(\frac{s^2 + 7s + 5}{s^2 + 7s + 10}\right) \\ &= \frac{s^2 + 7s + 5}{s(s^2 + 7s + 10)} \end{aligned}$$



*
 $\text{syms } s$
 $E = (s^2 + 7*s + 5)/(s*(s^2 + 7*s + 10))$

$$E = \frac{s^2 + 7s + 5}{s(s^2 + 7s + 10)}$$

* $\text{limit}(s*E, s, 0)$

$\text{ans} = \frac{1}{2}$ قيمة
عالية
النهايّة
تحقّق من
5% لـ
Value

(البيدي ياهـا نسبة 50%)

$$\lim_{s \rightarrow 0} s \times \left(\frac{s^2 + 7s + 5}{s(s^2 + 7s + 10)} \right)$$

استخرج ناتجها باستخدام limit Command في Matlab لـ $\lim_{s \rightarrow 0}$ بشرط أن $G(s) = \frac{(s+z_1)(s+z_2)}{s(s+p_1)(s+p_2)}$ ←
لـ و في طريقة ثانية أصول حذفها لقلم.

9

Errors in Closed-Loop Unity Feedback System I

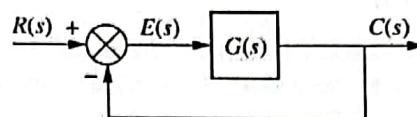
Closed-loop system ← جنسلوب ← with unity feedback (I)

$$E(s) = R(s) - C(s)$$

$$C(s) = E(s)G(s)$$

$$E(s) = \frac{R(s)}{1 + G(s)}$$

$$e(\infty) = \lim_{s \rightarrow 0} \frac{sR(s)}{1 + G(s)}$$



بۇنا تۈنۈن دىغىر وەزئاھو لا idle case بىس
بىلەققۇمۇسىدۇر وەزئاھو لا error

If the limit equals zero, this means there is no steady state error, otherwise we could have:

- a finite (small) error and we need to check if it is within the required error band
- or the error grows to infinity!

Finite error
Infinite error
(بىندىي يېرىن)

كيف يذهب Matlab الى limit و اذا قيمت النهاية من الممكن = قيمة النهاية من الميسار
فيطلع الجواب للنهاية طبعيًّا ما لو مش متساوي بيعطى NAN .

Ramp / Step / Parabola ← حشوف انه كل ما نخسي System's Integrator حين تدخل من ∞ الثالثة و الثانية و الأولى ادى الى Finite error .

10

Errors in Closed-Loop Unity Feedback System II

$R(s)$ could be a unit step, ramp, or parabola, therefore:

Step Input with $R(s) = 1/s$, we find

$$e(\infty) = e_{\text{step}}(\infty) = \lim_{s \rightarrow 0} \frac{s(1/s)}{1 + G(s)} = \frac{1}{1 + \lim_{s \rightarrow 0} G(s)}$$

Ramp Input with $R(s) = 1/s^2$, we obtain

$$e(\infty) = e_{\text{ramp}}(\infty) = \lim_{s \rightarrow 0} \frac{s(1/s^2)}{1 + G(s)} = \lim_{s \rightarrow 0} \frac{1}{s + sG(s)} = \frac{1}{\lim_{s \rightarrow 0} sG(s)}$$

Parabolic Input with $R(s) = 1/s^3$, we obtain

$$e(\infty) = e_{\text{parabola}}(\infty) = \lim_{s \rightarrow 0} \frac{s(1/s^3)}{1 + G(s)} = \lim_{s \rightarrow 0} \frac{1}{s^2 + s^2 G(s)} = \frac{1}{\lim_{s \rightarrow 0} s^2 G(s)}$$

Static Error Constants	$G(s) = \frac{(s+z_1)(s+z_2)\dots}{s^n(s+p_1)(s+p_2)\dots}$
K_p	Position
$\lim_{s \rightarrow 0} G(s) = \infty$	$G(s)$ denominator must have at least an s^1 at the denominator for the limit to be infinity (one integration)
K_v	Velocity
$\lim_{s \rightarrow 0} sG(s) = \infty$	$G(s)$ denominator must have an s^2 at the denominator for the limit to be infinity (two integrations)
K_a	Acceleration
$\lim_{s \rightarrow 0} s^2 G(s) = \infty$	$G(s)$ denominator must have an s^3 at the denominator for the limit to be infinity (three integrations)

بعندي قيم K_p ، K_v ، K_a يكواه كىشان $\frac{1}{s}$ = 0 فېقل نسبە الخطا لـ 0 .

لەتكەن $G(s) = \infty$ لام يكىن بىقاوا 0 كىن $s=0$ ← $G(s) \leftarrow s=0$ ← $e=0$ ← 0 .

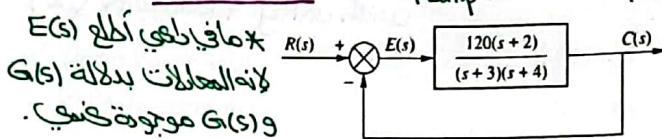
* وجىد s في المقام يېنى فيي Integration .

لَا زم يكون لا يشان أقر أحل.

11

Example: Steady-State Errors for Systems with No Integrations

Find the steady-state errors for inputs of $5u(t)$, $5tu(t)$, and $5t^2u(t)$ to the system shown assuming the system is stable.



Taking a look at $E(s)$, we notice that the denominator has no standalone s term. We expect that we have finite error for the unit step but infinite for the ramp and parabola inputs.

$$R(s) = 5/s$$

$$R(s) = 5/s^2$$

$$R(s) = 10/s^3$$

* Integrations لامشي اندليع اندليع *

يعني هنا اذا System يدخل ادخال unit step يخلع ادخال وفعلاً ذلك يكون فيه نسبة تطا.

```
syms s
G = (120*(s+2))/((s+3)*(s+4))
```

6 =

$$\frac{120s + 240}{(s+3)(s+4)}$$

$$E_{step} = 5 / (1 + \text{limit}(G, s, 0))$$

$$E_{step} = \frac{5}{21}$$

$$E_{ramp} = 5 / \text{limit}(s*G, s, 0)$$

$$E_{ramp} = \infty$$

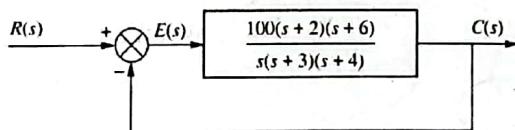
$$E_{parabola} = 10 / \text{limit}(s^2*G, s, 0)$$

$$E_{parabola} = \infty$$

Laplace لامشي
 $\int s^2$

Example: Steady-State Errors for Systems with One Integration

Find the steady-state errors for inputs of $5u(t)$, $5tu(t)$, and $5t^2u(t)$ to the system shown assuming the system is stable.



Taking a look at $E(s)$, we notice that the denominator has one term of s with a degree of s^1 . That is one integrator, we expect that we have zero for the unit step but infinite for the ramp and parabola inputs.

$$R(s) = 5/s$$

$$R(s) = 5/s^2$$

$$R(s) = 10/s^3$$

error لامشي اندليع اندليع Integrator لامشي اندليع اندليع
Parabola و بالا Ramp-Finitely unit step = 0 بار
Infinite بار Finite بار

```
syms s
G = (100*(s+2)*(s+6))/((s+3)*(s+4))
```

6 =

$$\frac{(100s + 200)(s+6)}{s(s+3)(s+4)}$$

$$E_{step} = 5 / (1 + \text{limit}(G, s, 0, 'right'))$$

$$E_{step} = 0 \quad \text{NAN لامشي اندليع}$$

$$E_{ramp} = 5 / \text{limit}(s*G, s, 0)$$

$$E_{ramp} = \frac{1}{20}$$

$$E_{parabola} = 10 / \text{limit}(s^2*G, s, 0)$$

$$E_{parabola} = \infty$$

* لازم نفحص الـ System اذا كان ملائماً لـ Stable ، اذا لا يتحقق ذلك ولا نوقف.

Example: Unstable Systems

الـ Stability كـ System ملائم لـ System.

A unity feedback system has the following forward transfer function:

$$G(s) = \frac{10(s+20)(s+30)}{s^2(s+25)(s+35)(s+50)}$$

Find the steady-state error for the following inputs: $15u(t)$, $15tu(t)$, and $15t^2u(t)$.

Notice, that unlike the previous examples, we did not assume or tell you that the system is stable, so we need to check if the system is stable, otherwise, nothing matters.

This is a feedback system with a transfer function Sys given as:

$$s = tf('s'); \\ G = (10 * (s + 20) * (s + 30)) / (s^2 * (s + 25) * (s + 35) * (s + 50)); \\ Sys = feedback(G, 1) \rightarrow \text{System}$$

استخدموها لـ $\text{tf}('s')$ من هنا

Continuous-time transfer function.

roots ([1, 110, 3875, 43760, 500, 6000])

ans = 5x1 complex
 -50.0064 + 0.0000i
 -34.9959 + 0.0000i
 -24.9984 + 0.0000i
 0.0004 + 0.3703i
 0.0004 - 0.3703i

\Rightarrow System unstable

فما يفعل

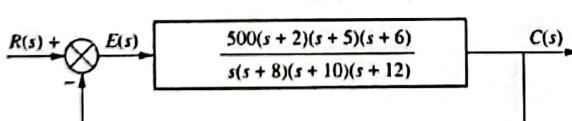
استخدام طريقة الـ Symbolic

Finding Static Error Constants using MATLAB

• Poly Command (يعرف المعدلة باستخدام roots) بـ

Evaluate the static error constants and find the expected error for the standard step, ramp, and parabolic inputs.

First, we need to know if the system is stable or not



* أطْلَقْتُهُ مُؤْسِسَ الـ System اذا كان ملائماً لـ Stable ، اذا لا يتحقق ذلك، انه Stable فيليش حل.

```
numg=500*poly([-2 -5 -6]); % Define numerator of G(s).
deng=poly([0 -8 -10 -12]); % Define denominator of G(s).
G=tf(numg,deng); % Form G(s).
'Check Stability' % Display label.
```

ans = 'Check Stability'

T=feedback(G,1) % Form T(s).

T =

$$\frac{500 s^3 + 6500 s^2 + 26000 s + 30000}{s^4 + 530 s^3 + 6796 s^2 + 26968 s + 30000}$$

Continuous-time transfer function.

poles=pole(T) % Display closed-loop poles.

poles = 4x1
 -516.9544
 -5.7623
 -5.4278
 -1.8554

Pole Command
 لـ pole يعطيه الـ roots دعوي
 الـ (real part) لها

roots ([1 530 6796 26968 30000])

ans = 4x1
 -516.9544
 -5.7623
 -5.4278
 -1.8554

Finding Steady-State Error Constants using MATLAB

Static Error Constants

$$K_p = \lim_{s \rightarrow 0} G(s)$$

$$K_v = \lim_{s \rightarrow 0} sG(s)$$

$$K_a = \lim_{s \rightarrow 0} s^2 G(s)$$

بيان قيمة K_p دفعي

لأنه في واحد Integration

$G(s)$ في s

وأحد Integration وجزء

Finite $\rightarrow \infty$ (infinite)

3 Integration لذات ∞ (غير متحدة) ←

تحسیں

```
%' Step Input'
Kp=dcgain(G) % Evaluate Kp=numg/deng for s=0.
```

Kp = Inf

```
e_step=1/(1+Kp) % Evaluate error for step input.
```

e_step = 0

```
%' Ramp Input'
numsg=conv([1 0],numg); % Define numerator of sG(s).
densg=poly([0 -8 -10 -12]); % Define denominator of sG(s).
sG=tf(numsg,densg); % Create sG(s).
Kv=dcgain(sG) % Evaluate Kv=sG(s) for s=0.
```

Kv = 31.2500

```
e_ramp=1/Kv % Evaluate steady-state error for ramp input.
```

e_ramp = 0.0320

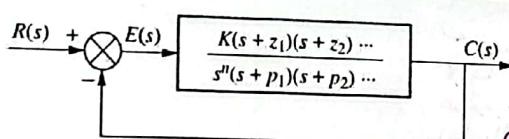
```
%' Parabolic Input'
nums2g=conv([1 0 0],numg); % Define numerator of s^2G(s).
dens2g=poly([0 -8 -10 -12]); % Define denominator of s^2G(s).
s2G=tf(nums2g,dens2g); % Create s^2G(s).
Ka=dcgain(s2G) % Evaluate Ka=s^2G(s) for s=0.
```

Ka = 0

e_parabola = 1 / Ka

e_parabola = Inf

System Types Summary



The number of integrations determine the number and value of errors we have in the system

$G(s)$ (مقدار) درج Integrations

2 Integrations
 $(s^2 \text{ في } G(s) \text{ مقدار})$

$G(s)$ Systems with no Integrators ← Type 0

(جزء s في $G(s)$)

Type 1

Input	Steady-state error formula	Static error constant	Error	Static error constant	Error	Static error constant	Error
Step, $u(t)$	$\frac{1}{1+K_p}$	$K_p = \text{Constant}$	$\frac{1}{1+K_p}$	$K_p = \infty$	0	$K_p = \infty$	0
Ramp, $tu(t)$	$\frac{1}{K_v}$	$K_v = 0$	∞	$K_v = \text{Constant}$	$\frac{1}{K_v}$	$K_v = \infty$	0
Parabola, $\frac{1}{2}t^2u(t)$	$\frac{1}{K_a}$	$K_a = 0$	∞	$K_a = 0$	∞	$K_a = \text{Constant}$	$\frac{1}{K_a}$

* كل مانزيد بعد ال Integrators error

* لمافي ولا Integrator أكيدتنا نسبة خطأ.

Interpretation Example 1

* بيجي زديه بالامتحان دوالن *

Interpreting the Steady-State Error Specification

Step input *Type 0* *Constant* *بما أن*

PROBLEM: What information is contained in the specification $K_p = 1000$?

SOLUTION: The system is stable. The system is Type 0, since only a Type 0 system has a finite K_p . Type 1 and Type 2 systems have $K_p = \infty$. The input test signal is a step, since K_p is specified. Finally, the error per unit step is

$$e(\infty) = \frac{1}{1 + K_p} = \frac{1}{1 + 1000} = \frac{1}{1001}$$

* أول خطوة لآن نقرحها
إنه معلم طبع عنك قيم
لل error ولا شغل تبعك
System II also
لما كان . Stable
ما يهدى التسلق Stable

Interpretation Example II

* system is stable . *

► What conclusions do we draw if a system has $K_v = 1000$

Constant as! بما
ramp ← Type 1 ←
input

1. The system is stable.
2. The system is of Type 1, since only Type 1 systems have K_v 's that are finite constants.
Recall that $K_v = 0$ for Type 0 systems, whereas $K_v = \infty$ for Type 2 systems.
3. A ramp input is the test signal. Since K_v is specified as a finite constant, and the steady-state error for a ramp input is inversely proportional to K_v , we know the test input is a ramp.
4. The steady-state error between the input ramp and the output ramp is $1/K_v$ per unit of input slope.

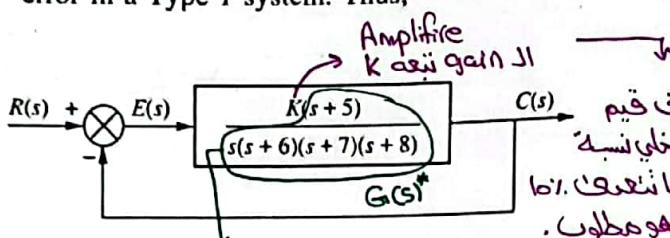
Designing for Gain, Error Specification and Stability

PROBLEM: Given the control system in Figure find the value of K so that there is 10% error in the steady state.

نجد K بحيث يكون الخطأ ثابت 10%

SOLUTION: Since the system is Type 1, the error stated in the problem must apply to a ramp input; only a ramp yields a finite error in a Type 1 system. Thus,

الإجابة هي K_v



لدي أعرف قيمة K التي يتخلص فيها الخطأ من 10% في ما هو مطلوب.

```
num=[1 5]; % Define numerator of G(s)/K.
den=pol([0 -6 -7 -8]); % Define denominator.
GdK=tf(num,den); % Create G(s)/K.
num=conv([1 0],num); % Define numerator of sG(s).
GdK=tf(num,den); % Create sG(s)/K.
e=0.1
```

e = 0.1000

$K_v = 1 / e$

$K_v = 10$

$K = K_v / \text{dcgain}(GdK)$

$K = 672$

```
% Check Stability ✓
T=feedback(K*GdK,1); % Form T(s).
poles=pole(T) % Display closed-loop poles.
```

```
poles = 4x1 complex
0.0000 + 0.0000i
-7.9956 + 25.5618i
-7.9956 - 25.5618i
-5.0088 + 0.0000i
```

في S بالمقام خانة

$K_v \leftarrow \text{Type 1}$

$$* e = 10\% \rightarrow K_v = \frac{1}{e} = 10$$

* المشكلة هي هل أنت متأكد من $G(s) = k \left(\frac{(s+5)}{s(s+4)(s+7)(s+8)} \right)$

مجهولة

فمن السهل هيكل $\frac{G(s)}{K}$ = System equation

لقد نقدر أكتب System بدون مجهول. ← لفهم أنت من

بعد كل

هذا إنشاء

System

من

Stable

Steady-State Errors with Disturbances I

+ أهم مصطلحات الخطا في التحكم نفسه والـ Systems كلياتها non-linear Systems

الأخطاء \rightarrow Feedback control systems are used to compensate for disturbances or unwanted inputs that enter a system. The advantage of using feedback is that regardless of these disturbances, the system can be designed to follow the input with small or zero error.

بتكون من \rightarrow The figure shows a feedback control system with a disturbance, $D(s)$, injected between the controller and the plant. We now derive the expression for steady-state error with the disturbance included. By setting each input $D(s)$ and $R(s)$ to zero one at a time, the transform of the output is given by:

(Disturbances)

$$\downarrow C(s) = E(s)G_1(s)G_2(s) + D(s)G_2(s)$$

تحتى

نأخذ دين

الاحتى

ونشوف

تأثير علاج

بعد تعويض

المعادلات يتحقق

طريق معالجة

$E(s)$

$$C(s) = R(s) - E(s)$$

$$E(s) = \frac{1}{1 + G_1(s)G_2(s)} R(s) - \frac{G_2(s)}{1 + G_1(s)G_2(s)} D(s)$$

الشق الأول هو زéro

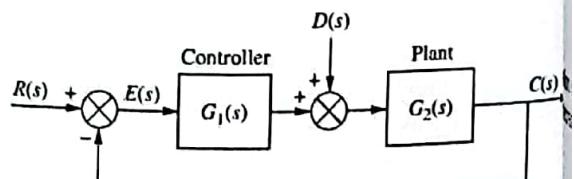
كل اثنى حملناه

الخطأ الناتج

من البداية

وزي معاللة

الـ feedback



* وجود الـ Disturbance ما يأثر على الأشياء

الـ \rightarrow التي كانت متوقعة (الأنظمة الناتجة من

الـ Input والـ non-linearity والـ

. ($K_v < K_p < K_a$) والـ Configuration

Steady-State Errors with Disturbances II

- To find the steady-state value of the error, we apply the final value theorem

$$e(\infty) = \lim_{s \rightarrow 0} sE(s) = \lim_{s \rightarrow 0} \frac{s}{1 + G_1(s)G_2(s)} R(s) - \lim_{s \rightarrow 0} \frac{sG_2(s)}{1 + G_1(s)G_2(s)} D(s)$$

$$= e_R(\infty) + e_D(\infty)$$

$$e_R(\infty) = \lim_{s \rightarrow 0} \frac{s}{1 + G_1(s)G_2(s)} R(s)$$

$$e_D(\infty) = - \lim_{s \rightarrow 0} \frac{sG_2(s)}{1 + G_1(s)G_2(s)} D(s)$$

- Notice that the first term relating the error to the input is the same as the one derived and analysed before (see slide 8), where G_1 and G_2 are in series and can be replaced with $G(s)$, and the input can be a unit step, a ramp, or a parabola

* للتبسيط حفظنا انه $D(s)$ كباقي $R(s)$ Step function .
 $(D(s) = \frac{1}{s})$

عشران s و $D(s)$ يلغوا بعض.
 بعددين بتنقسم البسط والمقام على (s) G_2 بطلع معادلة Slide 22

Steady-State Errors with Disturbances III

- Now, we only focus on the error due to the disturbance $D(s)$. To simplify the analysis, we only consider a unit step error, that is $D(s) = 1/s$. Substituting this value of $D(s)$ into the previous equation will yield:

$$e_D(\infty) = - \frac{1}{\lim_{s \rightarrow 0} \frac{1}{G_2(s)} + \lim_{s \rightarrow 0} G_1(s)}$$

نسبة الخطأ الناتجة عن الـ \leftarrow
 & Disturbance

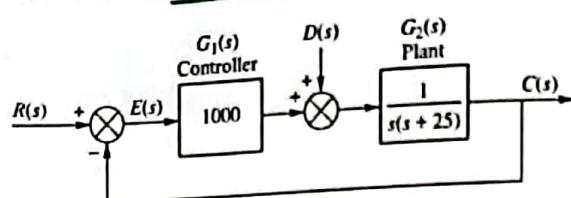
- This equation shows that the steady-state error produced by a step disturbance can be reduced by increasing the dc gain of $G_1(s)$ or decreasing the dc gain of $G_2(s)$.

(decrease) \leftarrow قيمة الـ

* زيادة $(s) G_1$ يقلل من نسبة الخطأ الخارجي.
 * زيادة $(s) G_2$ يزيد من نسبة الخطأ الخارجي.
 ← خارجية زراعة G_1 ونقل G_2
 لأن زراعة G_1 ونقل G_2
 لتقل نسبة الخطأ الخارجي.

Steady-State Error Due to Step Disturbance Example

- Find the steady-state error component due to a step disturbance for the system of the above figure



$$e_D(\infty) = -\frac{1}{\lim_{s \rightarrow 0} \frac{1}{G_2(s)} + \lim_{s \rightarrow 0} G_1(s)} = -\frac{1}{0 + 1000} = -\frac{1}{1000}$$

Steady-State Error Due to Step Disturbance Example

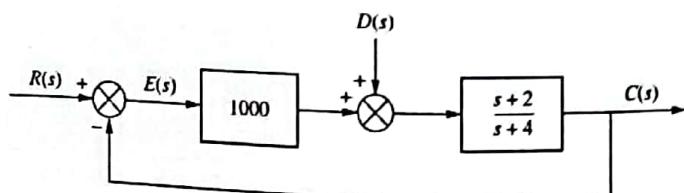
« حل بالماندوب »

- Evaluate the steady-state error component due to a step disturbance for the system shown in the figure using MATLAB. Assume the system is stable.

```

syms s
G1 = 1000;
G2 = (s+2)/(s+4);
1G1 = limit (G1, s, 0)
1G1 = 1000
1G2 = limit (1/G2, s, 0)
1G2 = 2
e_D = -1 / (1G1 + 1G2)
e_D =
-1/1002

```



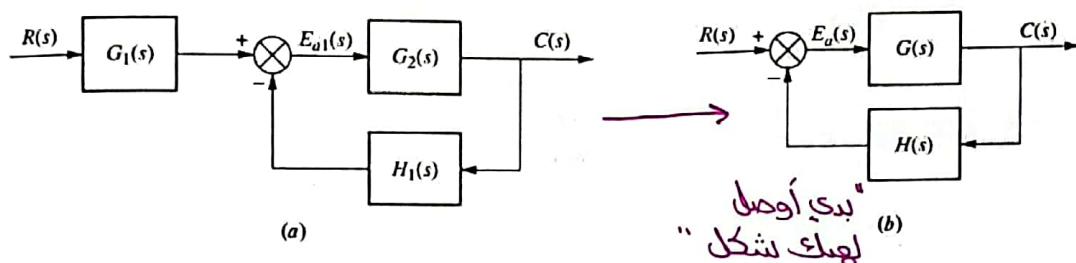
* إشارة الموجب بالنتائج بدل على أنه
لتفصل الـ signal كانت 5 ، لوالـ error
معجب بكل الخطأ = 4 لو كانت سالبة بكل
الخطأ أعلى من هيك.

* لو بلك تطهير الـ Total error
تطهير الـ error يعني أول وينتهي
الـ Disturbance error

• جملة الـ * يكتب في unity gain feedback

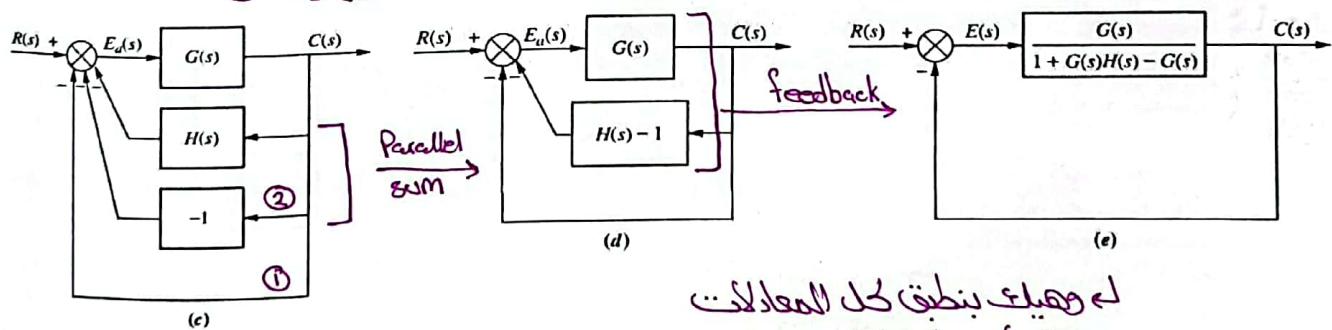
Steady-State Error for Nonunity Feedback Systems

- ▶ Control systems often do not have unity feedback because of the compensation used to improve performance or because of the physical model for the system. The feedback path can be a pure gain other than unity or have some dynamic representation.
- ▶ A general feedback system, showing the input transducer, $G_1(s)$, controller and plant, $G_2(s)$, and feedback, $H_1(s)$, is shown in the Figure (a) below, now we go through the steps to make it look like unity feedback systems



Steady-State Error for Nonunity Feedback Systems II

تحويل الـ System إلى Unity gain System



له وظيفة بنيت كل المعادلات
التي أخذناها على عادي.

Steady-State Error for Nonunity Feedback Systems Example

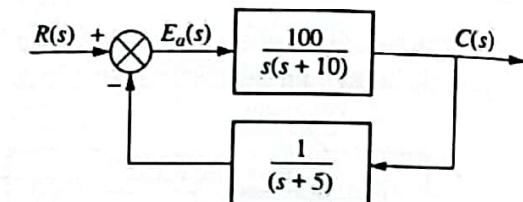
- For the system shown in the figure, find the system type, the appropriate error constant associated with the system type, and the steady-state error for a unit step input. Assume input and output units are the same.

$$G(s) = \frac{100}{s(s+10)}$$

$$H(s) = \frac{1}{(s+5)}$$

$$G_e(s) = \frac{G(s)}{1 + G(s)H(s) - G(s)} = \frac{100(s+5)}{s^3 + 15s^2 - 50s - 400}$$

$$K_p = \lim_{s \rightarrow 0} G_e(s) = \frac{100 \times 5}{-400} = -\frac{5}{4}$$



Type 0

$$e(\infty) = \frac{1}{1 + K_p} = \frac{1}{1 - (5/4)} = -4$$

ي يعني لو الدخل ثابت
كل منفوس تكون 5 هو

أعطاني 20 فأعطيه ناتج أكبر من
اللي بدبي عليه.

Steady-State Error for Nonunity Feedback Systems Example

- For the system shown in the figure, find the system type, the appropriate error constant associated with the system type, and the steady-state error for a unit step input. Assume input and output units are the same, and the system is stable. Use MATLAB

[] في طرق
كتير
للكتابة

```
G=zpk([], [0 -10], 100);
H=zpk([], -5, 1);
```

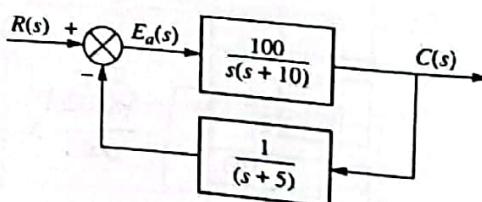
```
Ge=feedback(G, (H-1));
Ge = tf (Ge);
T=feedback (Ge, 1)
```

$$\begin{aligned} T = \\ \frac{100 s + 500}{s^3 + 15 s^2 + 50 s + 100} \end{aligned}$$

Continuous-time transfer function.

$$\begin{aligned} \text{pole}(T) \\ \text{ans} = 3 \times 1 \text{ complex} \\ -11.3780 + 0.0000i \\ -1.8110 + 2.3472i \\ -1.8110 - 2.3472i \end{aligned}$$

```
kp = dcgain (Ge)
kp = -1.2500
e_step = 1 / (1 + kp)
e_step = -4.0000
```



للتأكد
Stable.

Steady-State Error for Nonunity Feedback Systems Example 2

- Find the steady-state error for a unit step input given the nonunity feedback system. Repeat for a unit ramp input. Assume input and output units are the same.

```

G=zpk([],[-4],100);
H=zpk([],-1,1);

Ge=feedback(G,H-1);
Ge = tf (Ge);
T=feedback (Ge,1)

T =
    100 s + 100
    -----
    s^2 + 5 s + 104

Continuous-time transfer function.

pole(T)
ans = 2x1 complex
-2.5000 + 9.8869i
-2.5000 - 9.8869i

```

```

kp = dcgain (Ge)
kp = 25.0000

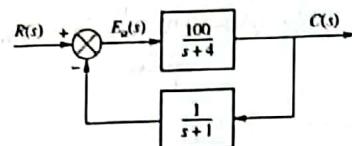
e_step = 1 / (1 + kp)
e_step = 0.0385

numf = [100, 100];
denf = [1, 5, 104];
numf = conv(numf, [1, 0]);
Gv = tf (numf, denf);
kv = dcgain(Gv)

kv = 0

e_ramp = 1 / kv
e_ramp = Inf

```



نعرف إن هنا خطأ بسبب R وأخطاء بسبب D (disturbances)

Steady-State Error for Nonunity Feedback Systems with Disturbances

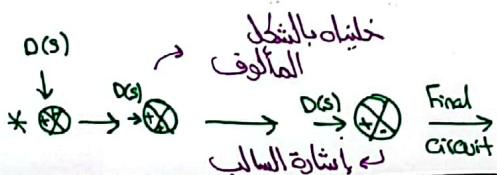
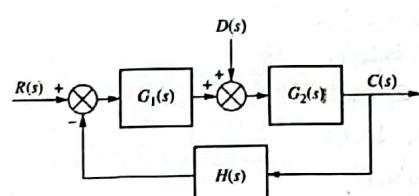
- To continue our discussion of steady-state error for systems with nonunity feedback, let us look at the general system shown in the figure, which has both a disturbance and nonunity feedback. We will derive a general equation for the steady-state error and then determine the parameters of the system in order to drive the error to zero for step inputs and step disturbances.

$$\begin{aligned}
 e(\infty) &= r(\infty) - c(\infty) \\
 e(\infty) &= \lim_{s \rightarrow 0} sE(s) = \lim_{s \rightarrow 0} s \left\{ \left[1 - \frac{G_1(s)G_2(s)}{1 + G_1(s)G_2(s)H(s)} \right] R(s) - \left[\frac{G_2(s)}{1 + G_1(s)G_2(s)H(s)} D(s) \right] \right\} \\
 e(\infty) &= \lim_{s \rightarrow 0} sE(s) = \left\{ \left[1 - \frac{\lim_{s \rightarrow 0} [G_1(s)G_2(s)]}{\lim_{s \rightarrow 0} [1 + G_1(s)G_2(s)H(s)]} \right] - \left[\frac{\lim_{s \rightarrow 0} G_2(s)}{\lim_{s \rightarrow 0} [1 + G_1(s)G_2(s)H(s)]} \right] \right\}
 \end{aligned}$$

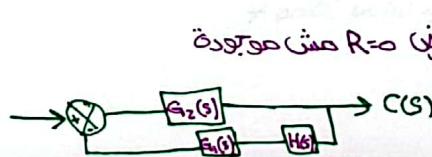
For zero error,

$$\frac{\lim_{s \rightarrow 0} [G_1(s)G_2(s)]}{\lim_{s \rightarrow 0} [1 + G_1(s)G_2(s)H(s)]} = 1 \quad \text{and} \quad \frac{\lim_{s \rightarrow 0} G_2(s)}{\lim_{s \rightarrow 0} [1 + G_1(s)G_2(s)H(s)]} = 0$$

* يطعن معادلة ٣ به يفرغ D = 0 مثل موجة كارينا.



* يطعن معادلة ٣ به يفرغ R = 0 مثل موجة كارينا.



← كارينا.

Steady-State Error for Nonunity Feedback Systems with Disturbances

- To continue our discussion of steady-state error for systems with nonunity feedback, let us look at the general system shown in the figure, which has both a disturbance and nonunity feedback. We will derive a general equation for the steady-state error and then determine the parameters of the system in order to drive the error to zero for step inputs and step disturbances.

بالنسبة لـ D \rightarrow
For zero error, $\lim_{s \rightarrow 0} [G_1(s)G_2(s)] = 1$ and $\lim_{s \rightarrow 0} \frac{G_2(s)}{1 + G_1(s)G_2(s)H(s)} = 0$

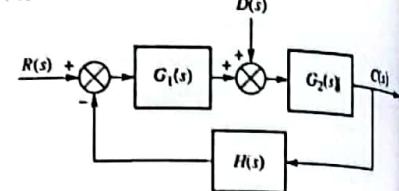
- 1 The two equations can always be satisfied if the system is stable.
- 2 $G_1(s)$ is a Type 1 system.
- 3 $G_2(s)$ is a Type 0 system, and
- 4 $H(s)$ is a Type 0 system with a dc gain of unity.

* تحقق كمان قيمة $R(s)$ و $D(s)$ العبرهم Step funct.

($\frac{1}{s}$) فبلغوا إلى الماء بالبسط.

فتحقق المعادلات الزعانية

للerror بالـ Nonunity feedback Systems with Disturbance.



كشان لا error
يكون ايساوي زي وتحقق
الحالتين لازم تكون
الشروط التالية ١، ٢، ٣، ٤
(محظاه بالامتحان)

Example

- Find the steady-state error for the following system assuming that it is stable, and that both the input and disturbance are unit step:

لـ $R(s)$ الحالات
بالنسبة

$$\text{sys}_R = \text{limit}(\text{sys}_R, s, 0)$$

$$G_1 = \frac{1}{s+5}$$

$$G_2 = \frac{100}{s+2}$$

$$H = 1$$

$$\text{sys}_R = (G_1 \cdot G_2) / (1 + G_1 \cdot G_2 \cdot H)$$

$$\text{sys}_R = \frac{100}{(s+2)(s+5) + 1}$$

$$e_R = \text{limit}(\text{sys}_R, s, 0)$$

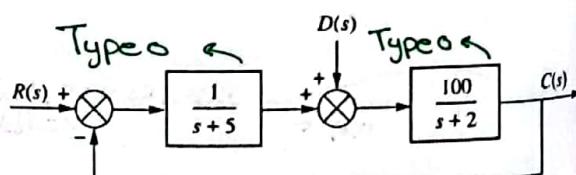
$$e_R = \frac{10}{11}$$

$$\text{sys}_D = \frac{100}{(s+2)(s+5) + 1}$$

$$e_D = \text{limit}(\text{sys}_D, s, 0)$$

$$e_D = \frac{50}{11}$$

لـ $R(s)$ فقط في الحالات.



unity gain

* ماتحقق كل الشروط ذيـفـر e مجموعـم

انـه المحصلة الزعانية الخطأ ما يكون Zero

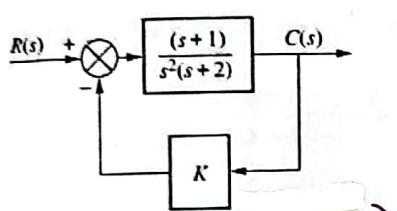
حيـكـونـا الـ قـيـمةـ.

* ممكنـ نـحلـهاـ جـالـيدـ هـولـتـ بـكـونـاـ أـسـرعـناـ.

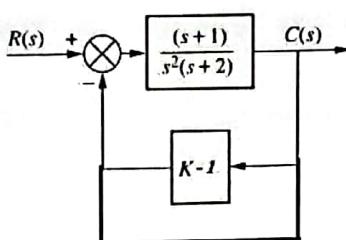
Additional Example I

Given the system shown below, find the system type and the value of K to yield 0.1% error in the steady state

↑ ramp or
Step or Parabola ?



ما يزيد نطح أو المدى
الذى نفس input
ملايينه المدى
يعطي ما يزيد ادخل Volt ٥٥٠ و ٥٥٠
تتحكم بال Sensor وال Motor
بال MV مشى بالـ V.

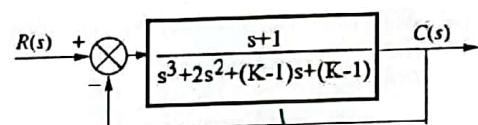


amplifier
ذى المدى
الذى نفس
input Sensor
فقدر نعم عالية الطرح

First step is to convert this system to a unity feedback system as follows:

كشان يهير

Second step is to evaluate the upper feedback loop which yields:



ما يزيد المدى
أو فجوة Integrator
step ← Type 0 → Kp

بـ Kp
الذى المناسب
نكرى ٠.١٪
الـ Steady-state error.

disturbance
بالـ System
زيـ ما أخذنا
بالـ disturbance
نستخرج

Additional Example I - Continued

Analysing G(s), it is clear that the system is Type 0, therefore, the static error constant mentioned in the question relates to K_p

$$e_{step}(\infty) = 0.001 = \frac{1}{1+K_p}$$

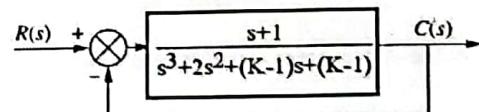
مشخصوها K الي
بالـ System انتوى

Therefore, K_p is 999 and we know that K_p is the limit of $G(s)$ as s goes to 0

It is easier to solve this mathematically, by substituting s by 0

$$K_p = \lim_{s \rightarrow 0} \left(\frac{s+1}{s^3 + 2s^2 + (K-1)s + (K-1)} \right) = \frac{1}{(K-1)} = 999, \text{ then } K = 1.001 \text{ and } G(s) \text{ is } \frac{s+1}{s^3 + 2s^2 + 0.001s + 0.001}$$

Check if the system is stable (remember to retrieve the transfer function for the whole system first):



System Stability
لا ينطع الـ
كامل .

$$\text{EIGEN} ([1 2 1.001 1.001])$$

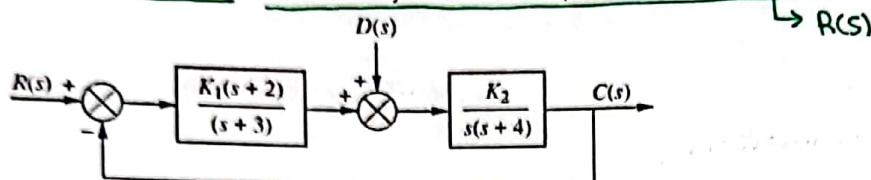
and

$$\begin{aligned} &-1.7546 + 0.00001 \\ &-0.1227 + 0.7453i \\ &-0.1227 - 0.7453i \end{aligned}$$

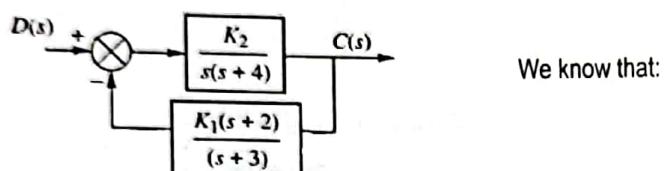
(بیجی زیبی های بالامتحان)

Additional Example II

- Design the values of K_1 and K_2 in the system of the adjacent figure such that the steady-state error component due to a unit step disturbance is 0.00001, and the steady-state error component due to a unit ramp input is 0.002.



- When analysing the steady-state error due to disturbance only, we assume that $R(s)$ is equal to zero, rearranging the shape:



We know that:

$$e_D(\infty) = -\lim_{s \rightarrow 0} \frac{sG_2(s)}{1 + G_1(s)G_2(s)} D(s)$$

"Disturbance due to $R(s)$ معرفی را"

Additional Example II - Continued

- Since $D(s)$ is assumed as a unit step disturbance, the equation becomes:

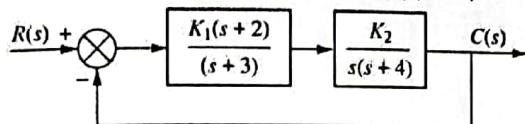
$$e_D(\infty) = -\lim_{s \rightarrow 0} \frac{G_2(s)}{1 + G_1(s)G_2(s)}$$

- Substituting: e_D , G_1 , G_2 and rearranging

$$0.00001 = \lim_{s \rightarrow 0} \left(\frac{\frac{K_2}{s(s+4)}}{1 + \frac{K_1 K_2 (s+2)}{s(s+3)(s+4)}} \right) \lim_{s \rightarrow 0} \frac{\frac{K_2(s+3)}{s(s+3)(s+4) + K_1 K_2 (s+2)}}{\frac{3}{2K_1}} = 0.00001, \text{ thus } K_1 = 150,000$$

Additional Example II - Continued

When analysing the steady-state error due to input only, we assume that $D(s)$ is equal to zero, and the system becomes:



Since the input is ramp, the error due to only the input is $e_R(\infty) = \frac{1}{K_v} = 0.002$ and we know that $K_v = \lim_{s \rightarrow 0}(sG(s)) =$

$$\lim_{s \rightarrow 0} \left(s \frac{K_1(s+2)}{s+3} \frac{K_2}{s(s+4)} \right) = \lim_{s \rightarrow 0} \left(\frac{K_1(0+2)}{0+3} \frac{K_2}{(0+4)} \right) = K_1 K_2 / 6$$

So, $6/(K_1 K_2) = .002$ and we already got $K_1 = 150,000$. therefore, $K_2 = 0.02$

أصلب أنواع الأسئلة → فيها مرجع إلى آخرناه (Time response)

Additional Example III

A second-order, unity feedback system is to follow a ramp input with the following specifications: the steady-state output position shall differ from the input position by 0.01 of the input velocity; the natural frequency of the closed-loop system shall be 10 rad/s.

Find the following:

$\downarrow K_u$

* The system type

Since we are talking about steady state-error from input velocity, and the error is finite (not infinite and not zero) → Type 1

* The exact expression for the forward-path transfer function

Integrator \rightarrow Type 1 \leftarrow Since we have a type 1 system, the simplest assumption about $G(s)$ is a pole with an integrator

For a Type 1 system, the error is $1/K_v$. The limit will give K/α so the error is $1/(K/\alpha) = 0.01$, or $K/\alpha = 100$.

The overall transfer function for the unity feedback system with system with $G(s) = \frac{K}{s(s+\alpha)}$ is $T(s) = \frac{G(s)}{1+G(s)} = \frac{K}{s^2 + \alpha s + K}$.

Since it is given that ω_n is 10, this means that $K = 100$, and therefore $\alpha = 1$

* The closed-loop system's damping ratio $2\zeta\omega_n = \alpha = 1$. Thus, $\zeta = \frac{1}{20}$

$$\frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

Additional Example III

- Now, as a new exercise, find K and a, that will yield K_v of 1000 and an overshoot of 20%

$$\%OS = e^{-(\zeta\pi/\sqrt{1-\zeta^2})} \times 100$$

For a 20% overshoot, this means $\zeta = 0.456$

$$② \underline{K_v = 1000 = \frac{K}{a}}$$

$$T(s) = \frac{K}{s^2 + as + K}$$

$$\omega_n = \sqrt{K}$$

$$① \underline{\frac{2\zeta\omega_n}{\omega_n} = a}$$

$$① \underline{a = 0.912\sqrt{K}}$$

① نحوه معادله *

② بعض بدل

a و K معاً

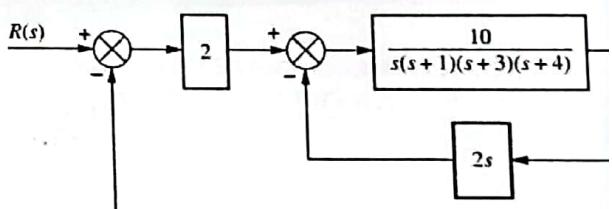
Substituting a in the K_v equation, this will yield $K = 831744$ and $a = 831.744$

Additional Example IV

Given the system in the adjacent figure, find the following:

- The closed-loop transfer function:

$$G_1(s) = \frac{\frac{10}{s(s+1)(s+3)(s+4)}}{1 + \frac{20}{s(s+1)(s+3)(s+4)}} = \frac{10}{s(s^3 + 8s^2 + 19s + 32)}$$



$$\text{Type 1} \leftarrow G_c(s) = \frac{20}{s(s^3 + 8s^2 + 19s + 32)}$$

$$T(s) = \frac{G_c(s)}{1+G_c(s)} = \frac{20}{s^4 + 8s^3 + 19s^2 + 32s + 20}$$

- The system type

Since there is one integrator in $G_c(s)$, this means a Type 1 system

Additional Example IV

Given the system in the adjacent figure, find the following:

- * The steady-state error for an input of $5u(t)$

$$E_{\text{Step}} = 0 \rightarrow \text{Type 1 ایکی}$$

- * The steady-state error for an input of $5tu(t)$

$$\text{From } G_e(s), K_v = \lim_{s \rightarrow 0} sG_e(s) = \frac{20}{32} = \frac{5}{8}. \text{ Therefore, } e_{ss} = \frac{5}{K_v} = 8.$$

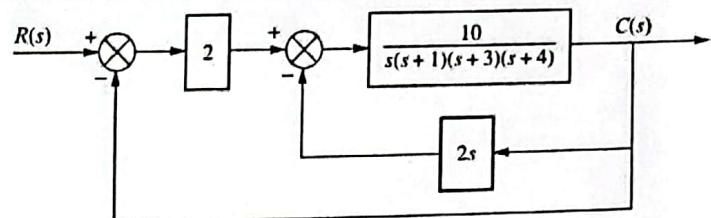
- * Discuss the validity of your answers to Parts c and d.

>> roots ([1 8 15 32 20])

ans =

```
-5.4755 + 0.0000i
-0.7622 + 1.7526i
-0.7622 - 1.7526i
-1.0000 + 0.0000i
```

System مذکور
نامناسب و لامناسب
کامل نیکوں.



References

- The material in these slides are based on:
Control Systems Engineering, Norman S. Nise, 7th Edition (2014), John Wiley And Sons
- Chapter 7 – Steady State Errors**
- Sections 7.1, 7.2, 7.3, 7.4, 7.5, and 7.6, (Students kindly note that some sections involve math derivations that we did not cover in class)

Quick Intro to PID Controllers

Proportional, Derivative and Integral Control

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DEPARTMENT OF COMPUTER ENGINEERING

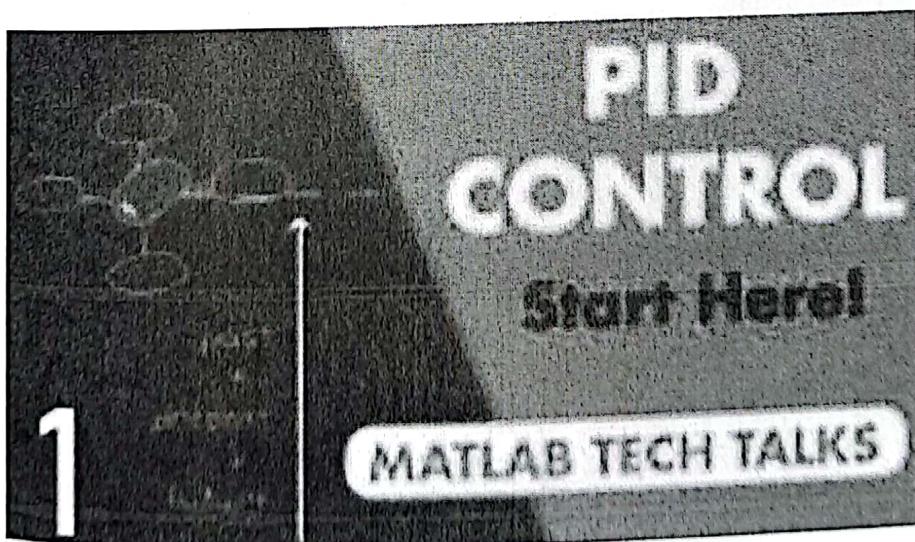
FALL 2022

PID Control – Intuitive Explanation

2

<https://www.youtube.com/watch?v=wkfEZmsQqiA&list=PLn8PRpmsu08pQBgjxYFXSsODEF3Jqmm-y>

Or in [presentation mode](#), click on the video below

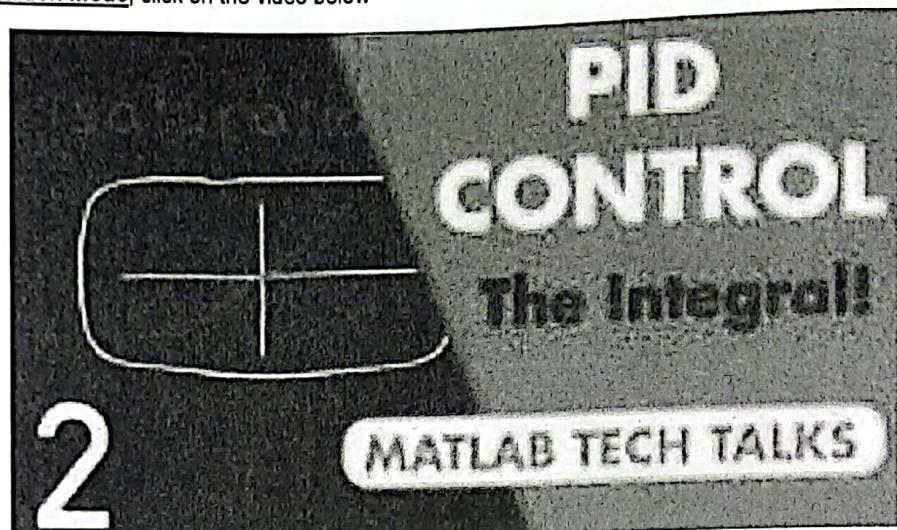


3

Integral Windup Problem- Intuitive Explanation

<https://www.youtube.com/watch?v=NVLXCwc8HzM&list=PLn8PRpmsu08pQBqjxYFXSsODEF3Jqmm-y&index=2>

Or in presentation mode, click on the video below

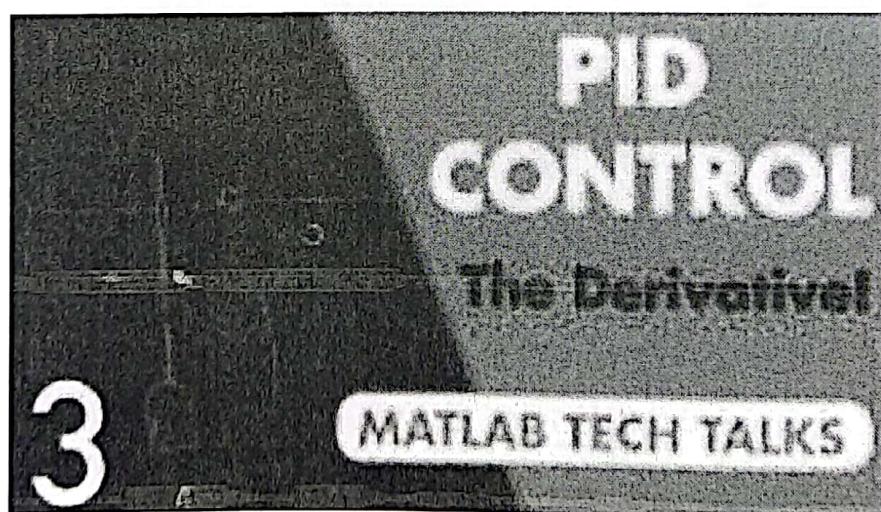


4

Noise Filtering in PID Control- Intuitive Explanation

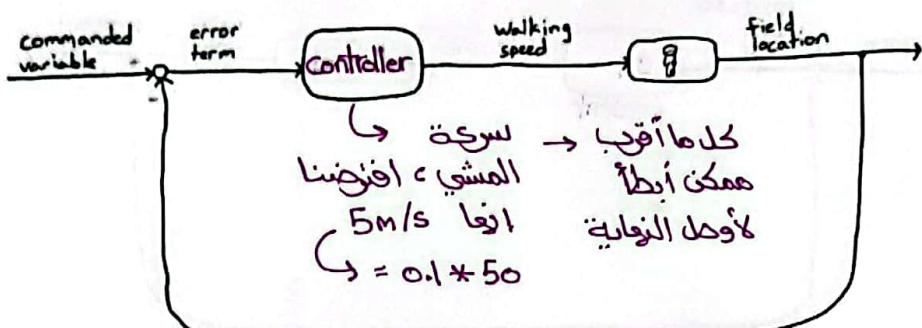
<https://www.youtube.com/watch?v=7dUVdrs1e18&list=PLn8PRpmsu08pQBqjxYFXSsODEF3Jqmm-y&index=3>

Or in presentation mode, click on the video below



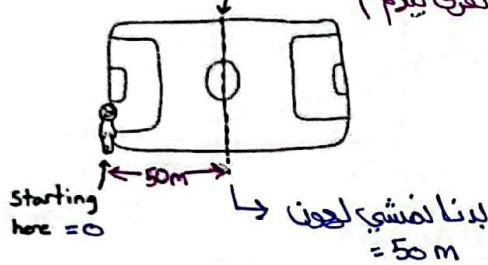
PID Controller Part 1

أنتو نوع الـ
Controller
(linear systems)



قيدي أنا بعيد عن المسافة النهائية \leftarrow feedback path

(دين بدري أروح ووين
أنا موجود الفرق بيني)

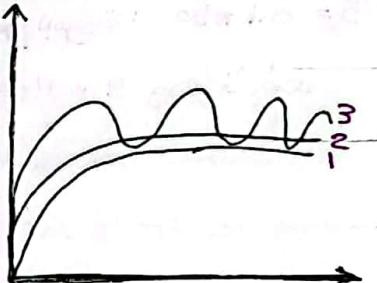


بتتحا كل ما يقرب للـ
Desired location

- * حفزني، إنه سرعة الإنسان
- * عند نقطة البداية مقدار الخطأ = 0
- * حصل أعشى لهد ما يغير الفرق = 0

بحب أشوف Transient response \rightarrow زي 2 (يكون الطوع
ولهم الشيء ما يكون Underdamped ولا يكون بطيء بسرعة قوية همك تأثير سلبي).

* وظيفتنا الآن حنخيف controller سوة كل SW أو HW نخيس لنفسن عن الشكل
تبع الـ System حق نحصل على الشكل اللي أنا بعي إيه. (حنخوه نظرياً)



* P : اللي كانينا شوفه طول الوقت (لا هو g / g)
Controllers هم مجموعة من الـ PID controller
(Amplifiers)

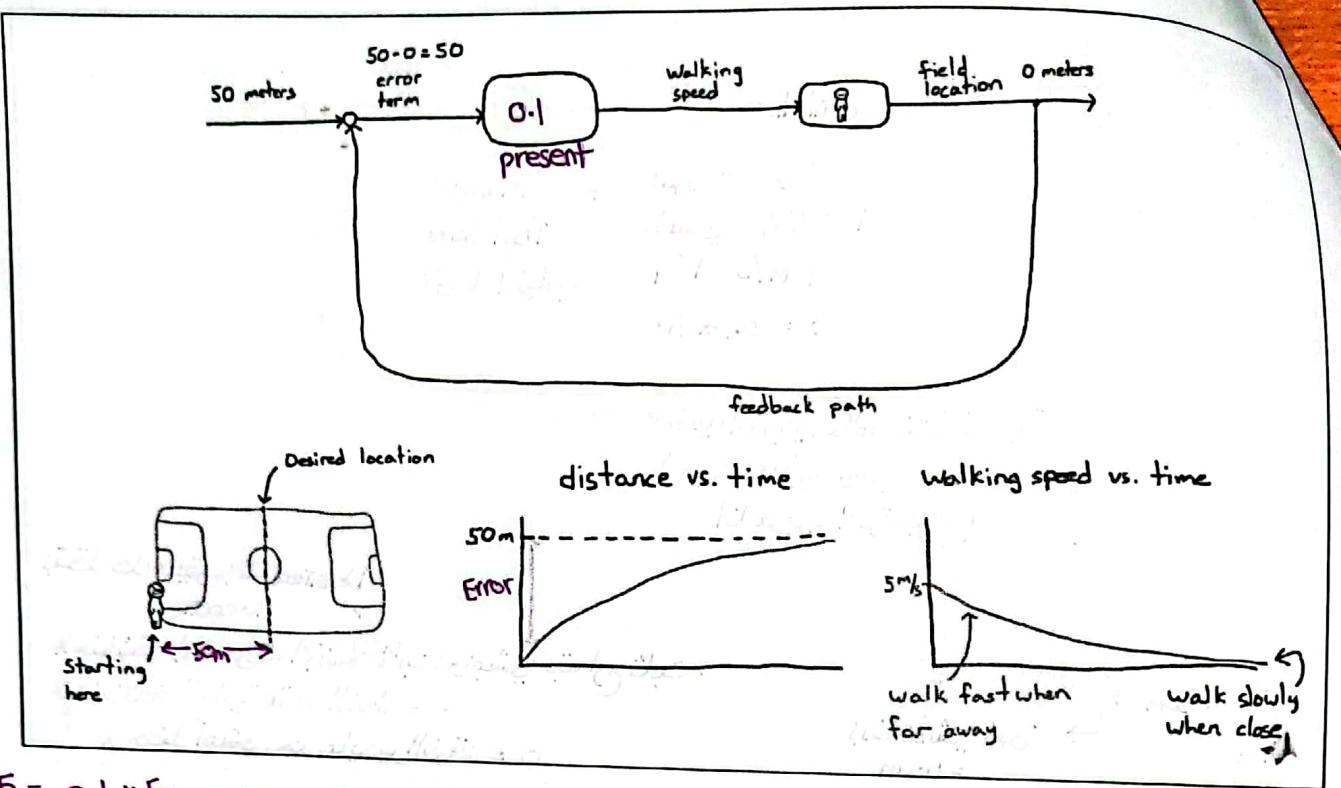
Integrator $\& I$ ←
Derivative $\& D$ ←

* دائماً يكون يعني يا إما PD / PI / P
PID /

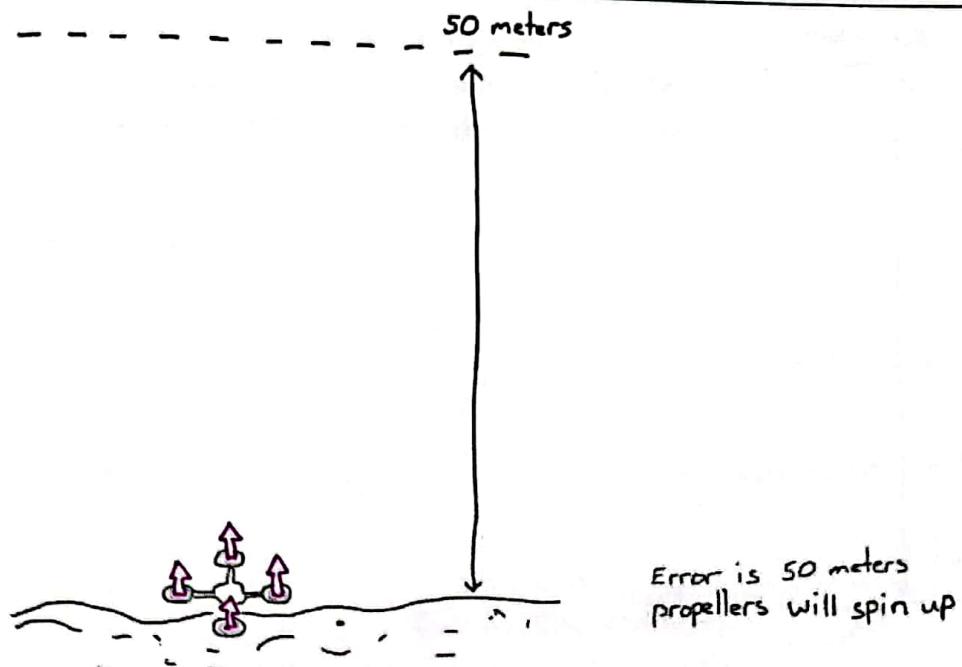
* لا P موجودها أساساً.

* أخذنا أنه دلائلاً علينا بالـ feedback sys فيه Plant فيه \rightarrow / Process / actuator (---).

* بواي المعرفة بنشوف كيف الـ P controllers يتبع الحالات بسا بعدين همك تفشد فشة نزيع الحالات
ثانية لو كانت الحالات.



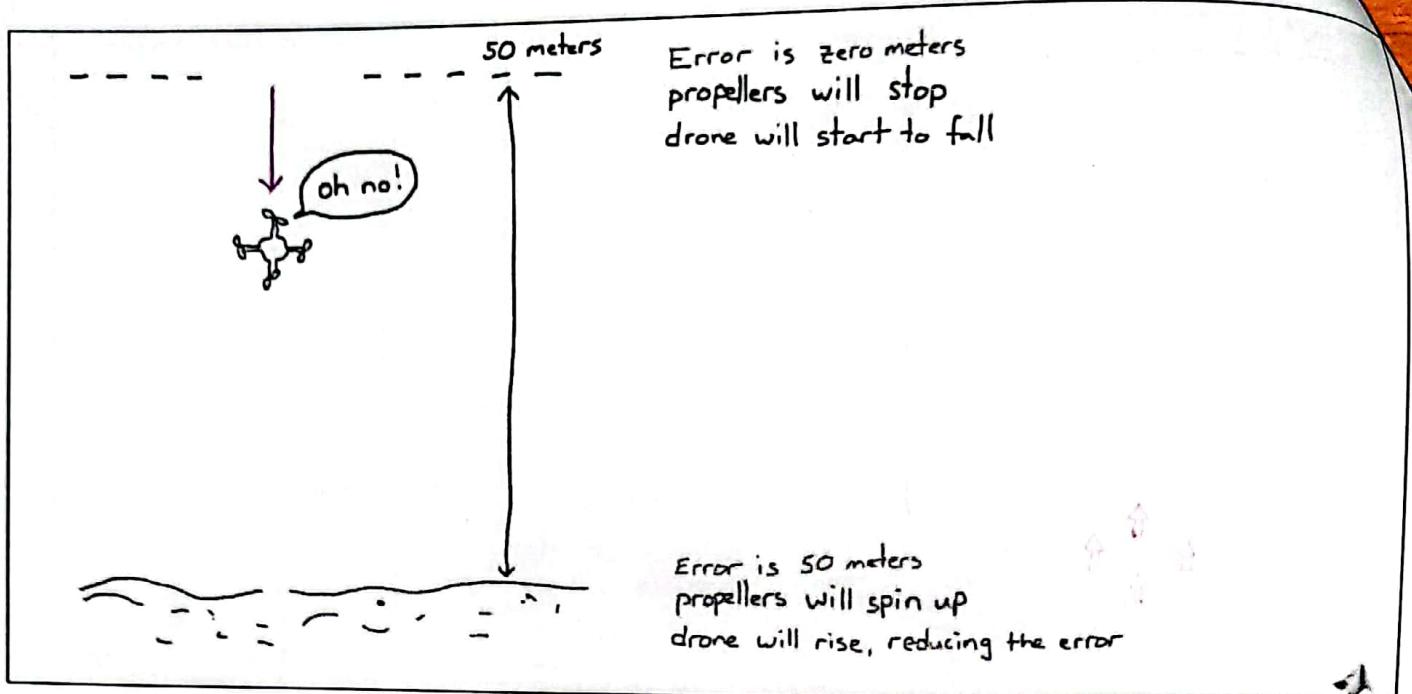
هون الا Controller نتعرى فقط كبارة ئى gain يتساوى ٠.١، بالبداية لمشي بسرقة :
 $5 = 0.1 * 50$
 بعدين كلها أقرب الى desired location ببطئ و هنكلنا لحد ما أوصل المكان المطلوب .
 * هون الا م لحاله كلن كافي



* نيزنا هون الـ App وبدل الانسان حطينا drone بدن ايهه يصل من الأرض لارتفاع ٥٠م .

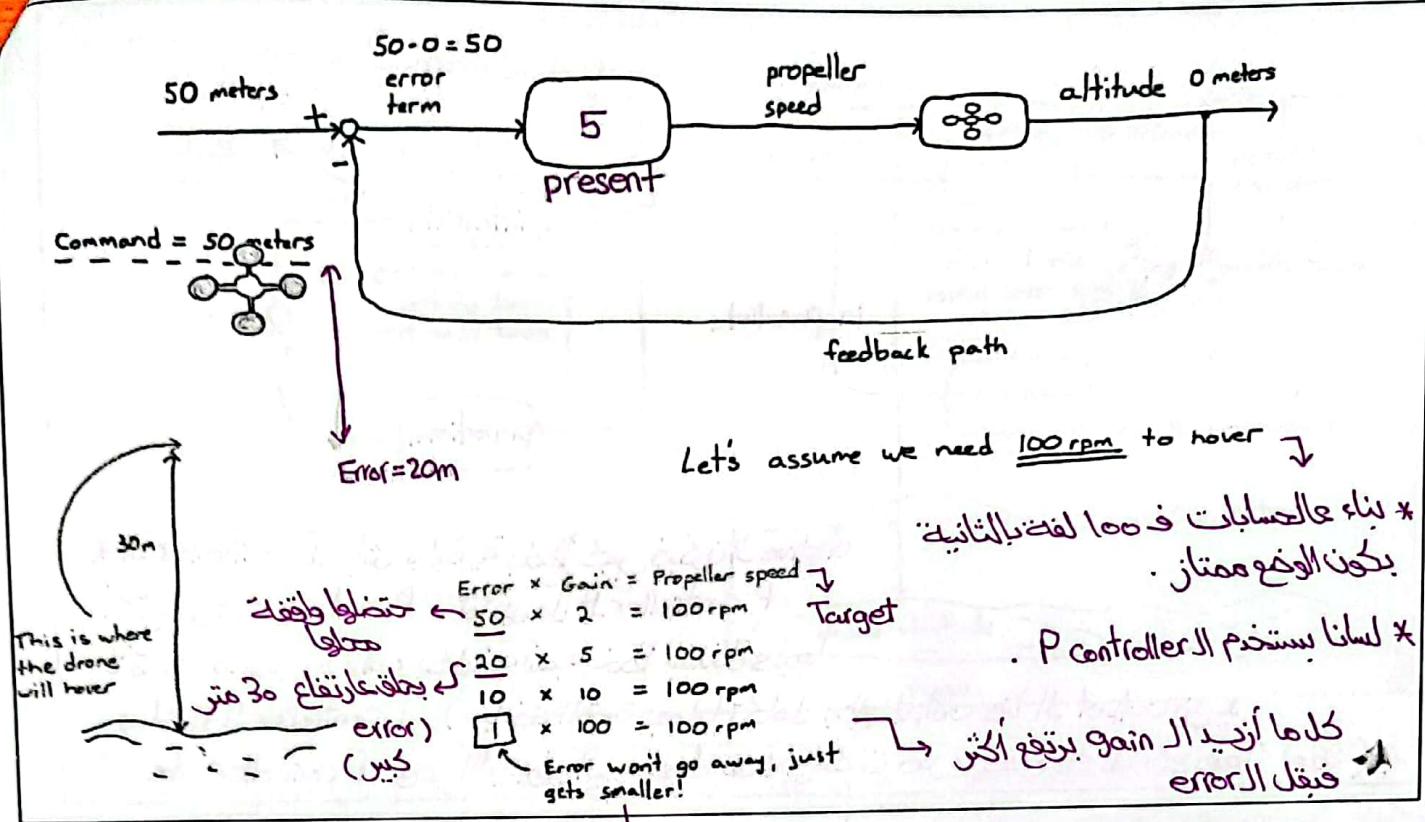
* الـ drone الـ المراح بتشغـل بسرقة معيـنة ، بالبداية الخطأ يكون ٥ بـ تشـغل بـ تـبـلـشـ تـطـيـن بـ سـرـقـةـ مـعـيـنةـ وكـلـ ماـ نـقـبـ يـتـقـلـ سـرـقـةـ لـدـدـ ماـ تـوـلـ الـ target . الـ error يكون = ٥ فـلـماـ وـدـلـ الـ target طـفاـ الـ drone لـاهـ الفـرقـ أـمـيـحـ ٥ فـالـ drone فـيـوـقـ الـ Controller Control Signal .

* فـلاـ لـحالـاـ هـونـ ماـ اـشـغـلتـ صـحـ .



* حنستخدمن هون الفيزياء شوي بحيث انه أنا بدي هذا drone يطير ويوقف بمكان معين يعني محصلة القوة عليه لازم تكون zero . محصلة القوة الموجدة على drone هي الجاذبية الأرضية فلازم يكون عندي قوة رفع متساوية لهاكيشان تهizin المحصلة = 0 و يخلي واقف مكانه.

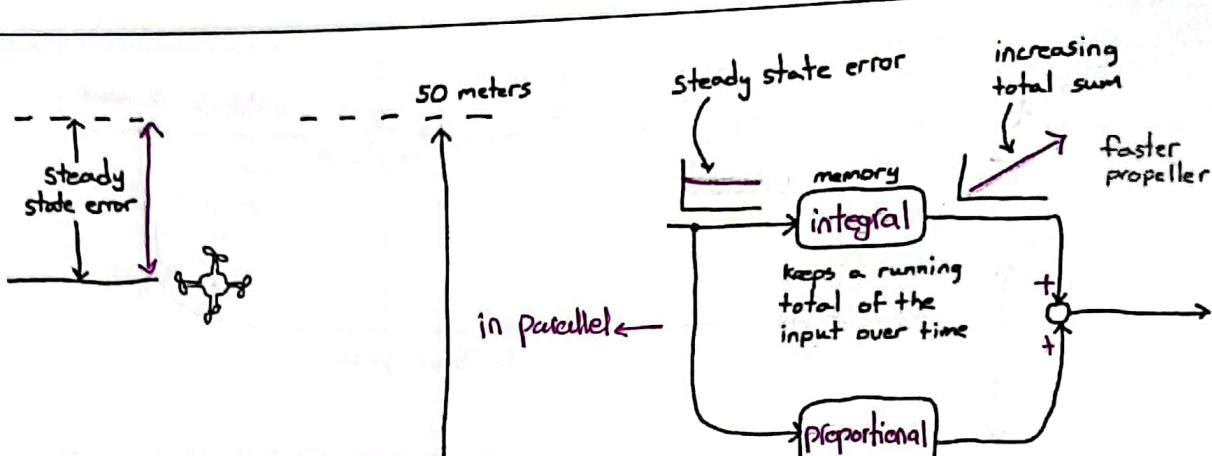
Reduced gravitational force due to the air resistance .
And relative to the ground it will stop .



* حقاً يكون الـ $o = \text{error}$ الـ gain لازم تكون 50
وهذا مستحيل يهين

* فإذا المحتوى هون انه باستخدام الـ P controller فقط بوندا ال App جيكون الخطأ طيبا Finite (يختفي فيه error) خارجاً مشاكيف لحاله فطلع الـ I controller .

* اضافية Integrator يعني انتهاقة block بالـ Software Code أو hardware أو مدمجة بمكتبات جاهزة.



* الـ I controller يلش بالبداية بخطأ كبير فتكون الأ

أكتر هو الـ P controller ، لما يوصل الـ P controller

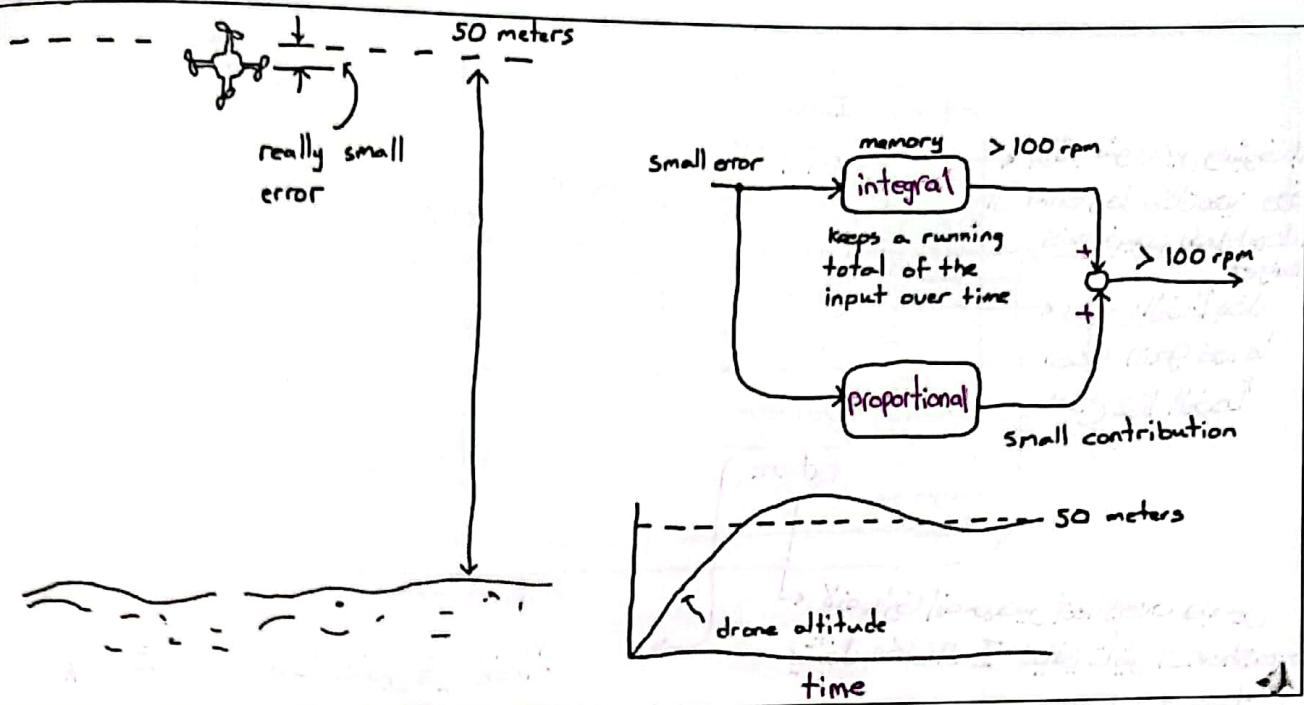
لا تصل حد بقدر يومه ، بدل فيه نسبة خطأ بتشغيل عليها

ساعتها الـ I controller (بياخذ القيمة بعملاها تكامل بطلع output هنالـ I controller يزيد

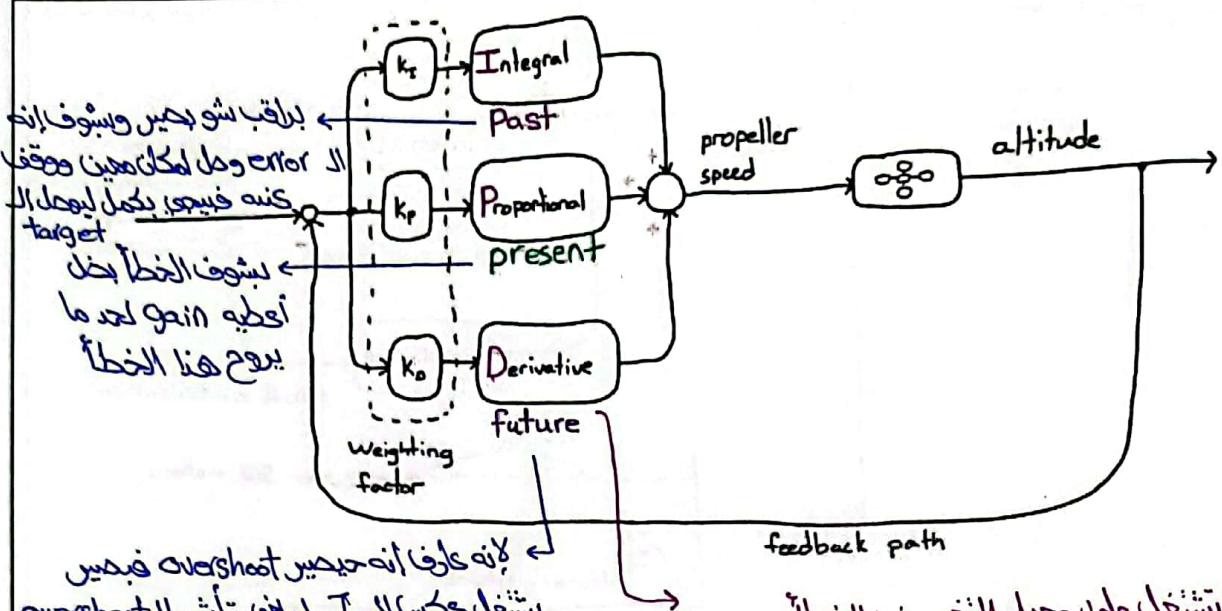
عازد المـ P controller وهو اللي بروح الـ drone من آخر مكان قدر يوقفي فيه لا (50) Target .

* هيك بديرين الـ steady-state-error = 0 وصار الواقع بطريق الرياضيات .

* هون وجدد الـ I مع الـ P كان أفعى من إنجاته لكن لحالها .



* هن بنوضح مشكلة ال I controller ؟ لما يليش يشنغل يوماناً (50) target بيقفز فجأة فواد بيتمامي overshoot ، مشابه لـ Steady-state ، لا تعودها لمان. بالأخير حصل لـ overshoot أكيد سوا لـ overshoot مشكلة يقين لأن رفعال drone لـ 52 فصار النطأ بساوي 2 - (يعني مثلاً بدل ما السرعة تقدر معا خلها 120) فعزا جيظل يعمل undershoot / overshoot وهنَا اشي مشا هنخ فحله كان بالضافة ال D controllers .



* يشتغل على معدل التغير في الخطأ.

* خلو بشوف انه معدل التغير كبير وسيج بتخاله وإذا كان بطريقه بسركه.
* تشغلا معاكس للcontroller I يشان تنحل المشاكل.

* فوكس ينزل هي تأثيرات الا overshoot اللي نتجت من الا Integral اللي خلاه drone يقف بسرعة.

* يوجد الـ controllers 3 هنولع بعض بحق ٩٥٪ تقريباً من المشاكل لأن كلهم بشغلو مع بعض.

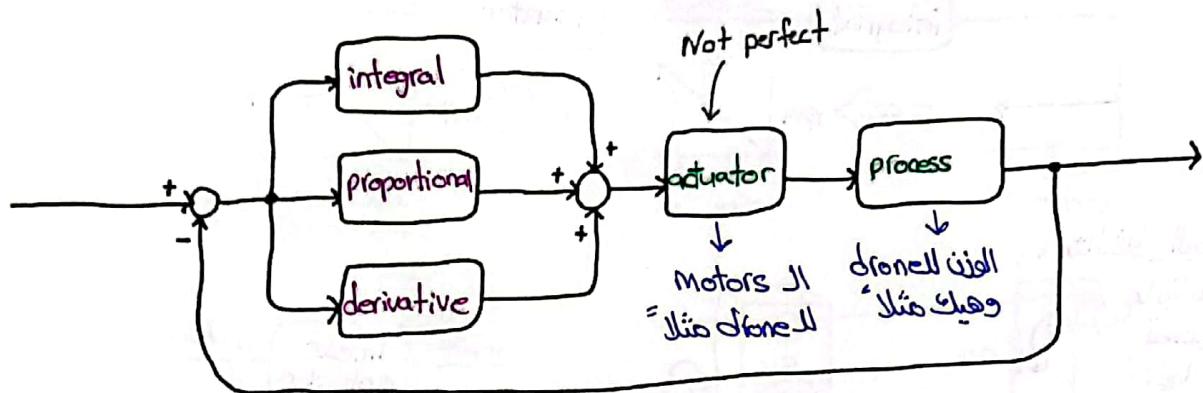
* من شرط بكل الا Apps يكونوا الا 3 مديون ، انا بنقر الأفضل بناء على الا App وتميمه .

* احنا من موقع الا Zeros ولا Pole كنانعرف شونوع الا System (Critically damped / damped Matlab root locus) ، بالكتاب بشرح عن اشي اسماء root locus بالـ tool يقينيا هناها الا root locus .

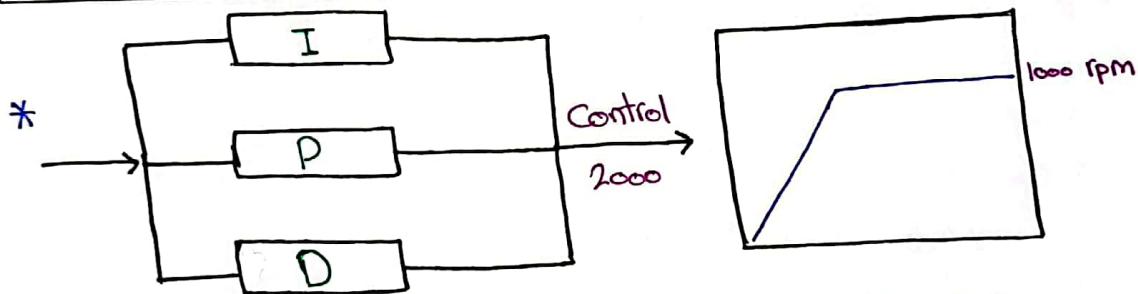
* root locus يشكل عالم كانوا يرسموا diagrams ويسموها بالـ Zeros ولا Poles بحيث يوصلوا الى بعض ياه ، كله بالرياضيات . أما احنا حنشغل بانه نعرف الا gain والـ parameters اللي لازم أحطوا بالـ Sys S و HW يشان تقطبني الشكل المعربي ياه . بعمنا بالـ turn off (k_p, k_d, k_i) هنولع مو تجين الخطأ تبعون الا gains .

Part 2

PID control



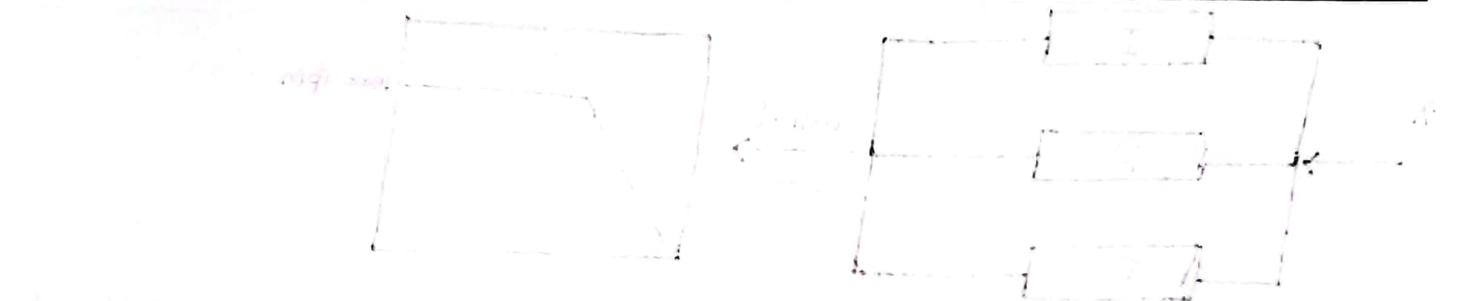
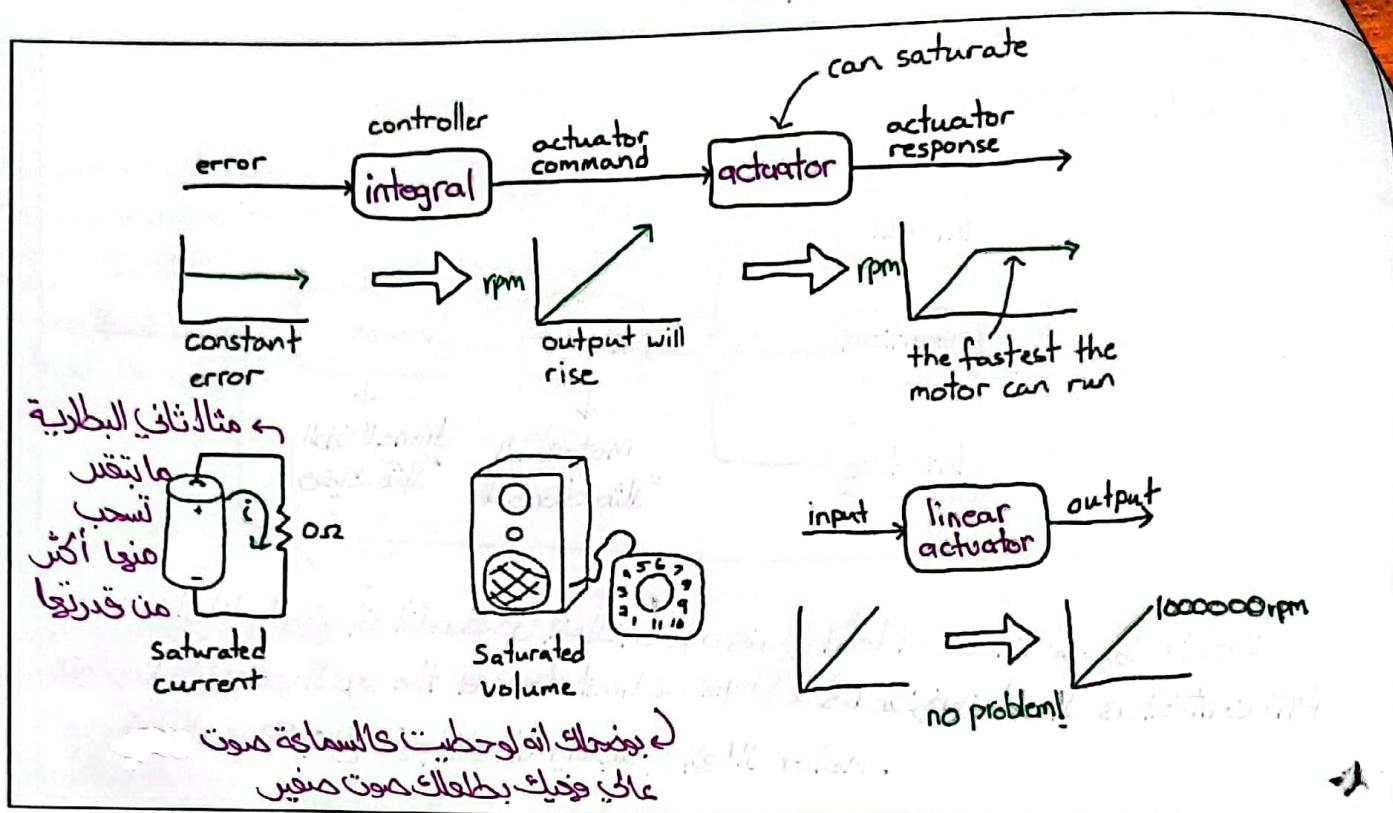
* من هنا كلنا بالواقع انه انا مفترض اه ال System ليس ideal و linear .
 بتواجهنا بالواقع هي العيوب Physical limitations of the sys كذا مفترضين اه ال
 PID controllers أي Signal بتطبع صور قدر انه يستجيب إلها ال Motor .



* لمفترضين انه actuator لا linear وهذا مش واقع (أي قيمة لا x بلاقي لها قيمة لا y) فعليا ما يجري هيك .

* فعليا ال Motors زي ما هو مرسوم (العاشرة قصوى) شو ما اعطيته أكثر منها حيصل بيفاكيلا بس (العاشرة) ، فعدم استجابةه للControl انه يكون 2000 هذه مشكلة .

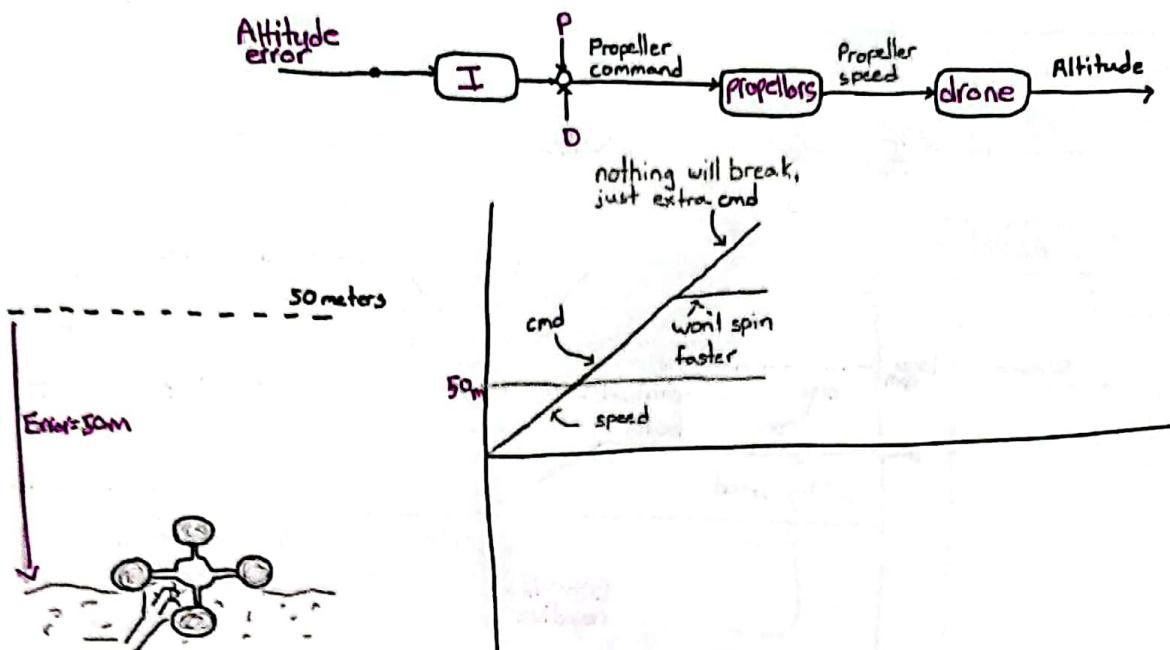
* بالوضع الطبيعي لو ما في Disturbance يعني بس بالواقع مش هيك المعنون .



الآن نحن نعلم أن المدخل هو موجة مربعة، ولذلك فإن المخرج هو موجة مربعة ولكن بتردد أقل.

لذلك إذا أردنا إزالة الترددات العالية، فلنطبق المكثف على المدخل.

لذلك إذا أردنا إزالة الترددات المنخفضة، فلنطبق المكثف على المدخل.

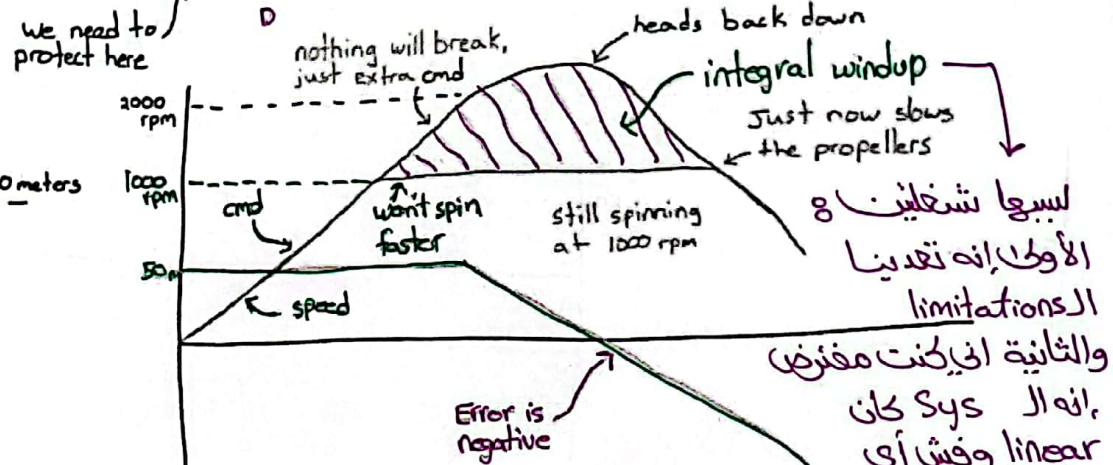
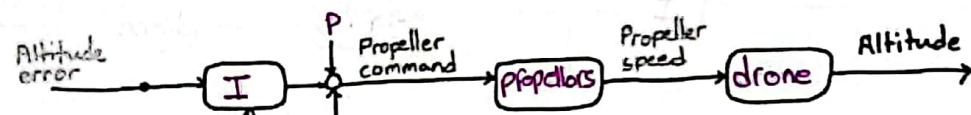


* هذه المشكلة مثلاً، إنك تشغلت drone بس ما ماسكه بتنقصه بيايدي لسبب ما.

بال P هو شايف انه في فرق وقادر بيعطي الا Signal اخر انه تطير.

بال I هو واقف براقب الملفي وشأيف انه الوقت لم يجي والerror ما زال نفسه فكر انه الشغل هلا عليه فبستله ويزيد سرعة ال motor وما زال ال drone ما يمكّن يتحرك لأن ما ماسكه بيايدي. بس أقرر أتركه يطلع فجأة بلاته طلع بسرعة مهولة جداً وسرعته كانت فوق اللي مطلوب وما شاف العطف steady-state وطلع زعا.

مع اني بزيد سرعته لانه كان ممسوك بالبيبة بس السرعة المحددة هنا لازم تكون 1000 كحد أقصى (مشيكسرة 1000 رغم انه older كان 2000 مثلاً) فهل ال Integral لما يشوف هذا الحد بيغير يعطي اوله لا drone انه ينزل من السرعة 1000 بعدينا 800 وهذا. فوزاً الوقت اللي يقضيه ال drone من بعد ما يطلع كثيـر لحد ما ينزل للحد الأعلى السرعة يسمى بالـ windup (المهوة المقابلة) حار يعكس شغلـه

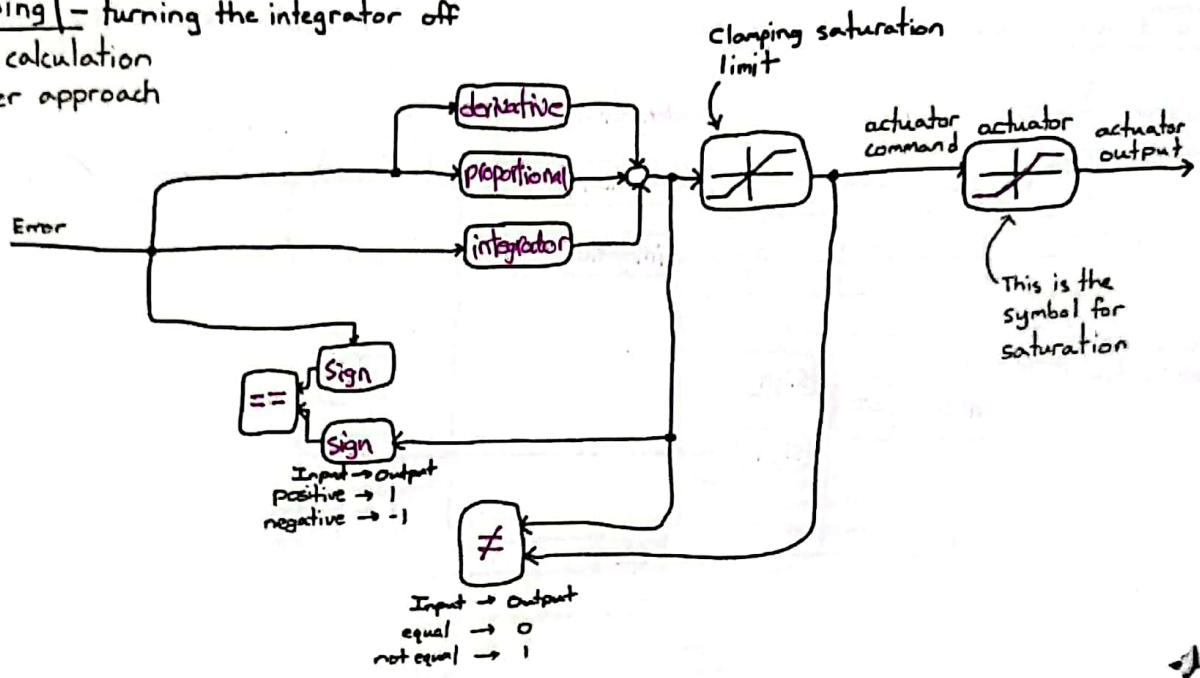


Integrator anti-windup - keep the integrated value from increasing past some specified limit.

Clamping - turning the integrator off

Back-calculation

Observer approach

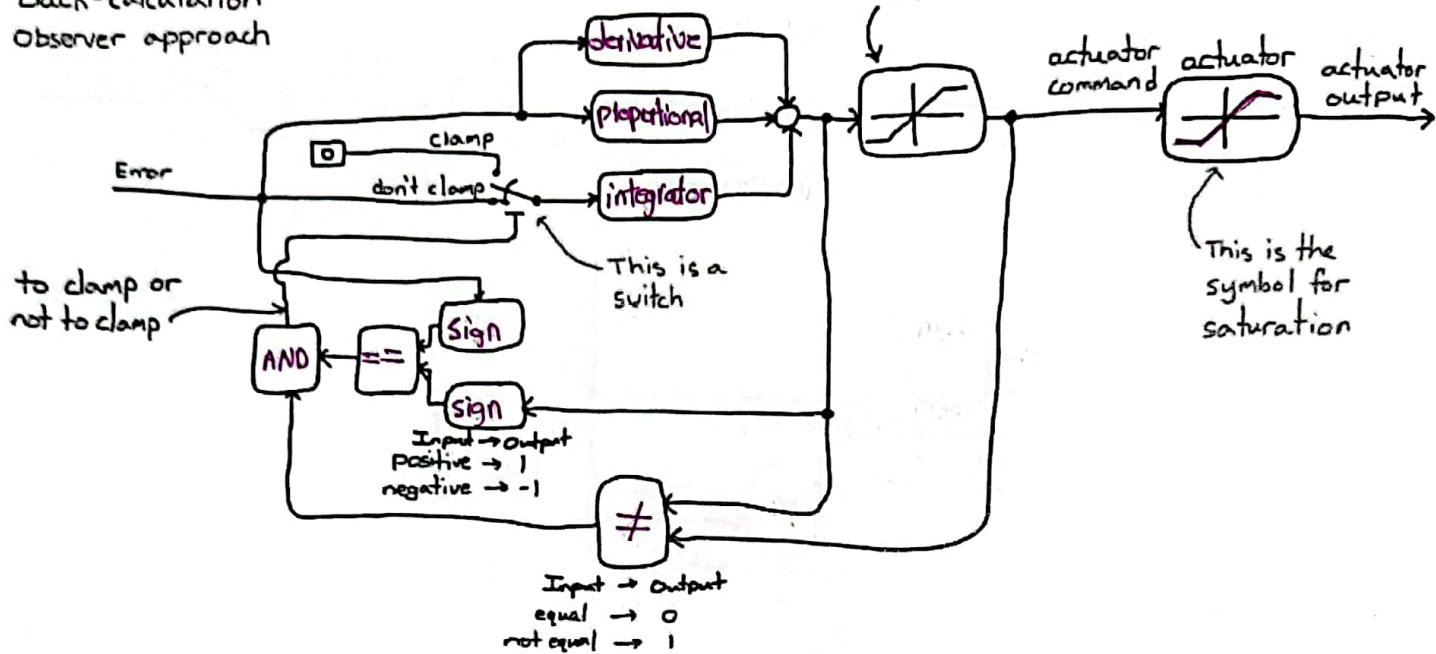


- * الحل المشكّلة الـ windup و إضافة Comparator أو SW (if statement) أو Circuit بدلًا من order مع المطبع.
- * هنّقى ليعرف إنّه فيه Saturation ؟ لما يكون يعطي أمر انه غير سوّي ٥٥ = output فما زلّي مشكّلة . (الّي يعطيه ياه هو نفسه اللي بطلع).
- * هنّقى ليعرف إنّه تجاوزت limitations ؟ لما أعطيه مثلًا طبعًا ١٢٠٠ او بطبعي ١٠٠٠ لا ١٣٠٠ وبطبعي ٩٠٠ وعكّا.
- * فاما ما يطلبني الـ Control اللي بطلع جابه مساوي للـ actual بعرف انه أنا بتعنى physical limitations .
- * لو الـ order مع الواقع متساوين الوضع تمام لو هو متساوين بعرف اني وصلت حد saturation .
- * المشكلة الثانية انه الـ error موجب (عندي خطأ) وما زال الـ controller بيعطي أمر طبعًا .
- * ففعلياً لو عندي error وما زال الـ windup Controller شغال ويعطي أمر طبع أو نزول (الـ error والـ controller نفس الإشارة منهاته بيعطّو signal order للطريق أو هنّك شغال يا controller يا integrator) كله تتحقق الأمورين بنعرف انه في هنا مشكلة بحلوها بإضافة switch لـ Integrator . (باللحظة اللي بطلع فيها الـ saturation شغال افضل لأن كل المشكلة مدهلاً لـ Integrator) هذا يسمى (Clamping)

Integrator anti-windup - keep the integrated value from increasing past some specified limit.

Clamping - turning the integrator off

Back-calculation
Observer approach

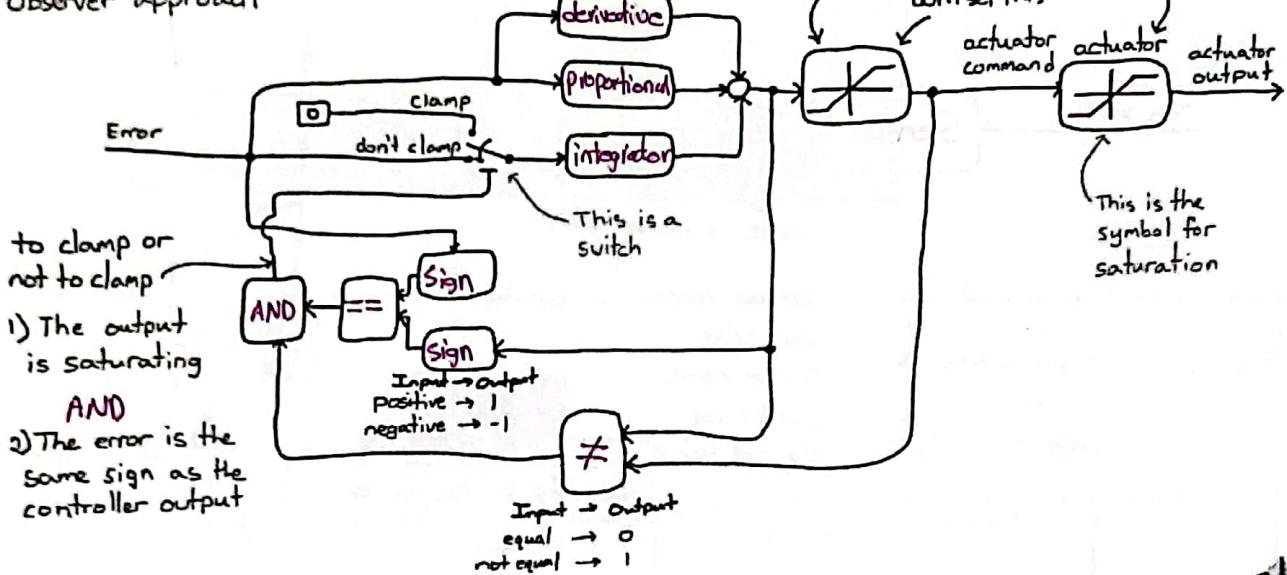


Integrator anti-windup - keep the integrated value from increasing past some specified limit.

Clamping - turning the integrator off

Back-calculation

Observer approach



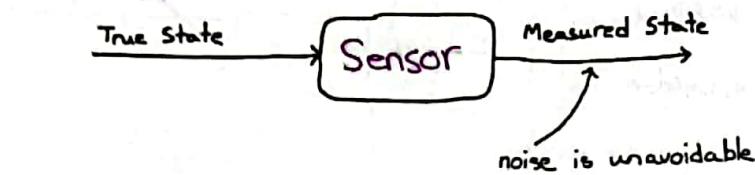
* شو القيمة اللي لازم أطحها لل Clamping؟ هنالك أنا حاكي إنه الطاقة الاستيعابية لمروحة لا drone هي 1000 rpm أقصى أشي، فول لا Clamping أطحها 1000 ولا أقل ولا أقل؟ دايماً بتحطه أقل (900) هنالك كشان في إيه تقدم بالهر فملح يعطيك قد ما كان يعطيك أولها استغل وحطها أقل من الهر اللي محدده بالبداية.

Part 3

* المشاكل اللي بنواجهها derivative controller وكيف نحلوها.

Noise is a random disturbance on a signal

Hey! Keep that noise down!



$$\text{“Noise”} = \begin{matrix} \text{Environment} \\ + \\ \text{Implementation} \\ + \\ \text{Defects} \end{matrix}$$

Thermal noise
Shot noise
Flicker noise
Burst noise
Coupled noise

white noise

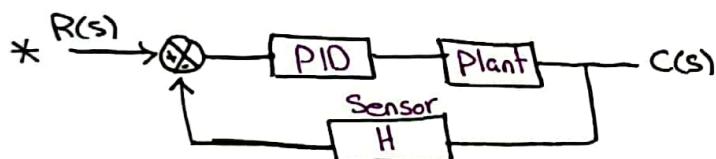


High and low frequencies
مثال عاليه نویس اول
ما نشفل التلفزيون.



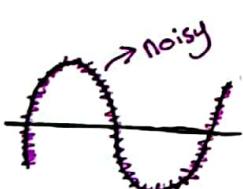
* feedback ينقيس فيه لا out وبنرجع لهجة الا in مشان نشوف الفرق بين الا desired Signal و الا actual feedback ونعدل بناء عونا الناتج ، وهذا الا feedback بنحصله بتركيب Sensor.

* في مشاكل بالقليعة من الا Sensor اللي بتعمل على Circuit وقوتها وتحصيموا.

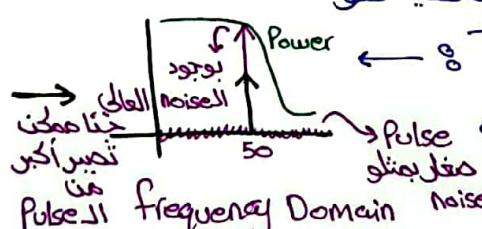


* مشكلتنا هي Derivative بتشغل مدخل \rightarrow
القيرين بالخطأ والalarm Proportional بخرب

الخطأ بـ Gain Integral الخطأ ويشغل عليه فلو
نشوف وجود الا noise بالtime/freq. domain



Time Domain
50 Hz \rightarrow Nyquist
 $= 100$ Hz

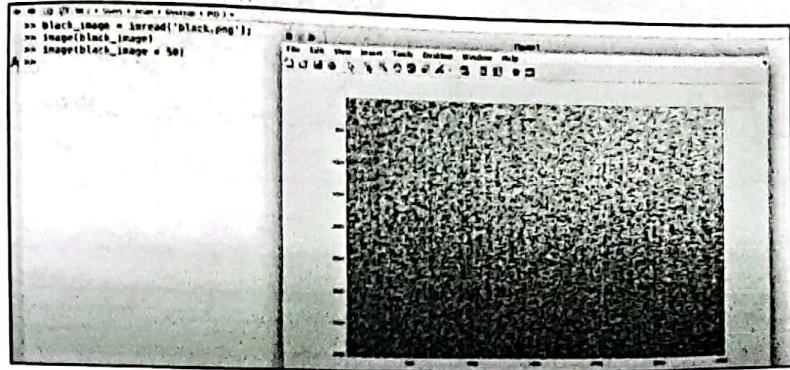


* المشكلة هنا بالController D اللي بتشغل noise بالSensor

كمشقة الخطأ ، لما الا Signal يجي عليعا noise سوا
(low freq) فمشقتة قليلة ، بينما لو كان noise كثير على فالمشقة
بتكون كبيرة فبتتأثر بشكل كبير فعندها بعدها باستخدام filters (ما يلغى noise بعدها تأثيرها)

لـ بدلها يعطيكي قيم صحيحة بهيس يعطيكي قيم خاطئة لـ الـ cutoff تبع احنا بقرره
حسب الـ System . هون مثال

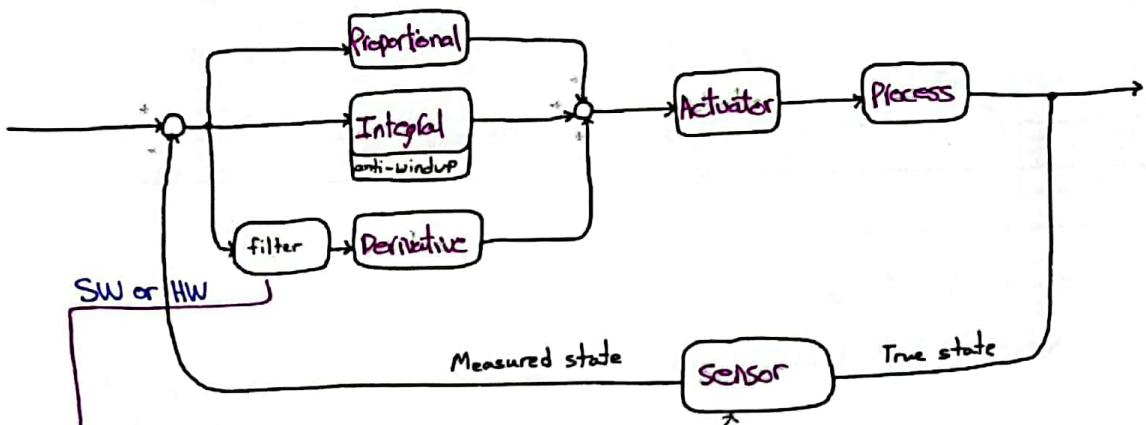
$$\frac{de}{dt}$$



تجربة كاميرا →
النلقوں کند تکوین
شاشة سطحه و زیادة
ال Brightness
لأعلى اشي
لوما ذخور حسیان

فجودة الكاميرا بتكون جيدة (low noise). (اد Noise بوضوح بالليل أكثر اشي)

PID control

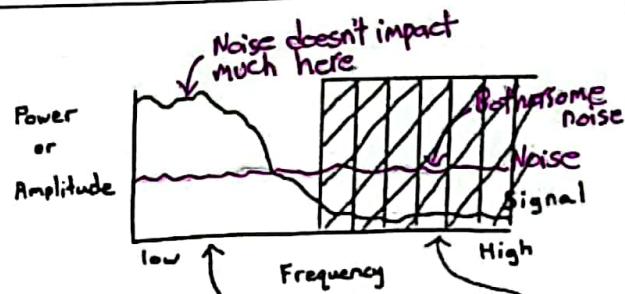
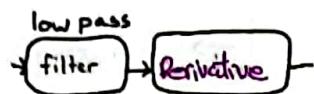


- low Pass: بمنع العالي وبسمح الواطي
- high Pass: بمنع الواطي وبسمح العالي can add noise into our loop (high frequency)
- band pass: بين 2 بسمح ranges

* احنا بوننا ينشئ الـ noise العالي قدر الامكان قبل ماتتحول للـ D controller

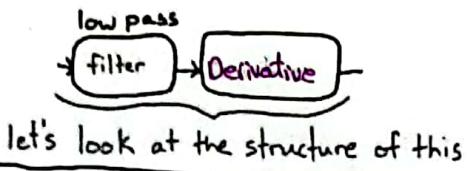
* يتعين بفتح فیلتر low pass



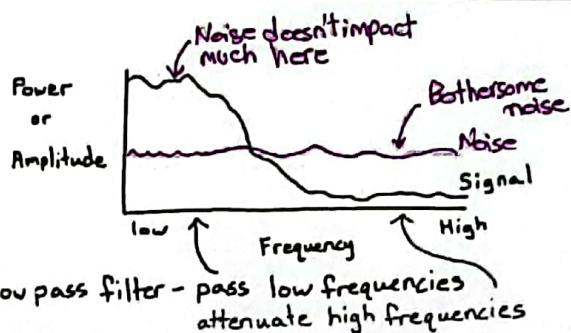
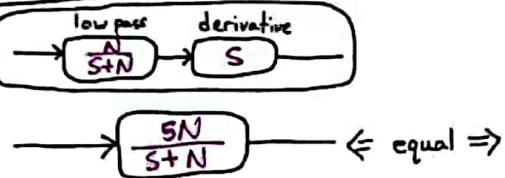


* الفكرة الأساسية هنا أن Signal
يُدَّى ياهابط كلُّها .
فلي يُدَّى ياهابط بعْرَقِ والَّتِي بعْدَ
يُتمَّ خلْتُرُوا .

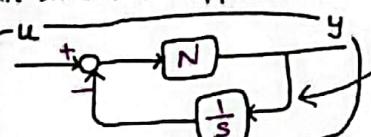
Frequency Domain \rightarrow Filter \rightarrow كيفية تحسين \rightarrow Cut-off Frequency



Laplace domain transfer function	what it is
S	derivative
$\frac{1}{S}$	integral
$\frac{N}{S+N}$	low pass filter with cut off at N rad/s



An alternative approach:



An integral in the feedback path

$$y = N(u - \frac{y}{S})$$

$$y + \frac{Ny}{S} = Nu$$

$$y = \frac{Nu}{1 + N\frac{1}{S}} u$$

$$\boxed{\frac{y}{u} = \frac{N}{1 + N\frac{1}{S}} u}$$

* نعرف من أول درس Laplace أنه المشتقة ينضرب بـ S والتكامل ينقسم S .
* وجدوا انه تعميم الـ Filter المناسب هو $\frac{N}{S+N}$ والا N هو الـ Cut off freq.

* فيقرر أحدهم الـ Filter باستخدام الـ IntegralDerivative \rightarrow low pass \rightarrow Feedback

5

p-Controller (Proportional Control)

يمثل الملف شفاه →
 (كان بالعادة يكتب فيه نسبة خطأ لثما أو م الحاله)

In proportional control, the controller produces a control action that is proportional to the error. There is a constant gain K_p acting on the error signal e and so:

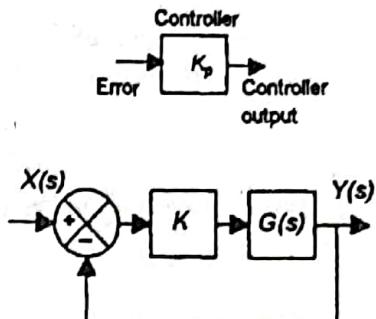
$$\text{controller output} = K_p e \rightarrow * \text{ مخصوص بالخطأ } *$$

We have steady-state error! How the P-control gain K_p affects the steady-state errors?

For a closed loop-system with a process transfer function $G(s)$, and unity feedback:

$$Y(s) = \frac{G(s)K_p}{1+G(s)K_p} X(s)$$

حلوا شو
مليك هون فرضها
 $1/(s+1)$



We need to determine the value of the output as the time t tends to an infinite value. To do this in the s-domain, we use the final value theorem (which if the limit exists):

$$\lim_{t \rightarrow \infty} f(t) = \lim_{s \rightarrow 0} sF(s)$$

P-Controller (Proportional Control)

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$$Y(s) = \frac{G(s)K_p}{1+G(s)K_p} X(s) \rightarrow y_{ss} = \lim_{s \rightarrow 0} s \frac{G(s)K_p}{1+G(s)K_p} X(s)$$

If $x(t) = u(t)$, then $X(s) = 1/s$

$$y_{ss} = \lim_{s \rightarrow 0} s \frac{K_p/(s+1)}{1+K_p/(s+1)} \frac{1}{s}$$

$$= \lim_{s \rightarrow 0} \frac{K_p}{s+1+K_p}$$

Suppose $G(s) = 1/(1+s)$ for this demo example

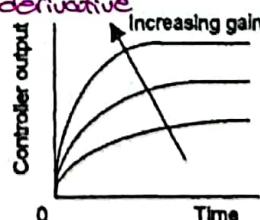
As the limit goes to zero

$$y_{ss} = K_p/(1+K_p)$$

الميحة النهاية
عد ما فعلنا
derivative

* شو ما حد قيمة لل gain
بضل فيينا نسبة خطأ

K_p	y_{ss}	Offset
1	0.5	0.5
4	0.8	0.2
10	0.91	0.09



* شو ما زينا بال gain
يتقل نسبة الخطأ بس مستحيل
نتحمل له إلا لو كانت ال gain
Infinite وهذا مستحيل.

Increasing the proportional gain decreases the steady state error, but it does not eliminate it to zero

* أهمية وجود ال K_p بكل أنواع ال Controller لازم تكون لأنه وحدنا انه هيكل افضل
 اشي ليقلل من مشاكل ال disturbance . (Slide 7 →)

P-Controller (Proportional Control and Disturbance Rejection)

Previously we considered the effect of a disturbance on the performance of a closed-loop control system. We have seen that closed-loop control systems are better at minimizing disturbances than an open-loop system.

Consider this closed-loop control system with two possible sources of disturbances, one being a disturbance affecting the input to the process and the other affecting its output.

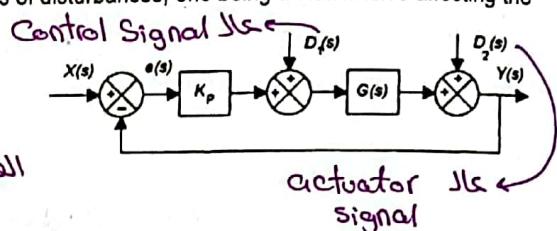
$$Y(s) \text{ is } K_p G(s)e(s) + G(s)D_1(s) + D_2(s)$$

Error = $X(s) - Y(s)$. → output \rightarrow Input المفهوم بين الـ

$$Y(s) = K_p G(s)[X(s) - Y(s)] + G(s)D_1(s) + D_2(s)$$

$$Y(s)[1 + K_p G(s)] = K_p G(s)X(s) + G(s)D_1(s) + D_2(s)$$

$$Y(s) = \frac{K_p G(s)}{1 + K_p G(s)} X(s) + \frac{G(s)}{1 + K_p G(s)} D_1(s) + \frac{1}{1 + K_p G(s)} D_2(s)$$



Increasing the proportional gain reduces the effect of disturbances.

The first term is the normal expression for the closed-loop system with no disturbances. The other terms are the terms arising from the two disturbances. The factor $1/[1 + K_p G(s)]$ is thus a measure of how much the effects of the disturbances are modified by the closed-loop.

* كل ما أزيد في كي بـ K_p يقل (يعني يقل) D_1 و D_2

Constant \rightarrow مخصوصية

PD-Controller (Proportional and Derivative Control)

In derivative control, the controller produces a control action that is proportional to the rate at which the error is changing.

$$K_d \frac{de}{dt}$$

where K_d is the derivative gain

Derivative control is not used alone but always in conjunction with proportional control and, often, also integral control.

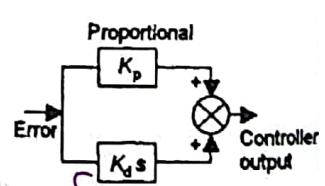
$$\text{controller output} = K_p e + K_d \frac{de}{dt}$$

The proportional element has an input of the error e and an output of $K_p e$. The derivative element has an input of e and an output which is proportional to the derivative of the error with time

In Laplace form: controller output (s) = $(K_p + K_d s)E(s)$

Can be rewritten as: controller output (s) = $K_p(1 + T_d s)E(s)$

* where $T_d = K_d/K_p$ and is called the derivative time constant.



مشتق المبدأ

* هنا وظيفتنا ناك في هي K_p و K_d و T_d ونعطيها

Controlled

هي فقط

استخدم

T_d بدلاً

من K_d

فائدته و

يعطي شكل الـ Signal و يقىم الـ oscillation

علية و هنا اشي كثير مرتقب ومغير

PD-Controller (Proportional and Derivative Control)

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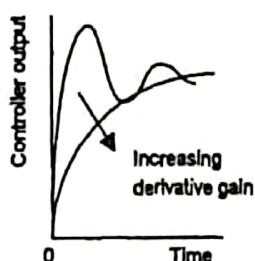
Derivative control has the controller effectively **anticipating** the way an error signal is growing and responding as the error signal begins to change (This is why we say that the derivative controller looks for the future).

A problem with this is that noise can lead to quite large responses.

Adding derivative control to **only** proportional control still leaves the output steady-state error and does not eliminate it.

Changing the amount of derivative control in a closed-loop system will change the damping ratio since increasing K_d increases the damping ratio.

oscillation
قبل الـ damped
under-damped
effects



10

→ constant / s

PI-Controller (Proportional and Integral Control)

In integral control, the controller produces a control action that is proportional to the integral of the error with time.

$$K_i \int e \, dt \quad \text{where } K_i \text{ is the integrating gain}$$

Integral control is not used alone but always in conjunction with proportional control and, often, also derivative control.

$$\text{controller output} = K_p e + K_i \int e \, dt$$

The proportional element has an input of the error e and an output of $K_p e$. The integral element has an input of e and an output which is proportional to the integral of the error with time

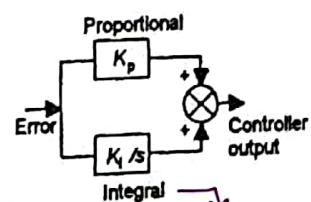
$$\text{In Laplace form: controller output (s)} = \left(K_p + \frac{K_i}{s} \right) E(s)$$

$$\text{Can be rewritten as: controller output (s)} = \frac{K_p}{s} \left(s + \frac{1}{T_i} \right) E(s)$$

where $T_i = K_p/K_i$ and is called the **integral time constant**.

The presence of integral control **eliminates** steady-state errors and this is generally an important feature required in a control system.

" K_i بدل T_i "



↓ وظيفتها الأساسية تقديم
الخطأ من الـ System و الـ steady-state.

تحل الخط.

* الـ P يقلل الـ oscillation ولا D يقلل الـ Disturbance ولا I يقىم الخطأ الميختل.

* لنفهم الـ Controller بالشكل المناسب يجب نعمل تجربة شوائي لازم نعرف نقاط البداية.

PID-Controller (Proportional, Derivative and Integral Control)

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The basic form is a *three-term controller*.

$$\text{output} = K_p e + K_i \int e dt + K_d \frac{de}{dt}$$

In Laplace form:

$$\text{output}(s) = K_p \left(1 + \frac{K_i}{K_p s} + \frac{K_d}{K_p} s \right) E(s)$$

$$\text{output}(s) = K_p \left(1 + \frac{1}{T_i s} + T_d s \right) E(s)$$

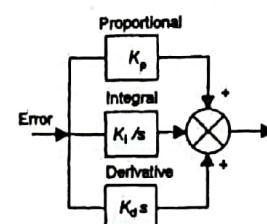


Table 7.4 Effect of Increasing the PID Gains K_p , K_D , and K_I on the Step Response

النسبة المئوية لزيادة التردد الباقي - Percent Steady-State Overshoot

Settling Time

Steady-State Error

PID Gain

Increasing K_p

Increases

Minimal impact

بسهاب مدخل زéro

Increasing K_I

Increases

Increases

Zero steady-state error

Increasing K_D

Decreases

Decreases

No impact

مشكلته بزيادة
الـ Signal damping
الـ overshoot هو
الـ الذي يقلل
الـ الخطأ

برهن
زيادة
الـ overshoot

إنه زاد
الـ overshoot
فحتاج
فترة أطول
إنه أقل
وقت أقل

لتحقيق
الـ overshoot = 0

PID-Controller (Proportional, Derivative and Integral Control)

12

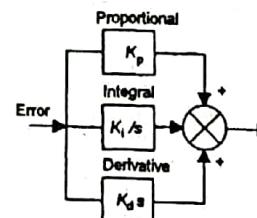
The basic form is a *three-term controller*.

$$\text{output} = K_p e + K_i \int e dt + K_d \frac{de}{dt}$$

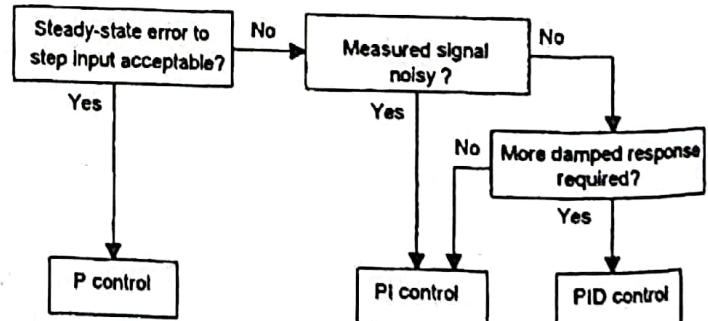
In Laplace form:

$$\text{output}(s) = K_p \left(1 + \frac{K_i}{K_p s} + \frac{K_d}{K_p} s \right) E(s)$$

$$\text{output}(s) = K_p \left(1 + \frac{1}{T_i s} + T_d s \right) E(s)$$



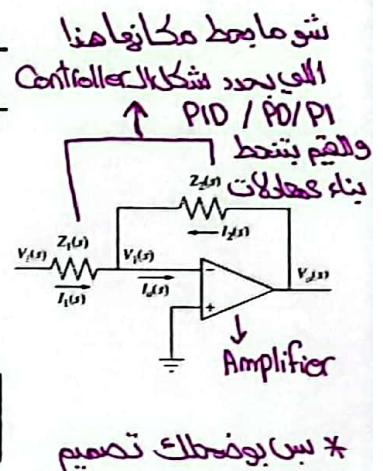
Controller Selection →



How are PID Controllers Realized (Classical Techniques)

This slide is just to have a quick idea, do not memorize the shapes or equations!

There are equations used to get the values for the resistors and capacitors from K_p , K_i , and K_d



kp بالا بتعطي قيم
Circuit *
و kD و kI و بتتحططها بالا
تعتلي س احنا بدنا نحل
بالا
SW

متاح
و بتتحطط
C و R
قيمة

Circuit Software

How are PID Controllers Realized (Computer Control)

We use libraries and functions, because the controller and system equations are all realized in software, so is the PID controller

Example, ARM DSP library has a function for PID Controller

https://www.keil.com/pack/doc/CMSIS/DSP/html/group_PID.html

مكتبة الـ CMSIS - DSP (مكتبة فيها أشياء

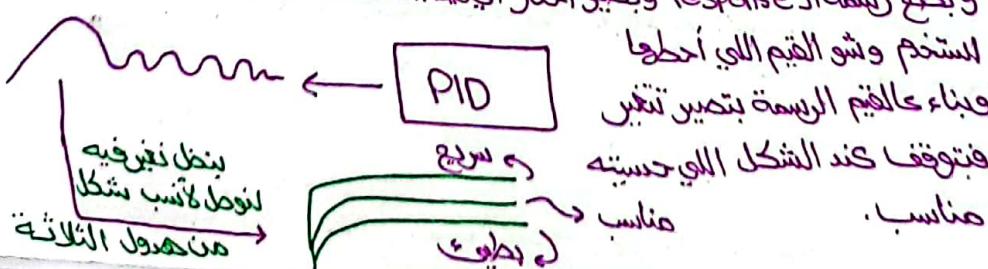
(PID controller)

* The function takes as input two parameters S and in: جارعاً عندها
__STATIC_FORCEINLINE float32_t arm_pid_f32 (arm_pid_instance_f32 * S, float32_t in)

in is the input signal object فيه قيم kp و ki و kd
And S is a struct which has the gains and the values of Kp, Kd, Ki
https://www.keil.com/pack/doc/CMSIS/DSP/html/structarm_pid_instance_f32.html لوهنا نجيبي

But where and how do we get the values of Kp, Kd, Ki?

MATLAB PID Tuner (DEMO In class) وشنقل الا block S domain System
وبطلع رسمة الـ response وبحسب اختيار أي controller



Data Fields

- float32_t A0
- float32_t A1
- float32_t A2
- float32_t state [3]
- float32_t Kp
- float32_t Ki
- float32_t Kd

جعدها
بالا
Self learning
9

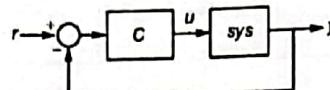
* بالطريقة الثانية لازم تدخل كينك طبعاً عال loop
 - stability
 * هيك مثال بالـ Continuous وكيف هي
 * discrete

Motors In real time يعني ال انه تشغيل Simulink

Initial Automatic PID Tuning

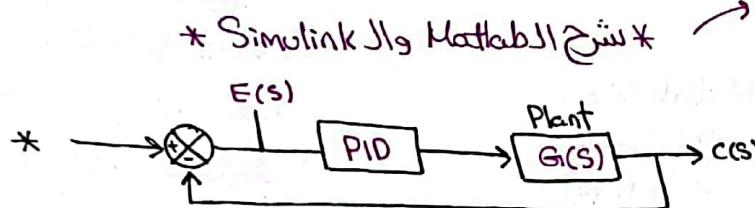
15

وال موجزة بالحقيقة Controls
 ٩٦١ مشتوك علىها ونفهم بالقيم ، فالسي منتج بالحقيقة مثل بس
 MATLAB has the `pidtune` command which takes in as input the plant system as input, and the type of controller you want to use.
 Notice, that MATLAB assumes a unity feedback design, so, if need be, transform your system to match a unity feedback design.
 Model to Sys.

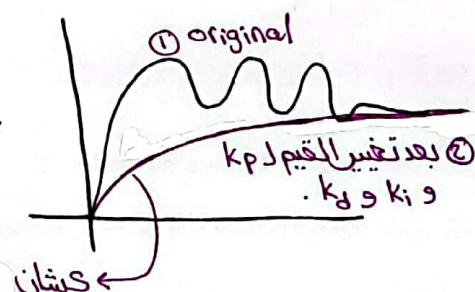
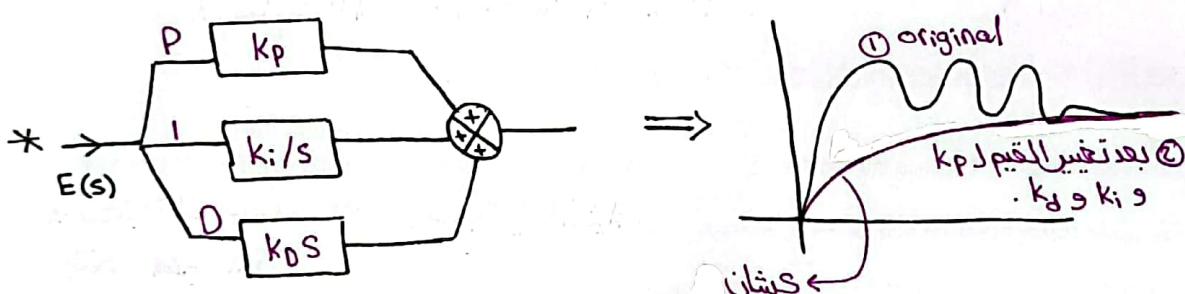


sys is the transfer function, it could be constructed using any of the techniques we have used before (tf, zpk)
 C could be any of the following controllers (in this course, we only cover 1-DOF types, and only those in red)

- 'P' — Proportional only
- 'I' — Integral only
- 'PI' — Proportional and integral
- 'PD' — Proportional and derivative
- 'PDF' — Proportional and derivative with first-order filter on derivative term
- 'PID' — Proportional, integral, and derivative
- 'PIDF' — Proportional, integral, and derivative with first-order filter on derivative term



محاضرة ٥/١/٢٠٢٣
 كيفية تطبيق المفهوم
 شرح الطريقة
 Matlab و Simulink



نحوها الحيله جنوف طرفيتين) الأولى والثانية
 الطريقة الأفضل والأحسن .

* فتح المانلايب وكمel Script وكتب المعادلة و

$$\frac{100}{s^2 + 15s + 100} \text{ of Sys}$$

(tot=feedback(sys,1) sys=100/(s^2+15*s+100) s=tf('s'))

بعين (sys) ← بعددين يستخدم pidtune Command (هاي الطريقة هش مفهومه بازخوا Manual)

يعطيك قيم K_p, K_i, K_d [c, info] = pidtune(sys, 'PID') ←

$MyController = 2.25 + 15.8/s + 0.0781*s$ ← Stepplot(new.sys) ← new.sys = mycontroller * sys ; ←

لشنوف تأثيرها Stepplot(new.sys) ← new.sys = mycontroller * sys ; ←

بنهين نغير قيم K_p, K_i, K_d لنعمل الاتي الانسب .

* الطريقة الثانية المستخدمة Apps → PIDTuner → Import for sys ← يدخل و Import the Type

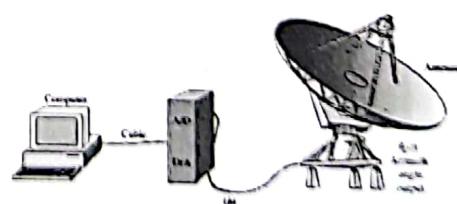
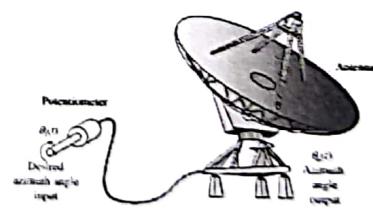
* ال Parallel form (K_p, K_d, K_i ماينعرف) اما الا Standard Form بوضوح D, T_d و T_i

* Show Parameters يظهر المعلمات وبهين أيير بالقيم لعمل اللي بدي ياه . (كملة فوق النوت)

Brief Introduction to Digital Control

Introduction

- ▶ In chapter 1, we learnt about the advantages of digital systems over analogue systems. We also learnt in this course and the embedded systems course about how to transform analogue data to digital data through sampling and quantization (A/D).
- ▶ We quickly reviewed how to determine the proper sampling rates in order to get digital representations that can faithfully represent the analogue signal.
- ▶ These days, many sensors are digital, that is, they have the A/D built-in and provide the sampled data at certain configured rates to the digital controller. These sensors can be used in the feedback loop or just to monitor the plant.
- ▶ In other cases, the A/D is built-in inside the controller chipset (SoC) and you can configure it to take samples in digital form at a required rate.

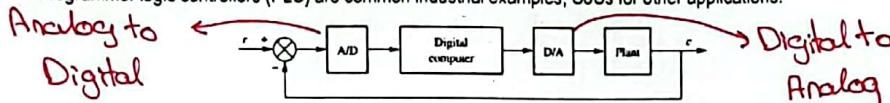


* الثالث أدوار الأساسية للـ Control ① Digital computer التي تكون جزء من
 الـ System. ② ما تعرفه العادة بالـ Control، موجة بين المدخل والمخرج وهي نفس SCADA (Supervision) وما يفترض تحكم بالقيم التي يتعرض لها. ③ الذي موجود بالواقع هو الـ SCADA الذي في عجزين بقراً لـ Data Supervisory وبحذفها يتيح للعامل وهو من نسخة بعدها أنه يتحكم بهذا الـ System Data Acquisition

3

Digital Computer Role in Control Systems

- A digital computer might not take a direct control role at all. The whole plant might be controlled through conventional classical and analogue controllers, however, computers take a supervisor role: That is, they simply collect data about the system operation, creating logs, or showing real-time sensory data input to remote screens where engineers can monitor the plant and see if there are hazards or safety actions that need to be taken.
- A digital computer can replace many analogue controllers and be placed in the forward path, where it takes inputs, feedback data in digital form, process them through software code, equations, libraries, software PID, and issue commands to the plant for direct control. Programmer logic controllers (PLC) are common industrial examples, SoCs for other applications.



- In Industrial automation, the computer role can be hybrid; that is, it can take both a supervisory role and direct control role. These systems are famous and are referred to as SCADA (Supervisory Control and Data Acquisition). They provide Human Machine Interfaces (HMI) to control PLCs, read sensors, log data, and issue alarms.

<https://inductiveautomation.com/resources/article/what-is-scada>

* مثل SCADA والـ النحوية التي مررنا بها في تحكم بـ المغناطيس.

* مثل تابي الـ "Program PLCs logic controllers"

Digital control Sampling rate نسبه بلا ارتو
 Analog control System unstable ، بلا ارتو
 كل يوم هنا قيم الـ gain مثلاً لإزعاجكانت مرات تخلي System
 . unstable

* يتم التبديل بين التحريك الـ A/D
 الـ D/A



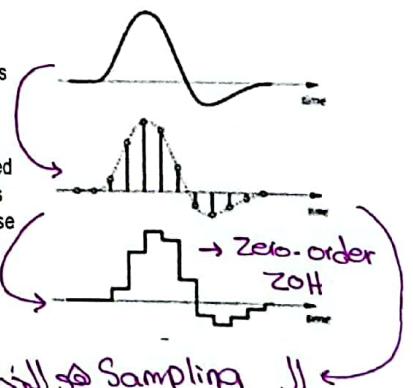
4

Review of the Sample and Hold

- In the embedded systems course, you learnt about the sample and hold circuit (SOH) which is the circuit that samples the analogue signal and holds the analogue value as steady as possible (unfortunately, we have drooping), then the A/D circuit converts this analogue value to a digital binary representation based on the resolution and voltage range of the ADC (quantization).
- Upon the complexity of this Sample and Hold circuit, they can be modeled and referred to as zero-order, first-order. In this course, we will assume the simplest circuit which is zero-order sample and hold, or "ZOH". You know this circuit from the embedded course as it generates the stair-step approximation.

SOH → أبسط أنواع الـ SOH
 "أبي الرج" → Stair-Step

after Sampling

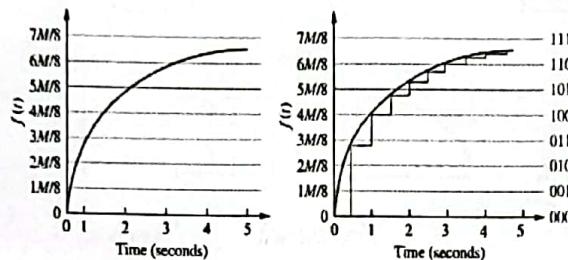


الـ Sampling هو الخطوة الأولى من الـ A to D ، هو بحول الـ Time domain من analog Sample بعدد نبذات على المدى المحدود في قيمة الـ Discrete binary ويعطي قيمة بالـ

* الـ Sampling بحد ذاته Circuit ويأخذ آخر قيمة لا Capacitor (Slide 14 embedded).
 * كلما كان الـ Sample rate على يتكمن الروجل أكثر فذلك أقرب للشكل الحقيقي.

Voltage References and Quantization

- We have voltage references that the ADC must use to know the range of the analogue signal, V_{ref+} and V_{ref-} , and we know the resolution of the ADC to be for example N bits, and this means we have $(V_{ref+} - V_{ref-}) / 2^N$ ranges, and each range will take one of the 2^N possible binary representations.
- There is always an error that is called the quantization error. No matter how much we increase the resolution, this error is always there and equals $\frac{1}{2} \times (V_{ref+} - V_{ref-}) / 2^N$ for a non-uniform quantizer



الخطوة المقدمة الـ 0.1, 0.2, ..., 0.9 → 000 = 0
فهي
هذا
الخطوة
الخطوة
موجود
وهو دائمًا
وقيمتها
التي طبعناها
 $= \frac{1}{2} * \frac{(V_{ref+} - V_{ref-})}{2^N}$

(resolution) 2^N س دوام (max/min) فنجزي quantization *

خواص محددة عند A/D بخلاف 5 = Max Volt و 0 = ref. Volt معناه يمكنني اخذ مساحة من 5-0 = 5 فاراً ، لو لا A/D يعطي النتيجة بشكل متدرج 16 bits فنجزي الى 16 على 2¹⁶ نعرف عدد ال ranges

Modeling the Sampler

- The simplest model for the sampler is that we have a pulse train (a series of rectangular pulses, on and off) multiplied by the signal. Effectively, this means we are reading the signal when we have a pulse (sampling it), or not reading the signal when the pulse is off.
- Ideally, the width of the rectangular pulse should be so small, more of an impulse train rather than a rectangular train. This ideal sampler can be mathematically described as

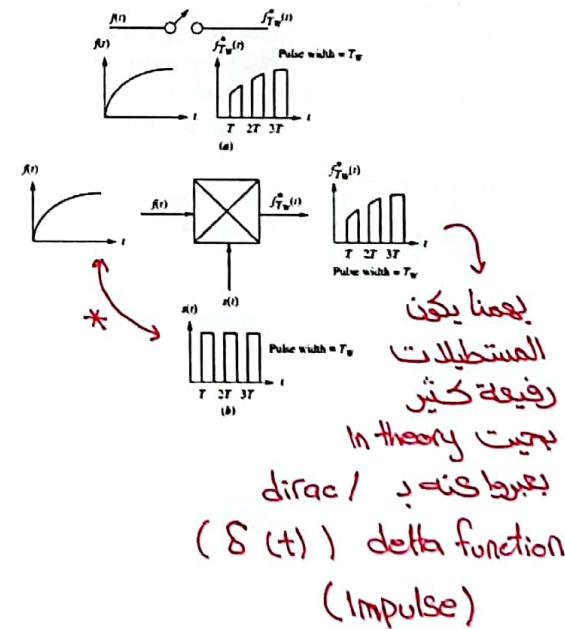
$$f_s(t) = \sum_{k=-\infty}^{\infty} f(kT) \delta(t - kT)$$

* اما في المجال العددي بال Time Domain يستخدم T

اما في المجال الـ Discrete domain يستخدم T

يأخذ قيم Discrete

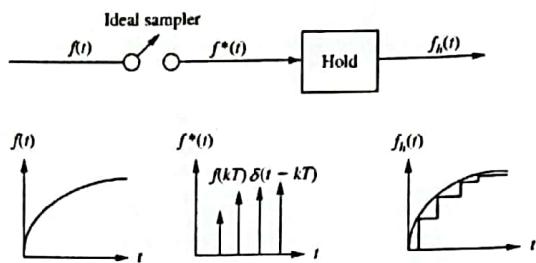
زوجي 0.25, 0.5 بما يلي



Modelling the Zero Hold Circuit

- Once we have taken a sample, we need to pass it to the quantizer to start converting it to a digital binary form. For this to happen, we need to keep the sampled value steady for the entire duration (in practice it drops a little due to capacitor discharge, i.e. drooping effect)
- The laplace transform for the zero-hold circuit is given by

$$G_h(s) = \frac{1 - e^{-Ts}}{s}$$



* المشكلة تجيء انه دخلت Computer ومن ثم كل
أجزاء الـ Motor هي digital System ، الـ digital Sensors
مش Analog Sensors الـ digital بعدها
Voltage divider Circuit الـ digital
Analog blocks بالـ analog blocks بالـ digital
ومن غيركروا بالـ S Domain زي هلا ، و
blocks بالـ Digital بعديروا بالـ Z Domain

Transient response وهذا يعني نفسه
بأنه مختلف الرسمة متغير زي
العن الأختبر



Samples الـ Samples . ونستوف انه سرقة الـ Samples ، الـSamples الجديد موجود في الـ System يمكن تطبيق

Digital Systems Modelling

System
Unstable
Sys
إيجار
ما يتحقق
كلها
في مصر
جور لـ
Sampling
وقد يتحقق
ذلك

- In this course, we mainly dealt with classical analogue control systems with continuous time-domain inputs and outputs. Once digital controllers (computers) are involved, many blocks in the system are using discrete time-domain samples of the original signal. As such, the Laplace transform can no longer be used to represent the system in the frequency domain.
- The discrete counterpart of the Laplace transform for digital control is the z-transform, and the new models we will develop in MATLAB will take into consideration the sampling rate F_s or sampling time T_s .
- We have already learnt how to properly choose a sampling rate based on the sampled signal characteristics (maximum frequency component) and applying Shannon theorem of sampling and Nyquist rate. We also studied the concepts of undersampling and oversampling.
- Now, whereas the stability and transient response of analog systems depend upon gain and component values, sampled-data system stability and transient response also depend upon sampling rate. So now, we have new things to consider when choosing the sampling rate, that is the system stability.

The z-transform

- Similar to the laplace transform, the z-transform converts the function from the sampled time-domain to the frequency domain.
- In the time-domain, we no longer use the variable t to represent time, since t is associated with continuous time signals. Instead, we use the variable n to denote sampled time instances. This is the default for MATLAB commands as well.
- MATLAB has two commands to move between the sampled-time domain and the z-domain, `ztrans` and `iztrans`

+ بدل

<pre>syms n f = sin(n); ztrans(f)</pre>	<pre>syms z F = 2*z/(z-2)^2; iztrans(F)</pre>
<pre>ans = (z^sin(1))/(z^2 - 2*cos(1)*z + 1)</pre>	<pre>ans = 2^n + 2^n*(n - 1)</pre>

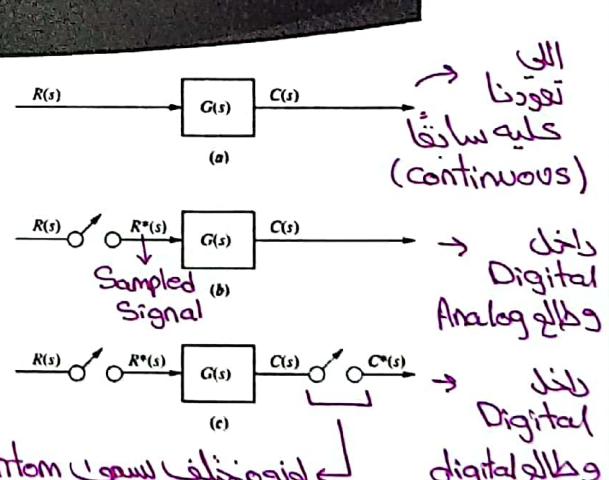
Samples

بحلول النهاية (n)

حسب التعاليم

The Transfer Function

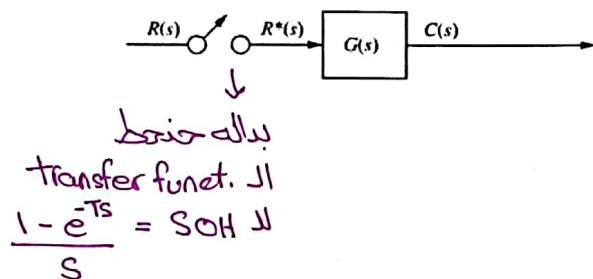
- In this course, we have learnt how to obtain the transfer functions for classical systems as shown in Figure a. We have continuous time input $R(s)$ going to an analogue system represented in the s-domain, and giving a continuous output.
- In Figure b; however, we have a sampled input $R^*(s)$ instead of a continuous time input $R(s)$. In this example, we assume that the output is continuous.
- However, in many cases, we might only be interested in seeing the output for those sampled inputs and not in between, in Figure c, an imaginary (phantom) sampler is added at the output (coloured) to denote that we are interested to see the discrete output to the discrete inputs only. This assumption further simplifies future analysis.



لأنه مختلف يسمى (phantom) Physically (وهي)،
لم تلاحظه كثيراً يعني أنه حالياً Input يسمى Discrete.
وشكل الـ output عند التقاطه الذي أنا مهتم به هو كما يذكر
فهو حتى اسمي Physically موجود ليس فقط في شكل يذكر
إنه الذي يوصل لما تبيح القراءة discrete يعني output الذي مصدره من هي القراءة
الـ discrete والمعنويدخل الـ Step output.

Converting $G(s)$ in Cascade with Z.O.H. to $G(z)$

- This figure below shows a discrete sampled input using a sample and hold circuit that goes into the s-domain block. Since the entire system is using sampled data, we need to find the overall new block $G(z)$ that represents this system.
- We can convert $G(s)$ in cascade with a zero-order hold (z.o.h.) to $G_n(z)$ using MATLAB's $G_n = c2d(G, T, 'zoh')$ command, where G_1 is an LTI continuous-system object and G is an LTI sampled-system object. T is the sampling interval and 'zoh' is a method of transformation that assumes $G(s)$ in cascade with a z.o.h. We simply put $G(s)$ into the command (the z.o.h. is automatically taken care of) and the command returns $G(z)$.



Example 1

Given a z.o.h. in cascade with $G_1(s) = (s+2)/(s+1)$ or

$$G(s) = \frac{1 - e^{-Ts}}{s} \frac{(s+2)}{(s+1)} \rightarrow \text{Z-Domain} \quad \text{لآن نعمل مatrix} \quad \text{ربط بين} \quad \text{بار} \quad \text{س}$$

find the sampled-data transfer function, $G(z)$, if the sampling time, T , is 0.5 second.

الإخضاعية ←
عن قبل يعني
حدد الـ sample
discrete
discrete
Continuous-time transfer function.
 $G_n = c2d(Gs, T, 'zoh')$ $\approx \frac{1 - e^{-Ts}}{s}$

```

T=0.5; % Input sampling interval.
numgs=[1 2]; % Define numerator of G(s).
dengs=[1 1]; % Define denominator of G(s).
Gs=tf(numgs,dengs) % Create G(s) and display.

```

→ Sys in
Z-Domain
↓
Sample time: 0.5 seconds
Discrete-time transfer function.

```

Gn =
z - 2.1131
-----
z - 0.6065

```

Example II

Sampling الـ Sys كعنوان قبل الـ System ← بمعنى أن الـ Sys ← System ← Sampling *

- Given $G(z)$ of a certain function, can we retrieve back the original block $G(s)$? Let us do this by reversing the previous example.
- Notice that we can use the familiar `zpk` command to represent transfer functions in the z -domain, the only change is that we need to specify the sampling period so that it knows it is dealing with sampled data, and give a representation in z instead of s

```

Zero ← num=0.2131;
Pdc ← den=0.6065;
gain ← K=1;
T=0.5;
Gz=zpk(num,den,K,T)
Gz =
(z-0.2131)
-----
(z-0.6065)

Sample time: 0.5 seconds
Discrete-time zero/pole/gain model.

```

هذا يعني
الزيادة الوحيدة هي
نحو زمرة
الـ Continuous time

$G_s = \frac{(s+2)}{(s+1)}$ → discrete to continuous
 زمورة، فهي كانت مستخدمة مع
 الـ Sys الـ ZOH فافعله في
 الـ Sys الأساسي
 وادعه في ياه شو كان بالـ S-Domain

Creating Digital Transfer Functions Polynomial Form

- A digital transfer function can be expressed as a numerator polynomial divided by a denominator polynomial, that is, $F(z) = N(z) / D(z)$. The numerator, $N(z)$, is represented by a vector, `numf`, that contains the coefficients of $N(z)$. Similarly, the denominator, $D(z)$, is represented by a vector, `denf`, that contains the coefficients of $D(z)$. We form $F(z)$ with the command, `F=tf(numf, denf, T)`, where T is the sampling interval. F is called a linear time-invariant (LTI) object. This object, or transfer function, can be used as an entity in other operations, such as addition or multiplication. We demonstrate with $F(z) = 150(z^2 + 2z + 7)/(z^2 - 0.3z + 0.02)$
- We MUST use an unspecified sampling interval, $T = []$ since we are already representing a digital system and not sampling a continuous one. It should be empty so that it creates a digital transfer function not a continuous one

```

numf=150*[1 2 7]; % Store 150(z^2+2z+7) in numf and
denf=[1 -0.3 0.02]; % Store(z^2-0.3z+0.02) in denf and
F=tf(numf ,denf,[])
F =

```

اضافة عنوان اخير، انه
احنا بالـ Z-Domain
(خديبي يكزن)

$$\frac{150 z^2 + 300 z + 1050}{z^2 - 0.3 z + 0.02}$$

Sample time: unspecified
Discrete-time transfer function.

Creating Digital Transfer Functions Directly Vector Method

- We also can create digital LTI transfer functions if the numerator and denominator are expressed in factored form. We do this by using vectors containing the roots of the numerator and denominator. Thus,

$$G(z) = 20(z+2)(z+4)/[(z-0.5)(z-0.7)(z-0.8)]$$

- can be expressed as an LTI object using the command, $G=zpk(\text{numg}, \text{deng}, K, T)$, where numg is a vector containing the roots of $N(z)$, deng is a vector containing the roots of $D(z)$, K is the gain, and T is the sampling interval.

```
numg=[-2 -4]; % Store (s+2) (s+4) in numg
deng=[0.5 0.7 0.8]; % Store (s-0.5)(s-0.7)(s-0.8) in deng
K=20; % Define K
G=zpk(numg,deng,K,[])
```

لأنني معرف \rightarrow
 $\frac{20 (z+2) (z+4)}{(z-0.5) (z-0.7) (z-0.8)}$ Sampling
 الـ $\frac{20}{(z-0.5) (z-0.7) (z-0.8)}$ Discrete-time zero/pole/gain model.
 خاصتي بـ $\frac{20}{(z-0.5) (z-0.7) (z-0.8)}$ Rate
 لذا تكون المكان كـ $\frac{20}{(z-0.5) (z-0.7) (z-0.8)}$ Domain

. \rightarrow Domain يكون المكان كـ $\frac{20}{(z-0.5) (z-0.7) (z-0.8)}$

Creating Digital Transfer Functions using the z-method

- Similar to before where we defined $s = tf('s')$, we can use $z = tf('z')$, then write the digital transfer function directly

كـ $\frac{150 z^2 + 300 z + 1050}{z^2 - 0.3 z + 0.02}$ Domain

ويندرينـ $\frac{150 z^2 + 300 z + 1050}{z^2 - 0.3 z + 0.02}$ Series
 أو $\frac{20 (z+2) (z+4)}{(z-0.5) (z-0.7) (z-0.8)}$ feedback

```
z=tf('z');
F=150*(z^2+2*z+7)/(z^2-0.3*z+0.02)
```

F =

$$\frac{150 z^2 + 300 z + 1050}{z^2 - 0.3 z + 0.02}$$

```
Sample time: unspecified
Discrete-time transfer function.
```

```
G=20*(z+2)*(z+4)/[(z-0.5)*(z-0.7)*(z-0.8)]
```

G =

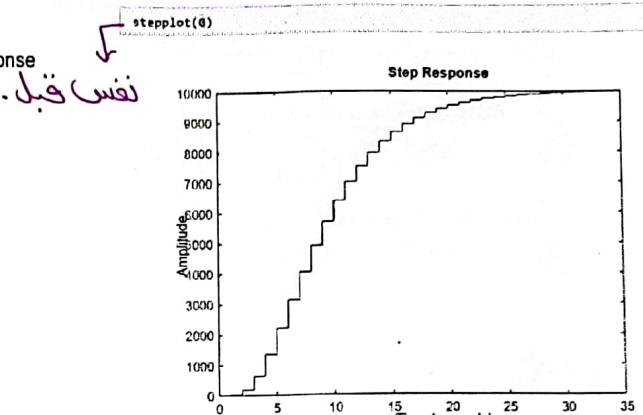
$$\frac{20 z^2 + 120 z + 160}{z^3 - 2 z^2 + 1.31 z - 0.28}$$

```
Sample time: unspecified
Discrete-time transfer function.
```

Command tf Python بالـ * في sympy بالـ *
 Matlab الـ *

Transient Response

- ▶ Similar to before, we can observe the step response using the stepplot command



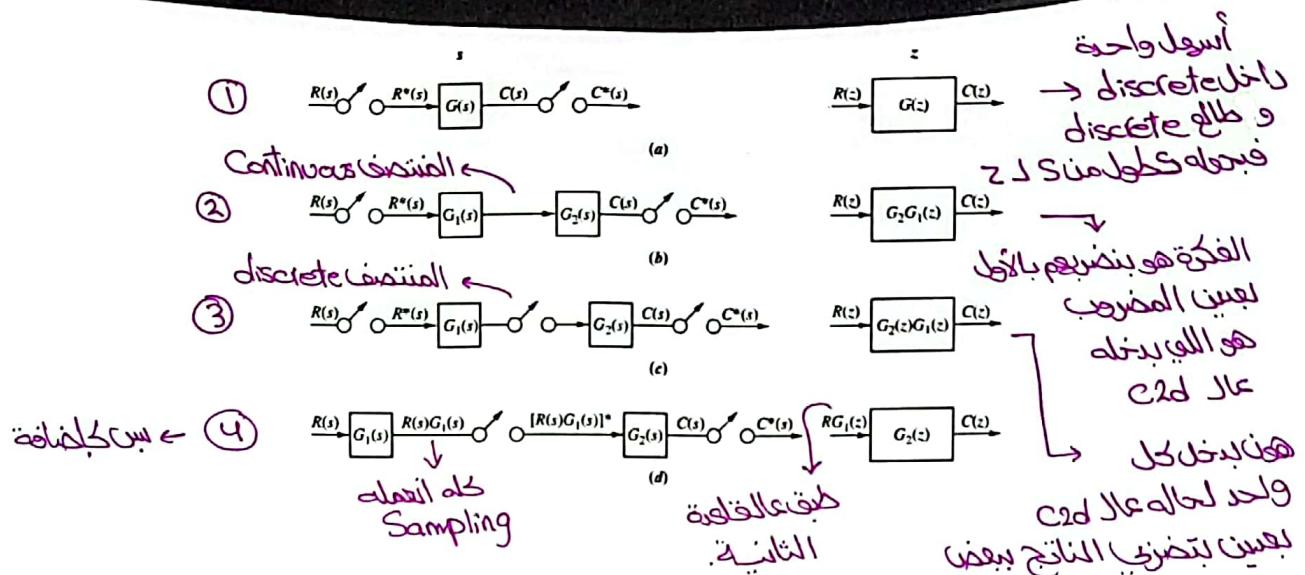
Block Diagram Reduction Important Note

- ▶ Our objective here is to be able to find the closed-loop sampled-data transfer function of an arrangement of subsystems that have a computer in the loop
- ▶ When manipulating block diagrams for sampled-data systems, you must be careful to remember the definition of the sampled-data system transfer function to avoid mistakes.
- ▶ For example, $z\{G_1(s)G_2(s)\} \neq G_1(z)G_2(z)$.
- ▶ The s-domain functions have to be multiplied together before taking the z-transform. In the ensuing discussion, we use the notation $G_1G_2(s)$ to denote a single function that is $G_1(s)G_2(s)$
- ▶ $z\{G_1(s)G_2(s)\} = z\{G_1G_2(s)\} = G_1G_2(z) \neq G_1(z)G_2(z)$

* نفس القواليـن اللي هـمـانـا قبل بالـblock Manipulation
 Combined blocks قبل تحضيرات و نشـيل الـSwitches و نـحـدـهـ مـكـافـهـ

19

Block Diagram Reduction Equivalent Blocks

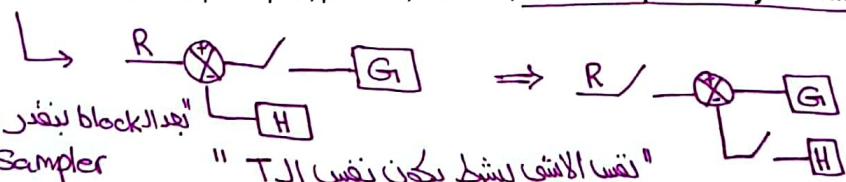


20

Rules for Adding Phantom Sampler

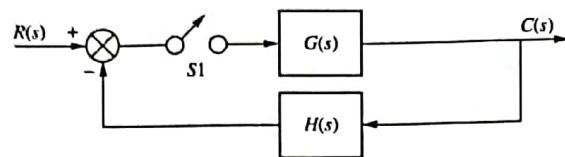
Simplification ↪

- A phantom sampler is an imaginary sampler that can be added to simplify block diagram reduction.
- Rule I: You can place a phantom sampler at the output of any subsystem that has a sampled input, provided that the nature of the signal sent to any other subsystem is not changed. For example, one can add phantom samplers at the output $C(s)$. The justification for this, of course, is that the output of a sampled-data system can only be found at the sampling instants anyway, and the signal is not an input to any other block.
- Rule II: Another operation that can be performed is to add phantom samplers at the input to a summing junction whose output is sampled. The justification for this operation is that the sampled sum is equivalent to the sum of the sampled inputs, provided, of course, that all samplers are synchronized



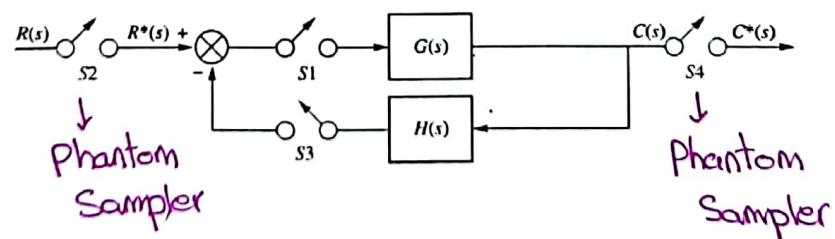
"لـهـذـهـ الـبـلـدـاـنـهـ وـهـيـ اـذـاـكـانتـ
 هـيـ الـبـلـدـاـنـهـ مـنـ مـشـبـكـهـ عـلـىـ
 مـنـأـبـلـهـ التـبـسيـطـ" (زـيـ ماـ)
 كـهـلـاـ بـعـدـ الـبـلـدـاـنـهـ (output)

Digital Block Simplification Example

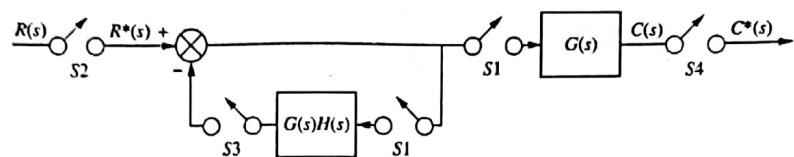


Digital Block Simplification Example – cont.

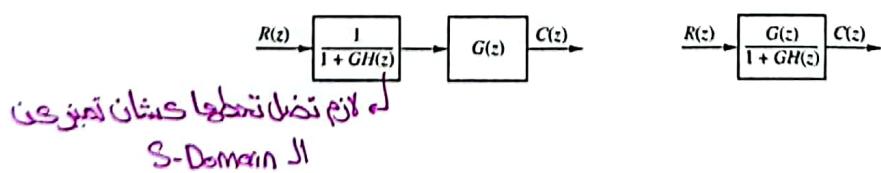
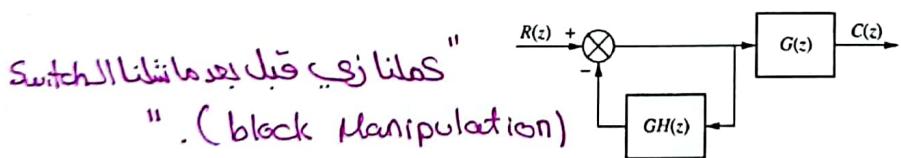
- ▶ Adding Phantom samplers per the rules



Digital Block Simplification Example – cont.



Digital Block Simplification Example – cont.



* بالـ S-Domain كان نسم ال Poles ونشوفهم لو بالـ LHP فهو Margin stable ولو بالـ RHP فهو unstable ولو جا في عـ jw-axis يكون غير Stable

* كيف أعرف الـ System اذا كان Stable

و لا جا بالـ z-Domain

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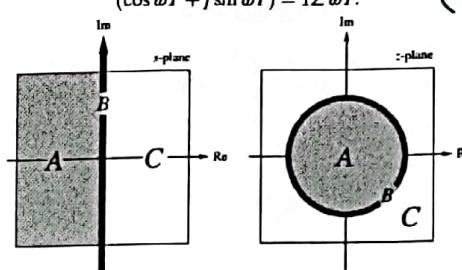
Digital System Stability via the z-Plane I

- The glaring difference between analog feedback control systems and digital feedback control systems, is the effect that the sampling rate has on the transient response. Changes in sampling rate not only change the nature of the response from overdamped to underdamped, but also can turn a stable system into an unstable one.
- In the s-plane, the region of stability is the left half-plane. If the transfer function, $G(s)$, is transformed into a sampled-data transfer function, $G(z)$, the region of stability on the z-plane can be evaluated from the definition, $z = e^{Ts}$. Letting $s =$, we obtain

* بالـ Z-Domain ما يجي RHP و LHP يعني unit circle

$$\begin{aligned} z &= e^{Ts} = e^{T(\alpha+j\omega)} = e^{\alpha T} e^{j\omega T} \\ &= e^{\alpha T} (\cos \omega T + j \sin \omega T) \\ &= e^{\alpha T} \angle \omega T \end{aligned}$$

العلاقة اللي
بتربط بين
معادلـ مع
الاستنفادـ



$$s = \frac{z - 1}{z + 1}$$

* صرنا نعرف او لا بناء كشطة بـ الـ zـPlane او جـ المـائـة او كـ حـيـطـ المـائـة

أـ أيـ نقطـةـ لـ المـائـةـ كـ اـنـهاـ RHPـ خـالـ Sysـ بـ كـونـهـ Unstableـ

أـ أيـ نقطـةـ لـ خـلـ المـائـةـ كـ اـنـهاـ LHPـ فـلاـ Sysـ بـ كـونـهـ Stableـ

أـ أيـ نقطـةـ كـ حـافـةـ المـائـةـ وـ لـ اوـ الـ rootـ متـكـرـراتـ فـوقـ بـعـدـ ماـ بـ كـونـهـ Stableـ

Margin stable

Digital System Stability via the z-Plane II

→ abs command → complex num لـ هناـ الـ magـ nitudeـ + pole command

* If the magnitude $e^{\alpha T} > 1$, this means that the poles are outside the unit circle. Any point outside the circle means the system is unstable. This is equivalent to having a pole in the right half plane in the s-domain

Poles او abs

* If the magnitude $e^{\alpha T} = 1$, the pole lies exactly on the circle. This is equivalent to having a pole on the jw axis in the s-domain

بعـدـ هـذـهـ الـ absـ

* If the magnitude $e^{\alpha T} < 1$, the pole lies inside the circle. This is equivalent to having a pole on the left half plane in the s-domain

وبـعـدـ هـذـهـ الـ absـ

* Stability criteria:

✓ (1) stable if all poles of the closed-loop transfer function, $T(z)$, are inside the unit circle on the z-plane,

قيـمـهـ لـ أـكـبـرـ

✓ (2) unstable if any pole is outside the unit circle and/or there are poles of multiplicity greater than one on the unit circle, and

منـ 1ـ فـلاـ Sysـ

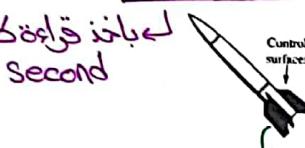
✓ (3) marginally stable if poles of multiplicity one are on the unit circle and all other poles are inside the unit circle.

بعـدـ هـذـهـ الـ absـ (لـ ولـقـيـتـ عـالـأـقـلـ عـاـلـهـ أـكـبـرـ) خـالـ Sysـ بـ كـونـهـ Unstableـ

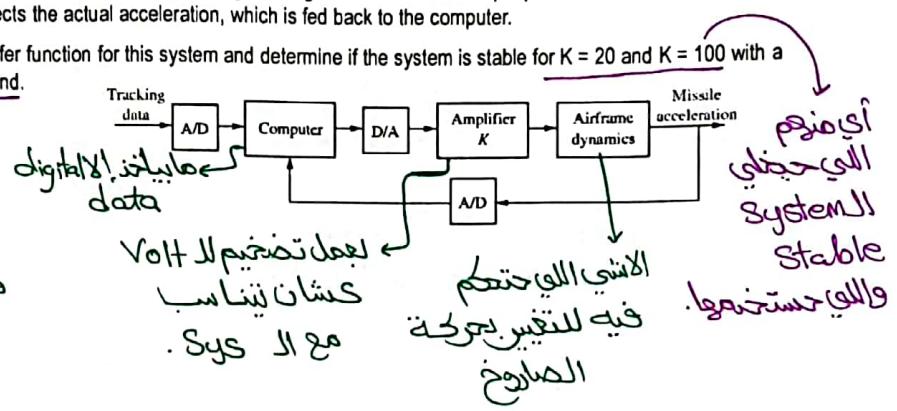
Example

- The missile shown can be aerodynamically controlled by torques created by the deflection of control surfaces on the missile's body. The commands to deflect these control surfaces come from a computer that uses tracking data along with programmed guidance equations to determine whether the missile is on track. The information from the guidance equations is used to develop flight control commands for the missile. A simplified model is shown as well.
- Here the computer performs the function of controller by using tracking information to develop input commands to the missile. An accelerometer in the missile detects the actual acceleration, which is fed back to the computer.
- Find the closed-loop digital transfer function for this system and determine if the system is stable for $K = 20$ and $K = 100$ with a sampling interval of $T = 0.1$ second.

لـ باخذ قراءة كل 0.1 Second

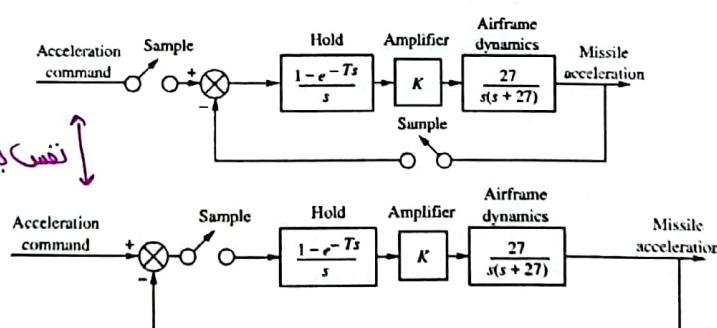


قطع مستمر للتحكم بحركة الصاروخ وويتوجه فالعم Controller للتحكم.

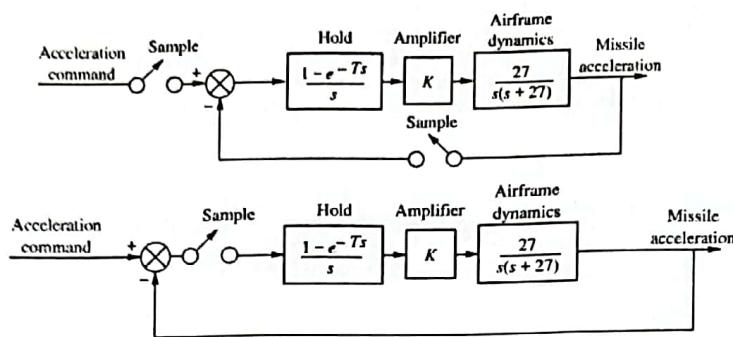


Example – cont.

نفس بعض بشرط تكون نفسموا حالاتن كشان أقدر أخليه زي نهت



Example – cont.



Example MATLAB Solution I

```

numg=27; % Define numerator of Ga(s).
deng=[1 27 0]; % Define denominator of Ga(s).
Ga=tf(numg,deng); % Create and display Ga(s).

Gz=c2d(Ga,0.1,'zoh') % Find G(z) assuming Ga(s) in cascade with z.o.h. and display.

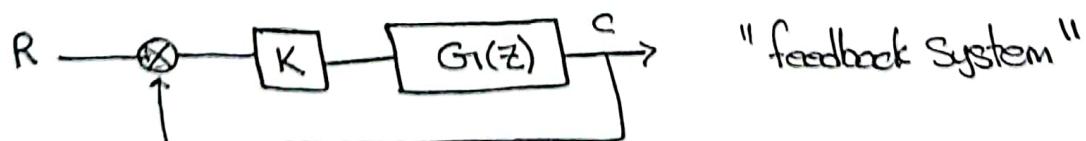
Gz =

```

$$\frac{0.06545 z + 0.02783}{z^2 - 1.067 z + 0.06721}$$

sample time: 0.1 seconds
Discrete-time transfer function.

بعد هادي الخطوة هذه الـ System زكي هيلك



Example MATLAB Solution II

* ببى ألا أشوف
شو قيم K اللي بتختلى
اللي يكون System II
Stable
 $K = 20$
(يمكن تختلوا)
 $K = 100$
و هى كده و هى
تطبع قيم K اللي
يكون تختلوا
واللي أكبر
هنا
(Unstable)

```

for K=1:0.1:50; % Set range of K to look for stability.
Tz=feedback(K*Gz,1); % Find T(z).
r=pole(Tz); % Get poles for this value of K.
rm=max(abs(r)); % Find pole with maximum absolute value for this value of K.
if rm>1, % See if pole is outside unit circle.
    break; % Stop if pole is found outside unit circle.
end
display(K)
K = 33.6000
display(r)

```

أى قيمة K تحتها يدخل الـ Sys Stable لو خرقوا بخلون . Unstable

النوى
Poles
Zeros
والـ
تقريباً

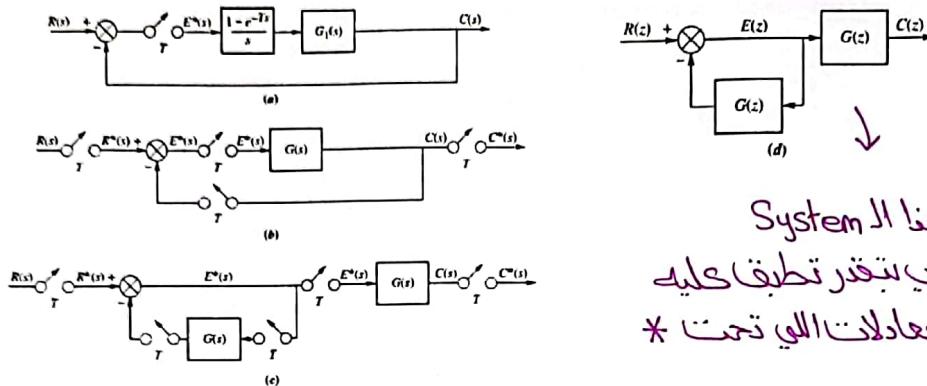
r = 2x1 complex
-0.5660 + 0.8257i
-0.5660 - 0.8257i

Steady-State Errors in Digital Systems

* نفس مفاهيم الـ K_a, K_v, K_p في طريقة استخراج الـ S-Domain لـ Z-Domain

- ▶ Any general conclusion about the steady-state error is difficult because of the dependence of those conclusions upon the placement of the sampler in the loop.
- ▶ Remember that the position of the sampler could change the open-loop transfer function.
- ▶ In the discussion of analog systems, there was only one open-loop transfer function, G(s), upon which the general theory of steady-state error was based and from which came the standard definitions of static error constants.
- ▶ For digital systems, however, the placement of the sampler changes the open-loop transfer function and thus precludes any general conclusions. In this chapter, we assume the typical placement of the sampler after the error and in the position of the cascade controller, and we derive our conclusions accordingly about the steady-state error of digital systems

Assumed System Model for Derivation



* هذا الـ System الى يتقدّم تطبيقاً عليه المعادلات الى تحت *

→ "In Z-Domain"

Steady State Error Equations

The final value theorem for discrete signals states that $\lim_{z \rightarrow 1} E(z) = S\text{-Domain}$ هذا اللي مختلف

The error due to a unit step input is defined as

The error due to a ramp input is defined as

The error due to a parabola input is defined as

* If $S \rightarrow 0$ So $Z \rightarrow 1$
because $Z = e^{Ts}$

$$\epsilon^*(\infty) = \frac{1}{1 + K_p}$$

$$K_p = \lim_{z \rightarrow 1} G(z)$$

$$e^*(\infty) = \frac{1}{K_v}$$

$$K_v = \frac{1}{T} \lim_{z \rightarrow 1} (z - 1) G(z)$$

$$\epsilon^*(\infty) = \frac{1}{K_a}$$

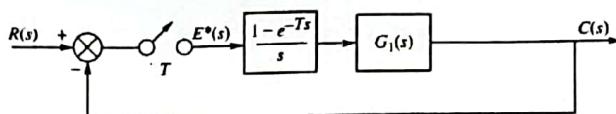
$$K_a = \frac{1}{T^2} \lim_{z \rightarrow 1} (z - 1)^2 G(z)$$

Example - MATLAB

- For step, ramp, and parabolic inputs, find the steady-state error for the feedback control system shown in the figure if

$$G_1(s) = \frac{20(s+3)}{(s+4)(s+5)}$$

- And the sampling rate is 0.1 sec, 0.5 sec



- We can use MATLAB's command `dcgain(Gz)` to find steady-state errors. The command evaluates the dc gain of G_z , a digital LTI transfer function object, by evaluating G_z at $z = 1$. We use the dc gain to evaluate, K_p , K_v , and K_a .

طابق الشكل اللي أخذناه أو لا، وهل لما بطبق بقدر أخليه يطابق
ولا، لازم أجواب المسؤولين هنول قبل ما أحل.

MATLAB Solution ($T = 0.1$)

```
T = 0.1; % Input sampling interval.
G1s= zpk(-3, [-4 -5], 20)
```

```
G1s =
20 (s+3)
-----
(s+4) (s+5)

Continuous-time zero/pole/gain model.
```

```
Gz=c2d(G1s,T,'zoh') % Convert G1(s) and z . o. h. to G(z)
```

```
Gz =
1.4994 (z-0.7465)
-----
(z-0.6783) (z-0.6065)

Sample time: 0.1 seconds
Discrete-time zero/pole/gain model.
```

```
Tz=feedback(Gz, 1) % Create and display T(z).
```

```
Tz =
1.4994 (z-0.7405)
-----
(z-0.7349) (z+0.9574)

Sample time: 0.1 seconds
Discrete-time zero/pole/gain model.
```

```
r=pole(Tz) % Check stability.
```

```
r =
0.7349
-0.9574
```

أقلمن ١
فال
stable
هلا بقدر أحسب
 K_v و K_a و K_p
الأخطاء

MATLAB Solution ($T = 0.1$) - cont

```

M=abs(r) % Display magnitude of roots.
M =
    2x1
    0.7349
    0.9574

Kp=dcgain(Gz) % Calculate Kp.
e=Finite <-- Kp = 3.0000

GzKv=(Gz*(z-1))/T;
Kv=dcgain(GzKv) % Calculate Kv.
e = ∞ <-- Kv = 0

GzKa=Gz*(1/T^2)*((z-1)^2);
Ka=dcgain(GzKa) % Calculate Ka
e = ∞ <-- Ka = 0

```

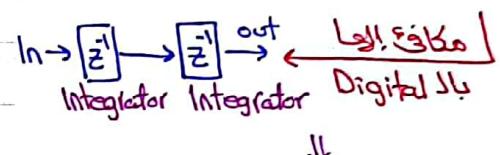
* note in Simulink :

$$x'' = bx' + 3 \quad \boxed{x}$$

بدهی مولعه اد x

$$x'' \rightarrow \begin{array}{c} \frac{1}{s} \\ \downarrow \end{array} x' \rightarrow \begin{array}{c} \frac{1}{s} \\ \downarrow \end{array} x$$

Integrator Integrator



بنده تفہیم اے \Rightarrow error
کام مالکنہا جا مقام s
منصفہ

* المکافع لای Integrator بالا فرضیتی
موجود معاملہ لای $\frac{1}{z}$ \Rightarrow z^{-1} \Rightarrow Type 0

و لو في حدود دنيا
و كلها للتكامل
کنا بتعمل بالا
block click
و بنكتب هایي الحدود
Physical limits

MATLAB Solution ($T = 0.5$) (UNSTABLE)

```

T = 0.5; % Input sampling interval.
G1s= zpk(-3, [-4 -5], 20)
G1s =
20 (s+3)
-----
(s+4) (s+5)

Continuous-time zero/pole/gain model.

Gz=c2d(G1s,T,'zoh') % Convert G1(s) and z . o. h. to G(z)
Gz =
3.02 (z-0.2116)
-----
(z-0.2085) (z+3.011)

Sample time: 0.5 seconds
Discrete-time zero/pole/gain model.

```

```

Tz=feedback(Gz, 1) % Create and display T(z).
Tz =
3.02 (z-0.2116)
-----
(z-0.2085) (z+3.011)

Sample time: 0.5 seconds
Discrete-time zero/pole/gain model.

r=pole(Tz) % Check stability.

```

$r = z \in$
 0.2085
 -3.0111

M=abs(r) % Display magnitude of roots.

M =
 2x1
 0.2085
 3.0111 \rightarrow اکبرون ۱

Sys unstable
فلتت الـ
فما بینکل جر.

* خیرنا الـ T
ھون وشننا
انه الـ System
unstable
کل

References

- ▶ The material in these slides are based on:
Control Systems Engineering, Norman S. Nise, 7th Edition (2014), John Wiley And Sons
 - **Chapter 13 – Digital Control Systems**
 - Sections 13.1, 13.2, 13.3, 13.4, 13.5, 13.6, 13.7 (Students kindly note that these sections involve lots of math, and we only described the ideas as we will use MATLAB instead)